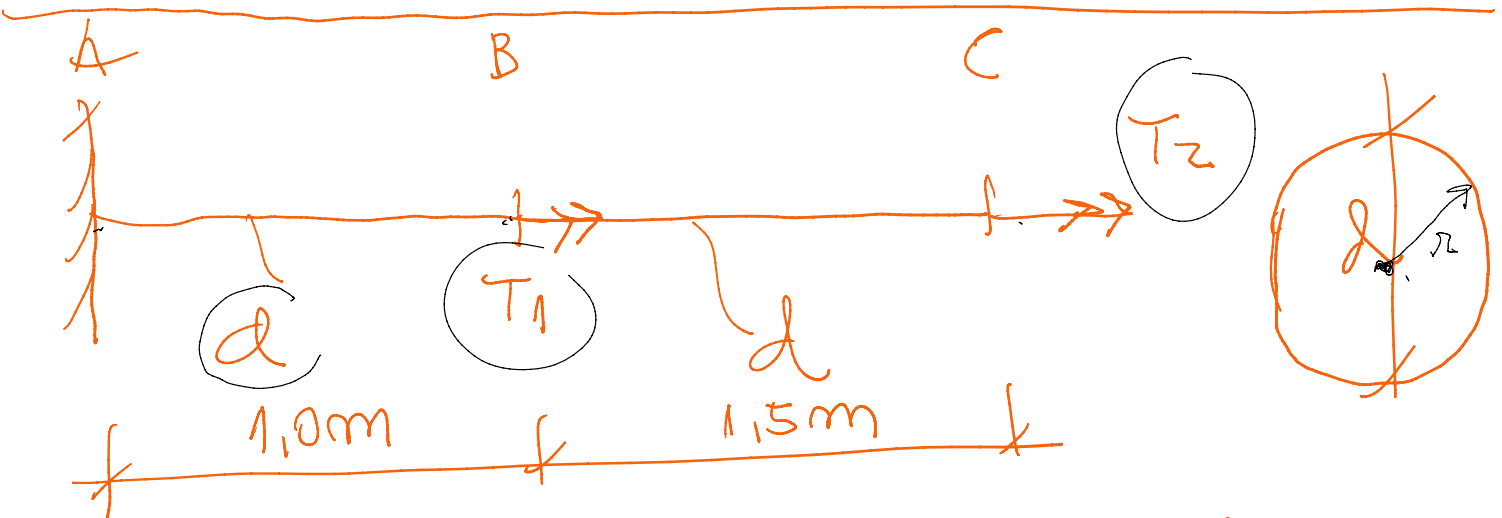


Tensões de cisalhamento

$$\phi = \frac{T \cdot L}{G \cdot I_p}$$

$$\tau = \frac{T \cdot \rho}{I_p}$$

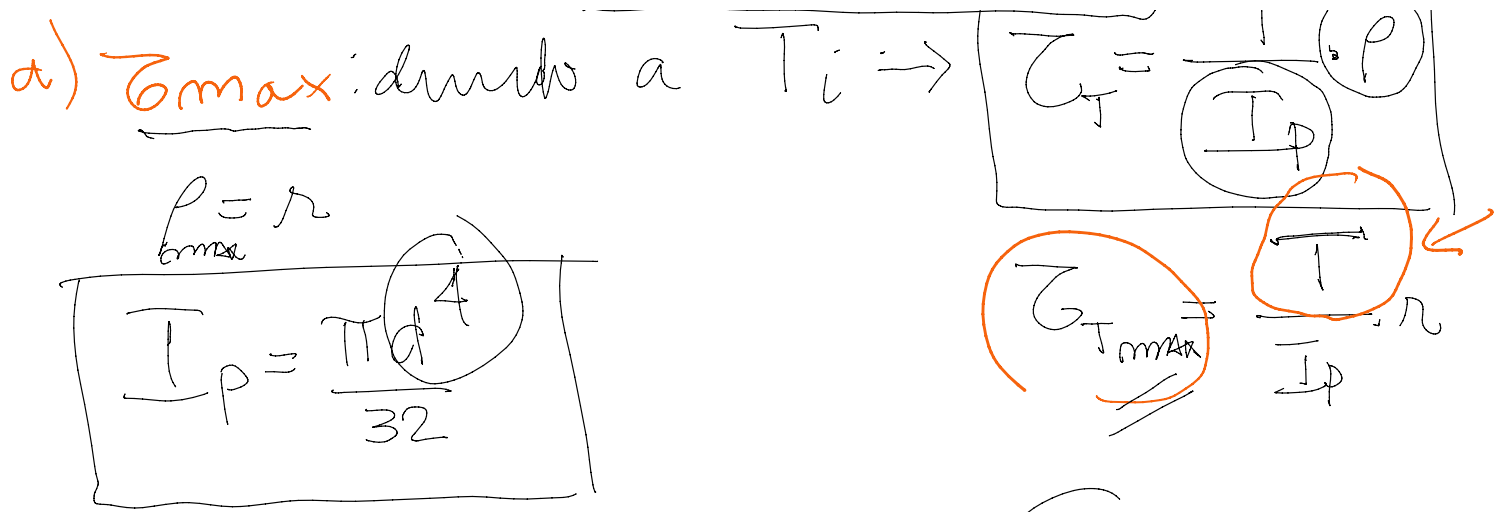


$T_1 = 10 \text{ kNm}$ $T_2 = 5 \text{ kN}$ $d = 5 \text{ cm}$

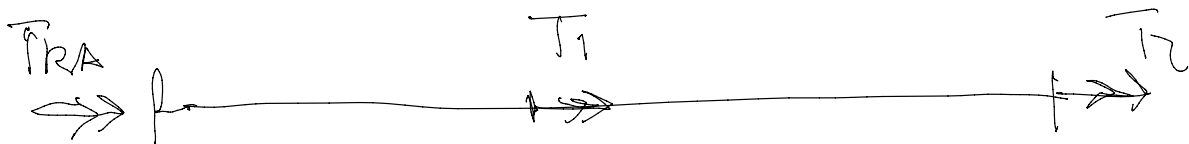
$G = 80 \text{ GPa}$

$\tau_{\text{m}} = 110 \text{ MPa}$

a) τ_{max} : devido a $T_1 \rightarrow \tau = \frac{T \cdot \rho}{I_p}$



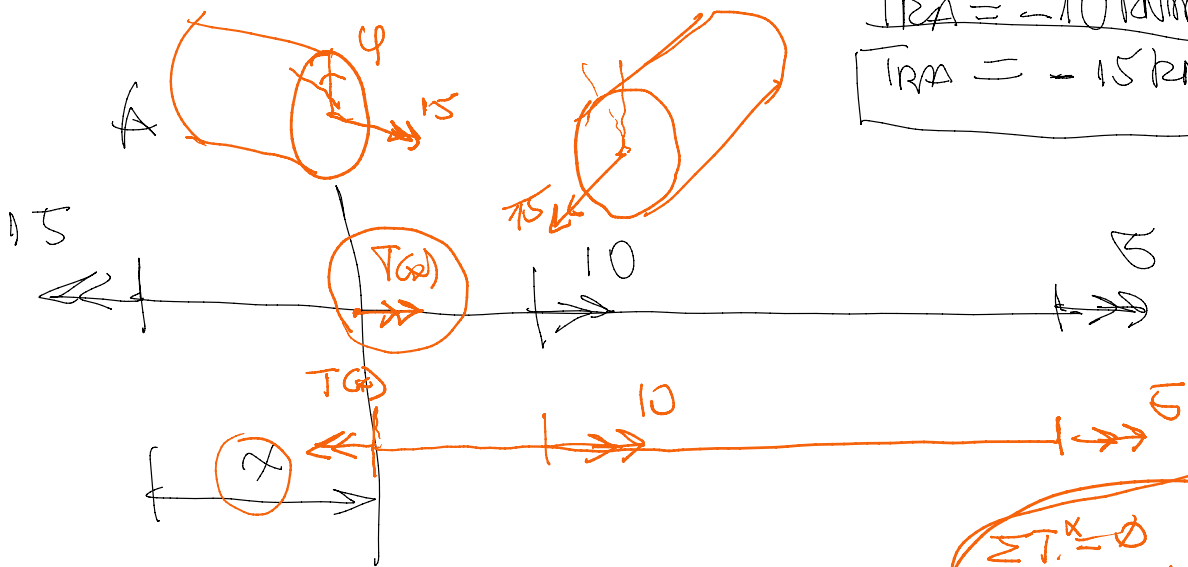
but a way more de torque T



$$\sum T^A = 0 \quad T_{RA} + T_1 + T_2 = 0 \quad \therefore T_{RA} = -T_1 - T_2$$

$$T_{RA} = -10 \text{ kNm} - 5 \text{ kNm}$$

$$T_{RA} = -15 \text{ kNm}$$



$$\sum T = 0 \quad T(x) - 15 = 0 \quad 0 < x < 1 \text{ m}$$

$$T(x) = 15 \text{ kNm}$$

$$\sum T^A = 0$$

$$T(x) - 10 - 5 = 0$$

$$T(x) = +15$$



$$\sigma_{all} = \sigma_{adm}$$

$$\frac{1}{I_p} \tau \leq \frac{\tau_{adm}}{\gamma_{coef. sup}}$$

o II a montagem e suficiente

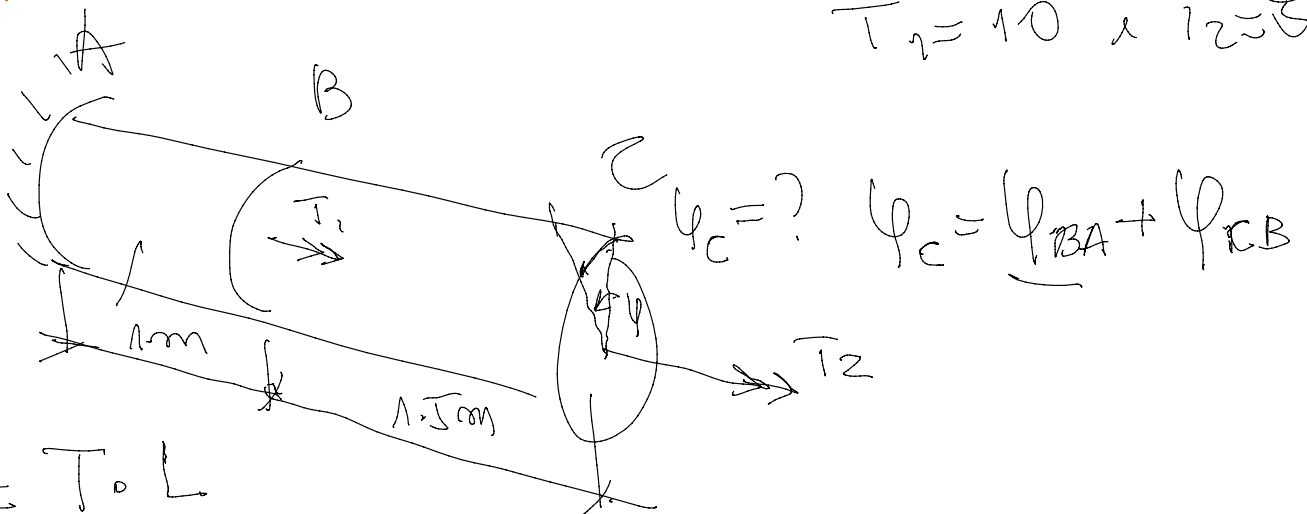
Dado o coef. de segurança de $\gamma = 2$, de τ o valor de d para o cisalhamento τ_1 e τ_2

$$\tau_{adm} \leq \tau_{adm}$$

$$\frac{16 \tau_{adm}}{\pi d^3} \leq \left(\frac{\sigma_u = 110}{2} \right) d^3 \geq \frac{16 \tau_{adm}}{55 \pi}$$

b) Calcular a posição por C (ext. da barra)

$$\tau_1 = 10 \text{ e } \tau_2 = 5$$



$$\phi = \frac{T \cdot L}{G \cdot I_p}$$

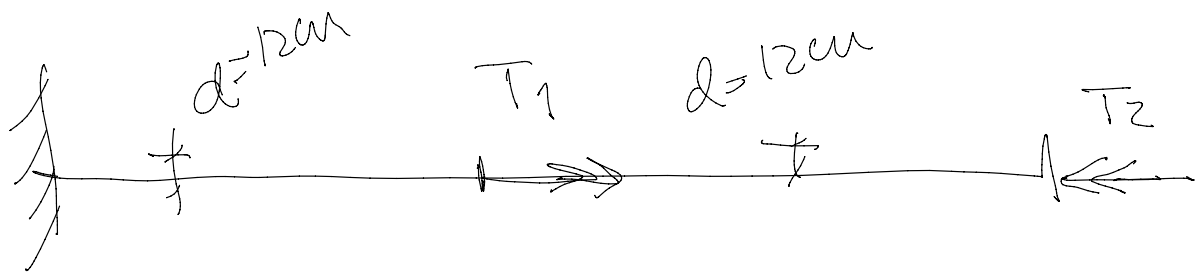
$$\phi_C = \frac{T_{AB} \cdot L_{AB}}{G \cdot I_p} + \frac{T_{BC} \cdot L_{BC}}{G \cdot I_p} = \left(\frac{15 \text{ kNm} \cdot 1 \text{ m}}{G \cdot I_p} \right) + \left(\frac{5 \text{ kNm} \cdot 1.5 \text{ m}}{G \cdot I_p} \right)$$

$$MD \rightarrow \frac{1}{G \cdot I_p} (15 \times 1 + 5 \times 1.5)$$

$$\varphi_c = \frac{1}{GJ\theta} (15 \times 1 + 5 \times 1,5) \uparrow \uparrow \quad \uparrow \uparrow$$

$$\varphi_c = 0,792 \text{ grau ou } \varphi_c = 0,14 \text{ rad}$$

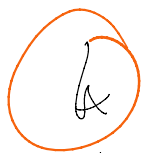
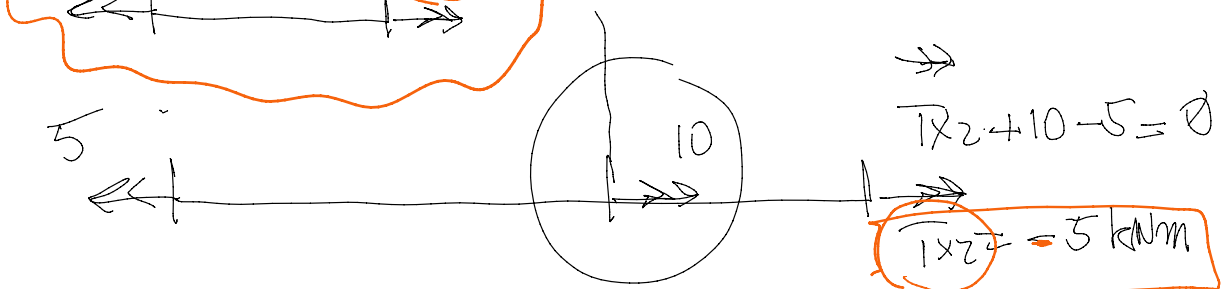
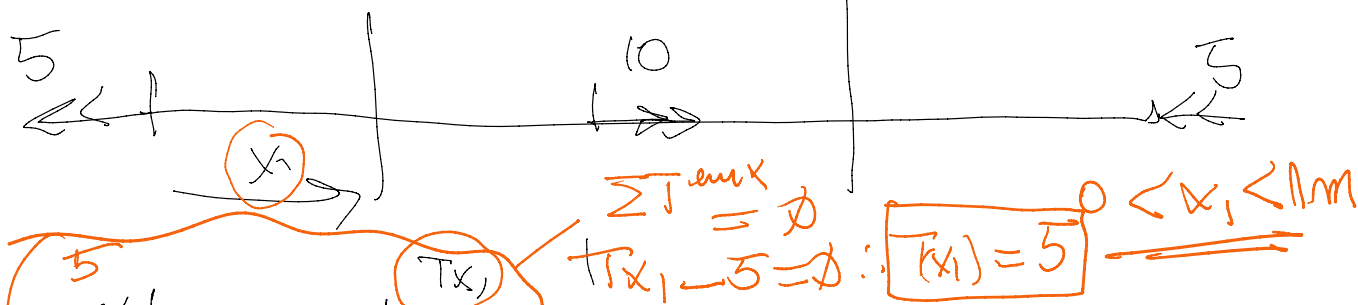
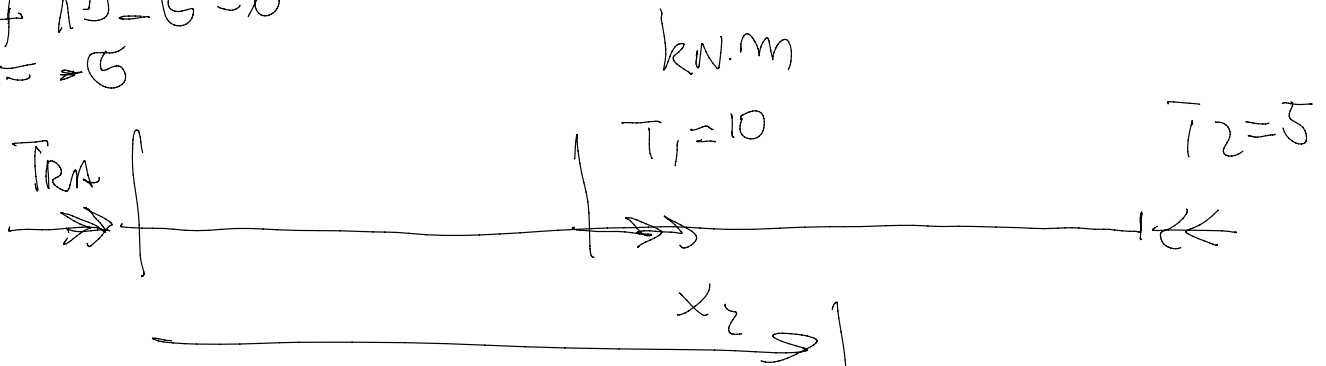
Ex 2



Calculo da rotacao

$$T_{RA} + 10 - 5 = 0$$

$$T_{RA} = -5$$



$$1 < x_2 < 2,5 \text{ m}$$



$$\varphi = \frac{T \cdot L}{G \cdot J_p}$$

