



ESCOLA POLITÉCNICA DA UNIVERSIDADE DE SÃO PAULO

Turbulência fantástica



aantunha

Tudo que existe no universo é fruto do acaso e da necessidade



Demócrito 460 a 385 a.C.

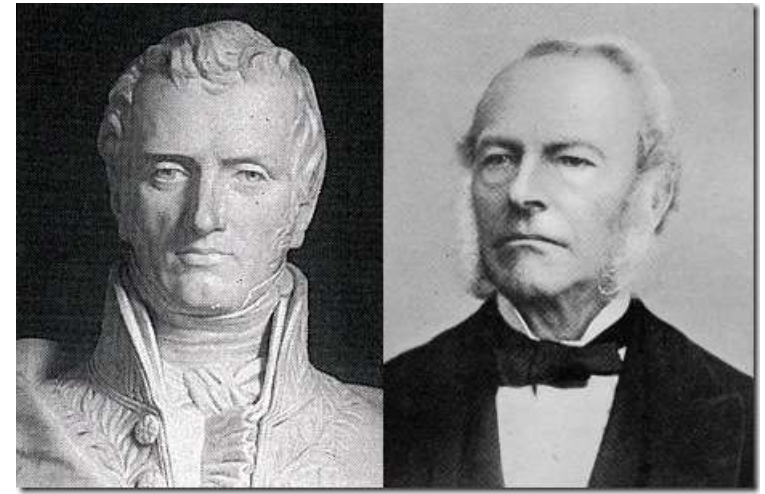
atomismo

da Vinci
1452-1519



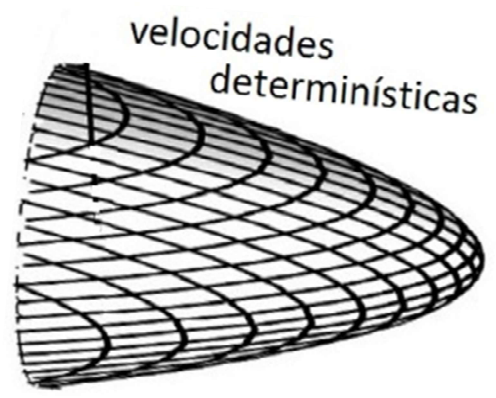
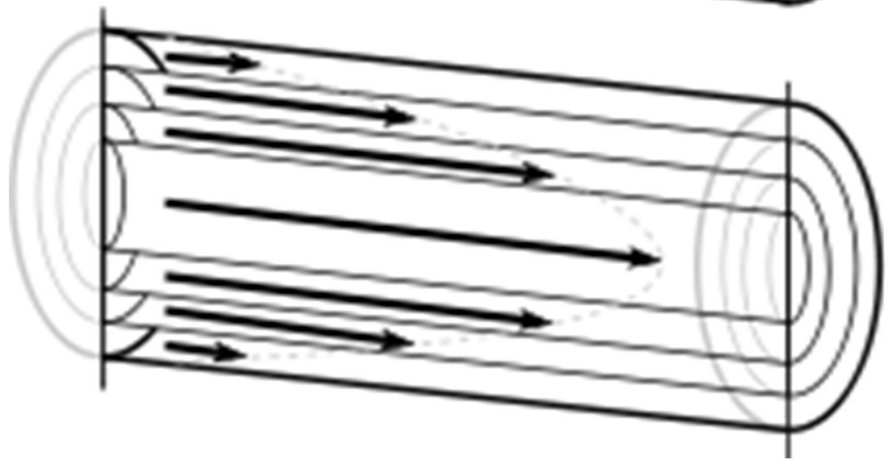
$$\underbrace{\frac{\partial(\rho \vec{v})}{\partial t}}_{\text{transiente}} = \underbrace{-\text{div}(\rho \vec{v} \vec{v})}_{\text{convecção}} - \underbrace{\text{div} \vec{\tau}}_{\substack{\text{força} \\ \text{contato} \\ \text{irreversível}}} - \underbrace{g \vec{r} \text{ad} p}_{\substack{\text{força} \\ \text{contato} \\ \text{reversível}}} + \underbrace{\rho \vec{g}}_{\text{força campo}}$$

$$\vec{\tau} = -\mu \frac{\partial v_z}{\partial r}$$

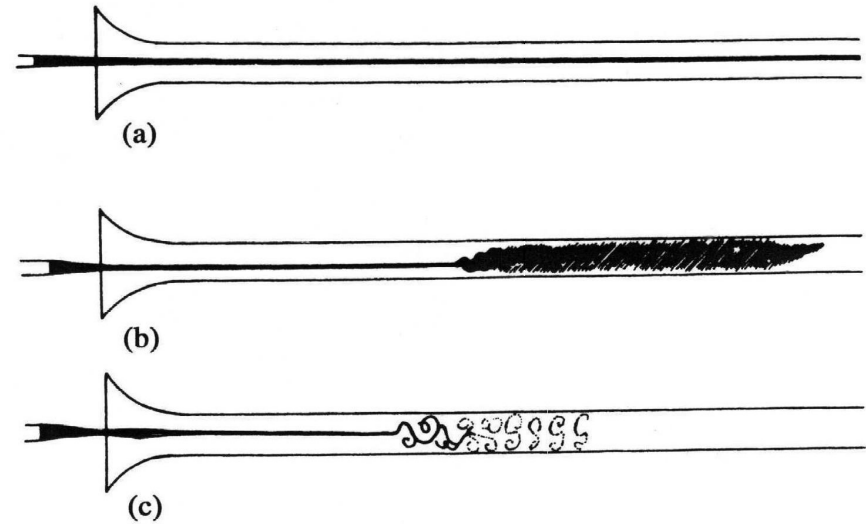
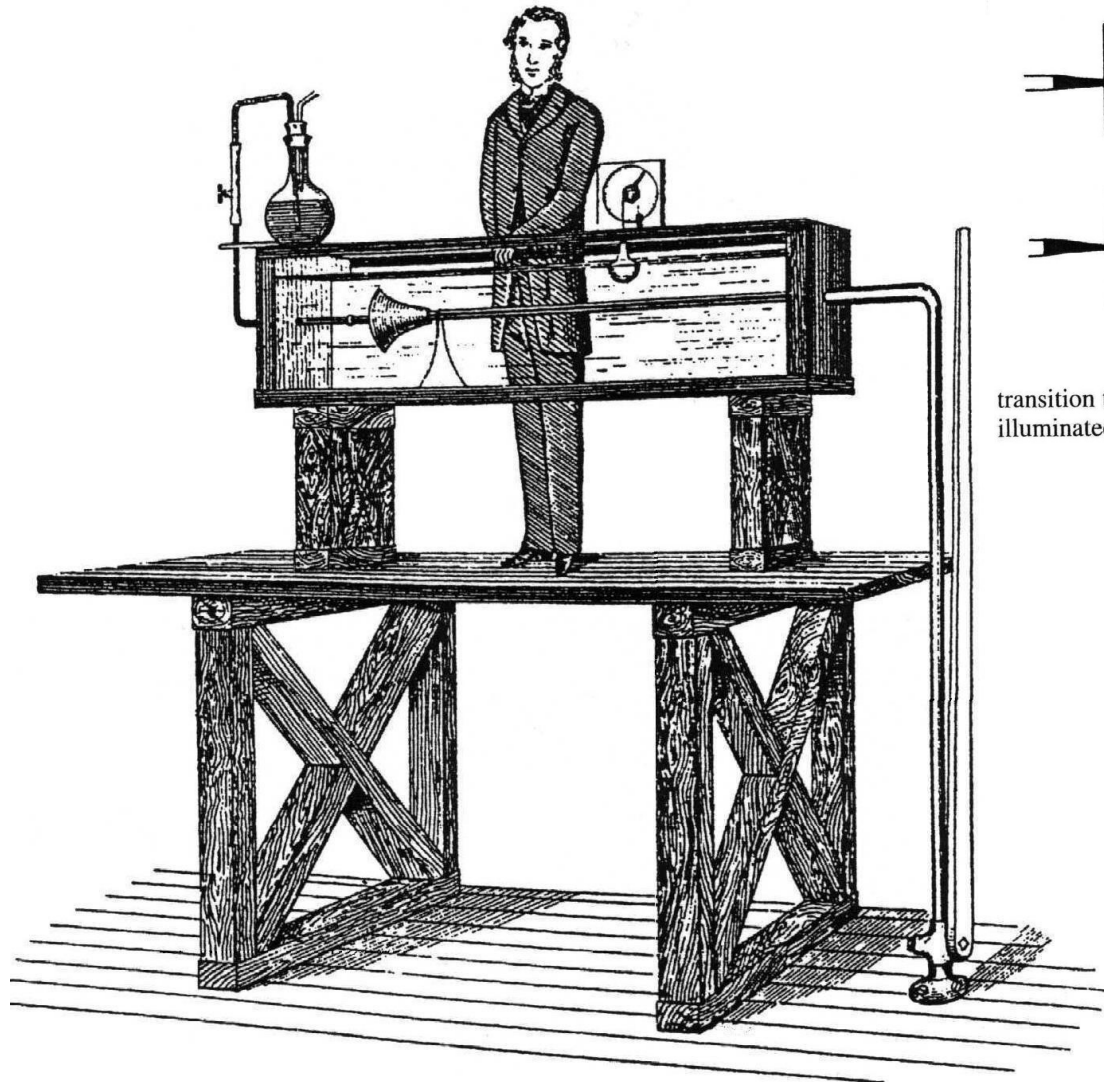


Navier
1785-1836

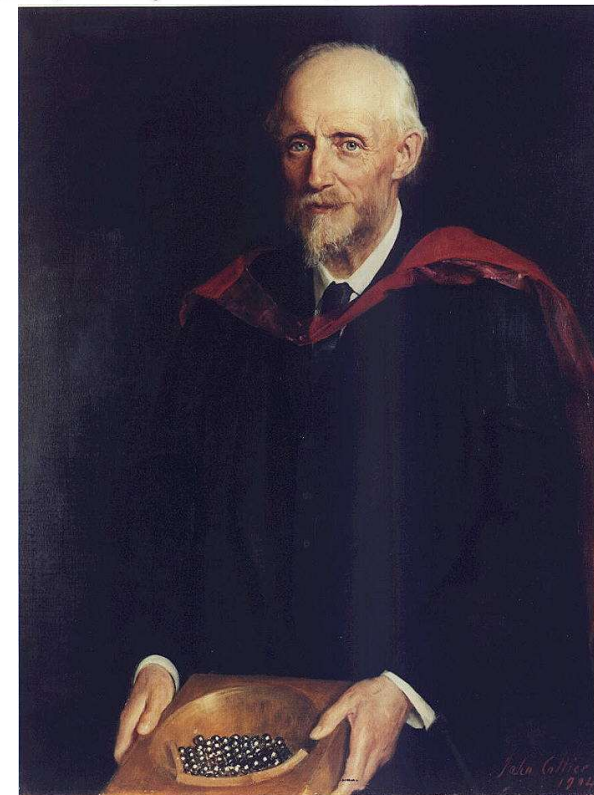
Stokes
1819 -1903



Osborne Reynolds 1842 1912



Sketches of (a) laminar flow in a pipe, indicated by a dye streak; (b) transition to turbulent flow in a pipe; and (c) transition to turbulent flow as seen when illuminated by a spark. (From Reynolds, 1883, Figs. 3, 4 and 5.)



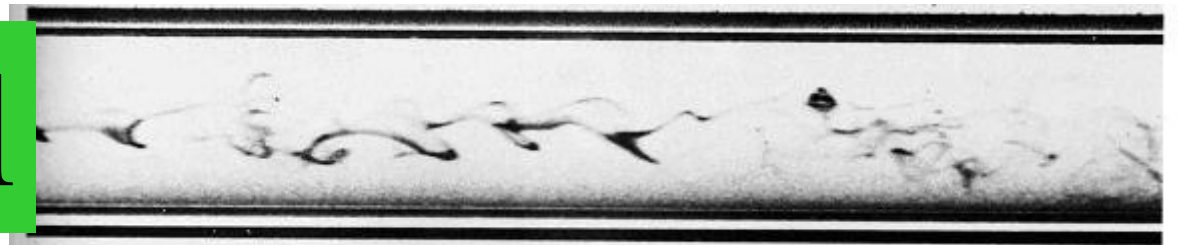
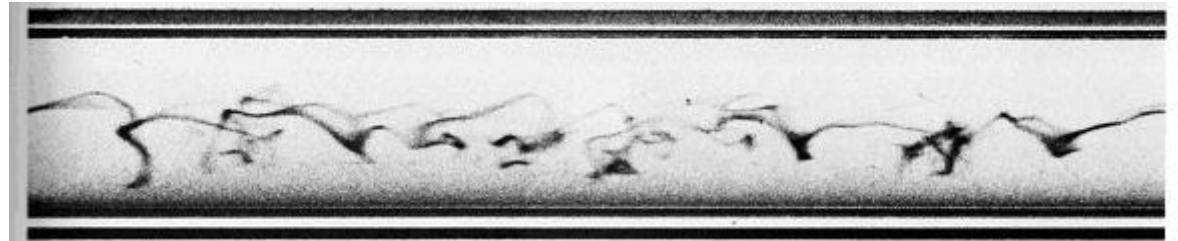
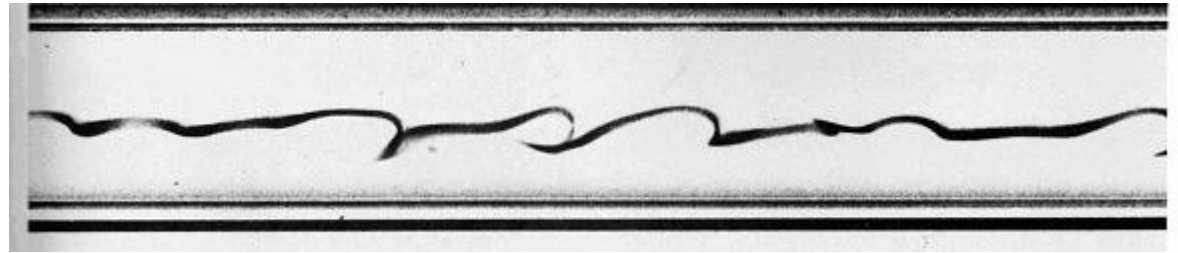
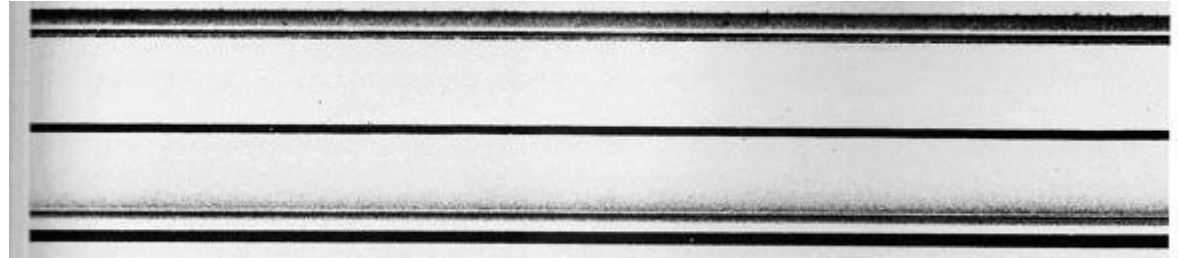
The configuration of Reynolds's experiment on flow along a pipe.

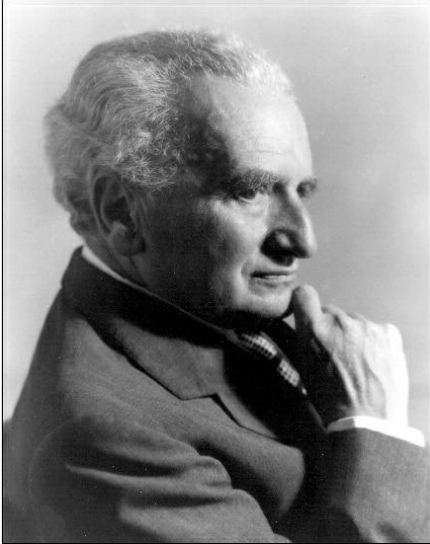
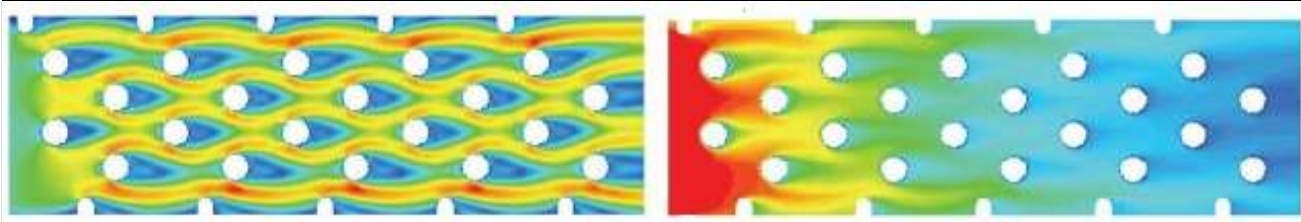
Reynolds
experimento
de 1883

rotacional

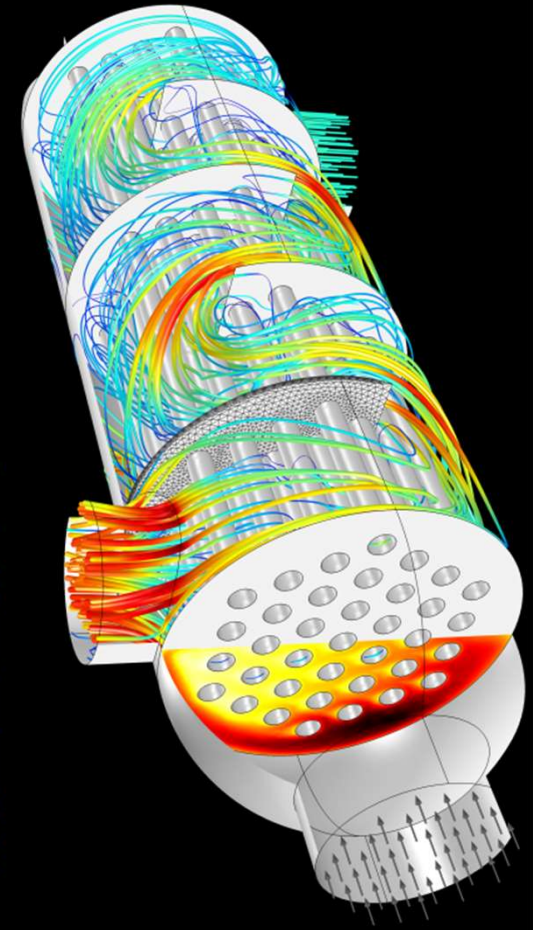
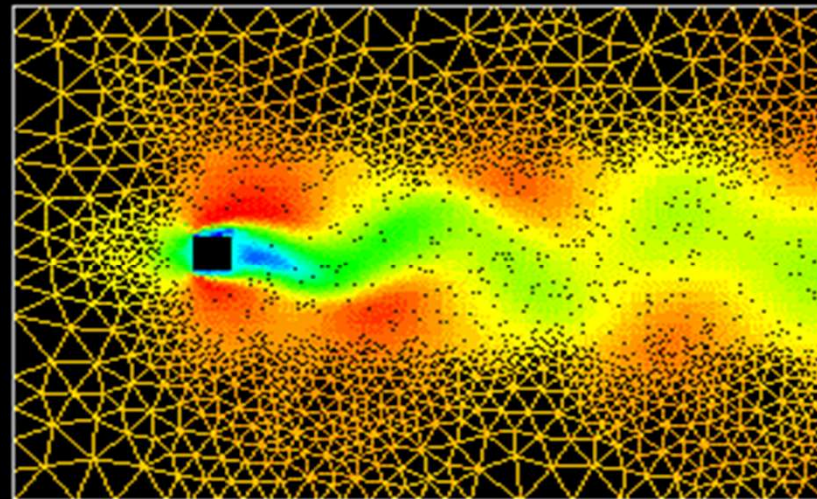
3D

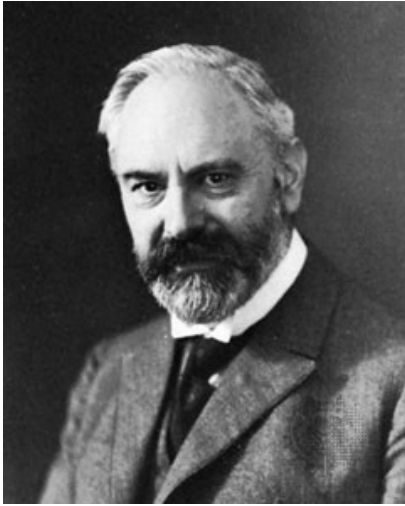
irreversible





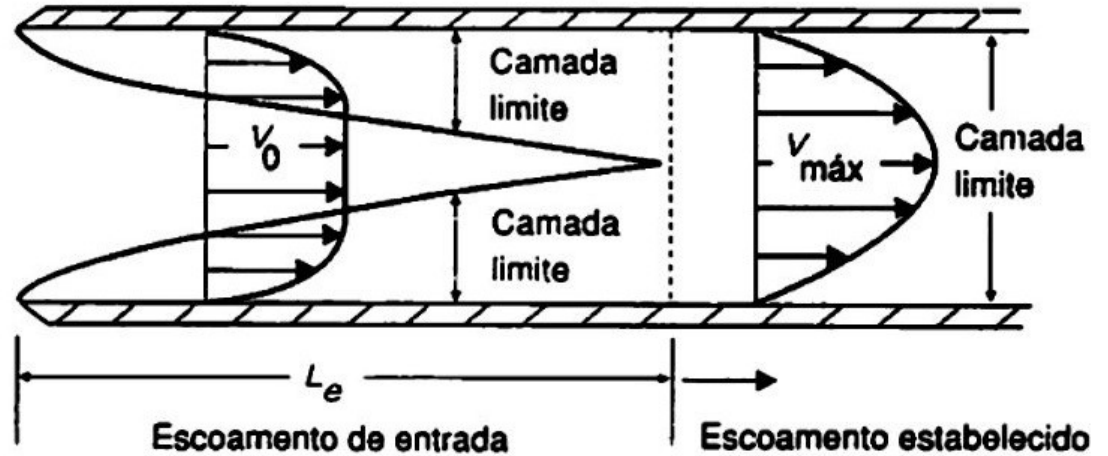
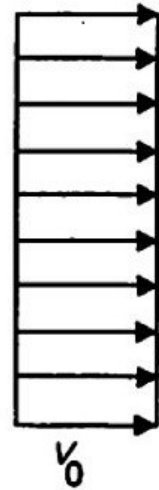
T. von Kármán
1881-1963



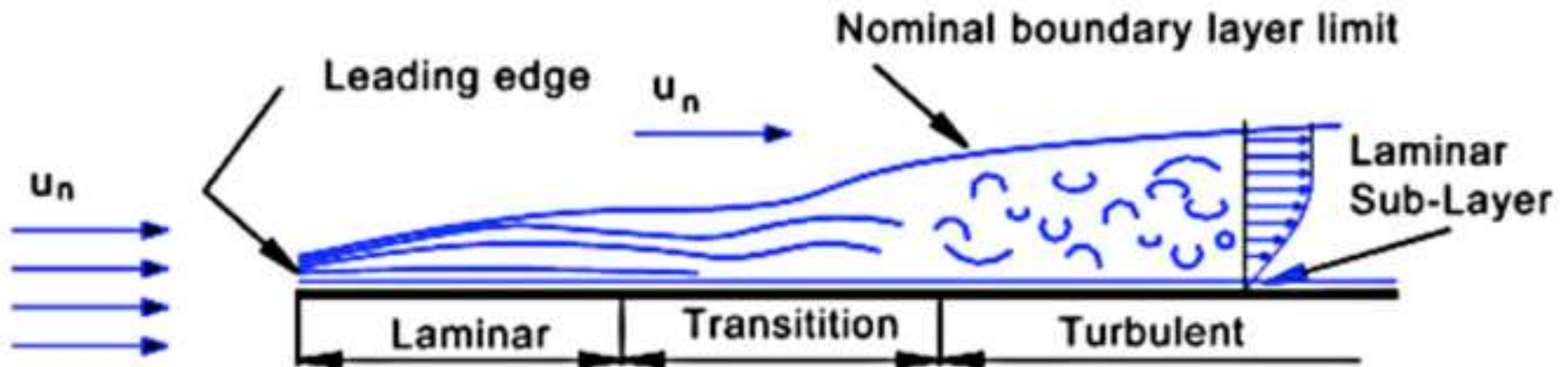


Prandtl
1875 – 1953

$$\frac{\partial \vec{v}}{\partial t} + \text{div } \vec{v} \vec{v} = \text{div} [\mu + \mu_T] \text{grad } \vec{v} + \vec{g}$$

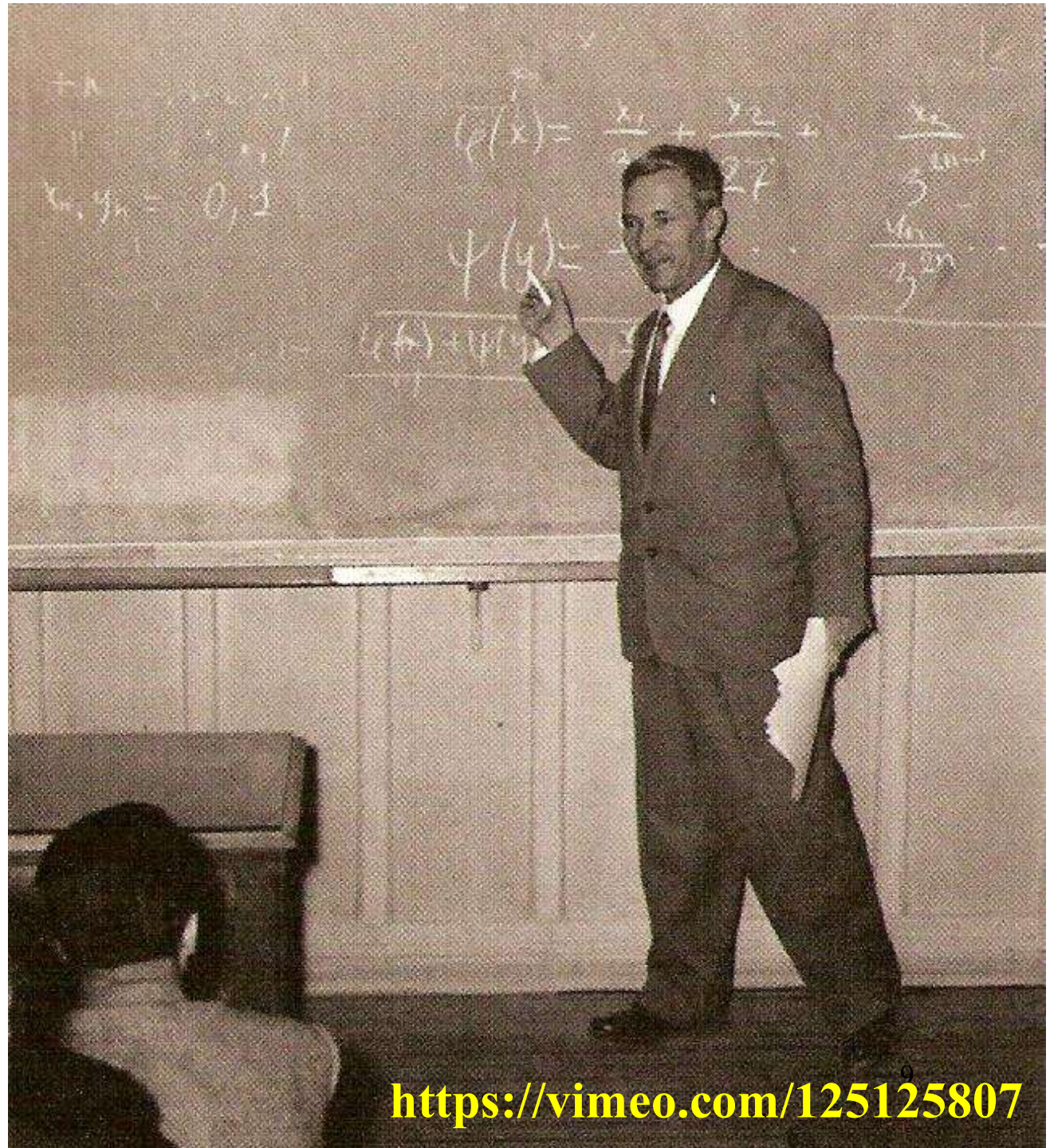


$$\mu_T = l \text{grad } \vec{v}$$



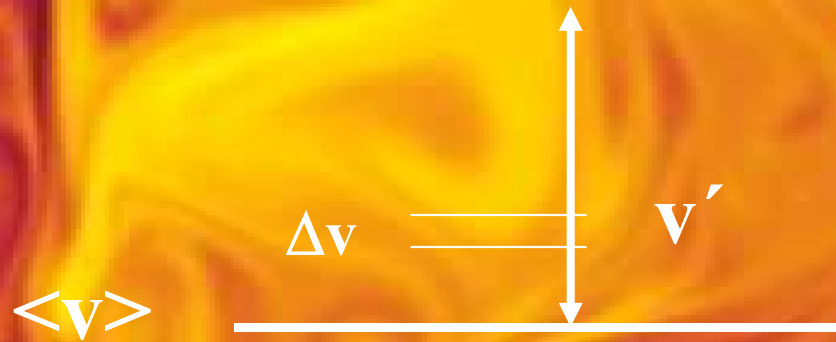
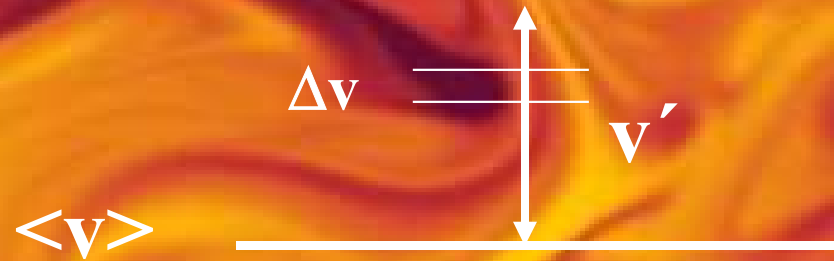
Kolmogorov 1903 -1987

“meu interesse pelo estudo dos escoamentos turbulentos surgiu no fim dos anos 30. Pareceu-me evidente que a técnica matemática principal deveria ser a teoria das **funções aleatórias de diversas variáveis**, que estava então nascendo.”



<https://vimeo.com/125125807>

$$\mathbf{v} = \langle \mathbf{v} \rangle + \mathbf{v}'$$

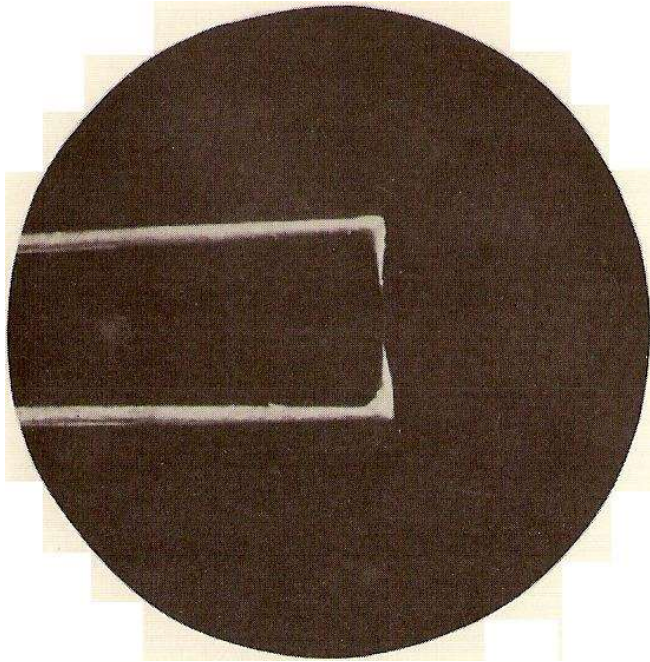


$$\mathbf{f}_v = \mathbf{f}_v(\mathbf{t}, \mathbf{r})$$

$$\mathbf{f}_v = \mathbf{f}_v[\mathbf{v}, \mathbf{v} + \Delta \mathbf{v}]$$

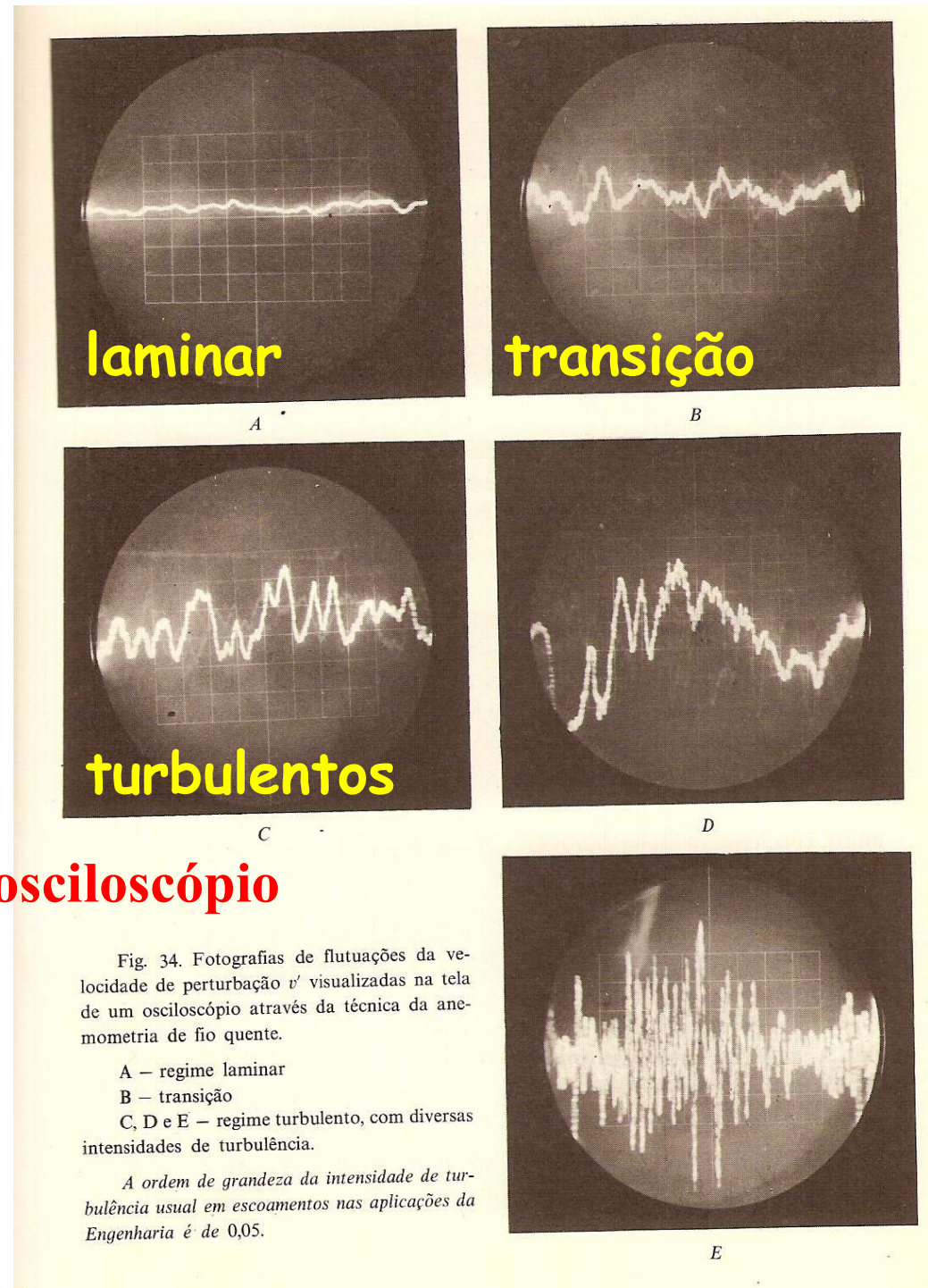
$\mathbf{f}_v =$ Probability Density Function = PDF

$$\pi = \pi' + \bar{\pi}$$

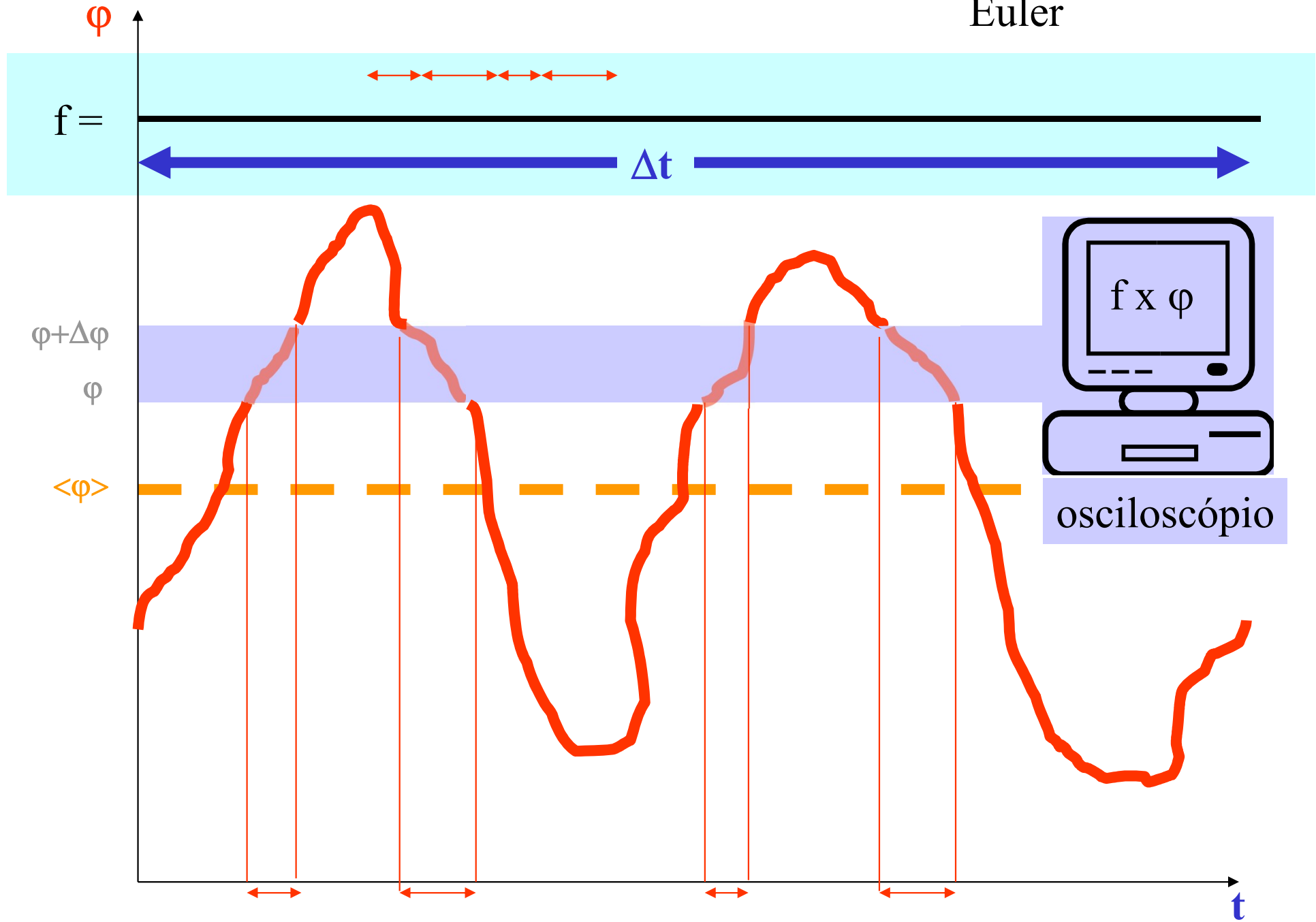


anemômetro de fio quente

$$\vec{V} = \vec{V}' + \bar{\vec{V}}$$



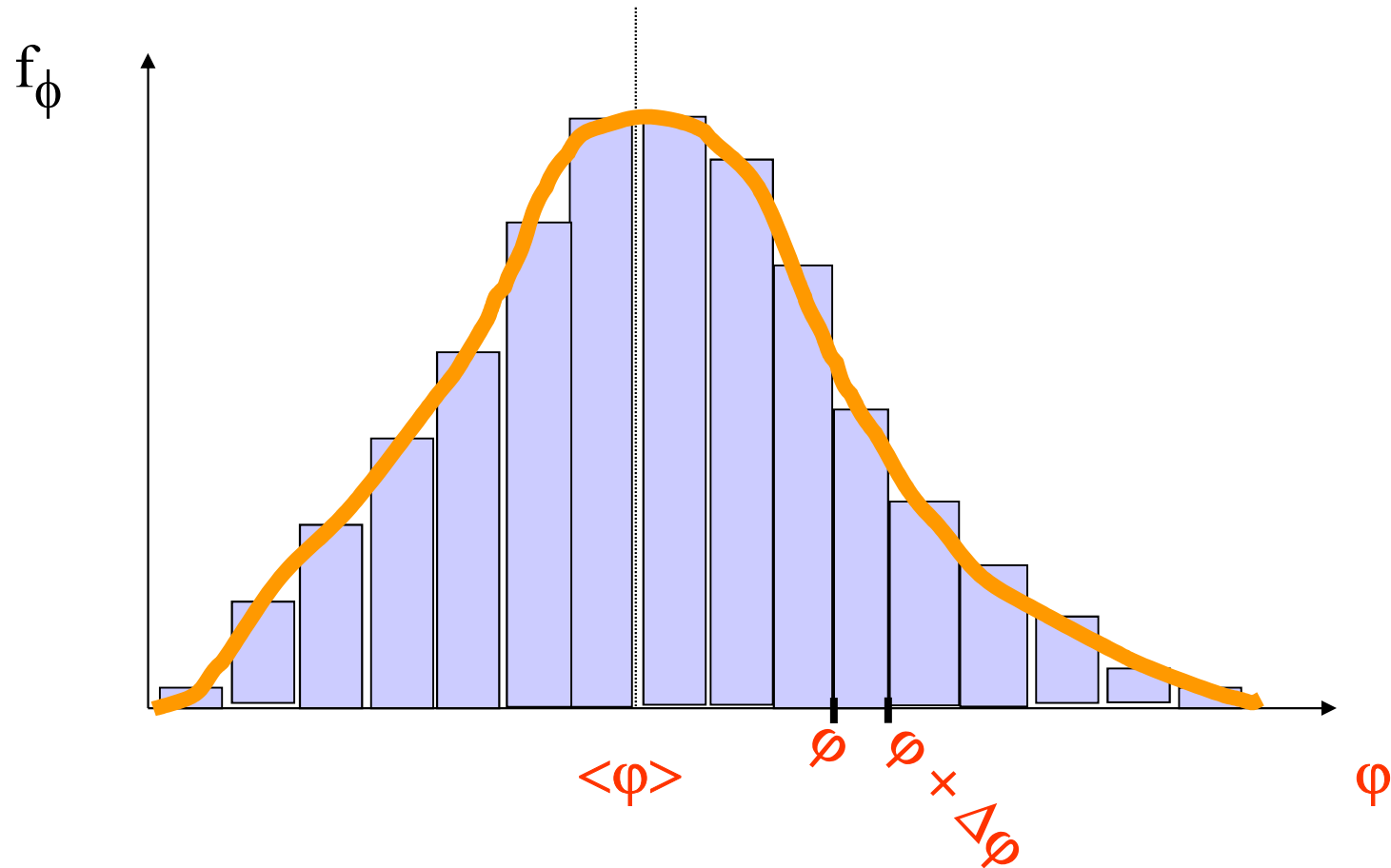
Euler



$f_\phi =$ Probability Density Function - PDF

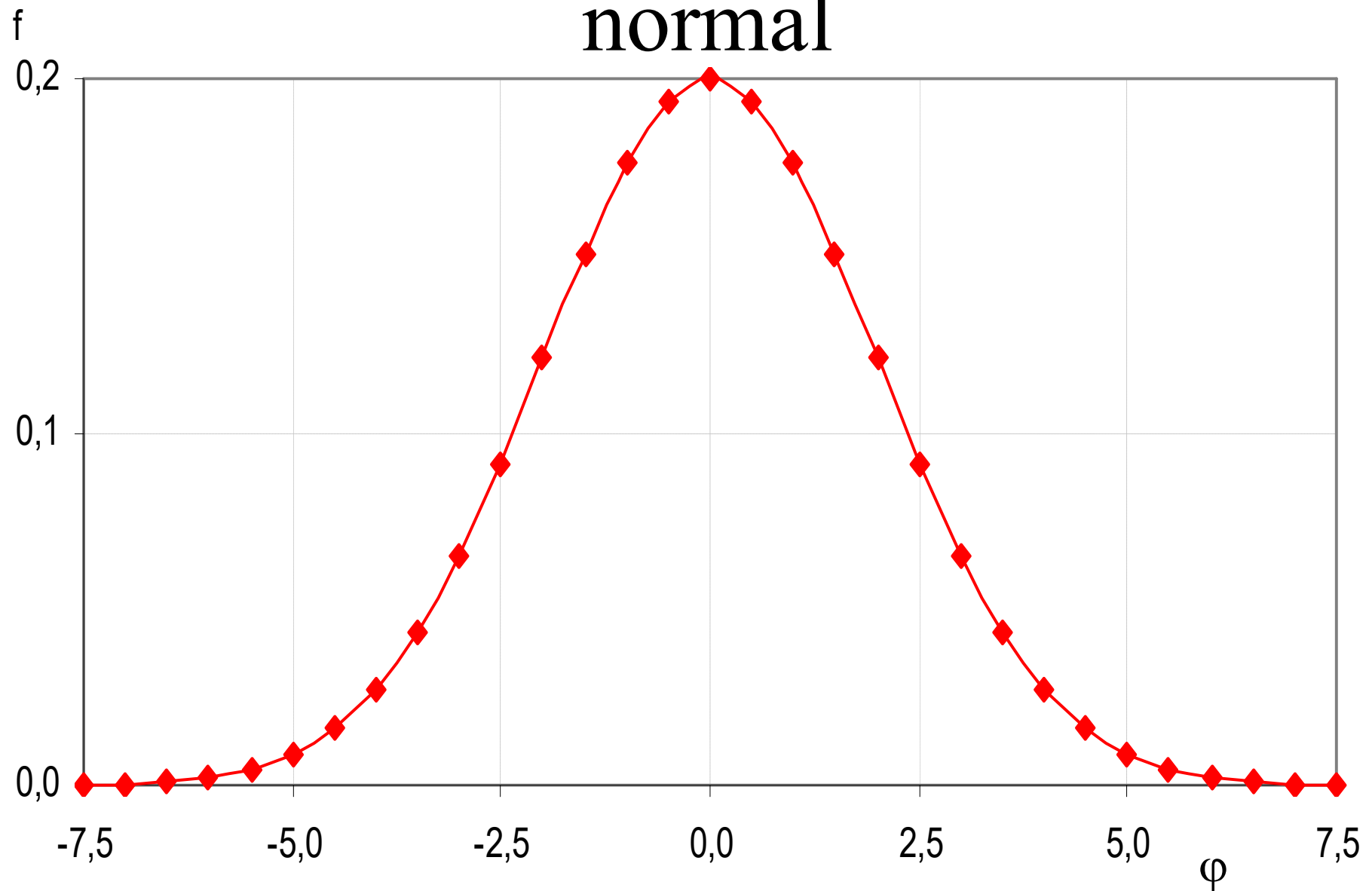
PDF

$$f_\phi = f_\phi [\phi , \phi + \Delta\phi] = f_\phi (t, r)$$



$f = \text{DIST.NORM}(\varphi ; \text{média} = 0 ; \text{variância} = 2 ; \tilde{n} \text{ cumulativo})$

normal



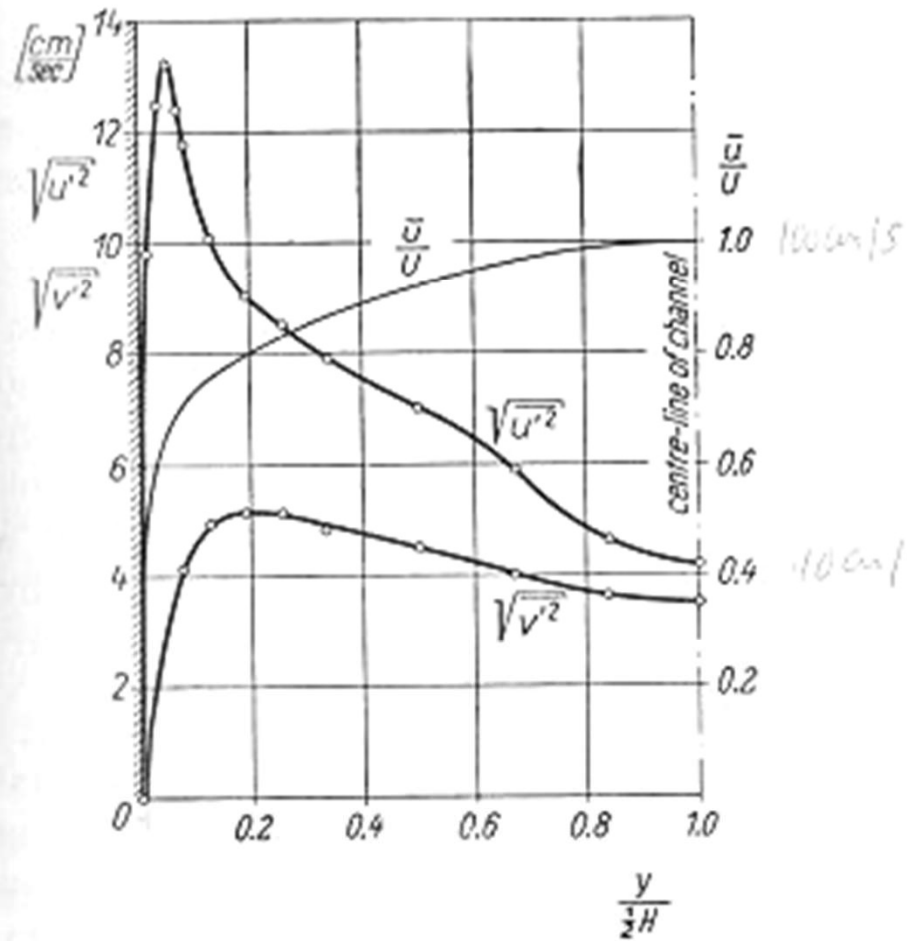


Fig. 18.3. Measurement of fluctuating turbulent components in a wind tunnel, at maximum velocity $U = 100 \text{ cm/sec}$ after Reichardt [41]

Root-mean-square of longitudinal fluctuation $\sqrt{u'^2}$, transverse fluctuation $\sqrt{v'^2}$, mean velocity \bar{u}

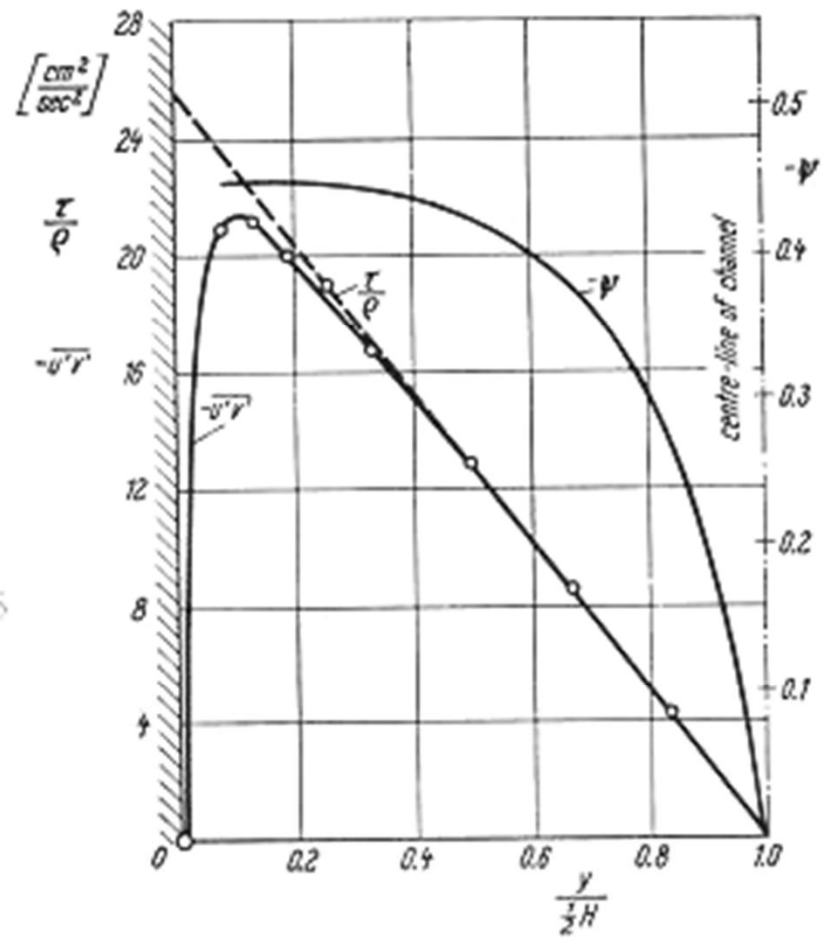
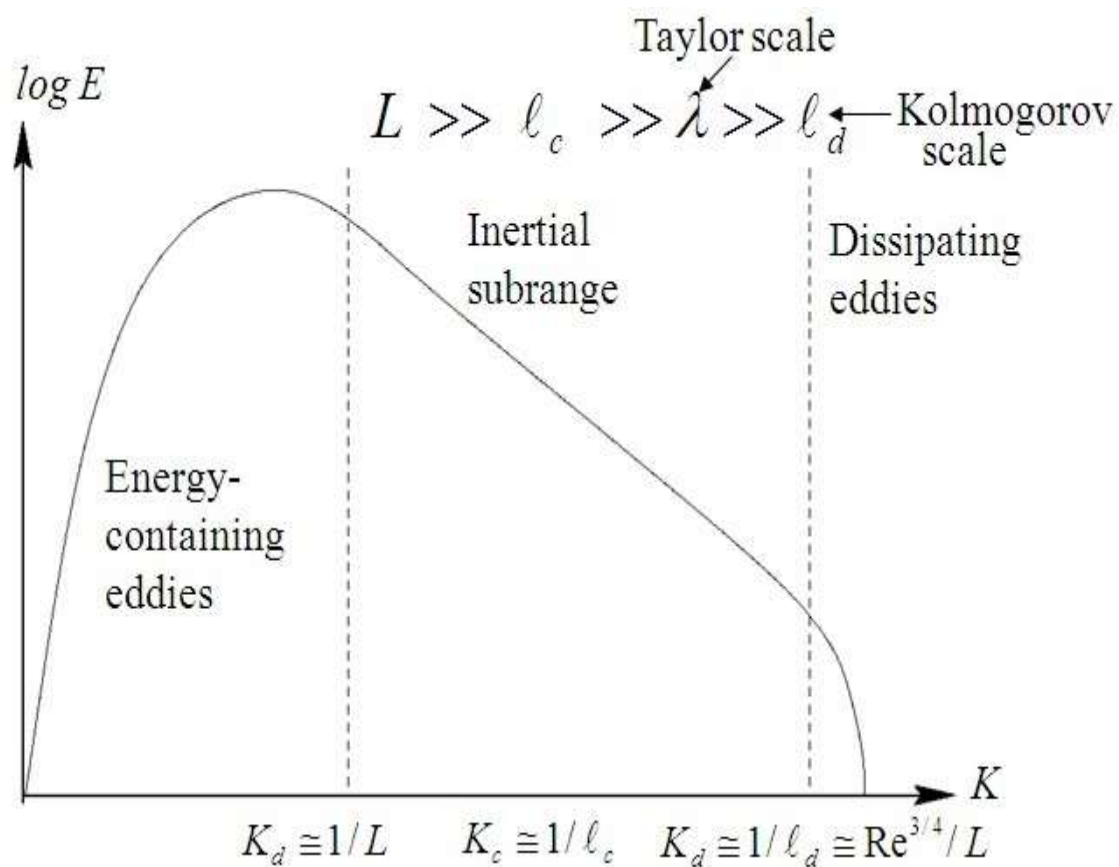
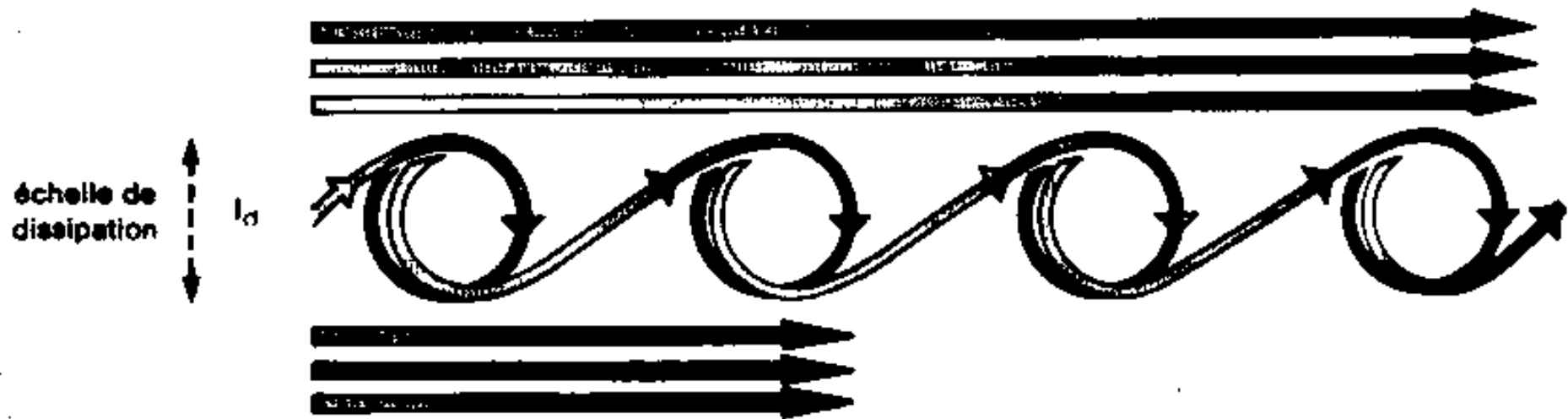
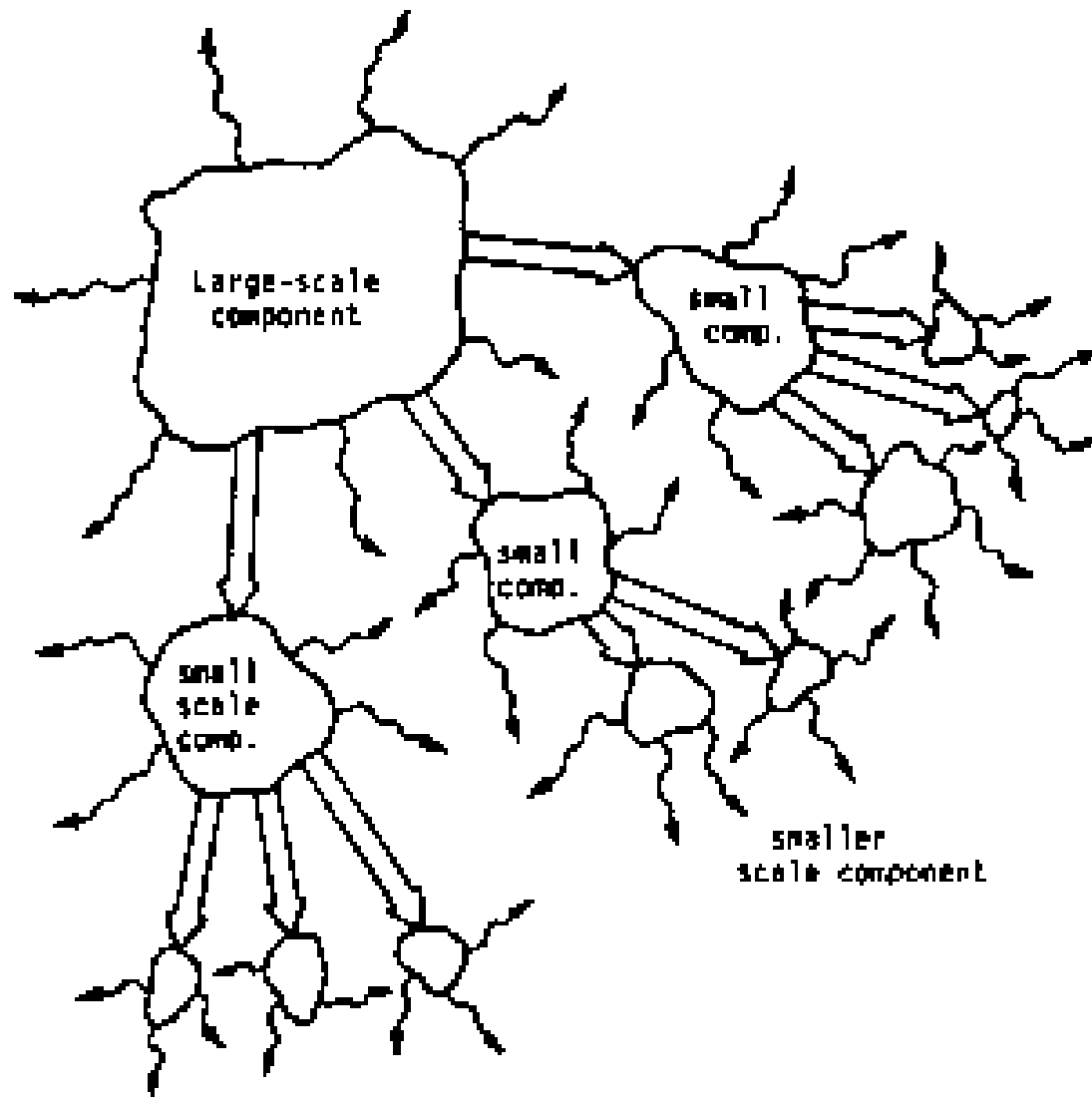


Fig. 18.4. Measurement of fluctuating components in a channel, after Reichardt [41]

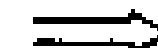

The product $\overline{u'v'}$, the shearing stress τ/ρ , and the correlation coefficient ψ

Intensidade de Turbulência



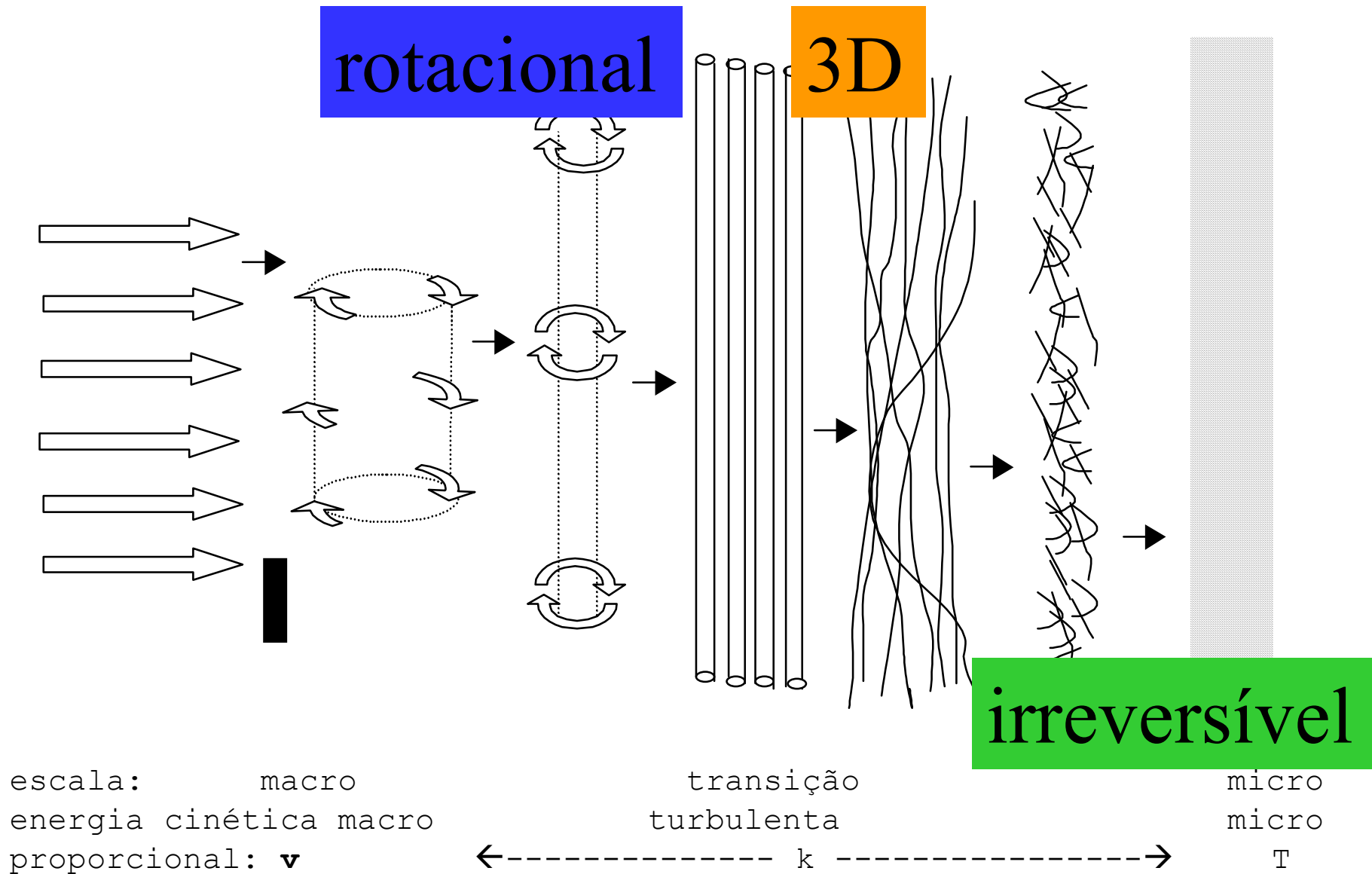


Distribution of energy between eddies

 denotes energy transfer between eddies
 denotes energy dissipated by the action of viscosity

recherche 139 p. 1422

Kolmogorov



escalas de turbulências

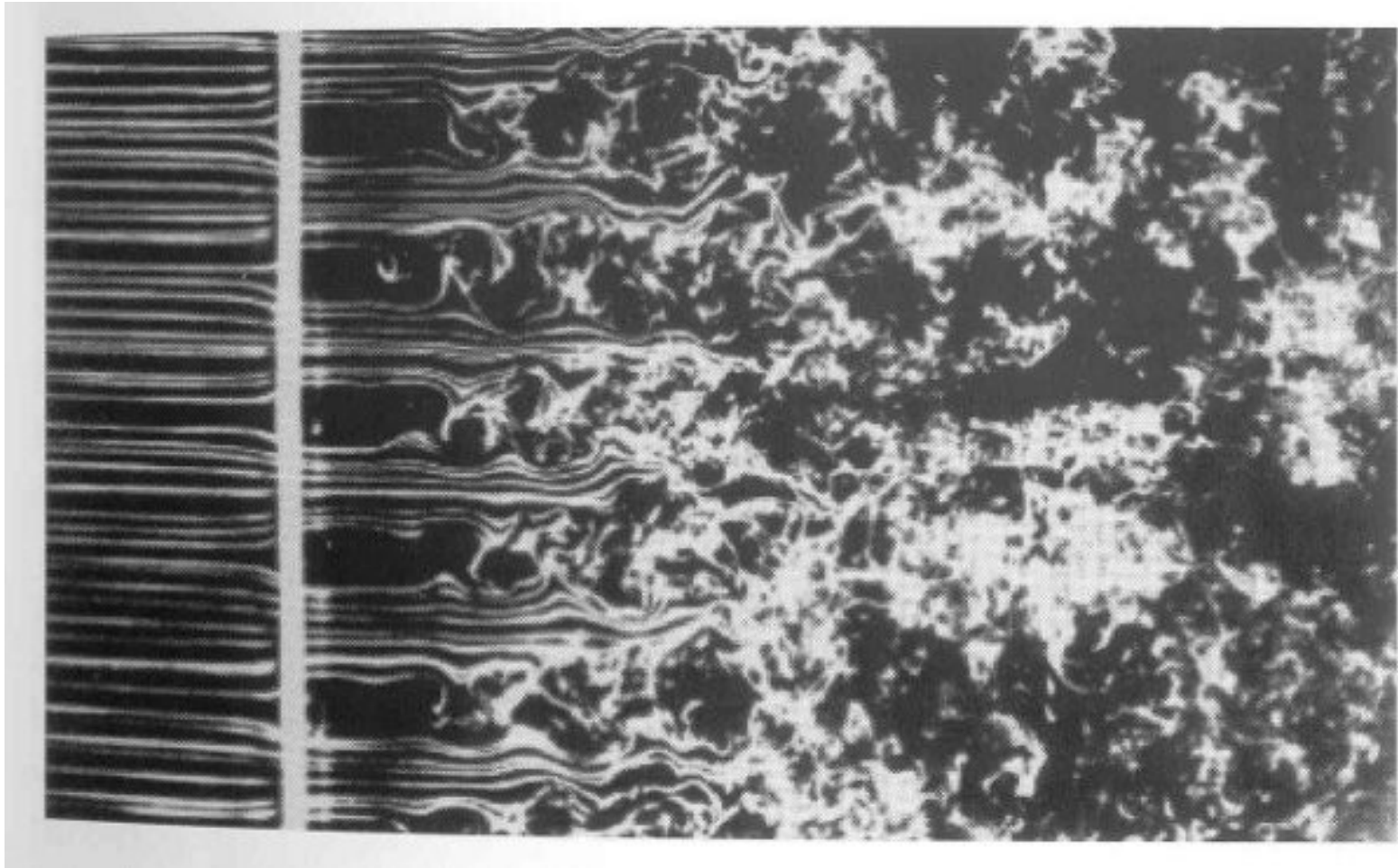
Escalas de Kolmogorov – menores escalas de turbulência

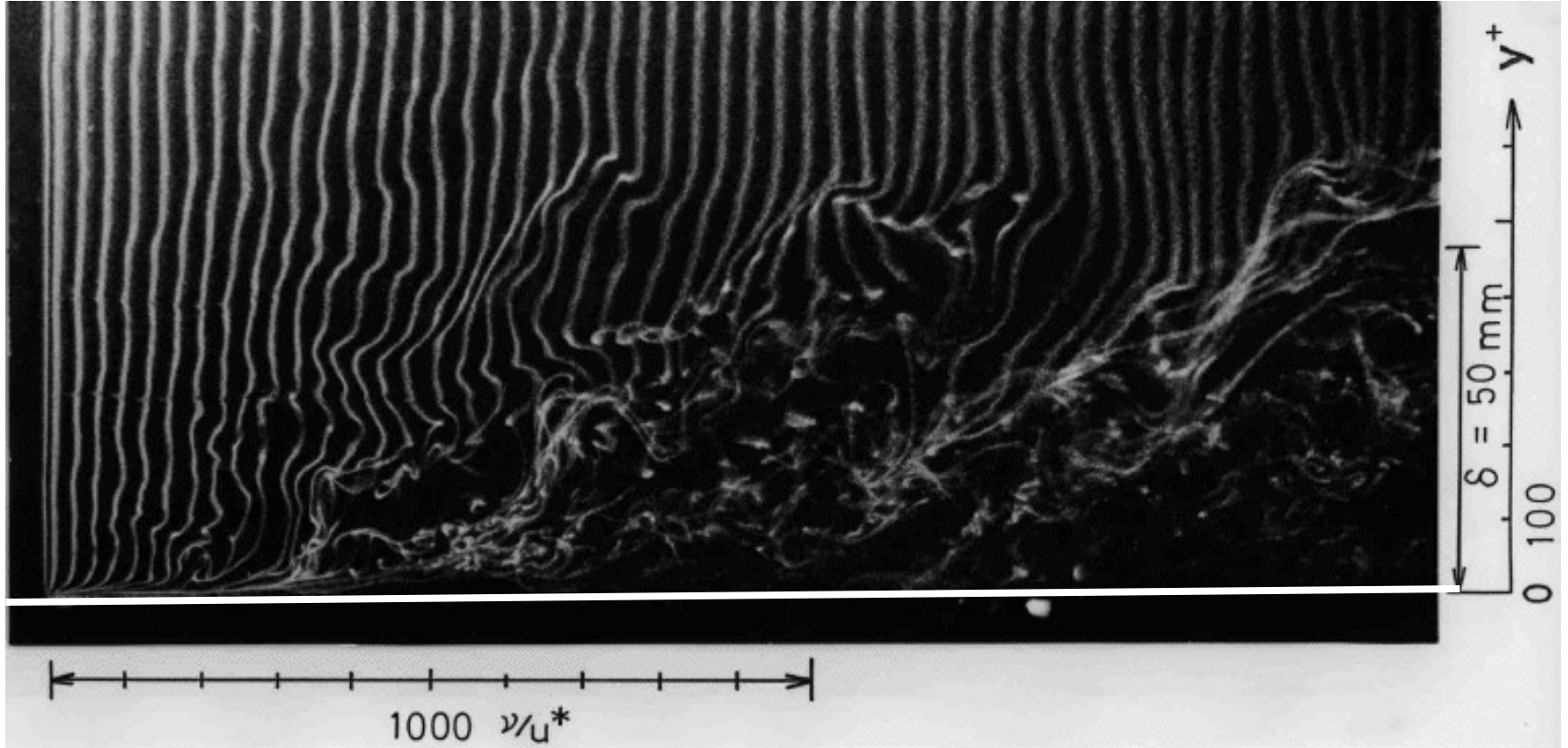
- Comprimento $\eta = \left(\frac{\nu^3}{\varepsilon} \right)^{1/4}$ $\frac{\eta}{l_0} = \text{Re}^{-3/4}$
- Velocidade $u_\eta = (\nu\varepsilon)^{1/4}$ $\frac{u_\eta}{u_0} = \text{Re}^{-1/4}$
- Tempo $\tau_\eta = \left(\frac{\nu}{\varepsilon} \right)^{1/2}$ $\frac{\tau_\eta}{\tau_0} = \text{Re}^{-1/2}$
- Reynolds $\text{Re}_\eta = \frac{u_\eta \eta}{\nu} = 1$

transition to turbulence



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sendo a difusividade turbulenta >> laminar:

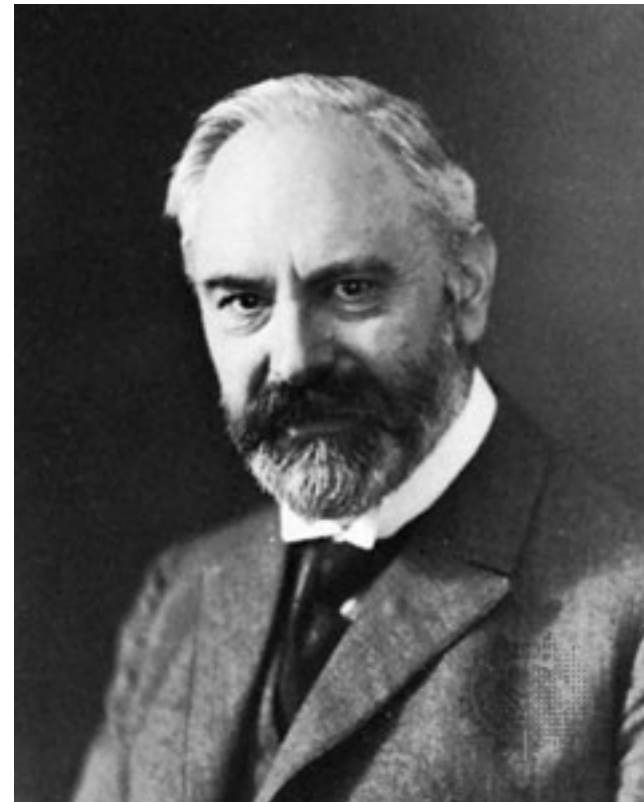
$$[\lambda_\phi + \lambda_{\phi T}] \approx \lambda_{\phi T}$$

$$\frac{\partial \bar{\phi}}{\partial t} + \text{div } \vec{v} \bar{\phi} = \text{div } \lambda_{\phi T} \text{ grad } \bar{\phi} + \overline{\dot{\sigma} M_\phi}$$

voltou à poderosa
agora super-poderosa


$$\lambda_{\vec{v}T} = \ell_m^2 \left| \text{grad } \vec{v} \right|$$

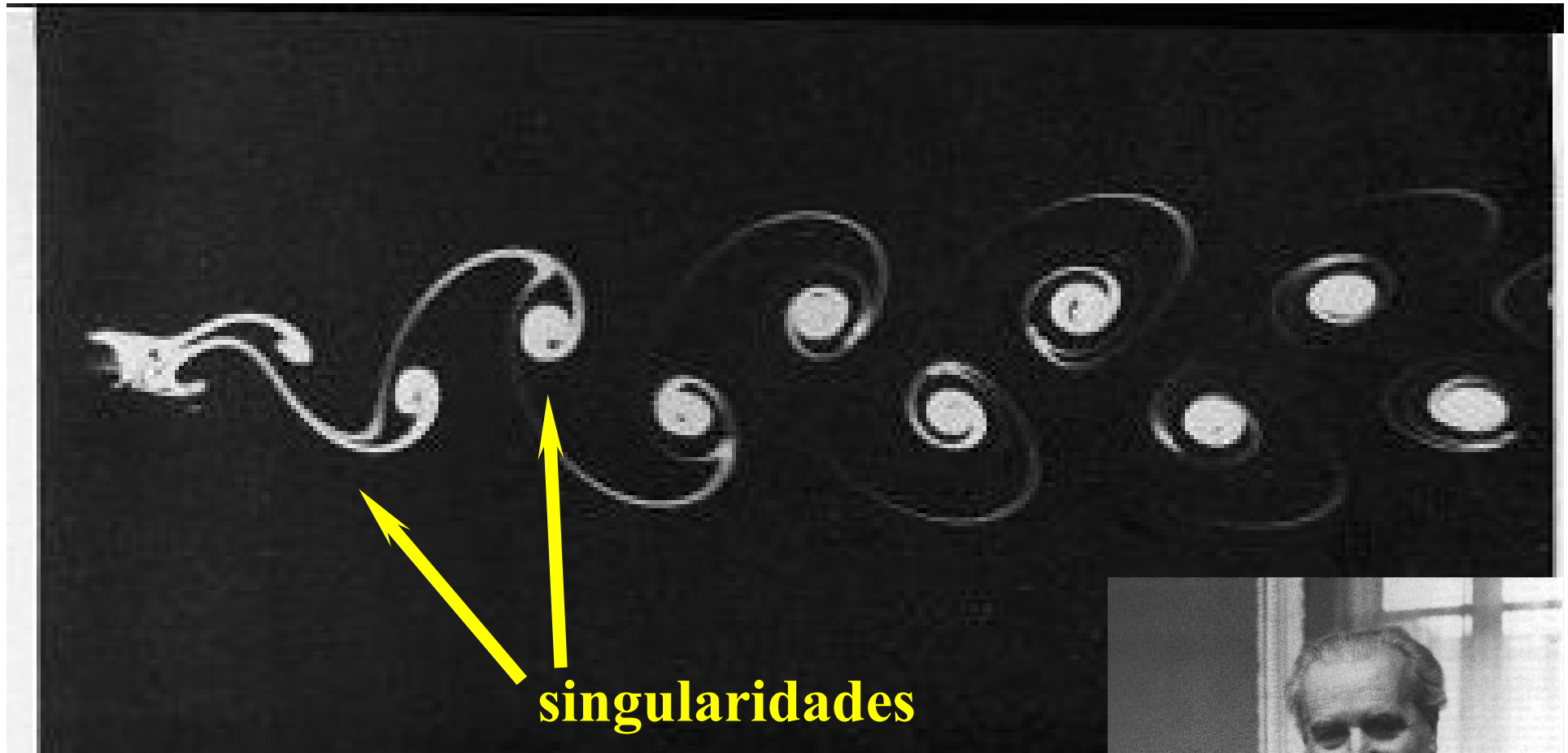
Prandtl mixing length



sendo $\varphi = v$:

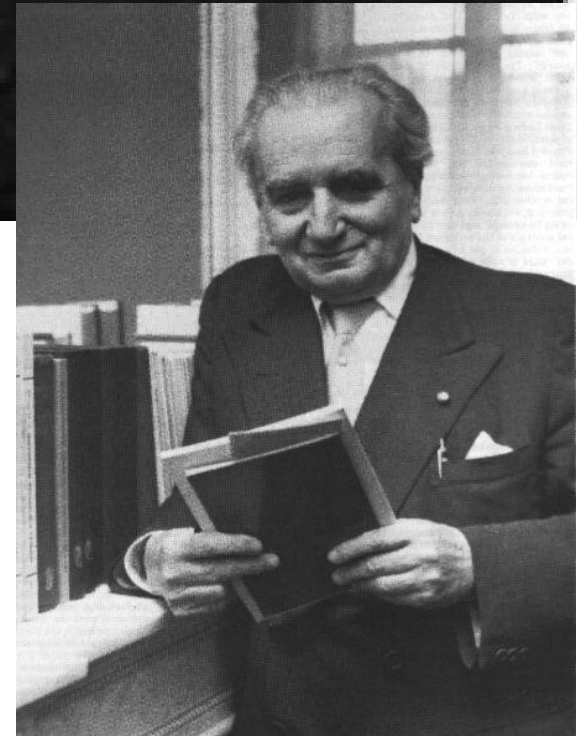
$$\frac{\partial \bar{\vec{v}}}{\partial t} + \text{div} \bar{\vec{v}} \bar{\vec{v}} = \text{div} \bar{\nu}_T \text{grad} \bar{\vec{v}} - \frac{\text{grad} \bar{p}}{\rho} + \bar{\vec{g}}$$

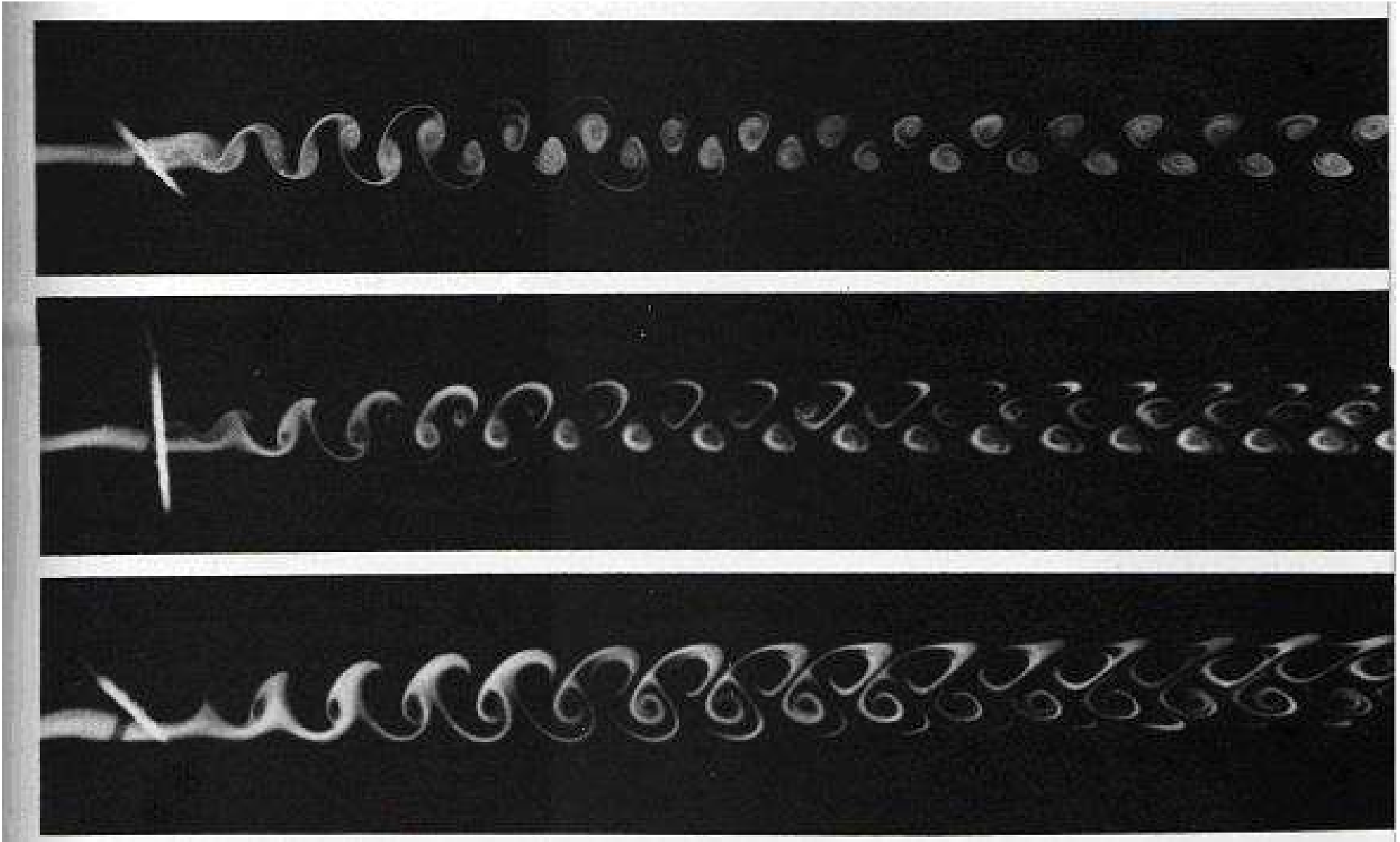
Prandtl mixing length	$\lambda_{\bar{\nu}_T} = \nu_T = \frac{\mu_T}{\rho} = l_m^2 \left \text{grad} \bar{\vec{v}} \right $
similaridade Prandtl	$\nu_T = c_\mu l \sqrt{K}$
k	 $k = \overline{e_{c_T}} = \frac{\overline{\vec{v}' \cdot \vec{v}'}}{2}$
	$\frac{D k}{Dt} = \text{div} \left(\frac{\nu_T}{\sigma_k} \text{grad} k \right) + \nu_T \left \text{grad} \bar{\vec{v}} \right ^2 - c_D \frac{k^{3/2}}{l}$
k ε	$\nu_T = c_\mu K^2 / \epsilon$



cilindro
 $Re = 105$

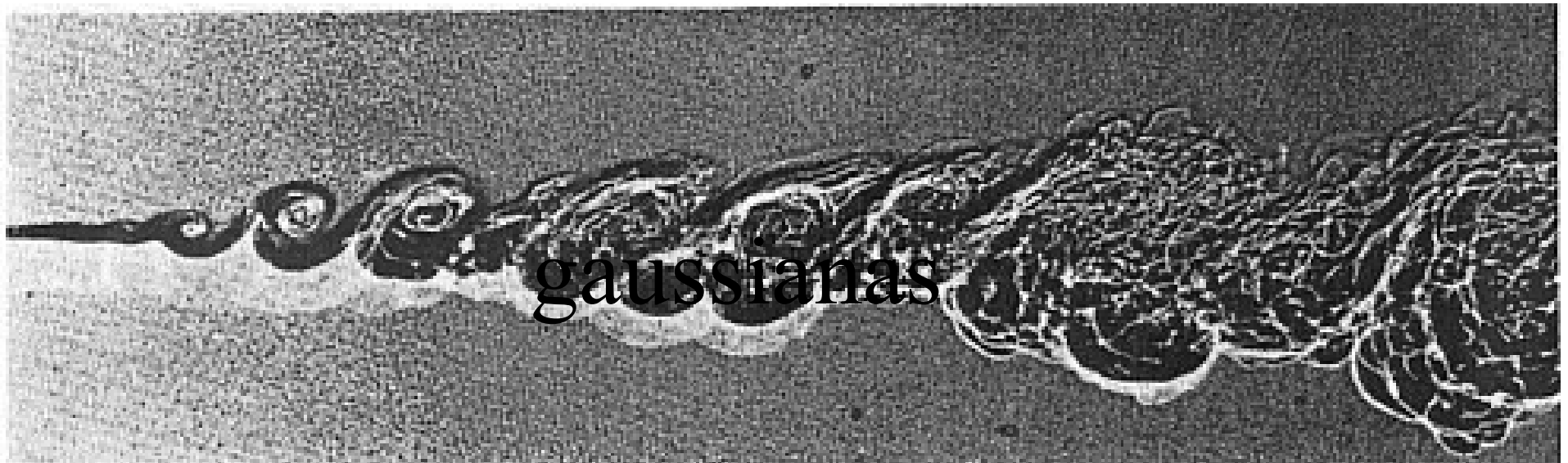
Von Karman
vortex





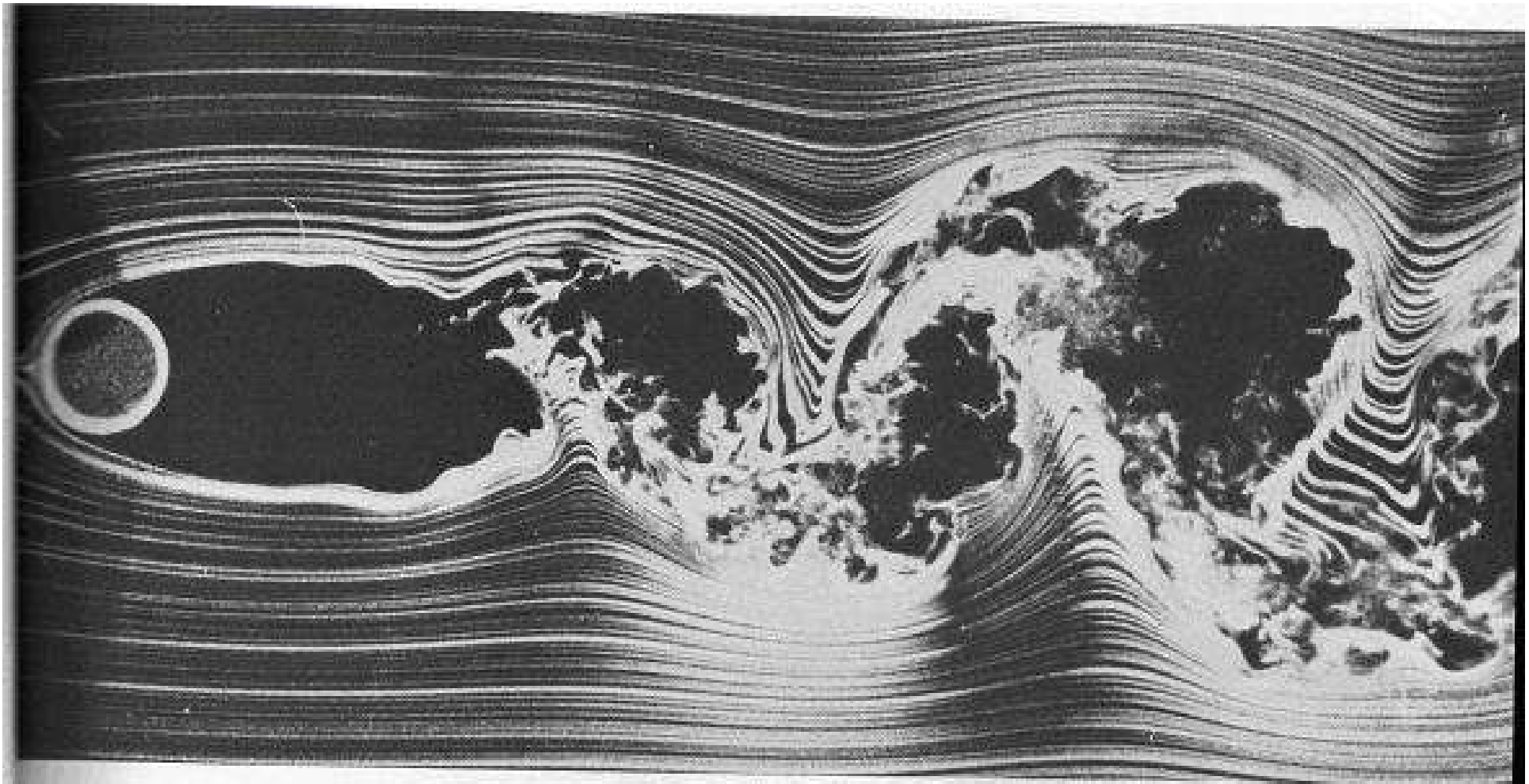
vortex

$Re = 100$



cilindro

$Re = 10.000$

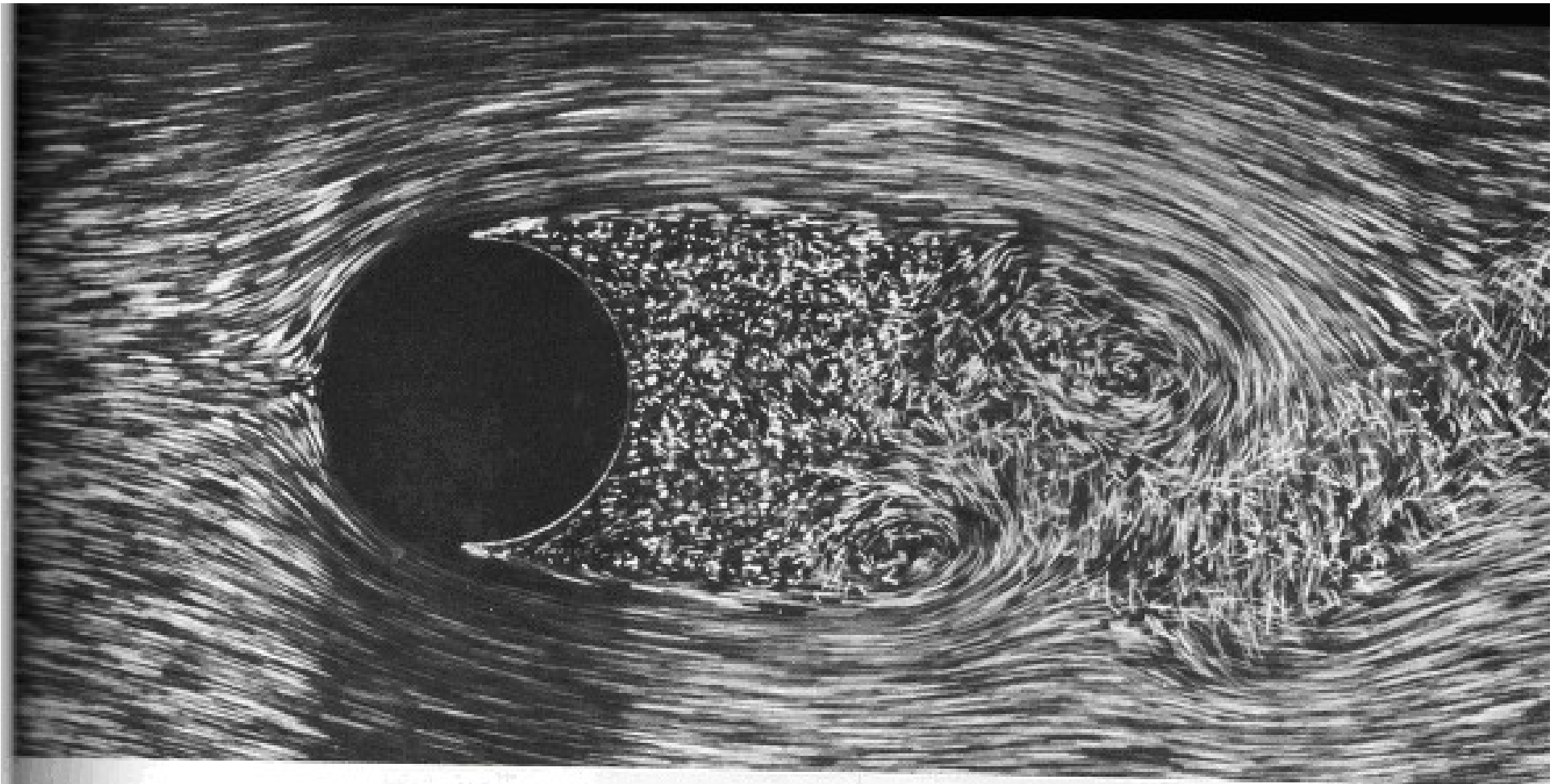


48. Circular cylinder at $R=10,000$. At five times the speed of the photograph at the top of the page, the flow pattern is scarcely changed. The drag coefficient consequently remains almost constant in the range of Reynolds

number spanned by these two photographs. It drops later when, as in figure 57, the boundary layer becomes turbulent at separation. Photograph by Thomas Corke and Hassan Nagib

cilindro

$Re = 2000$



47. Circular cylinder at $R=2000$. At this Reynolds number one may properly speak of a boundary layer. It is laminar over the front, separates, and breaks up into a turbulent wake. The separation points, moving forward as

the Reynolds number is increased, have now attained their upstream limit, ahead of maximum thickness. Visualization is by air bubbles in water. ONERA photograph, Werlé & Gallon 1972

$$v_z = \frac{\Delta p R^2}{2\mu L} \left[1 - \left(\frac{r}{R} \right)^2 \right]$$

$$\frac{v_z}{v_0} = 1 - \left(\frac{r}{R} \right)^2$$

$$\mathbf{V}_{\text{bulk}} = \dot{m} v_b = \rho v_b \pi R^2 = \int_0^R \rho v_z 2\pi r dr$$

$$v_b = \frac{\rho 2\pi}{\rho \pi R^2} \int_0^R v_z r dr = \frac{2}{R^2} \frac{\Delta p R^2}{2\mu L} \int_0^R \left(1 - \left(\frac{r}{R} \right)^2 \right) r dr$$

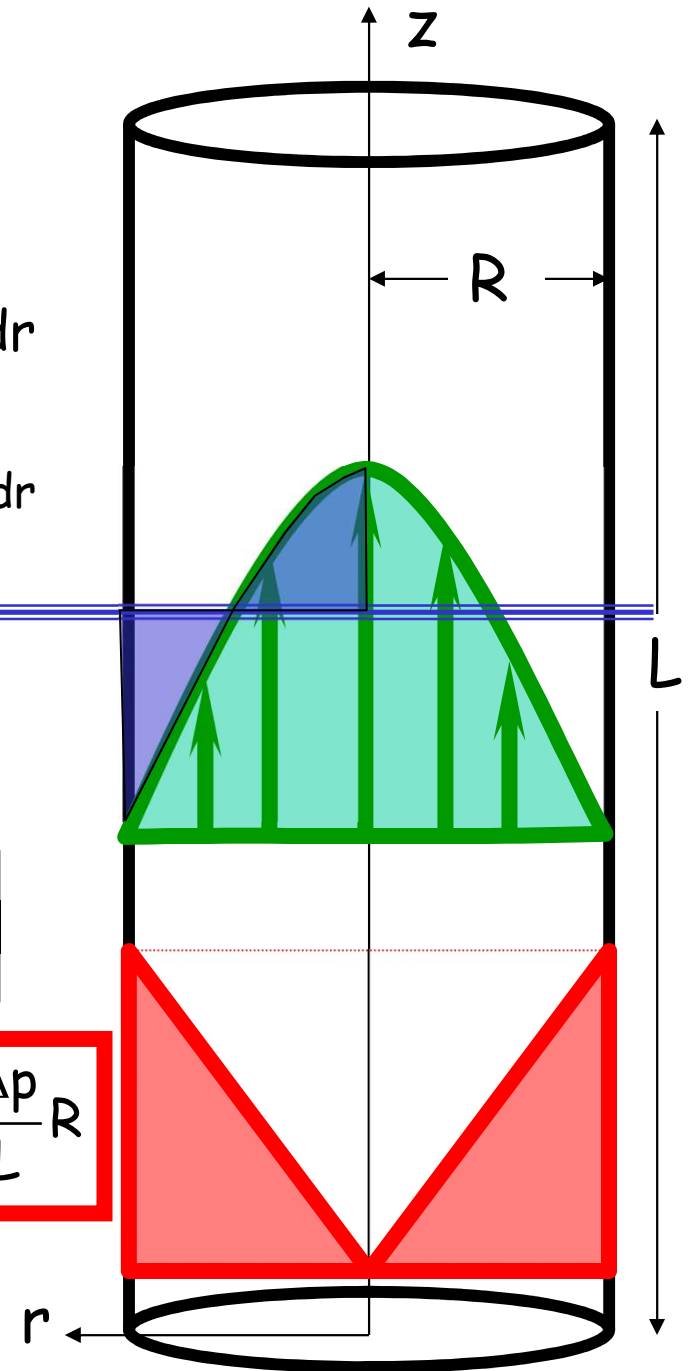
$$v_b = \frac{R^2 \Delta p}{4\mu L}$$

tensão

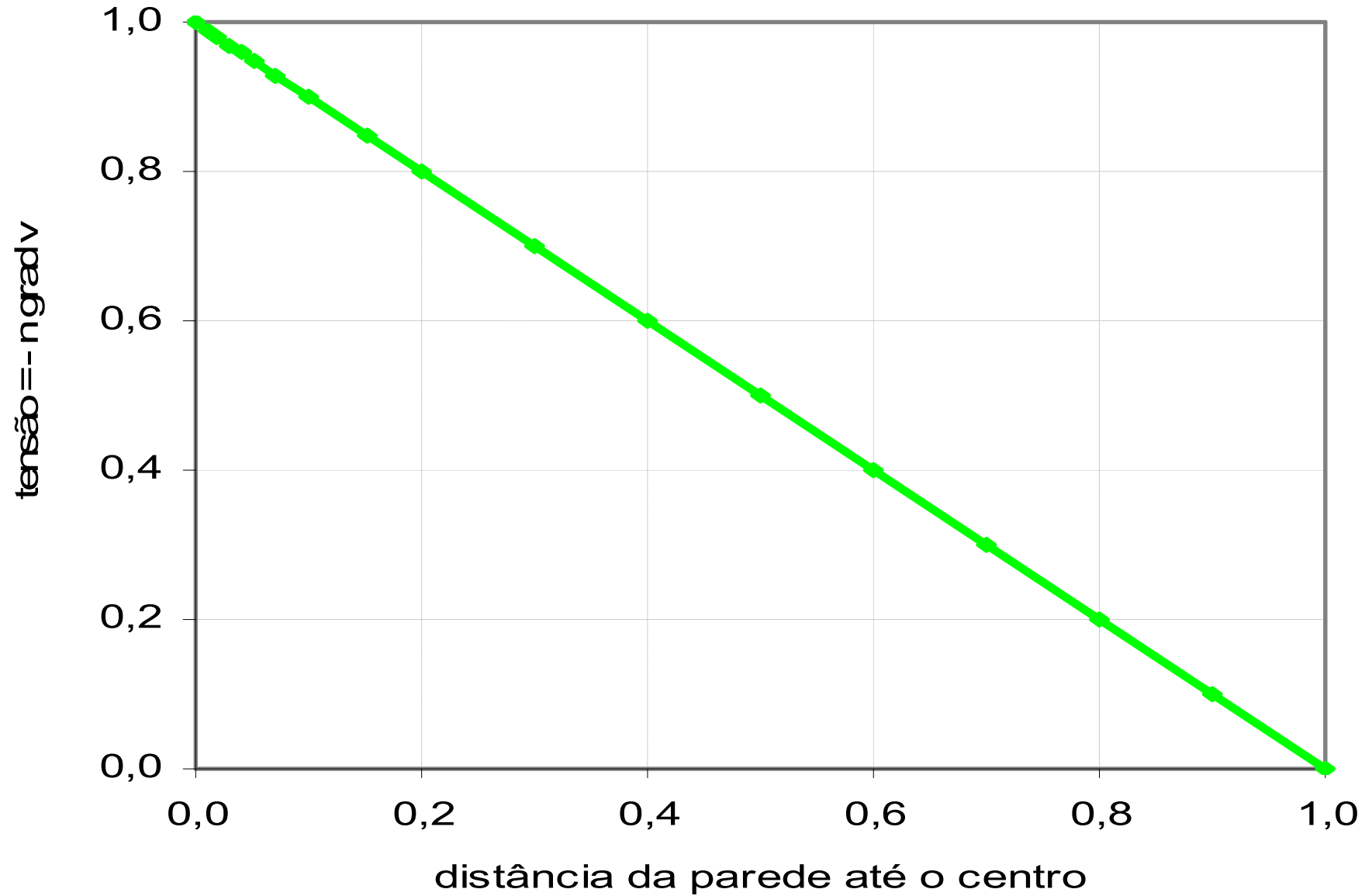
$$\zeta_{rz} = -\mu \frac{\partial v_z}{\partial r} = -\mu \frac{\partial}{\partial r} \left[\frac{\Delta p}{2\mu L} (R^2 - r^2) \right]$$

$$\zeta_{rz} = \frac{\Delta p}{L} r$$

$$\zeta_w = \frac{\Delta p}{L} R$$

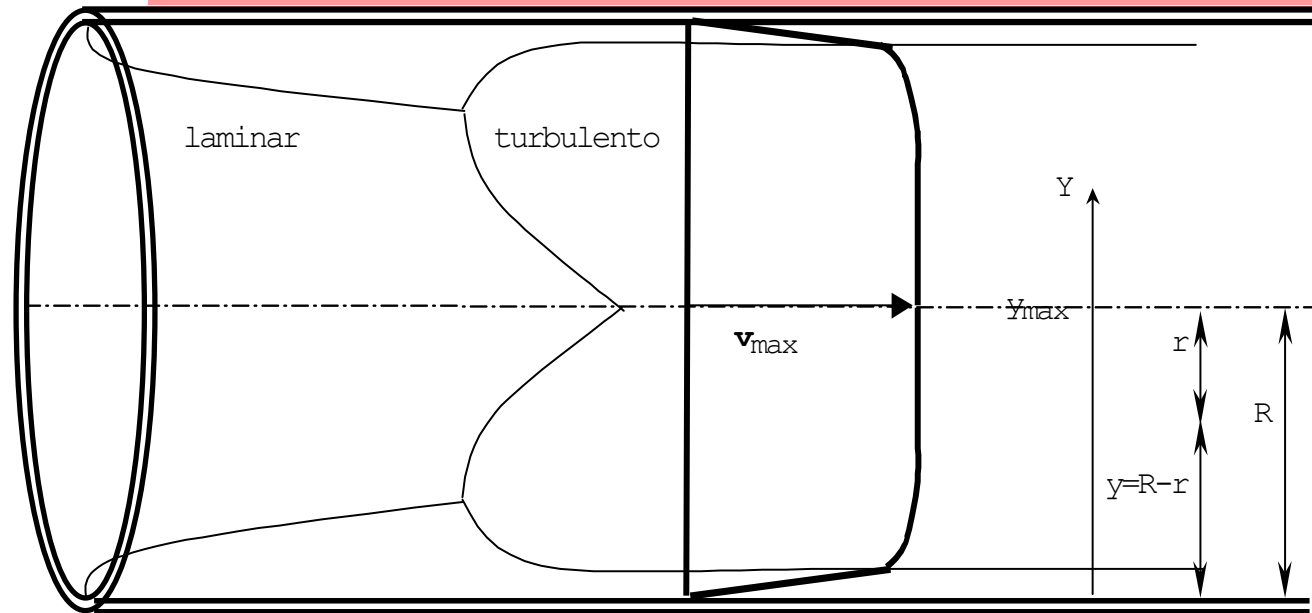


gradiente de velocidade adimensionalizado (laminar)



Turbulento

Escoamento unidimensional em estado estacionário sobre uma superfície (parede plana, tubo escoamento desenvolvido)



$$\frac{\partial \bar{\phi}}{\partial t} + \text{div } \bar{\vec{v}} \bar{\phi} = \text{div} [\lambda_{\phi} + \lambda_{\phi T}] \text{grad } \bar{\phi} + \bar{\dot{q}}_{M\phi}$$

$$\lambda_{\vec{v}T} = \nu_T = \ell_m^2 |\text{grad } \bar{\vec{v}}| \gg \lambda_{\vec{v}} = \nu$$

ℓ_m = comprimento de mistura de Prandtl

$$\text{div } \bar{\vec{v}} \bar{\vec{v}} = \text{div} \left\{ \ell_m^2 |\text{grad } \bar{\vec{v}}| \text{grad } \bar{\vec{v}} \right\}$$

$$\text{div } \bar{\vec{v}} \bar{\vec{v}} = \text{div} \left\{ l_m^2 \left| \text{grad } \bar{\vec{v}} \right| \text{grad } \bar{\vec{v}} \right\}$$

Próximo à parede (w)

$$l_m = \alpha y$$

$$\left\{ \text{div } \bar{\vec{v}} \bar{\vec{v}} \right\}_w = \text{div} \left\{ \alpha^2 y^2 \left(\frac{\partial \bar{v}_z}{\partial y} \right)^2 \right\} = \frac{1}{\rho} \text{div } \bar{\vec{\tau}}_w$$

$$\alpha^2 y^2 \left(\frac{\partial \bar{v}_z}{\partial y} \right)^2 = \frac{\tau_w}{\rho} \quad \rightarrow \quad y \frac{\partial \bar{v}_z}{\partial y} = \frac{1}{\alpha} \sqrt{\frac{\tau_w}{\rho}}$$

$$d \bar{v}_z = \frac{1}{\alpha} \sqrt{\frac{\tau_w}{\rho}} \frac{dy}{y} \quad \bar{v}_z = \frac{\sqrt{\tau_w / \rho}}{\alpha} \ln y + \text{cte}'$$

$$\bar{v}_z^+ = \frac{\bar{v}_z}{\sqrt{\tau_w / \rho}}$$

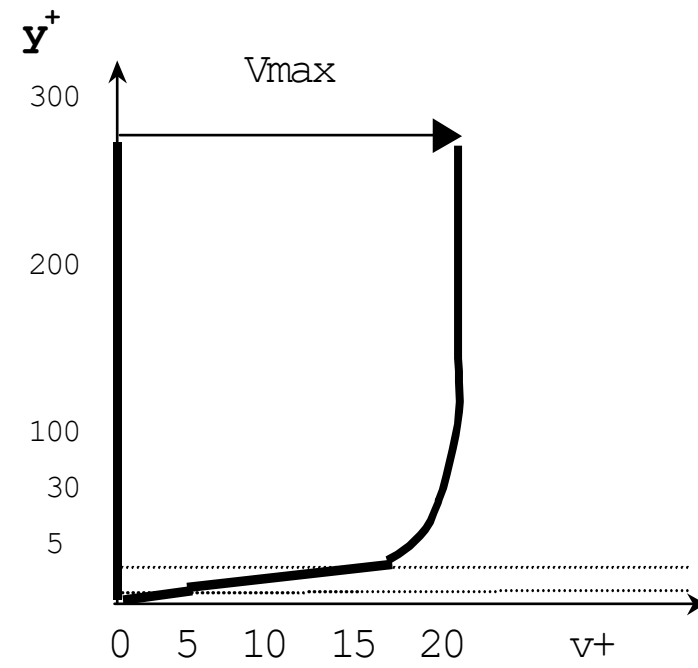
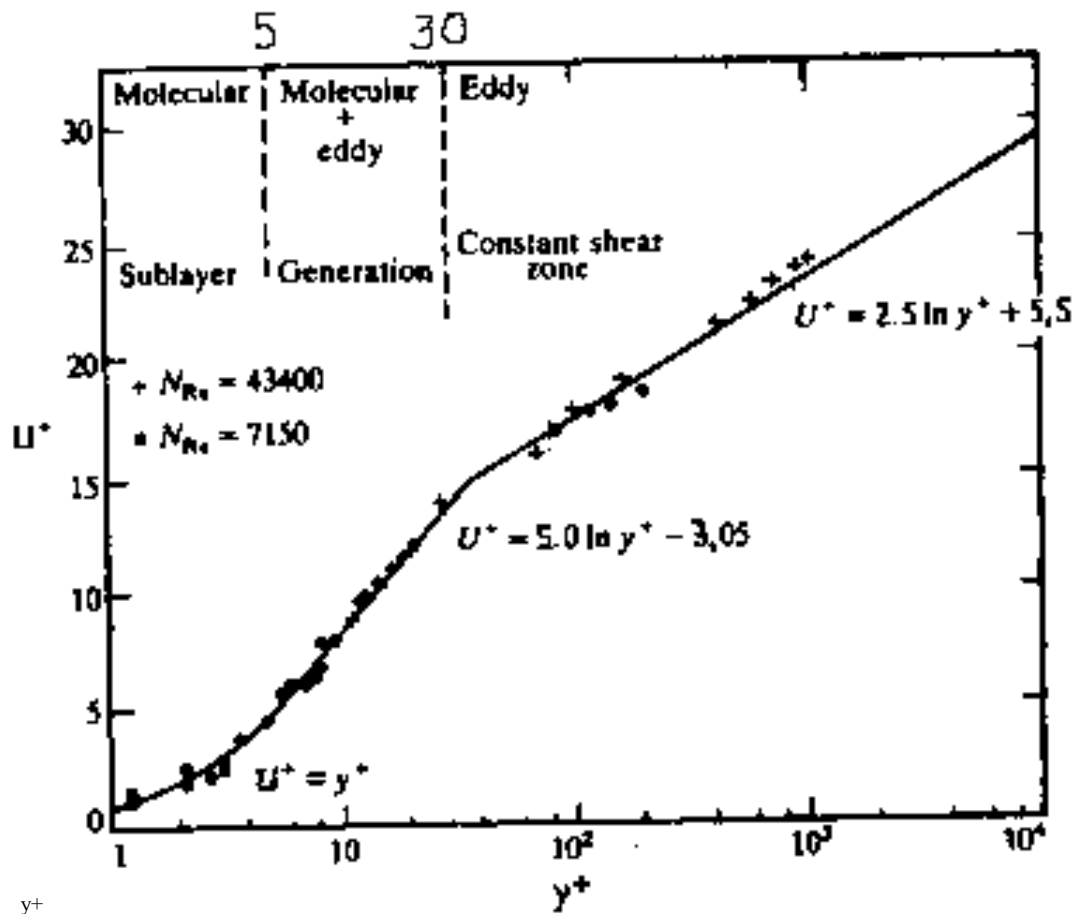
$$y^+ = \frac{y}{\nu} \sqrt{\frac{\tau_w}{\rho}} = \frac{y}{\mu} \sqrt{\rho \tau_w}$$

$$u_t = \sqrt{\tau_w / \rho}$$

“drift velocity”

$$\bar{v}_z^+ = \frac{1}{\alpha} \ln y^+ + \text{cte}$$

Região	subcamada	transição	"Core" turbulento
Mecanismo	molecular	molecular + eddy	eddy
propriedade	λ_Φ	$\lambda_\Phi + \lambda_{\Phi T}$	$\lambda_{\Phi T}$
v^+	y^+	$5 \ln y^+ - 3,05$	$2,5 \ln y^+ + 5,5$
y^+ min	0	5	30
y^+ max	5	30	∞



y^+

Perfil Universal de Velocidades

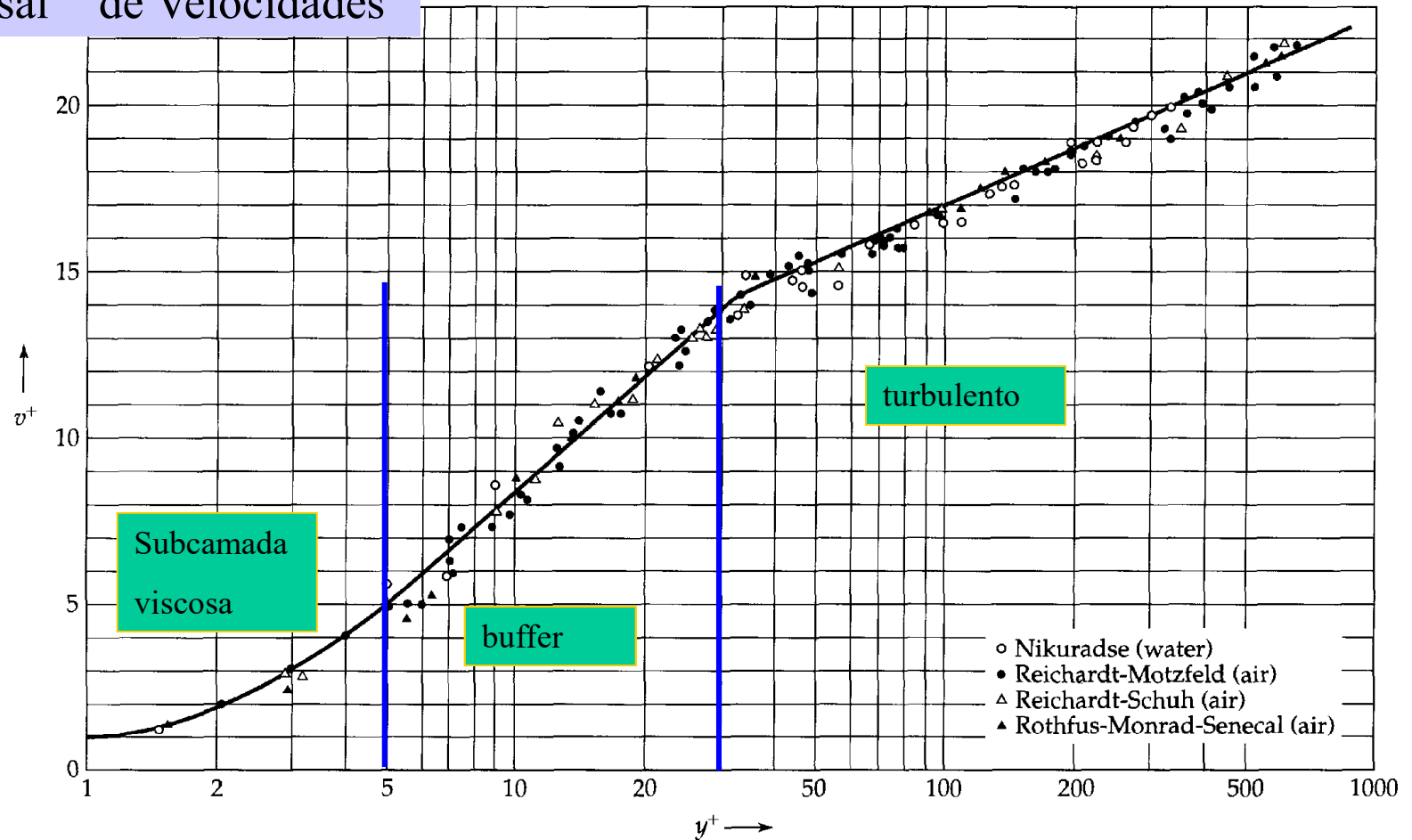


Fig. 5.5-3. Dimensionless velocity distribution for turbulent flow in circular tubes, presented as $v^+ = \bar{v}_z/v_*$ vs. $y^+ = yv_*\rho/\mu$, where $v_* = \sqrt{\tau_0/\rho}$ and τ_0 is the wall shear stress. The solid curves are those suggested by Lin, Moulton, and Putnam [*Ind. Eng. Chem.*, **45**, 636–640 (1953)]:

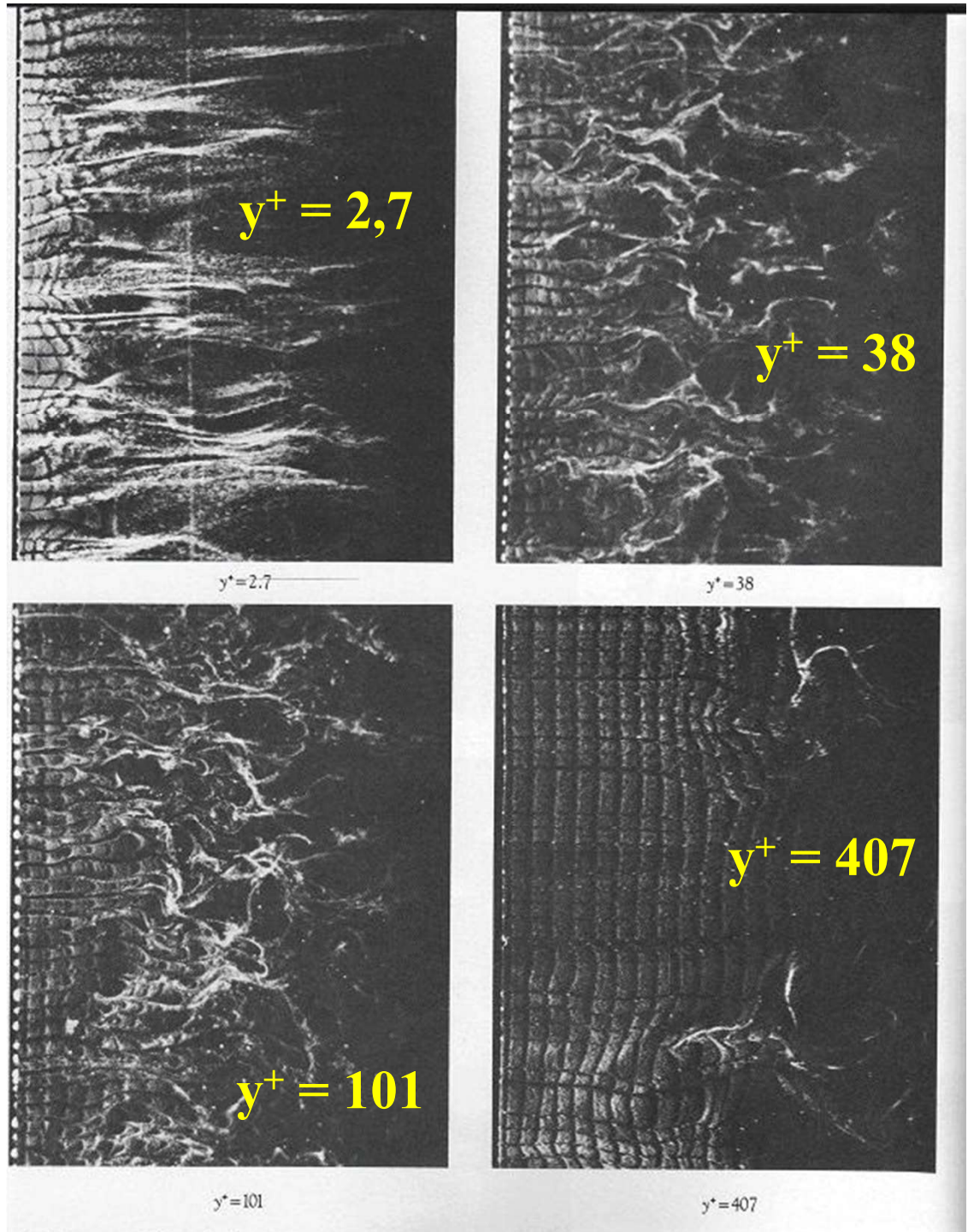
$$\begin{aligned}
 0 < y^+ < 5: & \quad v^+ = y^+ \left[1 - \frac{1}{4} (y^+/14.5)^3 \right] \\
 5 < y^+ < 30: & \quad v^+ = 5 \ln(y^+ + 0.205) - 3.27 \\
 30 < y^+: & \quad v^+ = 2.5 \ln y^+ + 5.5
 \end{aligned}$$

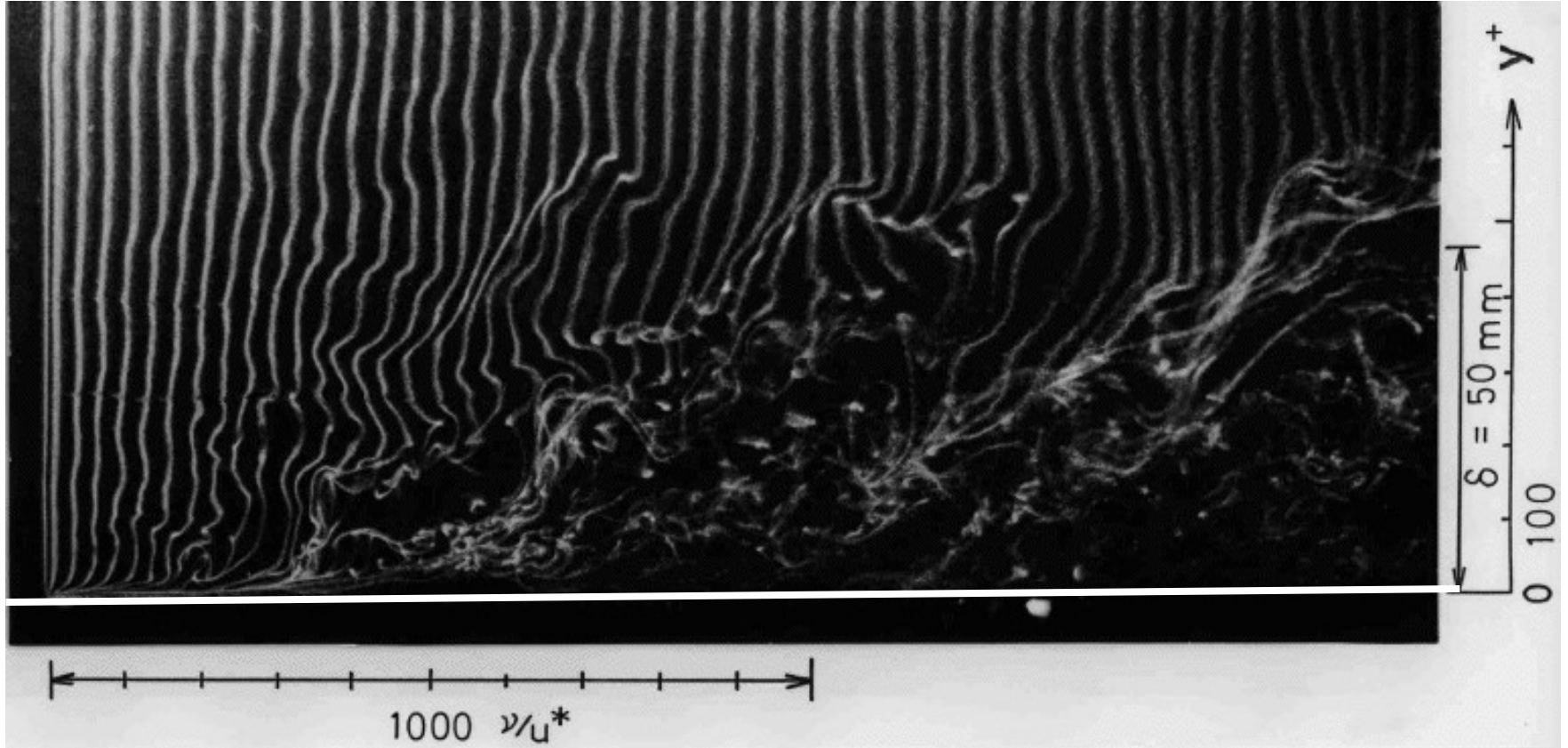
The experimental data are those of J. Nikuradse for water (○) [*VDI Forschungsheft*, **H356** (1932)]; Reichardt and Motzfeld for air (●); Reichardt and Schuh (△) for air [*H. Reichardt, NACA Tech. Mem. 1047* (1943)]; and R. R. Rothfus, C. C. Monrad, and V. E. Senecal for air (■) [*Ind. Eng. Chem.*, **42**, 2511–2520 (1950)].

161. Structure of a turbulent boundary layer. Successive layers of the flow near a flat plate in a water channel are shown by tiny hydrogen bubbles released periodically from a thin platinum wire seen at the left. The height $y^* = y u_* / \nu$ of the wire above the plate is shown in wall variables, where $u_* = (\tau_w / \rho)^{1/2}$ is the friction velocity. The

characteristic low- and high-speed streaks shown in the viscous sublayer at $y^* = 2.7$ become less noticeable farther away, and have disappeared in the logarithmic region at $y^* = 101$. In the wake region at $y^* = 407$ the turbulence is seen to be intermittent and of larger scale. Kline, Reynolds, Schraub & Runstadler 1967

y^+

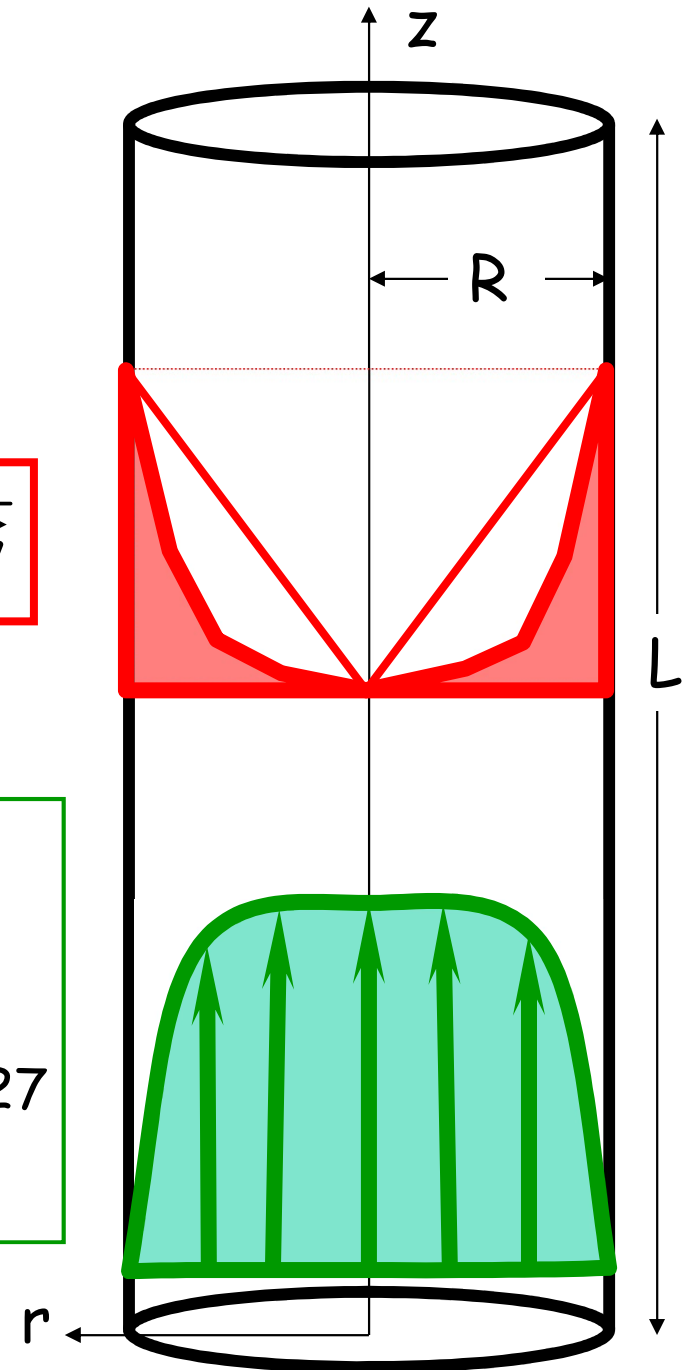




Turbulento

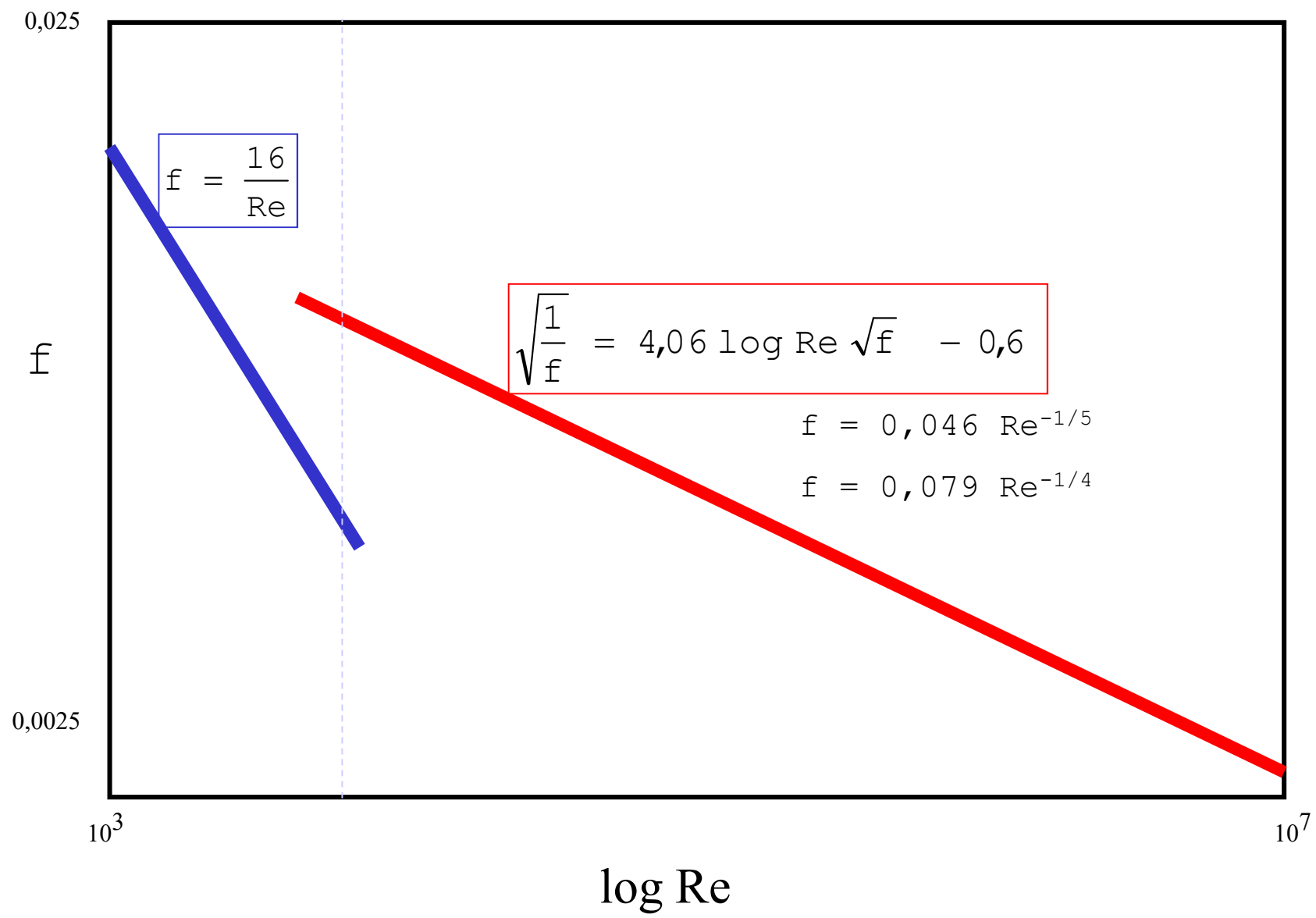
$$\vec{\zeta} = \nu \operatorname{grad} \vec{v} - \overline{\vec{v}' \vec{v}'} = \nu_T \operatorname{grad} \vec{v}$$

$$\begin{array}{ll} 0 > y^+ > 5 & \bar{v}^+ = y^+ \left[1 - \frac{1}{4} \left(\frac{y^+}{14,5} \right)^3 \right] \\ 5 > y^+ > 30 & \bar{v}^+ = 5 \ln(y^+ + 0,205) - 3,27 \\ y^+ > 30 & \bar{v}^+ = 2,5 \ln y^+ + 5,5 \end{array}$$



fator de atrito

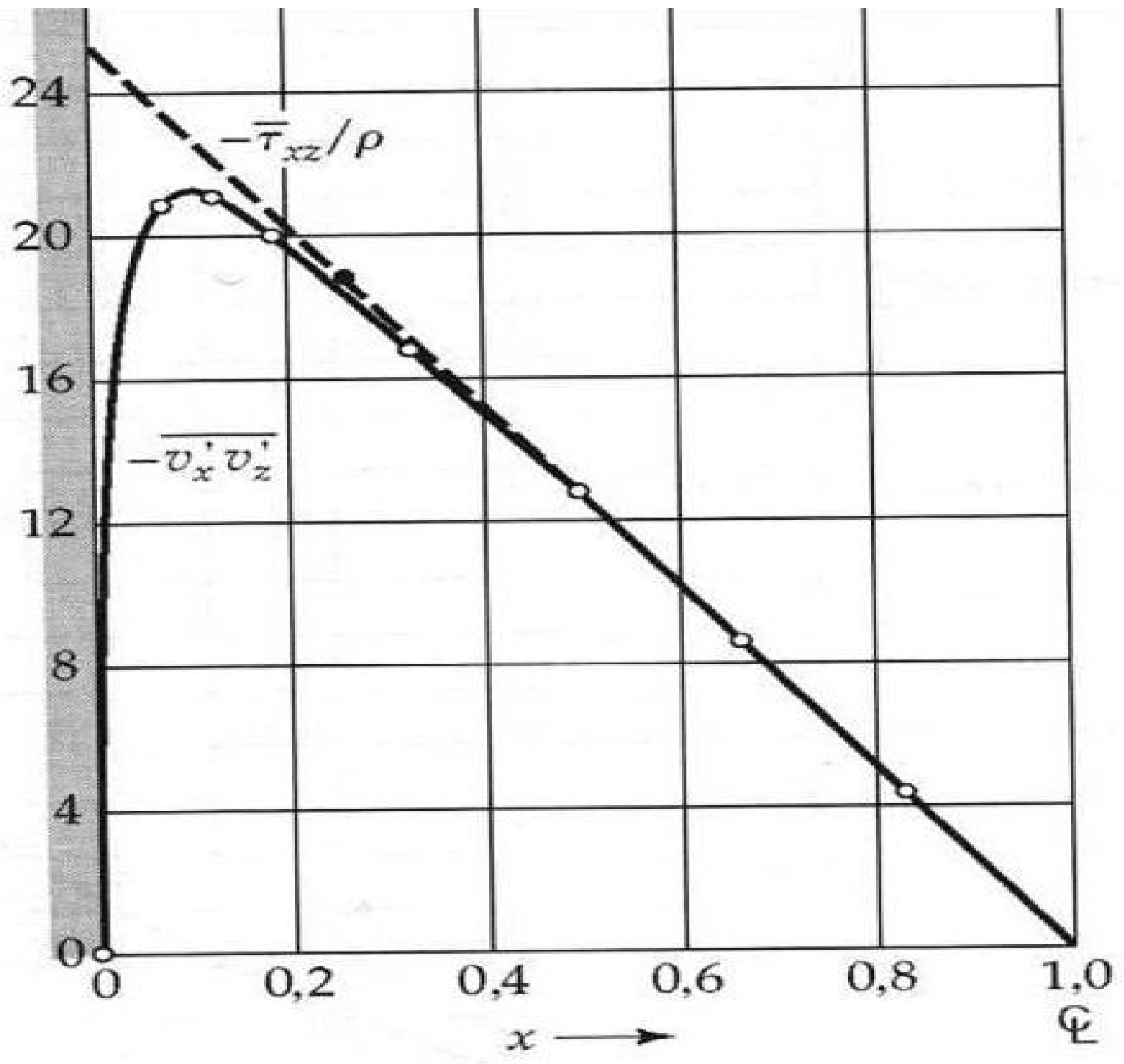
$$\tau_w = \frac{f}{2} \rho v_b^2$$



Escoamento
turbulento

Tensão

$z \uparrow$



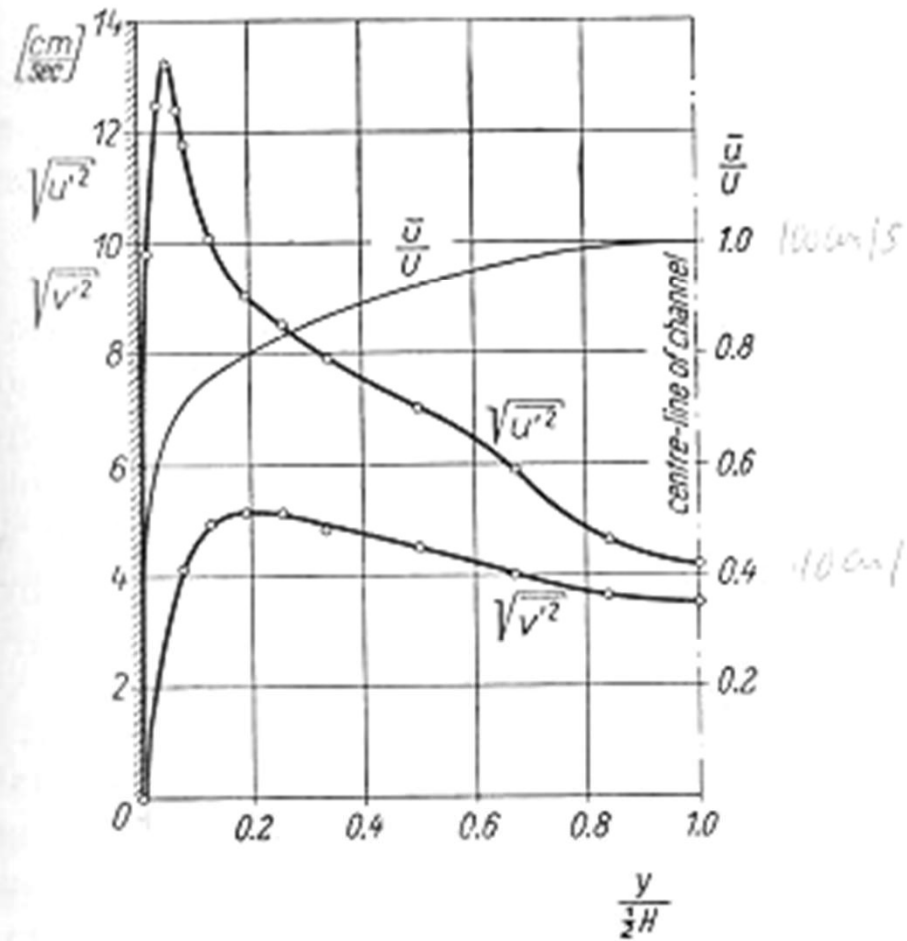


Fig. 18.3. Measurement of fluctuating turbulent components in a wind tunnel, at maximum velocity $U = 100 \text{ cm/sec}$ after Reichardt [41]

Root-mean-square of longitudinal fluctuation $\sqrt{u'^2}$, transverse fluctuation $\sqrt{v'^2}$, mean velocity \bar{u}

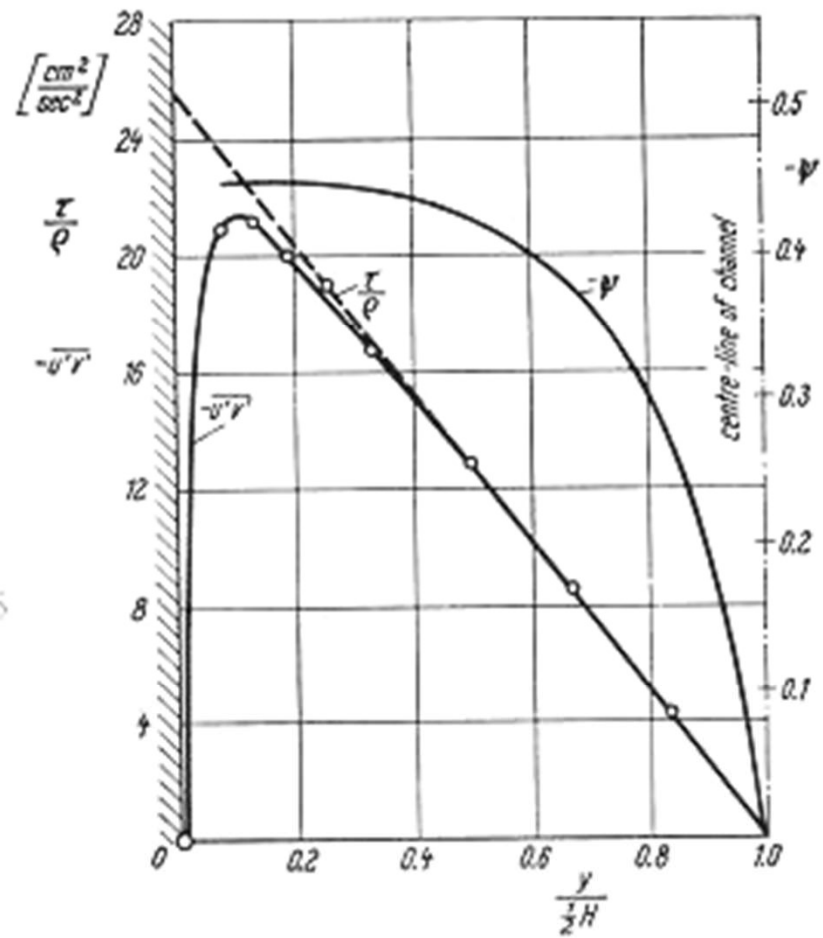


Fig. 18.4. Measurement of fluctuating components in a channel, after Reichardt [41]

The product $\overline{u'v'}$, the shearing stress τ/ρ , and the correlation coefficient ψ

Intensidade de Turbulência

Comprimento
o de Mistura
- Prandtl

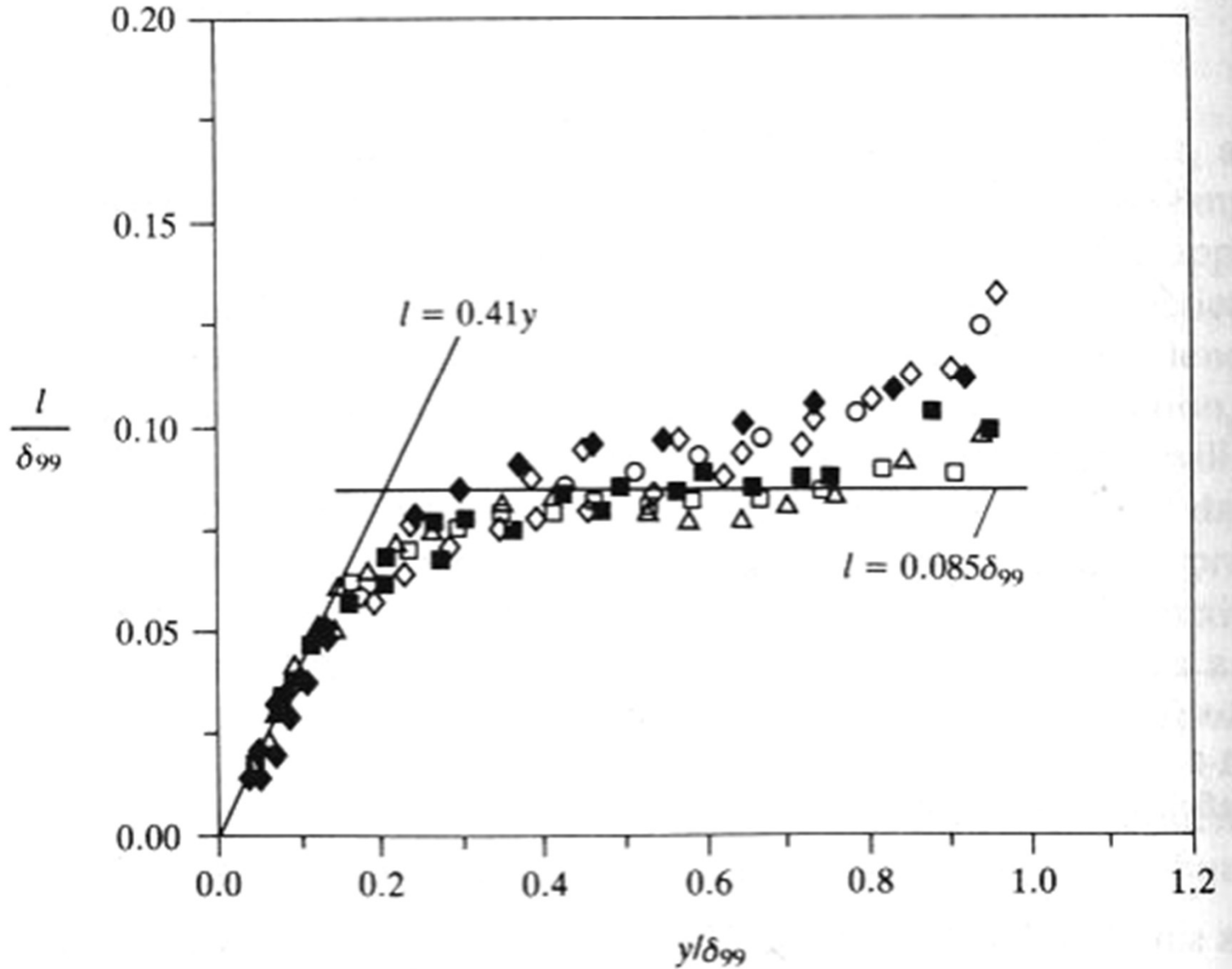


FIGURE 11-2

Mixing-length measurements of Andersen¹ for no pressure gradient, adverse gradient, blowing, and suction.

Comprimento de Mistura - Prandtl

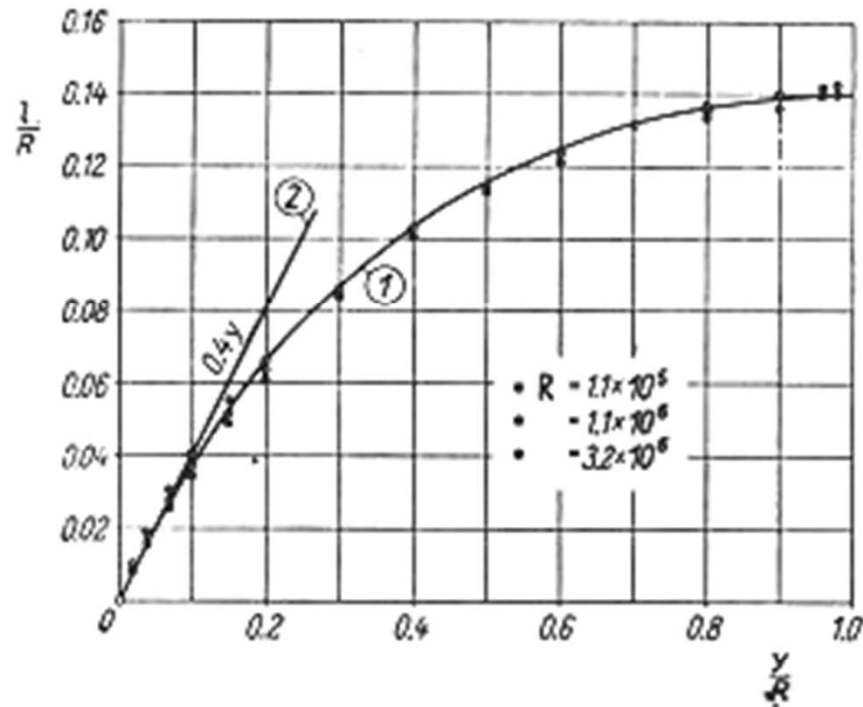


Fig. 20.5. Variation of mixing length over pipe diameter for smooth pipes at different Reynolds numbers

Curve (1) from eqn. (20.18)

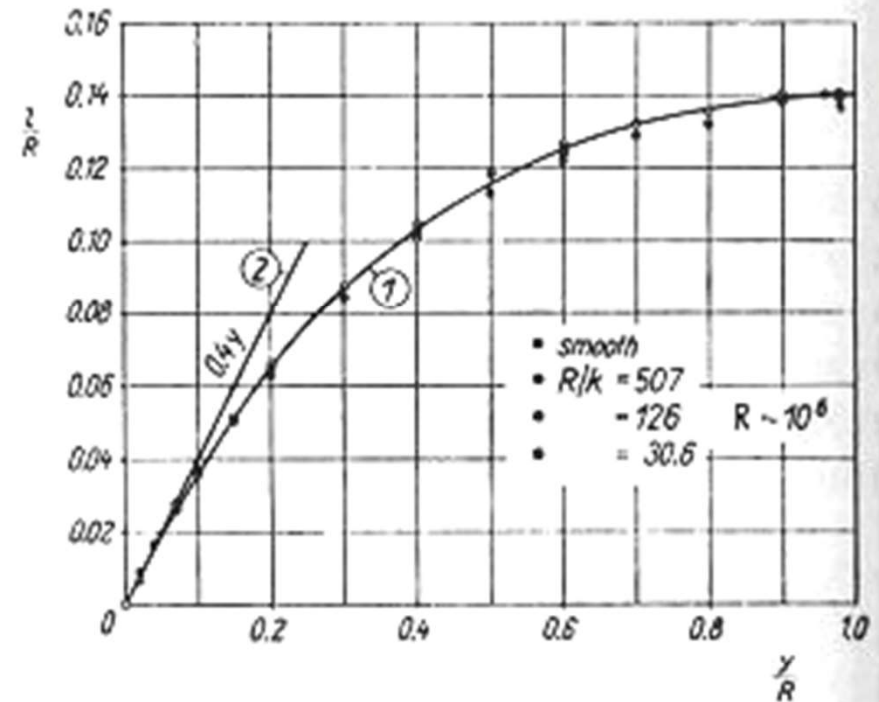
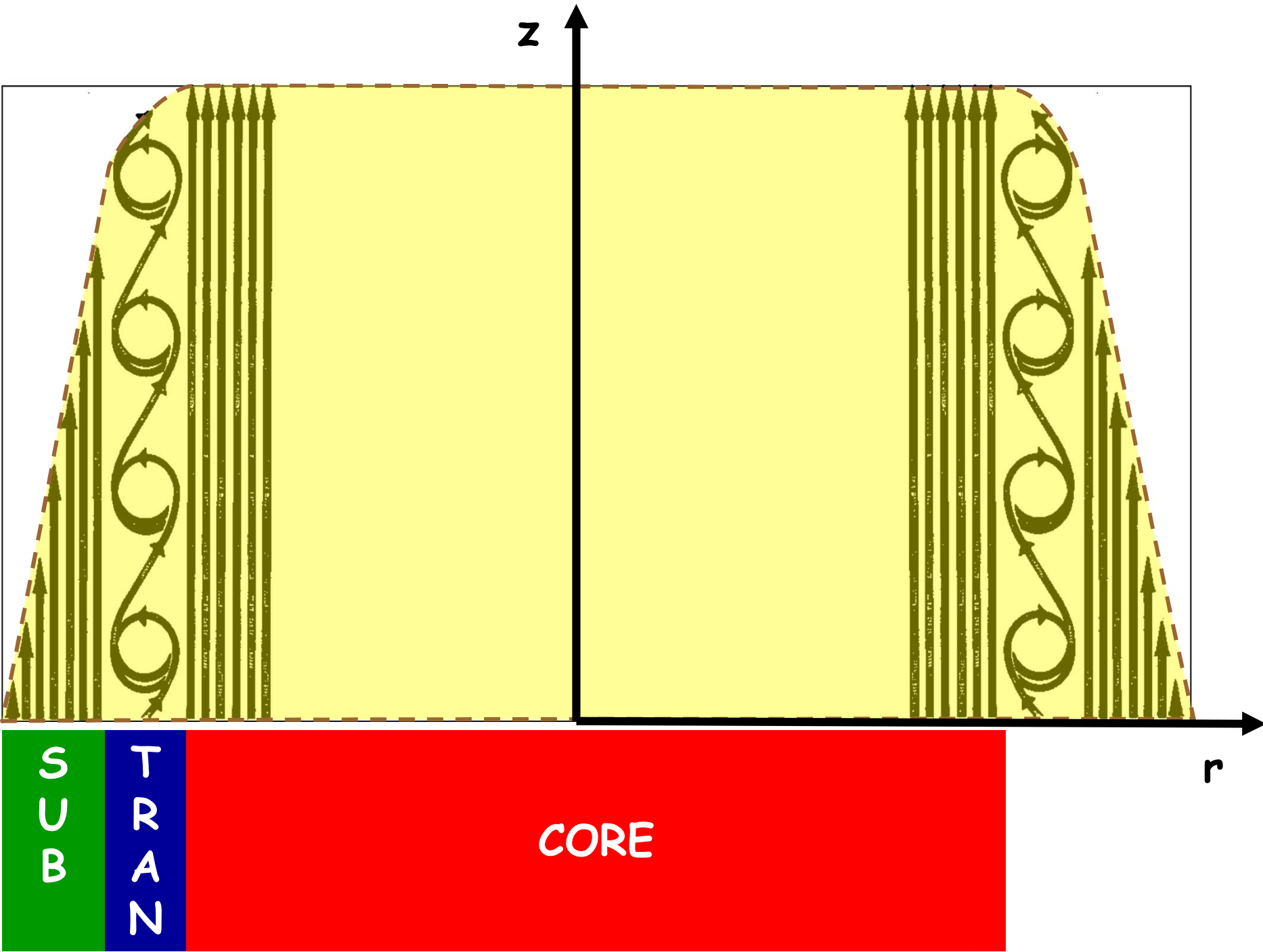


Fig. 20.6. Variation of mixing length over pipe diameter for rough pipes

Curve (1) from eqn. (20.18)



SUBSTANTIAL

CRUST

CORE

Modelos de Turbulência

Reynolds Stress

$$\overline{\vec{R}} = \overline{\vec{v}' \cdot \vec{v}'}$$

sete equações diferenciais	$\frac{D \overline{\vec{R}}}{Dt} = - \operatorname{div} \left[\overline{\vec{j}}_{\vec{R}} + \overline{\vec{\pi}}_{\vec{R}} + \overline{\vec{\Omega}}_{\vec{R}} \right] + \overline{\vec{\sigma}}_{M_{\vec{R}}} - \overline{\vec{\epsilon}}_{M_{\vec{R}}}$
algébrico	$\overline{\vec{R}} = \frac{2}{3} K \overline{\delta} + \left[\frac{C_D}{C_1 - 1 + p/\epsilon} \right] \left(\overline{\vec{\sigma}}_{M_{\vec{R}}} - \frac{2}{3} p \overline{\delta} \right) \frac{K}{\epsilon}$
2 eq. $K \epsilon$	$v_T = C_\mu \frac{K^2}{\epsilon}$
0 eq. Prandtl mixing length	$v_T = l_m^2 \left \operatorname{grad} \overline{\vec{v}} \right $
large eddy simulation LES	por hora apenas fornecem parâmetros

Rayleigh

$$\Phi_v = 2 \left[\left(\frac{\partial v_x}{\partial x} \right)^2 + \left(\frac{\partial v_y}{\partial y} \right)^2 + \left(\frac{\partial v_z}{\partial z} \right)^2 \right] + \left[\frac{\partial v_y}{\partial x} + \frac{\partial v_x}{\partial y} \right]^2 + \left[\frac{\partial v_z}{\partial y} + \frac{\partial v_y}{\partial z} \right]^2 + \left[\frac{\partial v_x}{\partial z} + \frac{\partial v_z}{\partial x} \right]^2 - \frac{2}{3} \left[\frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z} \right]^2$$

Dissipação da energia cinética de turbulência - ε

$$\varepsilon = \nu \left[2 \left\langle \left(\frac{\partial v'_x}{\partial x} \right)^2 \right\rangle + 2 \left\langle \left(\frac{\partial v'_y}{\partial y} \right)^2 \right\rangle + 2 \left\langle \left(\frac{\partial v'_z}{\partial z} \right)^2 \right\rangle + \left\langle \left(\frac{\partial v'_y}{\partial x} + \frac{\partial v'_x}{\partial y} \right)^2 \right\rangle + \left\langle \left(\frac{\partial v'_z}{\partial y} + \frac{\partial v'_y}{\partial z} \right)^2 \right\rangle + \left\langle \left(\frac{\partial v'_z}{\partial x} + \frac{\partial v'_x}{\partial z} \right)^2 \right\rangle \right]$$

Energia cinética de turbulência - k

$$k = \frac{1}{2} (v'_x{}^2 + v'_y{}^2 + v'_z{}^2)$$

$$\frac{\partial \bar{k}}{\partial t} + \text{div } \vec{v} \bar{k} = \text{div} \frac{v_T}{\sigma_k} \text{grad } \bar{k} + P_k - \varepsilon$$

$$\frac{\partial \varepsilon}{\partial t} + \text{div } \vec{v} \varepsilon = \text{div} \frac{v_T}{\sigma_\varepsilon} \text{grad } \varepsilon + C_{\varepsilon 1} \frac{\varepsilon}{\bar{k}} P_k - C_{\varepsilon 2} \frac{\varepsilon^2}{\bar{k}}$$

$$P_k = -\bar{\rho} v_T \left(\text{grad } \vec{v} : \text{grad } \vec{v} \right)$$

$$v_T = C_\mu \frac{\bar{k}^2}{\varepsilon}$$

$$\sigma_k = 1,0 ; \sigma_\varepsilon = 1,217 ;$$

$$C_{\varepsilon 1} = 1,44 ; C_{\varepsilon 2} = 1,92 ; C_\mu = 0.09$$

constantes experimentais

sendo a difusividade turbulenta >> laminar:

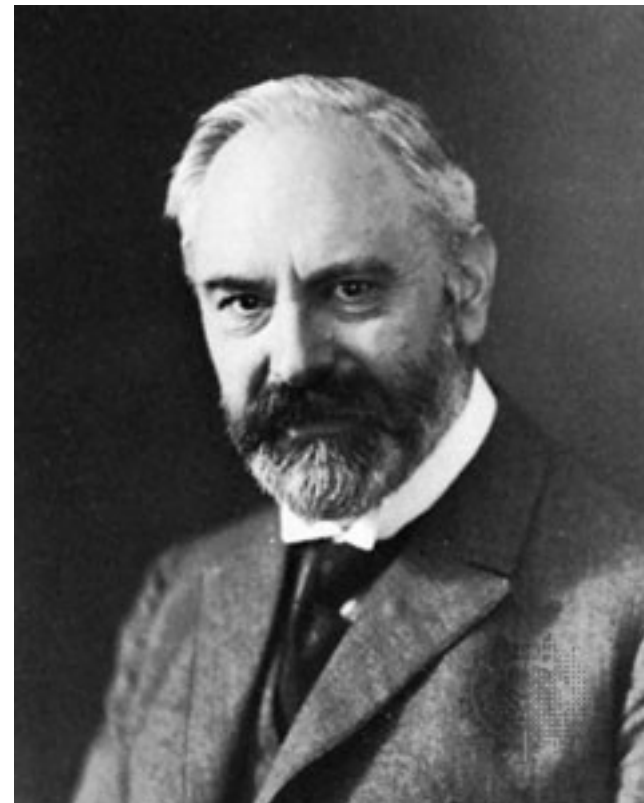
$$[\lambda_\phi + \lambda_{\phi T}] \approx \lambda_{\phi T}$$

$$\frac{\partial \bar{\phi}}{\partial t} + \text{div } \vec{v} \bar{\phi} = \text{div } \lambda_{\phi T} \text{ grad } \bar{\phi} + \overline{\dot{\sigma} M_\phi}$$

voltou à poderosa
agora super-poderosa


$$\lambda_{\vec{v}T} = \ell_m^2 \left| \text{grad } \vec{v} \right|$$

Prandtl mixing length



sendo $\varphi = v$:

$$\frac{\partial \bar{\vec{v}}}{\partial t} + \text{div} \bar{\vec{v}} \bar{\vec{v}} = \text{div} v_T \text{ grad } \bar{\vec{v}} - \frac{\text{grad } \bar{p}}{\rho} + \bar{\vec{g}}$$

Prandtl mixing length	$\lambda_{\bar{\vec{v}}_T} = v_T = \frac{\mu_T}{\rho} = l_m^2 \text{grad } \bar{\vec{v}} $
similaridade Prandtl	$v_T = c_\mu l \sqrt{K}$
k	 $k = \overline{e_{c_T}} = \frac{\overline{\vec{v}' \cdot \vec{v}'}}{2}$
	$\frac{D k}{Dt} = \text{div} \left(\frac{v_T}{\sigma_k} \text{ grad } k \right) + v_T \text{grad } \bar{\vec{v}} ^2 - c_D \frac{k^{3/2}}{l}$
k ε	$v_T = c_\mu K^2 / \epsilon$

Modelos de Turbulência

Reynolds Stress

$$\vec{R} = \overline{\vec{v}' \cdot \vec{v}'}$$

sete equações diferenciais	$\frac{D \vec{R}}{Dt} = - \operatorname{div} \left[\vec{j}_{\vec{R}} + \vec{\pi}_{\vec{R}} + \vec{\Omega}_{\vec{R}} \right] + \overline{\vec{\sigma}_{M_{\vec{R}}}} - \overline{\vec{\epsilon}_{M_{\vec{R}}}}$
algébrico	$\vec{R} = \frac{2}{3} K \vec{\delta} + \left[\frac{C_D}{C_1 - 1 + p/\epsilon} \right] \left(\overline{\vec{\sigma}_{M_{\vec{R}}}} - \frac{2}{3} p \vec{\delta} \right) \frac{K}{\epsilon}$
2 eq. $K \epsilon$	$v_T = C_\mu \frac{K^2}{\epsilon}$
0 eq. Prandtl mixing lenght	$v_T = l_m^2 \left \operatorname{grad} \vec{v} \right $
large eddy simulation LES	por hora apenas fornecem parâmetros

Rayleigh

$$P = \mu \Phi_v = \rho \nu \left(\text{grad} \vec{v} : \text{grad} \vec{v} \right)$$

turbulento

$$\overline{\zeta}_T = -\rho \nu_T \text{grad} \vec{v}$$

comprimento de
mistura de Prandtl

$$\nu_T = \ell_m^2 \left| \text{grad} \vec{v} \right|$$

dissipação de energia cinética de turbulência

$$P_k = -\bar{\rho} \ell_m^2 \left(\text{grad} \vec{v} : \text{grad} \vec{v} \right)$$

Pressure, Pa

Probe value

1144.653

1116.380

1032.765

Average value

920.8766

572.1378

808.9884

697.1002

585.2120

473.3239

361.4357

249.5475

137.6593

25.77116

-86.11702

-198.0052

-309.8934

-421.7816

-533.6697

-645.5579

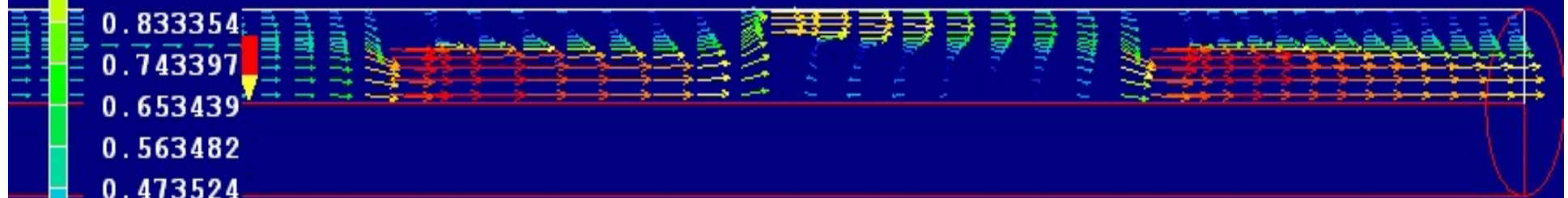
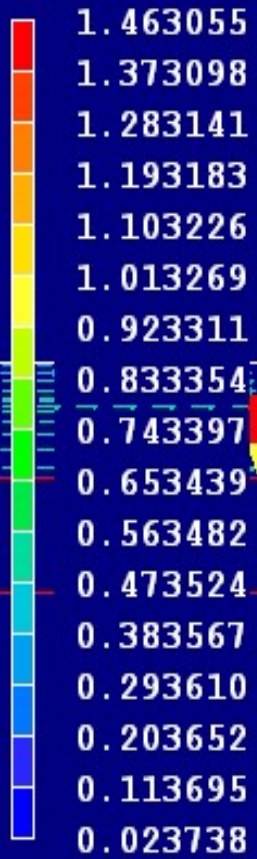


pressão

Velocity, m/s

Probe value

0.552486



velocidades

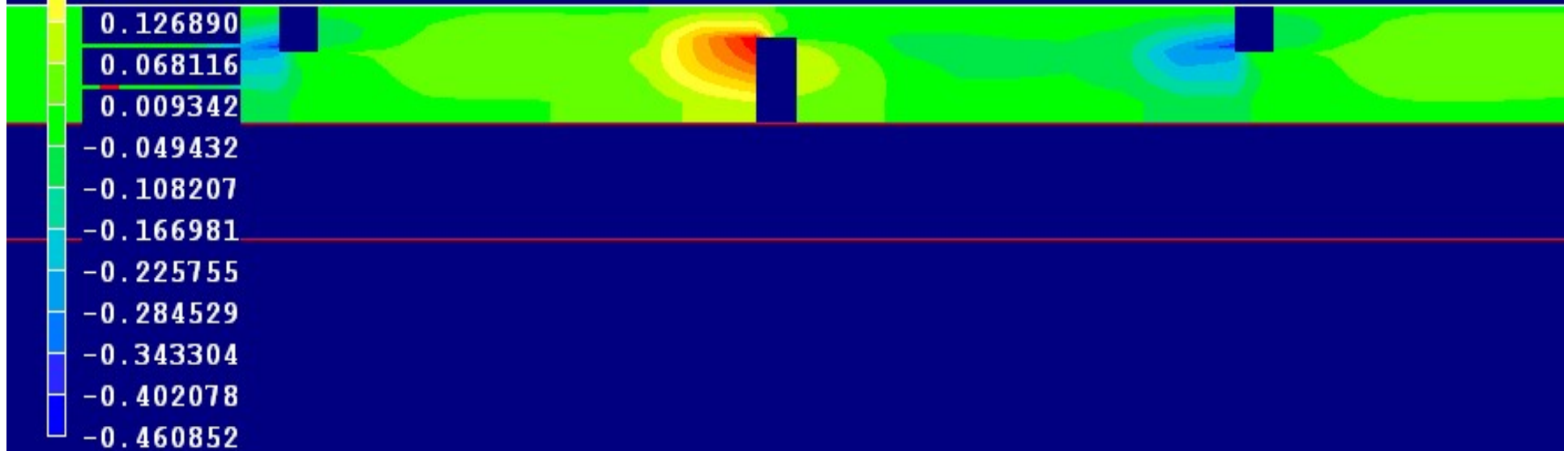
Y-Velocity, m/s

Probe value

-0.002395

Average value

-0.002428



velocidade radial

KE



Probe value

2.756E-4

Average value

0.017677

energia cinética de
turbulência k

EP

phoenics

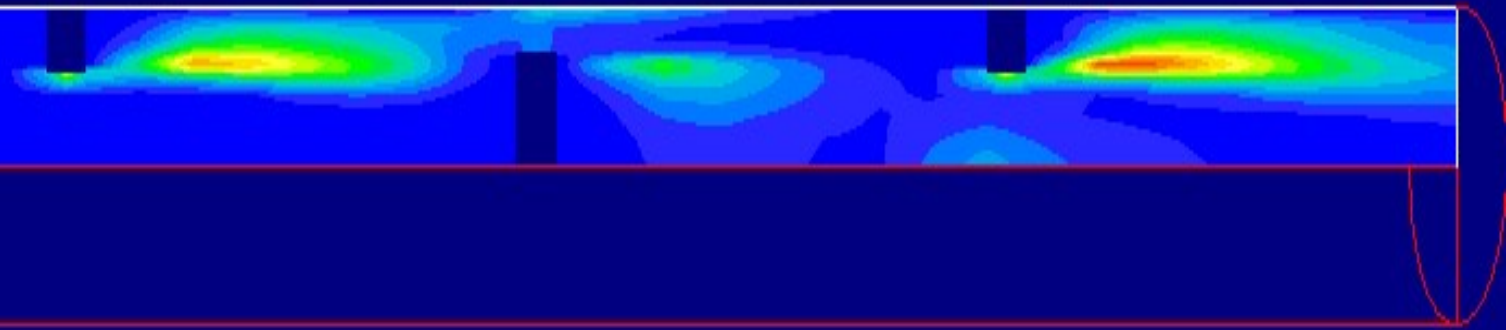


Probe value

8.640E-4

Average value

0.854358



dissipação de k