

Lecture 1-b

Catenary Risers: Global Analysis

Dynamic Problem **equations and time scales**

Celso P. Pesce

Professor of Mechanical Sciences

PhD in Ocean Engineering, MSc Marine Hydrodynamics, Naval Architect

ceppesce@usp.br

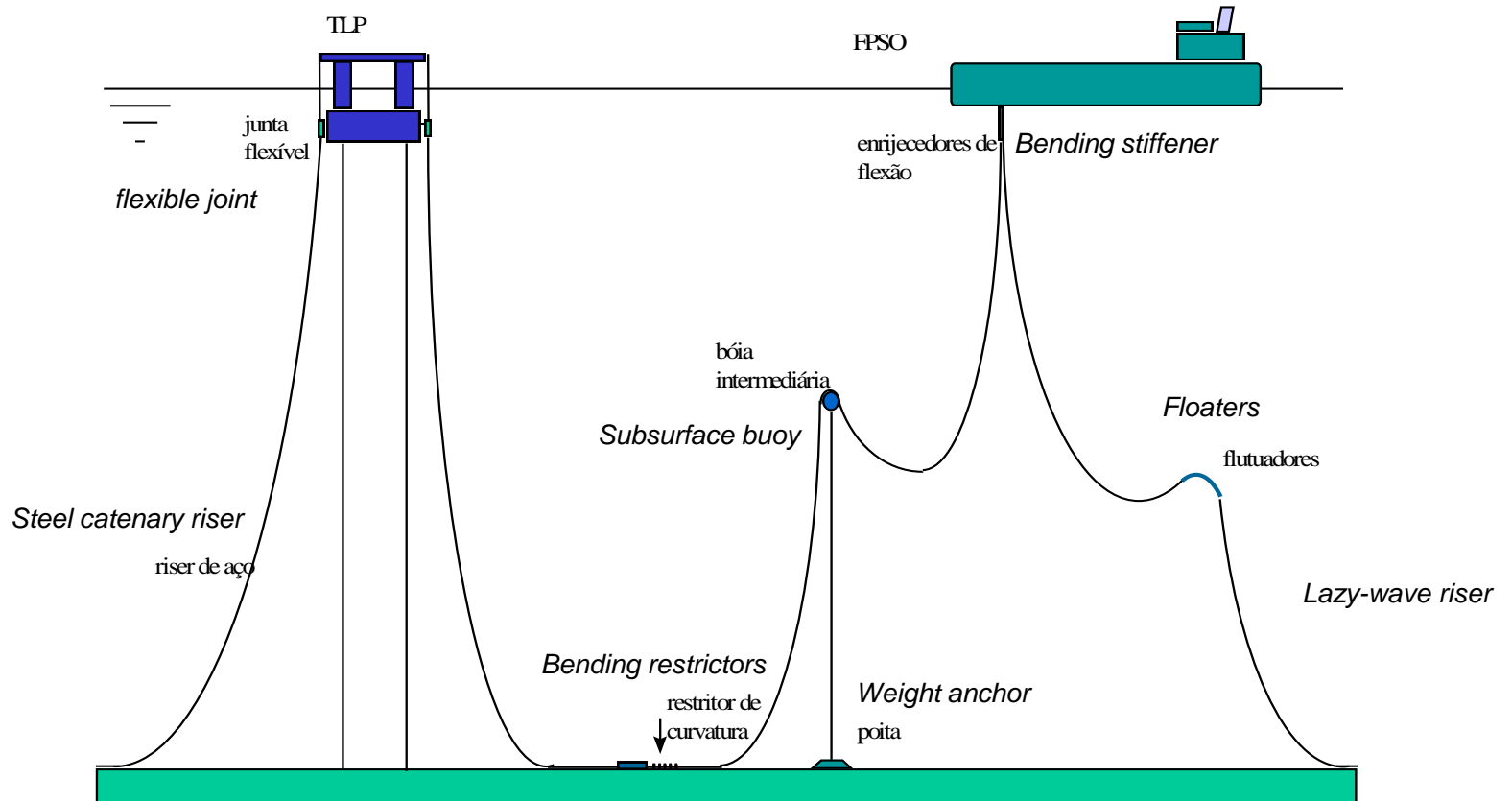
LMO - Offshore Mechanics Laboratory

Escola Politécnica

University of São Paulo

Brazil

Catenary lines



General Kirshoff-Clebsh-Love Equations (KCL equations)

Curved bars – large displacements, small deformations

$$\frac{\partial T}{\partial s} - Q_v \kappa_w + Q_w \kappa_v + f_u(t) = m \frac{\partial^2 u}{\partial t^2}$$

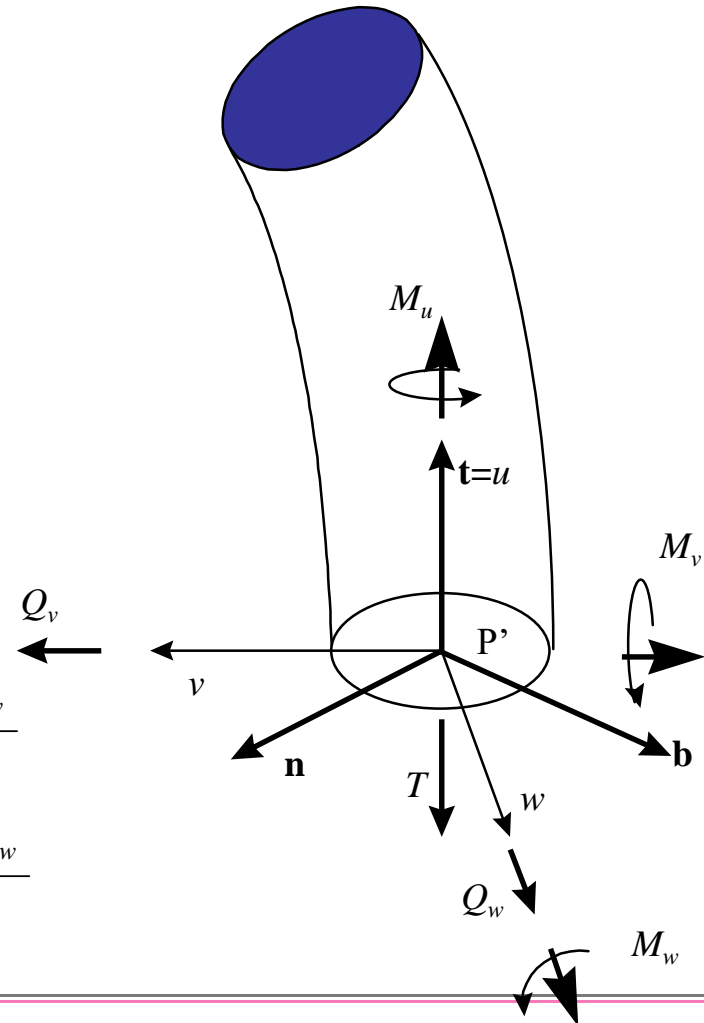
$$\frac{\partial Q_v}{\partial s} - Q_w \kappa_u + T \kappa_w + f_v(t) = m \frac{\partial^2 v}{\partial t^2}$$

$$\frac{\partial Q_w}{\partial s} - T \kappa_v + Q_v \kappa_u + f_w(t) = m \frac{\partial^2 w}{\partial t^2}$$

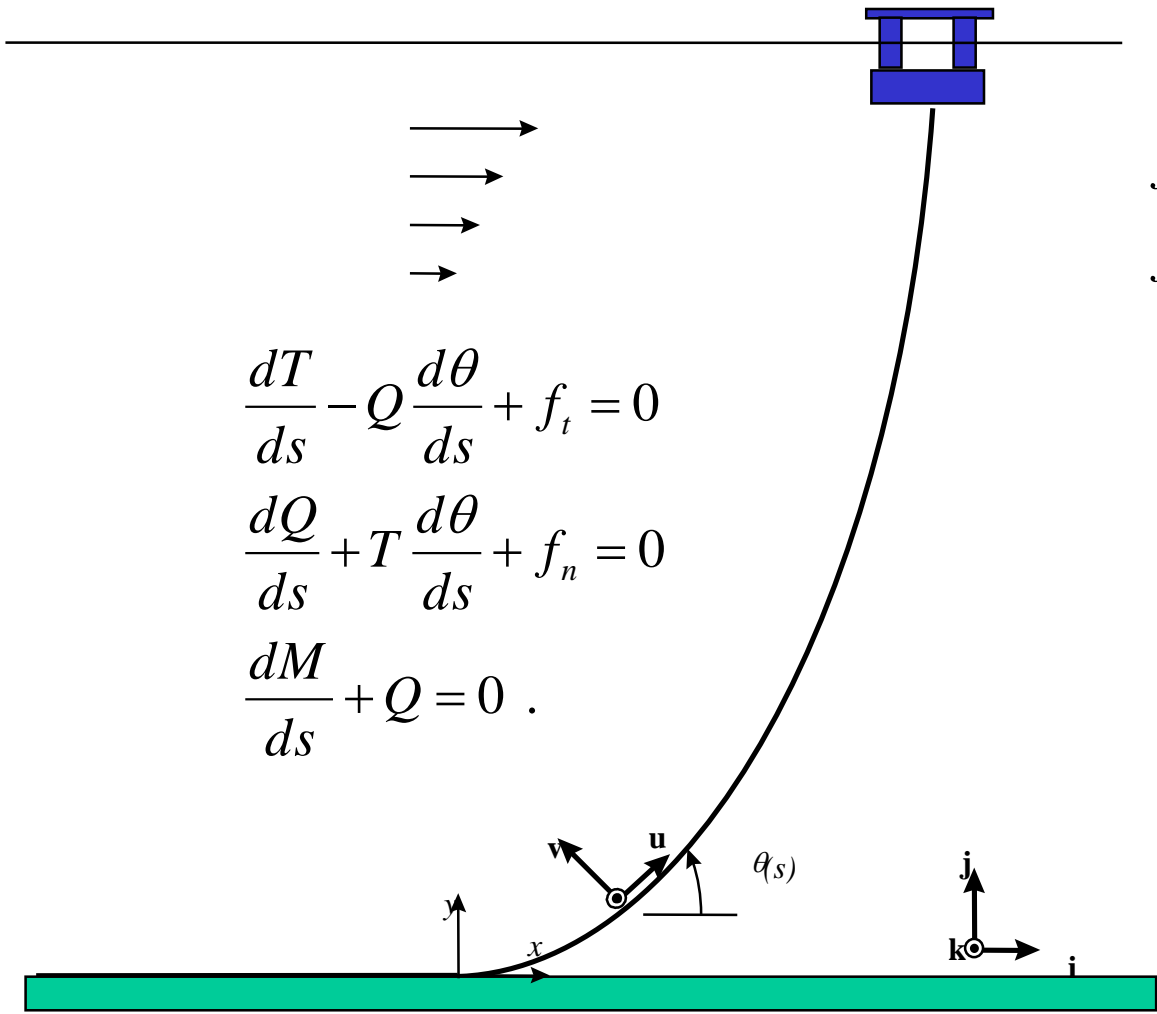
$$\frac{\partial M_u}{\partial s} - M_v \kappa_w + M_w \kappa_v + K_u(t) = I_{uu} \frac{\partial^2 \varphi_u}{\partial t^2}$$

$$\frac{\partial M_v}{\partial s} - M_w \kappa_u + M_u \kappa_w - Q_w + K_v(t) = I_{vv} \frac{\partial^2 \varphi_v}{\partial t^2}$$

$$\frac{\partial M_w}{\partial s} - M_u \kappa_v + M_v \kappa_u + Q_v + K_w(t) = I_{ww} \frac{\partial^2 \varphi_w}{\partial t^2}$$



The planar static problem



with:

$$f_t = -q \sin \theta + c_t(s)$$

$$f_n = -q \cos \theta + c_n(s)$$

$$[c_t(s); c_n(s)]$$

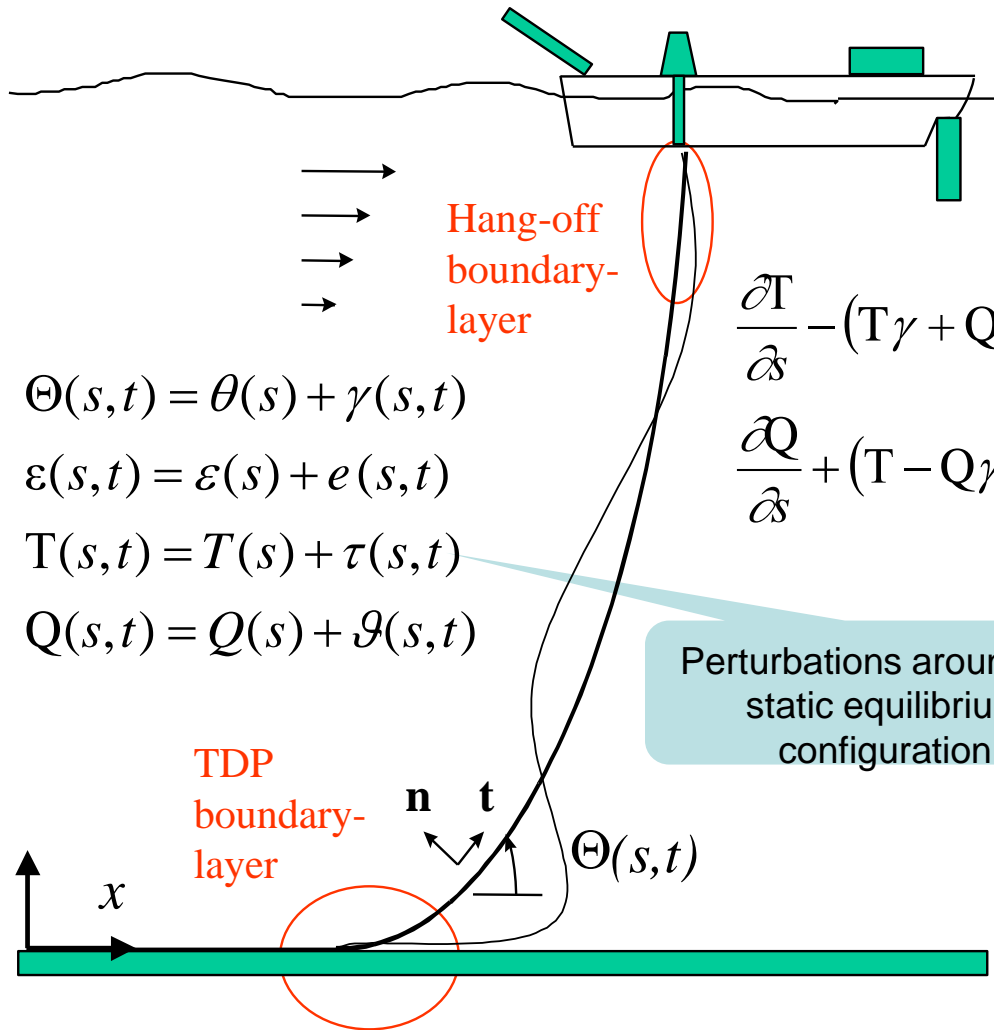
$$\frac{dT}{ds} - Q \frac{d\theta}{ds} + f_t = 0$$

$$\frac{dQ}{ds} + T \frac{d\theta}{ds} + f_n = 0$$

$$\frac{dM}{ds} + Q = 0 .$$

Static forces due to current

The planar dynamic problem



$$\Theta(s, t) = \theta(s) + \gamma(s, t)$$

$$\varepsilon(s, t) = \varepsilon(s) + e(s, t)$$

$$T(s, t) = T(s) + \tau(s, t)$$

$$Q(s, t) = Q(s) + \mathcal{Q}(s, t)$$

$$\frac{\partial T}{\partial s} - (T\gamma + Q) \frac{d\theta}{ds} - \frac{\partial}{\partial s} (Q\gamma) + c_t + \varpi_t - q \sin \theta = m \frac{\partial^2 u_t}{\partial t^2}$$

$$\frac{\partial Q}{\partial s} + (T - Q\gamma) \frac{d\theta}{ds} + \frac{\partial}{\partial s} (T\gamma) + c_n + \varpi_n - q \cos \theta = m \frac{\partial^2 u_n}{\partial t^2}$$

Oscillating hydrodynamic forces

Perturbations around the static equilibrium configuration

$$\frac{\partial u_n}{\partial s} + u_t \frac{d\theta}{ds} = (1 + \varepsilon(s)) \gamma(s, t)$$

$$\frac{\partial u_t}{\partial s} - u_n \frac{d\theta}{ds} = e$$

Kinematic relations

Separating static and dynamic parts:

Static equilibrium,
tangential direction

$$\left\{ \frac{\partial T}{\partial s} - Q \frac{d\theta}{ds} + c_t - q \sin \theta \right\} + \longrightarrow \boxed{\{\bullet\} \equiv 0}$$

$$+ \left\{ \frac{\partial \tau}{\partial s} - (T\gamma + \varrho) \frac{d\theta}{ds} - \frac{\partial}{\partial s} (Q\gamma) + \varpi_t \right\} = m \frac{\partial^2 u_t}{\partial t^2}$$

Static equilibrium,
normal direction

$$\left\{ \frac{\partial Q}{\partial s} + T \frac{d\theta}{ds} + c_n - q \cos \theta \right\} + \longrightarrow \boxed{\{\bullet\} \equiv 0}$$

$$+ \left\{ \frac{\partial \varrho}{\partial s} + (\tau - Q\gamma) \frac{d\theta}{ds} + \frac{\partial}{\partial s} (T\gamma) + \varpi_n \right\} = m \frac{\partial^2 u_n}{\partial t^2}$$

Linearized Dynamic Equations

Neglecting second-order terms

$$\begin{aligned} & \frac{\partial \tau}{\partial s} - ((T + \tau)\gamma + \mathcal{G}) \frac{d\theta}{ds} - \\ & - \frac{\partial}{\partial s} ((Q + \mathcal{G})\gamma) + \varpi_t = m \frac{\partial^2 u_t}{\partial t^2} \\ & \frac{\partial \mathcal{G}}{\partial s} + (\tau - (Q + \mathcal{G})\gamma) \frac{d\theta}{ds} + \\ & + \frac{\partial}{\partial s} ((T + \tau)\gamma) + \varpi_n = m \frac{\partial^2 u_n}{\partial t^2} \end{aligned}$$

$$\begin{aligned} & \frac{\partial \tau}{\partial s} - (T\gamma + \mathcal{G}) \frac{d\theta}{ds} - \frac{\partial}{\partial s} (Q\gamma) + \varpi_t = m \frac{\partial^2 u_t}{\partial t^2} \\ & \frac{\partial \mathcal{G}}{\partial s} + (\tau - Q\gamma) \frac{d\theta}{ds} + \frac{\partial}{\partial s} (T\gamma) + \varpi_n = m \frac{\partial^2 u_n}{\partial t^2} \end{aligned}$$

Two dynamic equilibrium equations, linearized around the static equilibrium configurations

Linearized Dynamic Equations: neglecting bending stiffness

Neglecting bending stiffness effects: \longrightarrow

$$\frac{\partial \tau}{\partial s} - \gamma T \frac{d\theta}{ds} = -\varpi_t + m \frac{\partial^2 u_t}{\partial t^2}$$

$$\tau \frac{d\theta}{ds} + \frac{\partial}{\partial s} (\gamma T) = -\varpi_n + m \frac{\partial^2 u_n}{\partial t^2}$$

Two dynamic equilibrium equations, linearized around the static equilibrium configurations

$$\frac{\partial u_t}{\partial s} - u_n \frac{d\theta}{ds} = e$$

$$\frac{\partial u_n}{\partial s} + u_t \frac{d\theta}{ds} = (1 + \varepsilon) \gamma$$

Kinematic relations

$$\tau(s, t) = EAe(s, t) + q_f (u_t \sin \theta + u_n \cos \theta)$$

$$q_f = (\rho_a S_o - \rho_f S_i) g$$

Constitutive equations

Mean current hydrodynamic forces

$$c_t(s) = \frac{1}{2} \rho_a D U(s) |U(s)| C_T \cos^2 \theta(s)$$

$$c_n(s) = -\frac{1}{2} \rho_a D U(s) |U(s)| C_D \sin^2 \theta(s)$$

Oscillating hydrodynamic forces

Parcelas dinâmicas das forças devido à correnteza e às ondas:

Viscous parcels

$$\varpi(s, t) = \varpi^{visc}(s, t) + \varpi^{in}(s, t)$$

$$\varpi_t^{visc}(s, t) = -\frac{1}{2} C_T \rho_a D \text{sinal}(v_t) \left(-2U(s) \cos\theta \left(\frac{\partial u_t}{\partial t} + U(s) \sin\theta \frac{\partial u_n}{\partial s} - w_t \right) \right)$$

$$\varpi_n^{visc}(s, t) = -\frac{1}{2} C_D \rho_a D \text{sinal}(v_n) \left(2U(s) \sin\theta \left(\frac{\partial u_n}{\partial t} + U(s) \cos\theta \frac{\partial u_n}{\partial s} - w_n \right) \right)$$

$$\varpi_t^{in}(s, t) = 0$$

$$\varpi_n^{in}(s, t) = -m_a \left(\frac{\partial^2 u_t}{\partial t^2} + U(s) \cos\theta \frac{\partial \gamma}{\partial t} - \frac{\partial w_n}{\partial t} \right)$$

Inertial parcels

Relative velocity components to the flow

$$v_t = \frac{\partial u_t}{\partial t} - U \cos\theta - w_t$$

$$v_n = \frac{\partial u_n}{\partial t} + U \sin\theta - w_n$$

Gravitational waves velocity field

The bending stiffness effect

From the classic linear constitutive equations:

$$M = EI\chi(s, t) = EI\left(\frac{d\theta}{ds} + c(s, t)\right)$$

e

$$Q = \frac{\partial M}{\partial s} = EI \frac{\partial \chi}{\partial s} = Q + \mathcal{Q} = EI \frac{d^2\theta}{ds^2} + EI \frac{\partial c}{\partial s}$$

The dynamic equation in the normal direction may be written:

$$\boxed{-EI \frac{\partial^2 \chi}{\partial s^2} + T\chi + \gamma \left(\frac{\partial \Gamma}{\partial s} - EI \frac{d\theta}{ds} \frac{\partial \chi}{\partial s} \right) + c_n + \varpi_n - q \cos \theta = m \frac{\partial^2 u_n}{\partial t^2}}$$

The bending stiffness effect

Defining:

$$\lambda = \sqrt{\frac{EI}{T_0}}$$

Flexural length at TDP

$$\chi_0 = \frac{q}{T_0}$$

Small length parameter

$$\varepsilon = \frac{\lambda}{L}$$

Static curvature at TDP

$$\hat{t} = \frac{c_0}{L} t$$

Transversal wave celerity due to geometric stiffness, at TDP

$$c_0 = \sqrt{\frac{T_0}{m + m_a}}$$

Second-order term

The nondimensional dynamic equation in the normal direction reads:

$$-\varepsilon^2 \frac{\partial^2 \hat{\chi}}{\partial \hat{s}^2} + \hat{T} \hat{\chi} + \left(\frac{\partial \hat{T}}{\partial \hat{s}} - \varepsilon^2 \frac{d\theta}{d\hat{s}} \frac{\partial \hat{\chi}}{\partial \hat{s}} \right) \gamma + \hat{c}_n + \hat{\omega}_n - \hat{\chi}_0 \cos \theta = \frac{\partial^2 \hat{u}_n}{\partial \hat{t}^2}$$

The bending stiffness effect

Rigid risers (Steel): $\varepsilon \approx O(10^{-2})$

Flexible risers: $\varepsilon \approx O(10^{-3})$

Neglecting secon-order terms (bending stiffness):

$$\hat{T}\hat{\chi} + \frac{\partial \hat{T}}{\partial \hat{s}} \gamma + \hat{c}_n + \hat{w}_n - \hat{\chi}_0 \cos \theta = \frac{\partial^2 \hat{u}_n}{\partial \hat{t}^2} (1 + O(\varepsilon^2))$$

Time scales

$$t_1 = \frac{L}{c_g}$$

$$\bar{c}_g = \sqrt{\frac{\bar{T}}{(m + m_a)}}$$

Wave celerity associated to geometric rigidity

$$t_2 = \frac{\lambda}{c_g}$$

$$\bar{c}_f^{(i)} = \frac{2\pi}{\lambda_f^{(i)}} \sqrt{\frac{EI}{(m + m_a)}}$$

Wave celerity associated to bending stiffness

$$t_3^{(i)} = \frac{L}{c_f^{(i)}}$$

$$c_a = \frac{EA}{m}$$

Wave celerity associated to axial stiffness

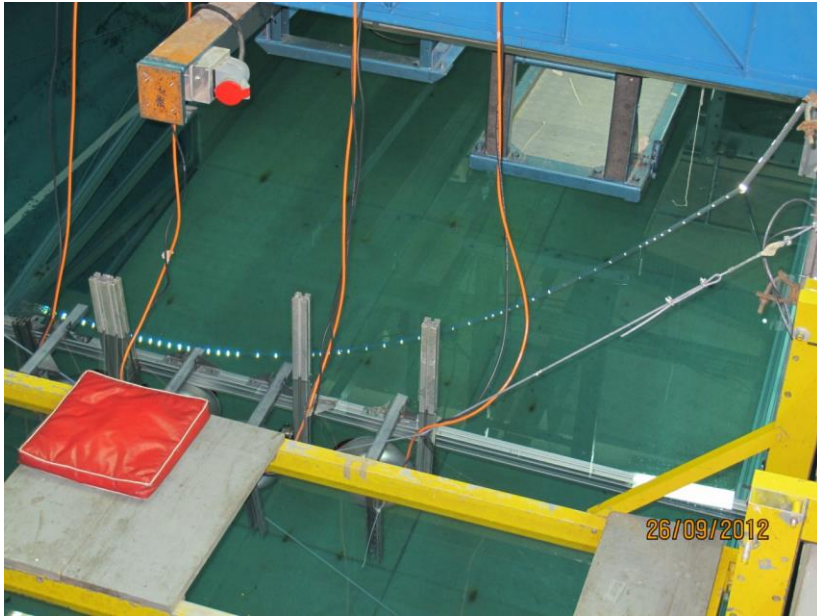
$$t_4^{(i)} = \frac{\lambda}{c_f^{(i)}}$$

$$t_5 = \frac{L}{c_a}$$

$$\lambda(s) = \sqrt{\frac{EI}{T(s)}}$$

Local flexural length

Model similarity and nondimensional group



Number	Symbol and definition	Representation
Froude Number	$F_r = \frac{\omega A}{\sqrt{gL}}$	Dynamic motion in waves
Reynolds Number	$Re = \frac{UD}{\nu}$	Viscous forces
Strouhal Number	$S_t = \frac{f_s D}{U}$	Vortex shedding frequency
Keulegan-Carpenter Number	$KC = \frac{2\pi A}{D}$	Inertial forces vs. drag forces
Structural Damping	$\zeta = \frac{c}{c_c}$	Linear structural damping
Reduced Velocity	$V_r = \frac{U}{f_n D}$	Normalized velocity in VIV
Reduced Shedding Frequency	$f_s^* = \frac{f_s}{f_n} = S_t \frac{U}{f_n D} = S_t V_r$	Vortex shedding normalized frequency
Reduced Mass	$m^* = \frac{m}{m_D}$	Riser mass vs. displaced mass
Added Mass	$a = \frac{m_a}{m}$	Added mass vs. riser mass
Bending Stiffness	$K_f = \frac{\lambda_f}{L} = \frac{1}{L} \sqrt{\frac{EI}{T}}$	Bending vs. geometrical stiffness
Axial Stiffness	$K_a = \frac{EA}{T}$	Axial vs. geometrical stiffness
Soil Stiffness	$K_s = \frac{k_s EI}{T^2}$	Soil vs. bending and geometrical stiffness

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Summary

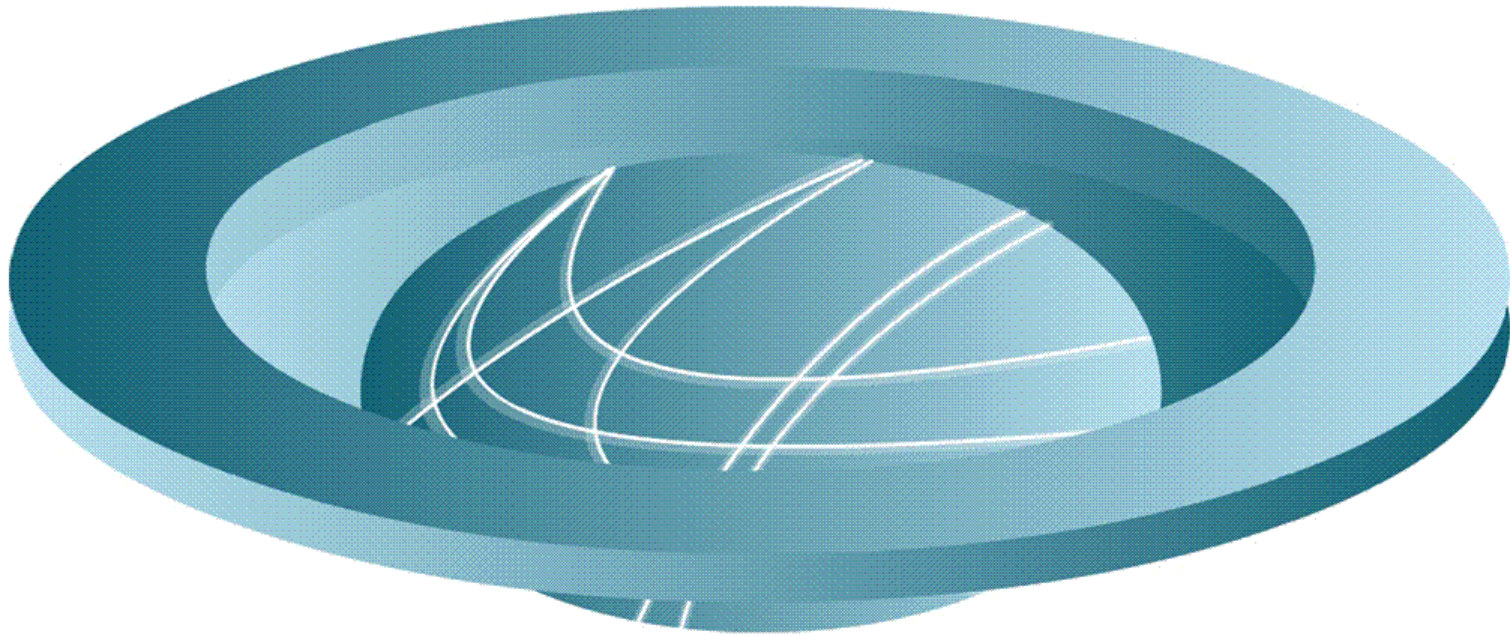
- The global dynamics is governed by the geometric stiffness or catenary stiffness.
 - Bending stiffness effects are important at the extremities and TDP, or for high-order vibration modes for which the mode vibration length is of same order of the local flexural length.
 - There are several time scales that govern the overall dynamics of a riser.
 - A large nondimensional parameters group govern the overall riser dynamics
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Acknowledgements





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