

Testando ② e ③ com ⑥, tem-se:

$$\dot{\beta}_f - \dot{\beta}_r = \frac{L}{R} \Rightarrow$$

$$\Rightarrow \dot{\beta} + \frac{L_f \times h}{U} - \dot{\beta} + \frac{L_r \times h}{U} = \frac{L}{R} \Rightarrow$$

$$\frac{(L_f + L_r) \times h}{U} = \frac{L}{R}. \text{ Como } \frac{h}{U} = \frac{1}{R}, \text{ temos:}$$

$$\frac{L \times \frac{1}{R}}{R} = \frac{L}{R} \text{ c.g.d.}$$

De ④ \Rightarrow

$$\Rightarrow \delta = \beta_f - \alpha_f \quad ⑦. \text{ Substituindo ⑥ em ⑦:}$$

$$\Rightarrow \boxed{\delta = \frac{L}{R} + \alpha_r - \alpha_f} \quad ⑧$$

Calculo das forças laterais:

$$\dot{F}_{yf} = C_{\alpha_f} \cdot \dot{\alpha}_f \quad ⑨ \text{ - equação válida na região linear de } F_{yx\alpha}.$$

Substituindo ④ em ⑨ \Rightarrow

$$\Rightarrow F_{yf} = C_{\alpha_f} \times \left(\beta + \frac{L_f \times h}{U} - \delta \right) \Rightarrow$$

$$\Rightarrow \boxed{F_{yf} = C_{\alpha_f} \cdot \beta + C_{\alpha_f} \cdot \frac{L_f \times h}{U} - C_{\alpha_f} \cdot \delta} \quad ⑩$$

$$F_{yr} = C_{\alpha_r} \cdot \dot{\alpha}_r \quad ⑪$$

Substituindo ⑨ em ⑪ tem-se:

$$F_{yr} = C_{\alpha_r} \cdot \left(\beta - \frac{L_r \times h}{U} \right) \Rightarrow$$

$$\Rightarrow \boxed{F_{yr} = C_{\alpha_r} \cdot \beta - C_{\alpha_r} \cdot \frac{L_r \times h}{U}} \quad ⑫$$

Somando ⑩ e ⑫ tem-se:

$$\boxed{F_y = C_{\alpha_f} \cdot \beta + C_{\alpha_f} \cdot \frac{L_f \times h}{U} - C_{\alpha_f} \cdot \delta + C_{\alpha_r} \cdot \beta - C_{\alpha_r} \cdot \frac{L_r \times h}{U}} \quad ⑬$$

Escrivendo:

$$F_y = \frac{\partial F_y}{\partial \beta} \cdot \beta + \frac{\partial F_y}{\partial r} \cdot r + \frac{\partial F_y}{\partial \delta} \cdot \delta \Rightarrow \text{ma regras linear } F_{y \text{xd}}$$

$$\Rightarrow F_y = Y_\beta \cdot \beta + Y_r \cdot r + Y_\delta \cdot \delta \quad (14)$$

Substituindo (13) em (14) \Rightarrow

$$\Rightarrow Y_\beta = C\alpha_f + C\alpha_r \quad (15)$$

$$\Rightarrow Y_r = \frac{1}{L} (C\alpha_f \cdot L_f - C\alpha_r \cdot L_r) \quad (16)$$

$$\Rightarrow Y_\delta = -C\alpha_f \quad (17)$$

Calculo dos torques laterais:

$$N_f = C\alpha_f \cdot \alpha_f \cdot L_f \quad (18) \text{ ma regras linear } F_{y \text{xd}}$$

$$N_r = C\alpha_r \cdot \alpha_r \cdot L_r \quad (19)$$

Somando os torques:

$$N = C\alpha_f \left(\beta + \frac{L_f \times h}{L} - \delta \right) \cdot L_f - C\alpha_r \left(\beta - \frac{L_r \times h}{L} \right) \cdot L_r \Rightarrow$$

$$N = C\alpha_f \cdot \beta \cdot L_f + C\alpha_f \cdot \frac{L_f \times h}{L} \cdot L_f - C\alpha_f \cdot \delta \cdot L_f - C\alpha_r \cdot \beta \cdot L_r + C\alpha_r \cdot \frac{L_r \times h}{L} \cdot L_r \quad (20)$$

Escrivendo:

$$N = N_\beta \cdot \beta + N_r \cdot r + N_\delta \cdot \delta \quad (21)$$

De (20) e (21) \Rightarrow

$$\Rightarrow N_\beta = (C\alpha_f \cdot L_f - C\alpha_r \cdot L_r) \quad (22)$$

$$\Rightarrow N_r = \frac{1}{L} (C\alpha_f \cdot L_f^2 + C\alpha_r \cdot L_r^2) \quad (23)$$

$$\Rightarrow N_\delta = -C\alpha_f \cdot L_f \quad (24)$$

Resumindo as derivadas de controle e estabilidade:

$$y_\beta = C_{\alpha_f} + C_{\alpha_t} \quad \text{e} \quad N_h = \frac{1}{L} \left(C_{\alpha_f} \cdot L_f^2 + C_{\alpha_t} \cdot L_t^2 \right)$$

derivadas de arranque/cimento

$$y_\delta = -C_{\alpha_f} \quad \text{e} \quad N_g = -C_{\alpha_f} \cdot L_f$$

derivadas de controle

$$y_h = \frac{1}{L} \left(C_{\alpha_f} \cdot L_f - C_{\alpha_t} \cdot L_t \right) \quad \text{e} \quad N_\beta = (C_{\alpha_f} \cdot L_f - C_{\alpha_t} \cdot L_t)$$

derivadas de variação

Métrica de estabilidade direcional estática

$$\text{De } ③ \rightarrow \delta = \frac{L}{R} - \bar{\alpha}_f + \bar{\alpha}_t = 0$$

$$F_y f = m_f \cdot \frac{V_{CG}^2}{R} \Rightarrow \frac{W_f}{g} \cdot \frac{V_{CG}^2}{R} \quad \text{e} \quad F_y f = C_{\alpha_f} \alpha_f \text{ na região limpos}$$

$$\Rightarrow \bar{\alpha}_f = \frac{W_f}{C_{\alpha_f}} \cdot \frac{V_{CG}^2}{Rg}$$

$$\Rightarrow \bar{\alpha}_t = \frac{W_t}{C_{\alpha_t}} \cdot \frac{V_{CG}^2}{Rg}$$

$$\Rightarrow \delta = \frac{L}{R} \left(\frac{W_f}{C_{\alpha_f}} + \frac{W_t}{C_{\alpha_t}} \right) \cdot \bar{\alpha}_y = 0$$

$$\delta = \frac{L}{R} + K \cdot \bar{\alpha}_y \text{ onde}$$

$$K = \left(\frac{W_t}{C_{\alpha_t}} - \frac{W_f}{C_{\alpha_f}} \right) \quad ⑤$$

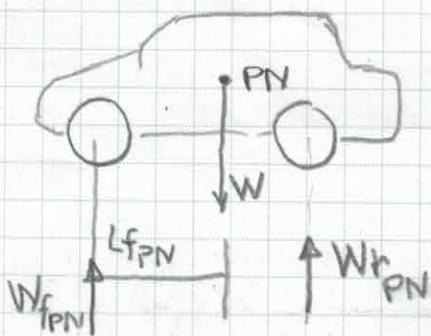
quando $K > 0 \Rightarrow \bar{\alpha}_y, \bar{V}_{CG} \Rightarrow \delta \uparrow$

K é chamado de gradiente de estacionamento. É a métrica de estabilidade direcional estática.

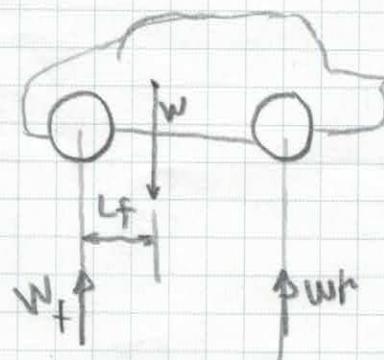
Esta métrica pode ser vista de uma forma diferente, a margem de estabilidade direcional estática SM.

Existe um ponto na direção x do veículo que se o centro de massa do veículo estiver sobre

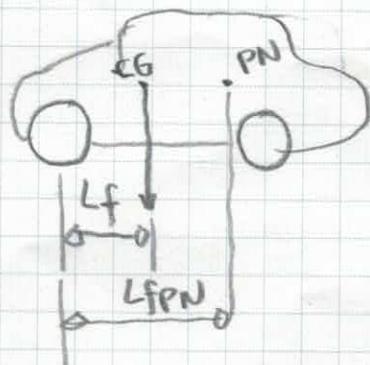
de, $K=0$. Este ponto recebe o nome de "ponto neutro de manobra". Sua posição será determinada tendo o eixo dianteiro como referência:



Com o centro de gravidade do veículo em outra posição tem-se:



A margem de estabilidade direcional estática é definida como:



$$SM = \frac{L_fPN - L_f}{L} \times 100$$

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Da equação 25,

$$K = \left(\frac{W_{fPN}}{G_{ext}} - \frac{W_{fPN}}{G_{df}} \right) = 0 \rightarrow \text{definição de ponto neutro de manobra.}$$

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 \Rightarrow

$$\frac{W_{fPN}}{-\alpha_f} + \frac{W_{tPN}}{\alpha_r} = 0 \quad (28)$$

semedo $y_s = -\alpha_f$

$$y_B = \alpha_f + \alpha_r \Rightarrow \alpha_r = y_B - \alpha_f = 0$$

$$\alpha_r = y_B + y_s \quad (29)$$

De (28) e (29) tem-se

$$\frac{W_{fPN}}{y_s} + \frac{W_{tPN}}{y_s + y_B} = 0 \quad (30), \text{ semedo } W_{fPN} = \frac{W}{L} \cdot L_{fPN} \Rightarrow$$

$$W_{fPN} = \frac{W}{L} L_{fPN}$$

$$\Rightarrow \frac{W}{L} \left(\frac{L_{fPN}}{y_s} + \frac{L_{fPN}}{y_s + y_B} \right) = 0 = 0$$

$$\frac{L_{fPN}}{y_s + y_B} + \frac{L - L_{fPN}}{y_s} = 0 \Rightarrow$$

$$\Rightarrow \frac{L_{fPN}}{y_s + y_B} + \frac{L}{y_s} - \frac{L_{fPN}}{y_s} = 0 \Rightarrow$$

$$\Rightarrow L_{fPN} \left(\frac{1}{y_s + y_B} - \frac{1}{y_s} \right) + \frac{L}{y_s} = 0$$

$$L_{fPN} \left(\frac{y_s - (y_s + y_B)}{y_s(y_s + y_B)} \right) + \frac{L}{y_s} = 0 \Rightarrow$$

$$\Rightarrow L_{fPN} \cdot \left(\frac{-y_B}{y_s(y_s + y_B)} \right) = -\frac{L}{y_s} \Rightarrow$$

$$L_{fPN} = \left(-\frac{L}{y_s} \right) \frac{y_s(y_s + y_B)}{(-y_B)} = \boxed{L_{fPN} = L \cdot \left(\frac{y_s + y_B}{y_B} \right)} \quad (31)$$

$$\Rightarrow SM = \frac{-L_f + L_{fPN}}{L} \times 100 \Rightarrow \boxed{SM = \frac{-L_f + L \left(\frac{y_s + y_B}{y_B} \right)}{L} \times 100} \quad (32)$$

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$$SM = \frac{1}{L} \left(-L_f + L \left(\frac{y_B + y_S}{y_B} \right) \right) \times 100$$

$$SM = \frac{1}{L} \left(\frac{-y_B \cdot L_f + L y_S + L y_B}{y_B} \right) \times 100 = 0$$

$$\Rightarrow SM = \frac{1}{L} \left(\frac{-y_B \cdot L_f + L_f y_B + L y_B + L y_S}{y_B} \right) \times 100 = 0$$

$$\Rightarrow SM = \frac{1}{L} \left(\frac{L_f y_B + L y_S}{y_B} \right) \times 100 = 0$$

$$\Rightarrow SM = \frac{1}{L} \left(\frac{L_f y_B + L_f y_S + L y_S}{y_B} \right) \times 100 = 0$$

$$\Rightarrow SM = \frac{1}{L} \left(\frac{L_f (y_B + y_S) + L y_S}{y_B} \right) \times 100 = 0$$

$$\Rightarrow SM = \frac{1}{L} \left(\frac{L_f y_S + L_f (y_B + y_S)}{y_B} \right) \times 100$$

-> testado
na
simulador

Resumindo :

$$K = \left(\frac{W_h}{G_{dt}} + \frac{W_f}{y_S} \right)$$

$$G_{dt} = y_B - G_{dt} = y_B + y_S$$

$$\Rightarrow K = \frac{W_h}{y_B + y_S} + \frac{W_f}{y_S}$$

$$SM = \frac{1}{L} \left(\frac{L_f y_S + L_f (y_B + y_S)}{y_B} \right) \times 100$$

$$mV(t+\beta) - Y_B \cdot \beta - Y_t \cdot t = Y_S \cdot \delta \quad ①$$

$$I_{zz} \dot{r} - N_r r - N_B \beta = N_S \delta \quad ②$$

$$\text{De } ① \rightarrow mVt + mv\beta - Y_B \beta - Y_t t = Y_S \delta \Rightarrow$$

$$\Rightarrow mv\beta - Y_B \beta + mv t - Y_t t = Y_S \delta \Rightarrow \text{Aplicando desloc}$$

$$\Rightarrow mvS\beta - Y_B \beta + mv t - Y_t t = Y_S \delta \Rightarrow$$

$$\Rightarrow (mvS - Y_B) \cdot \beta + (mv - Y_t) t = Y_S \delta \quad \div mv =$$

$$\Rightarrow \left(s - \frac{Y_B}{mv} \right) \cdot \beta + \left(1 - \frac{Y_t}{mv} \right) t = \frac{Y_S}{mv} \cdot \delta \quad ③$$

$$\text{De } ② \rightarrow Sr - \frac{Nr}{I_{zz}} \cdot r - \frac{Nb}{I_{zz}} \cdot \beta = \frac{Ns}{I_{zz}} \cdot \delta$$

Na forma matricial

$$\begin{bmatrix} \left(s - \frac{Y_B}{mv} \right) & \left(1 - \frac{Y_t}{mv} \right) \\ \frac{-Nb}{I_{zz}} & \left(s - \frac{Nr}{I_{zz}} \right) \end{bmatrix} \begin{bmatrix} \beta(s) \\ t(s) \end{bmatrix} = \begin{bmatrix} \frac{Y_S}{mv} \cdot \delta \\ \frac{Ns}{I_{zz}} \cdot \delta \end{bmatrix}$$

A matriz inversa de

$$\begin{bmatrix} \left(s - \frac{Y_B}{mv} \right) & \left(1 - \frac{Y_t}{mv} \right) \\ \frac{-Nb}{I_{zz}} & \left(s - \frac{Nr}{I_{zz}} \right) \end{bmatrix}^{-1} = \frac{\begin{bmatrix} \left(s - \frac{Nr}{I_{zz}} \right) & \left(1 - \frac{Y_t}{mv} \right) \\ \frac{Nb}{I_{zz}} & \left(s - \frac{Y_B}{mv} \right) \end{bmatrix}}{\left(s - \frac{Y_B}{mv} \right) \left(s - \frac{Nr}{I_{zz}} \right) - \left(1 - \frac{Y_t}{mv} \right) \left(\frac{-Nb}{I_{zz}} \right)}$$

onde o polinomio característico é:

$$D(s) = \left(s - \frac{Y_B}{mv} \right) \left(s - \frac{Nr}{I_{zz}} \right) - \left(1 - \frac{Y_t}{mv} \right) \left(- \frac{Nb}{I_{zz}} \right) \quad ④$$

$$\beta(s) = \frac{\left(s - \frac{Nr}{I_{zz}} \right) \times \frac{Y_S}{mv} - \left(1 - \frac{Y_t}{mv} \right) \cdot \frac{Ns}{I_{zz}}}{D(s)} \cdot \delta(s) \therefore$$

$$\frac{\beta(s)}{\delta(s)} = \frac{\frac{Y_S}{mv} \cdot s + \left(\frac{Y_t \cdot N_S - N_r \cdot Y_S - mV \cdot N_S}{mV I_{zz}} \right)}{D(s)}$$

$$\frac{h(s)}{\delta(s)} = \frac{(s - \frac{y_B}{mV}) \cdot \frac{N\beta}{I_{zz}} - \frac{N\beta}{I_{zz}} \cdot \frac{y_S}{mV}}{D(s)} = 0$$

$$\frac{h(s)}{\delta} = \frac{\frac{N\beta}{I_{zz}} \cdot s - \left(\frac{y_B \cdot N\beta + N\beta \cdot y_S}{mV I_{zz}} \right)}{D(s)}$$

Calculo do fator de amorteecimento, frequencia natural ω_m e amorteecida ω_a

$$D(s) = 0$$

$$(s - \frac{y_B}{mV})(s - \frac{N\beta}{I_{zz}}) - \left(1 - \frac{y_F}{mV} \right) \left(\frac{-N\beta}{I_{zz}} \right) = 0$$

$$s^2 - \frac{N\beta}{I_{zz}} s - \frac{y_B}{mV} s + \frac{y_B}{mV} \cdot \frac{N\beta}{I_{zz}} + \frac{N\beta}{I_{zz}} - \frac{y_F}{mV} \cdot \frac{N\beta}{I_{zz}} = 0$$

$$\Rightarrow s^2 + \frac{-(mV \cdot N\beta + y_B \cdot I_{zz})s}{mV I_{zz}} + \frac{y_B \cdot N\beta + N\beta \cdot mV - y_F \cdot N\beta}{mV I_{zz}} = 0$$

fazendo analogia

$$s^2 + 2\xi \omega_m s + \omega_m^2 = 0$$

$$2\xi \omega_m = - \frac{mV N\beta + I_{zz} \cdot y_B}{mV I_{zz}} = 0$$

$$\xi = - \frac{mV N\beta + I_{zz} \cdot y_B}{2 \cdot \omega_m \cdot mV I_{zz}}$$

$$\omega_m^2 = \frac{y_B \cdot N\beta + N\beta \cdot mV - y_F \cdot N\beta}{mV I_{zz}} = 0$$

$$\omega_m^2 = \frac{y_B \cdot N\beta - N\beta \cdot y_F}{mV I_{zz}} + \frac{N\beta}{I_{zz}}$$

$$\omega_a = \omega_m \cdot \sqrt{1 - \xi^2}$$

Screvendo SM de outra forma:

$$SM = \frac{1}{L} \left(\frac{Lf \cdot y\delta + Lt \cdot (y\beta + y\delta)}{y\beta} \right) \times 100$$

$$Lf = \frac{Wf \cdot L}{W}; \quad Lt = \frac{Wf \cdot L}{W}$$

$$SM = \frac{1}{L} \left(\frac{\frac{Wf \cdot L \cdot y\delta}{W} + \frac{Wf \cdot L \cdot (y\beta + y\delta)}{W}}{y\beta} \right) \times 100$$

$$SM = \frac{L}{W} \cdot \frac{1}{W} \left(\frac{Wf \cdot y\delta + Wf \cdot (y\beta + y\delta)}{y\beta} \right) \times 100$$

$$SM = \frac{1}{W} \left(\frac{Wf \cdot y\delta + Wf \cdot y\beta + Wf \cdot y\delta}{y\beta} \right) \times 100$$

$$SM = \frac{1}{W} \left(\frac{W \cdot y\delta + Wf \cdot y\beta}{y\beta} \right) \times 100$$

$$SM = \left(\frac{W \cdot y\delta}{W \cdot y\beta} + \frac{Wf \cdot y\beta}{W \cdot y\beta} \right) \times 100 \Rightarrow$$

$$SM = \left(\frac{y\delta}{y\beta} + \frac{Wf}{W} \right) \times 100$$

O.K.
testado no simulador

Qndo $y\delta \uparrow$ $SM \downarrow$

Qndo $y\beta \uparrow$ $SM \uparrow$

$$\text{Qndo } SM=0 \Rightarrow \frac{y\delta}{y\beta} = -\frac{Wf}{W}$$

$$\frac{Wf}{W} = -\frac{y\delta}{y\beta} = \frac{C_{df}}{C_{df} + C_{dt}}$$

$$K = 0 \Rightarrow \frac{Wf}{y\beta + y\delta} = -\frac{Wf}{y\delta} \Rightarrow K = \frac{W}{y\beta + y\delta} = \frac{W}{y\beta}$$

$$K = \frac{W}{y\beta + y\delta}$$

Para a solução das equações de movimento, usou-se:

Na forma matricial:

$$\begin{bmatrix} (s+a_{11}) & a_{12} \\ a_{21} & (s+a_{22}) \end{bmatrix} \begin{Bmatrix} y_1(s) \\ y_2(s) \end{Bmatrix} = \begin{bmatrix} b_{11}U_1(s) + b_{12}U_2(s) \\ b_{21}U_1(s) + b_{22}U_2(s) \end{bmatrix}$$

$$\begin{Bmatrix} y_1(s) \\ y_2(s) \end{Bmatrix} = \begin{bmatrix} (s+a_{11}) & a_{12} \\ a_{21} & (s+a_{22}) \end{bmatrix}^{-1} \begin{bmatrix} b_{11}U_1(s) + b_{12}U_2(s) \\ b_{21}U_1(s) + b_{22}U_2(s) \end{bmatrix}$$

A matriz inversa é:

$$\begin{bmatrix} (s+a_{11}) & a_{12} \\ a_{21} & (s+a_{22}) \end{bmatrix}^{-1} = \frac{\begin{bmatrix} (s+a_{22}) & -a_{12} \\ -a_{21} & (s+a_{11}) \end{bmatrix}}{(s+a_{11})(s+a_{22}) - a_{12}a_{21}} \quad \ddots$$

$$\begin{Bmatrix} y_1(s) \\ y_2(s) \end{Bmatrix} = \frac{\begin{bmatrix} (s+a_{22}) & -a_{12} \\ -a_{21} & (s+a_{11}) \end{bmatrix}}{(s+a_{11})(s+a_{22}) - a_{12}a_{21}} \begin{bmatrix} b_{11}U_1(s) + b_{12}U_2(s) \\ b_{21}U_1(s) + b_{22}U_2(s) \end{bmatrix}$$

onde $D(s) = (s+a_{11})(s+a_{22}) - a_{12}a_{21}$ é comum a $y_1(s)$ e $y_2(s)$, sendo chamado de polinômio característico.

Assim,

$$y_1(s) = \frac{(s+a_{22})b_{11} - a_{12}b_{21}}{D(s)} U_1(s) + \frac{(s+a_{22})b_{12} - a_{12}b_{22}}{D(s)} U_2(s)$$

$$y_2(s) = \frac{(s+a_{11})b_{21} - a_{21}b_{11}}{D(s)} U_1(s) + \frac{(s+a_{11})b_{22} - a_{21}b_{12}}{D(s)} U_2(s)$$

Da equações de movimento,

$$a_{11} = \frac{-\gamma \beta}{mv} ; a_{12} = \left(1 - \frac{\gamma r}{mv} \right) ; b_{11} = \frac{\gamma \delta}{mv} ; b_{12} = 0$$

$$a_{21} = \frac{-N\beta}{I_{zz}} ; a_{22} = \frac{-Nr}{I_{zz}} ; b_{21} = \frac{Ns}{I_{zz}} ; b_{22} = 0$$

 X