

# **Introduction – Optimization techniques applied to projects of mechanical parts**

**PMR - 5215 – Optimization applied to projects of  
mechanical systems**

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Mecânicos**

## **Presentation outline**

- Introduction
- Parametric optimization
- Shape optimization
- Topology optimization (OT)
- Industrial applications of topology optimization

# What is optimization concerned with?

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Consider the following problem: to maximize the stiffness of an automobile car chassis?

Parameters that may be altered:

- width of reinforcers ( $b_1, b_2$ )
- moment of inertia of the reinforcers ( $I$ );
- distance between reinforcers ( $L_1$  e  $L_2$ );
- position of the reinforcers ( $L_3$  e  $L_4$ );
- sheet thickness at different points ( $h_1, h_2$ )
- chassis material ( $E$ );

Total: 10 parameters subject to modification

Suppose that each parameter may adopt 10 fixed values.

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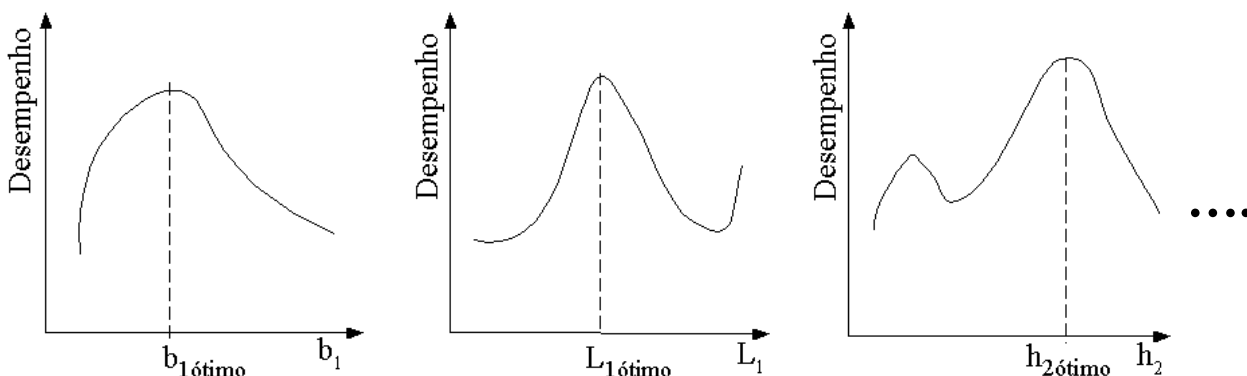
# What is optimization concerned with?

4

First approach to solve the problem:

## Analysis

- Execute analysis for each combination of parameters  $b_1, b_2, L_1, L_2$ , etc.;
- Plot the performance of each parameter;
- Find the combination of parameters that offers the best performance.



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## Consequences of the analysis-based approach:

Considering only 3 parameters → analysis of  $10^3$  combinations of parameters.

If the time spent for each analysis = 0,1 s → Total time: 100s

Now considering only 10 parameters → analysis of  $10^{10}$  Combinations of parameters.

If the time spent for each analysis = 10s → Total time =  $10^{11}$ s = 3200 anos!!

**This approach is unfeasible for a large number of parameters!!**

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## Second approach:

### Optimization (or Synthesis)

- Use of computational methods that search, in a rational way, an optimal solution;
- Renders the search for the optimal point automatic and systematic, i.e., independent of the experience of the designer;
- The time required for the solution of the previous problem would be shortened to a few hours, for example.

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## Basic definition of an optimization problem

**Minimize (or Maximize) Objective Function (stiffness, (project variables) resonance frequency, etc.)**

**Subject to Restrictions (maximum mass or volume, displacement, mechanical tension, etc...)**

**Project variables (parameters subject to modification during the optimization):**

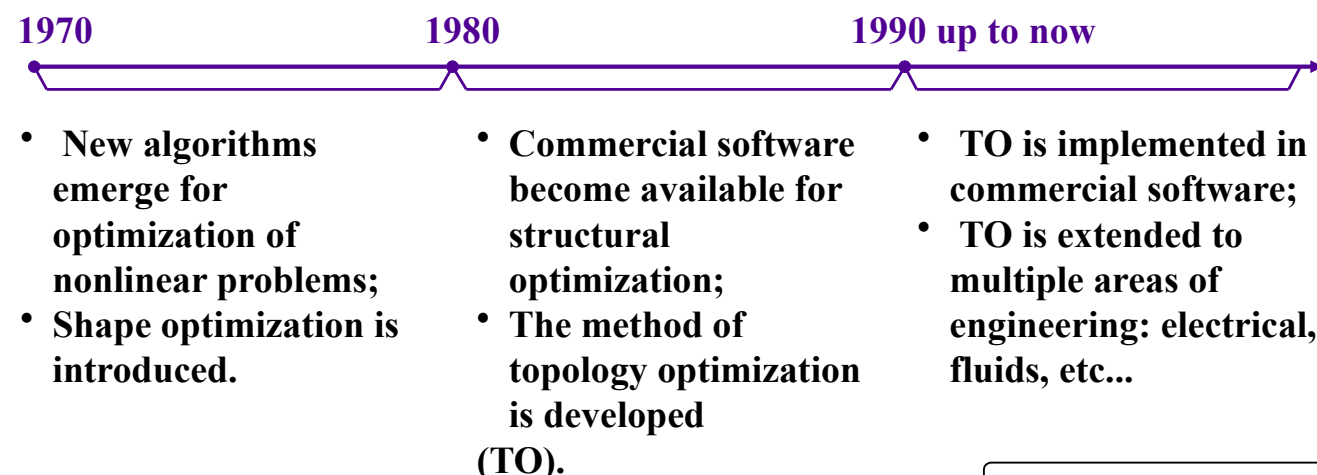
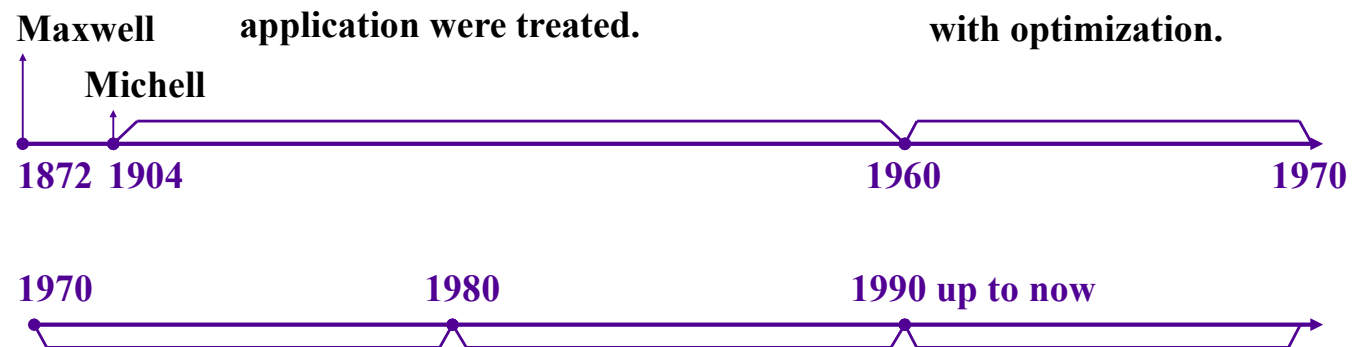
- part dimensions;
- curve parameters representative of the part shape;
- material distribution of a part;

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## History of Optimization in Mechanical Engineering

There was no evolution in studies of structural optimization. Only academic problems with no practical application were treated.

Development of FEM: Practical problems started to be studied with optimization.

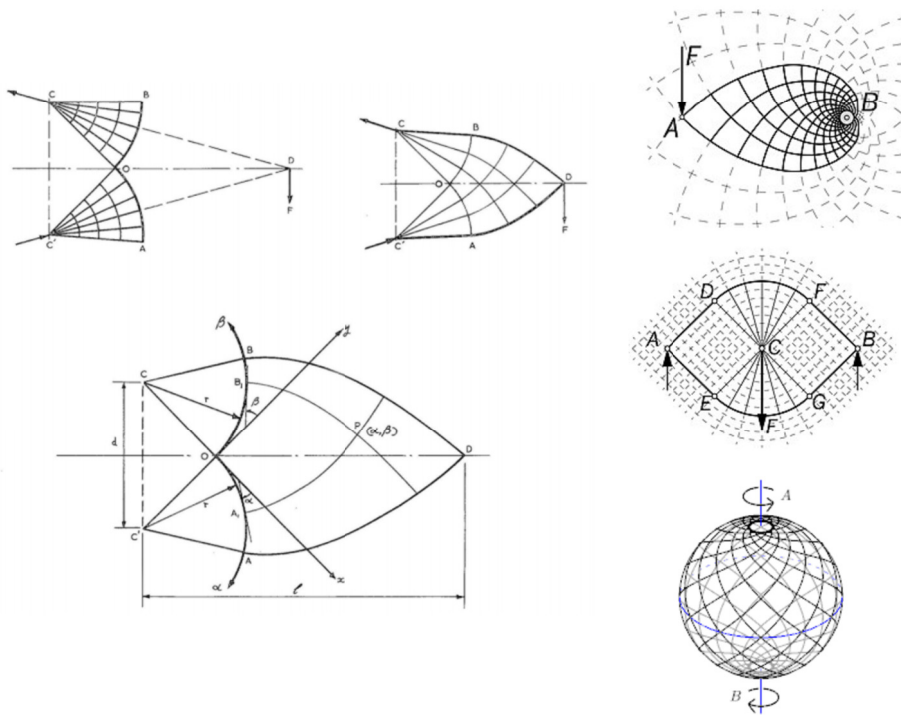


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- “The limits of economy of material in frame structures”, (1904)

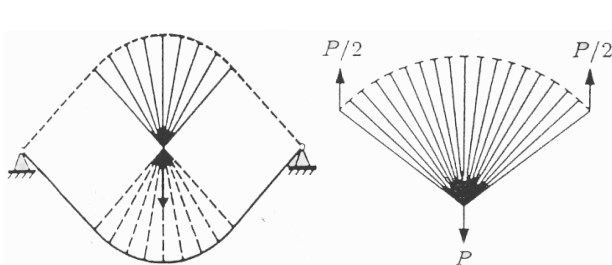


Anthony G.M. Michell  
(1870-1959)



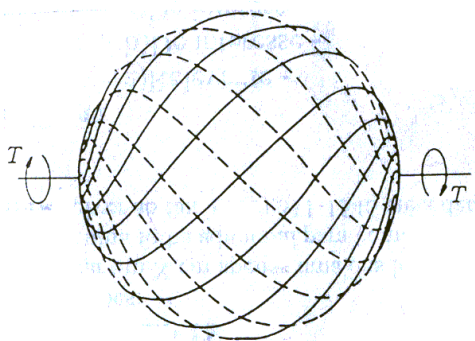
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## Michell (1904)

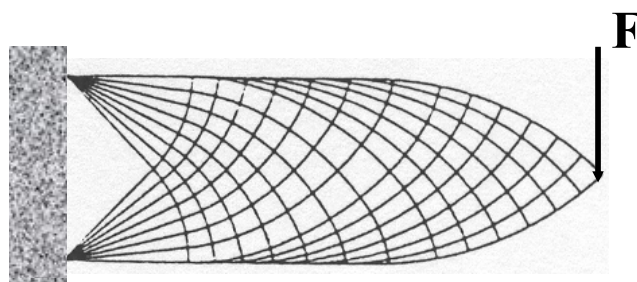


Bi-supported structure

Handle

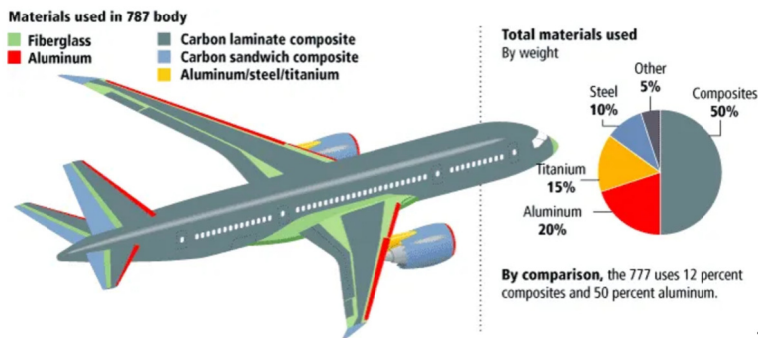
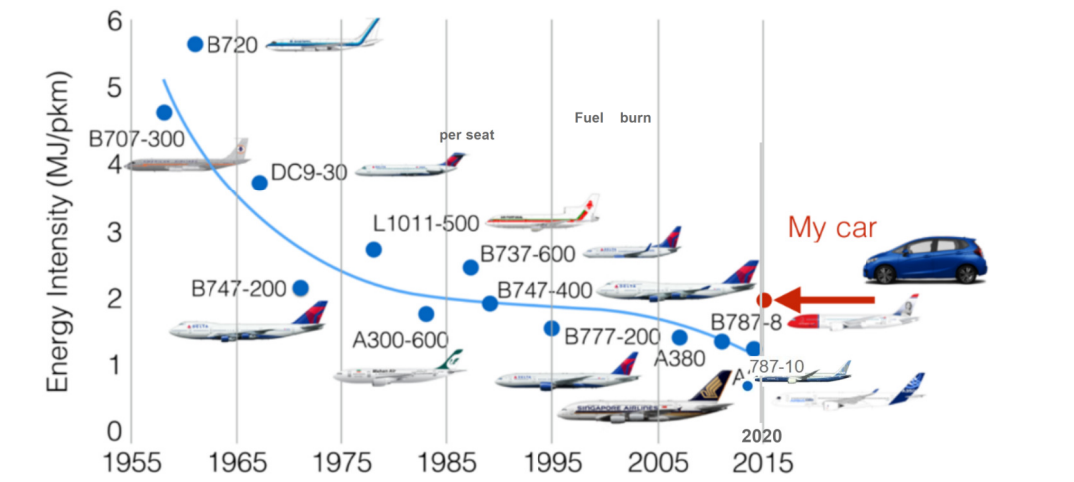


Structure under torsion



Fixed-support structure

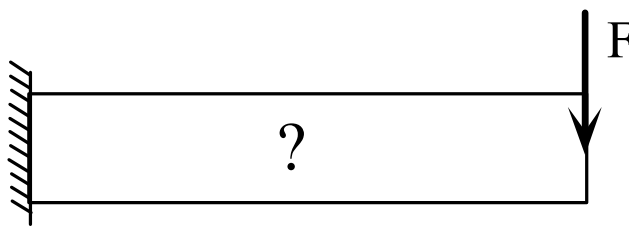
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## Available Optimization Techniques

Consider the following problem:

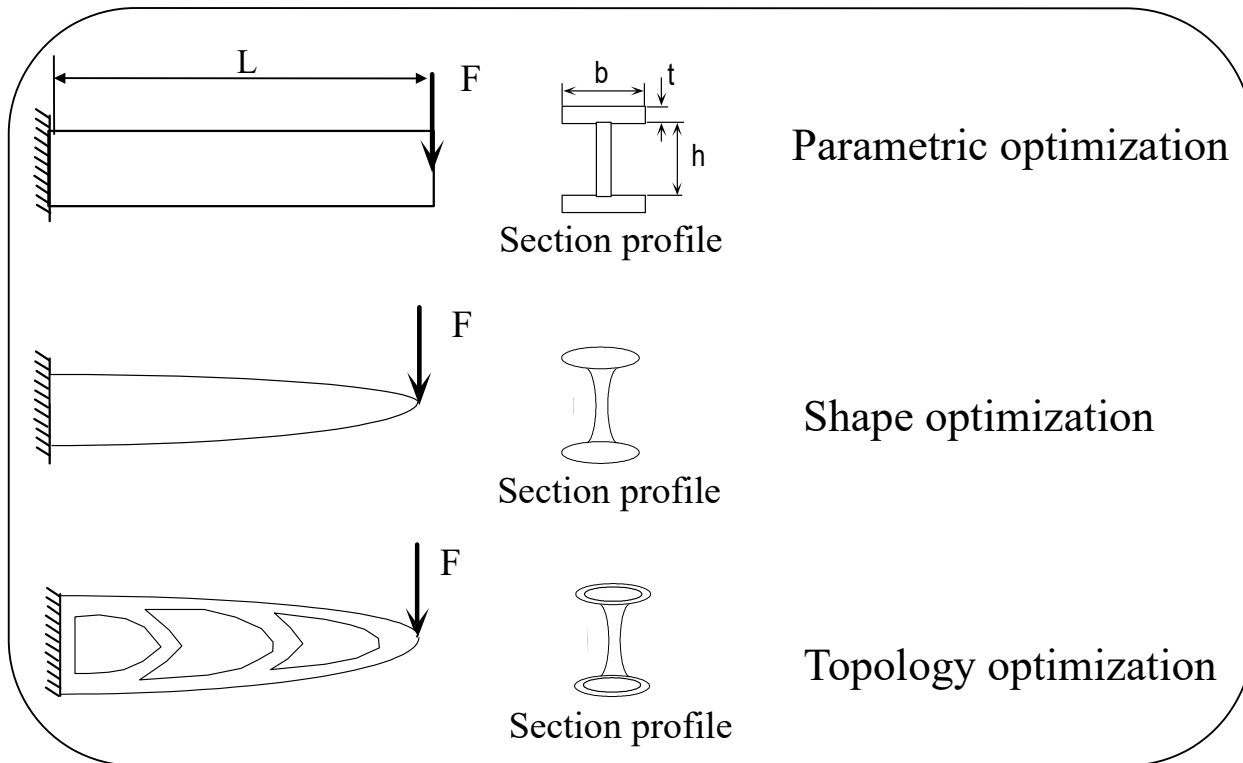


Find the structure that:

*Minimizes Compliance (or maximizes stiffness)  
subject to Restriction on volume of material*

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Possible approaches toward the solution:



Parametric optimization

Section profile

Shape optimization

Section profile

Topology optimization

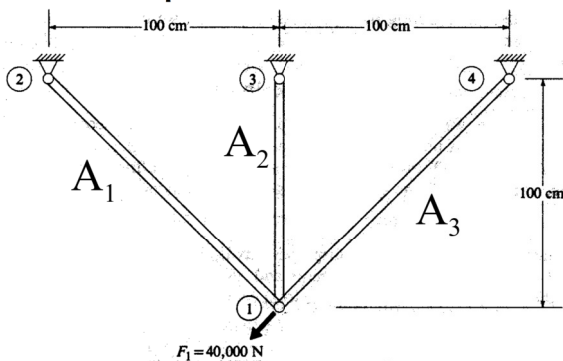
Section profile

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## Parametric Optimization

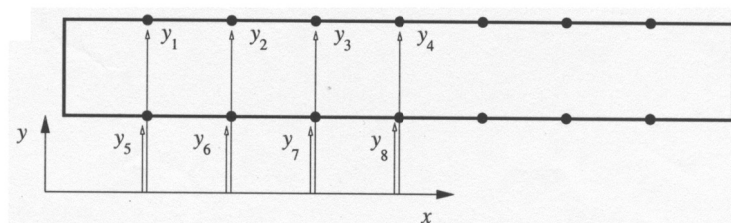
- Project variables are the dimensions or ratio of the dimensions of the part;
- It does not alter the shape of the part, only its aspect;

Examples:



Variables are the areas  $A_1$ ,  $A_2$  and  $A_3$ .

Variables are the coordinates of the nodes

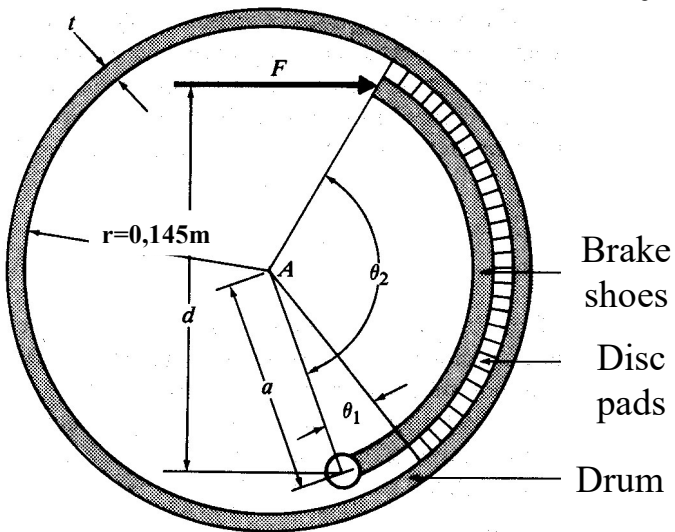


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## Project of an automobile breaking system

Rotation direction

Vehicle of 2725 Kg at 90 Km/h must stop in 20s.



Min Mass of the drum  
such that

Breaking time  
Parts' restrictions

Project variables: dimensions above

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## Parametric Optimization

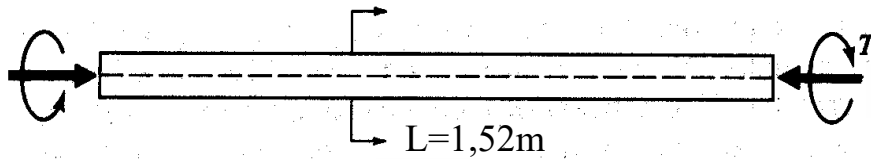
Results obtained with an optimization software that uses finite elements:

Parameters	Initial value	Optimal value
Width $b$	0.08 m	0.08 m
angle $\theta_2$	2.10 rad	1.92 rad
Ratio $a/r$	0.75	0.755
Drum's thickness $t$	0.010 m	0.020 m
Force $F$	2815 N	2086 N
Breaking time $tm$	7.04 s	7.00 s
Maximum temperature $T$	348.7°C	229.2°C
Drum's volume $V$	754 cm <sup>3</sup>	1590 cm <sup>3</sup>

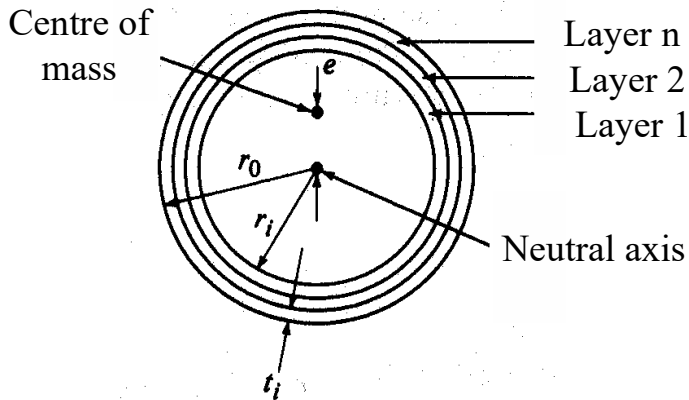
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## Project of a composite axis



Axial force  
and torque



Axis with 4 layers:

- 1<sup>st</sup> layer is parallel to the axis;
- 4<sup>th</sup> layer is normal to the axis;
- 2<sup>nd</sup> and 3<sup>rd</sup> layers are oriented with an angle  $\theta$  and  $-\theta$  relative to the axis;

**Project conditions:** to transmit 1000 HP at 6000 rpm with axial load of 13350 N and to transmit 1500 HP at 8000 rpm without axial load.

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# Parametric optimization

Min Axis mass

subject to

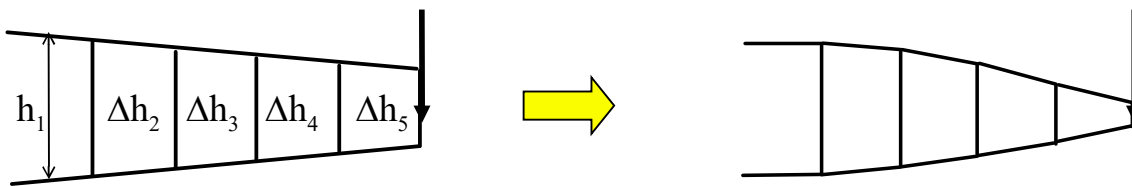
Restrictions on buckling mechanical stress  
Restrictions on critical frequency response

Results obtained with an optimization software used together with an analytical model of the axis:

	Steel	Composite
Radius $r_1$ , cm	12.7	6.91
Thickness $t_1$ , cm	0.1258	0.3834
Thickness $t_2$ , cm	—	0.0396
Thickness $t_3$ , cm	—	0.0396
Thickness $t_4$ , cm	—	0.0
Layer angle $\theta$ , degrees	12.03	4.90
Mass, Kg		

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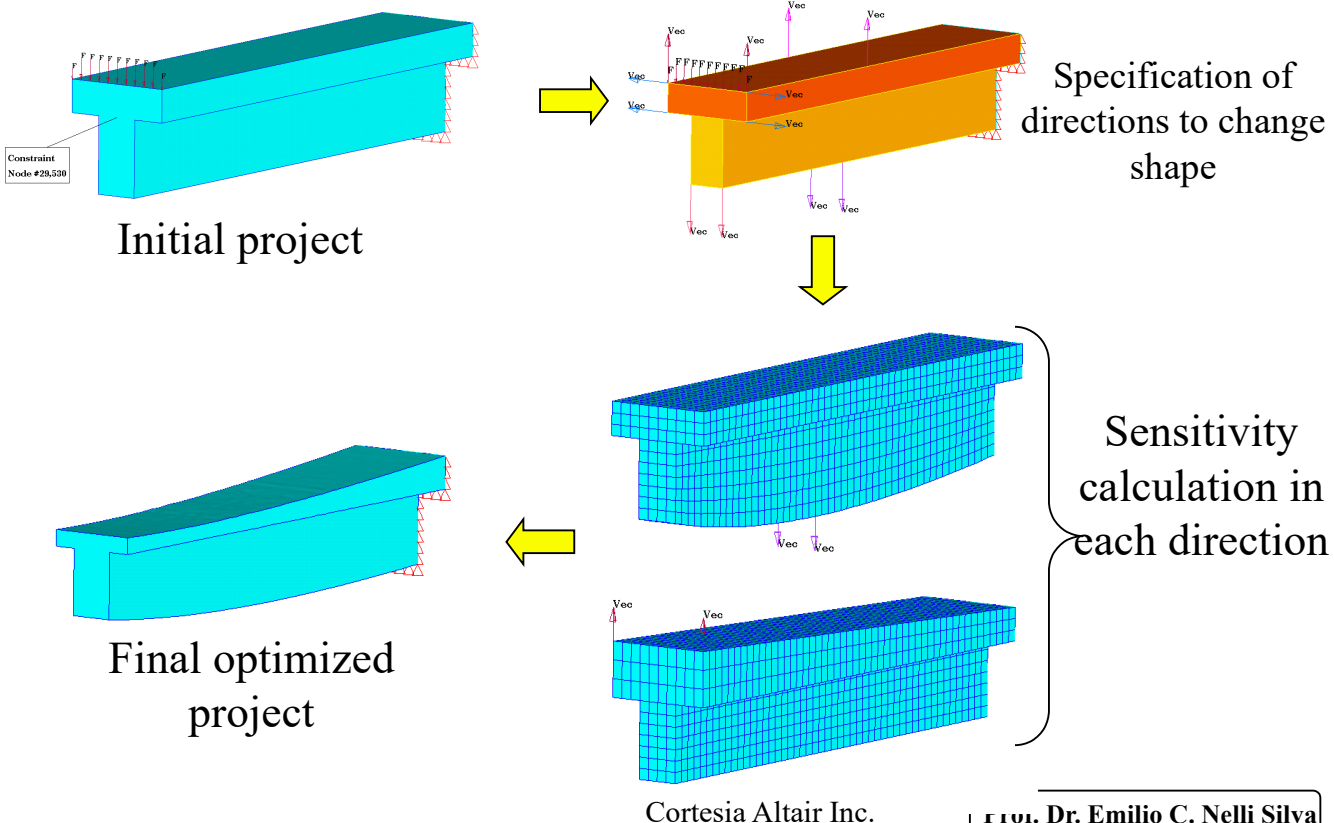
- Alters only the structure's shape, and does not allow to find new 'hole' in the domain;
- Project variables are coordinated by grid nodes of FEM or curve coefficients representative of the structure's shape (e.g., 'spline' curves);
- Requires techniques for FEM grid update, because it is distorted during optimization;



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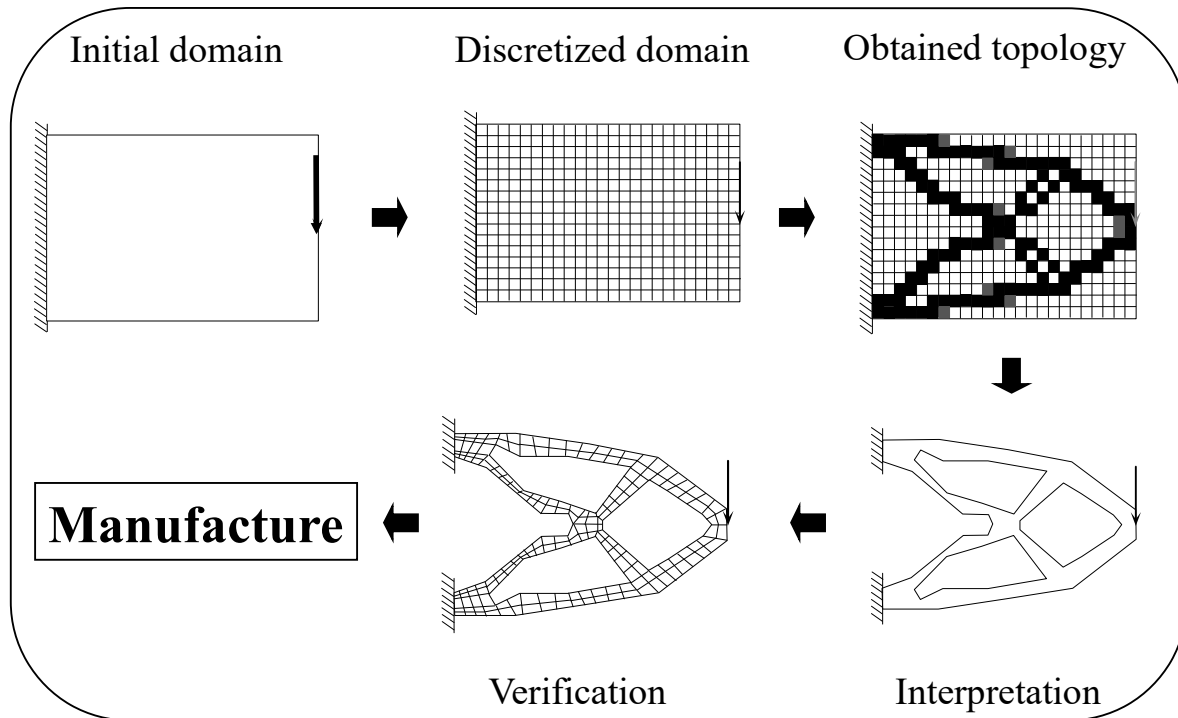
# Shape optimization

## Example of procedure



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Typical procedure:



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# Topology Optimization (TO)

Topology optimization (TO) combines:

- Finite Element Methods (FEM);
- Optimization algorithms (mathematical programming, etc....);

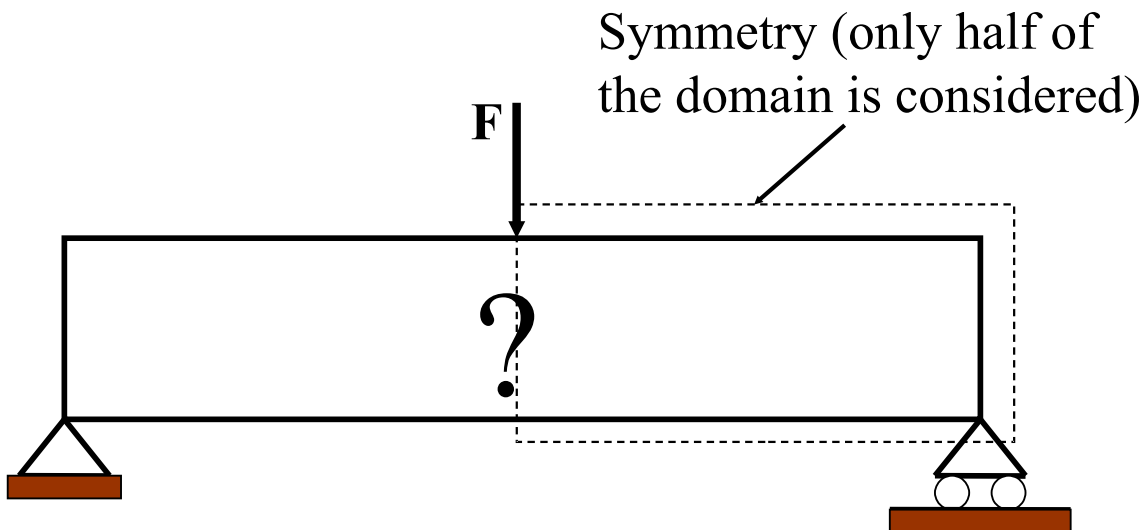
To find an optimal distribution of material for a **fixed** domain.

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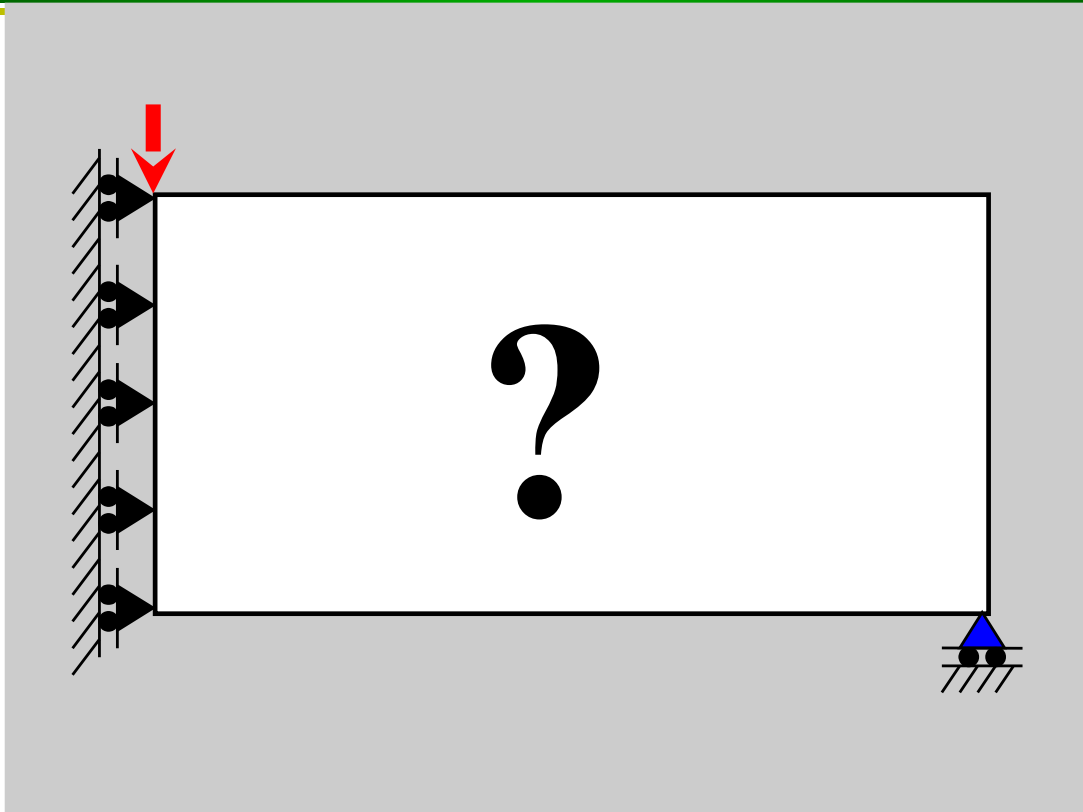
Find the structure of the domain below that:

*Minimizes Compliance*

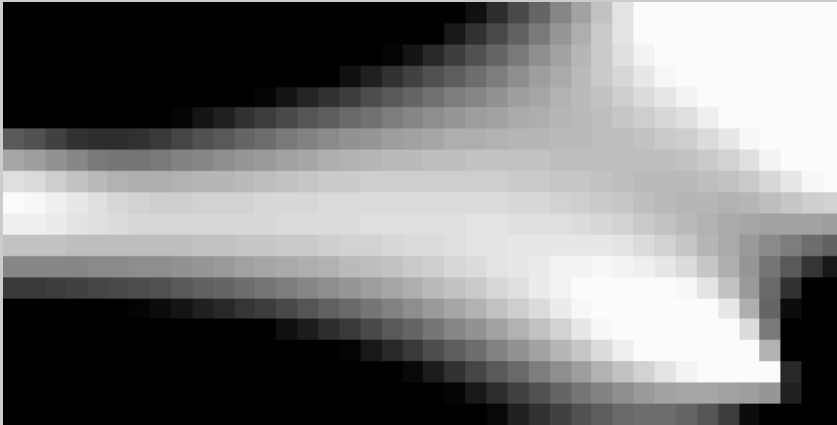
*Subject to Volume restriction*



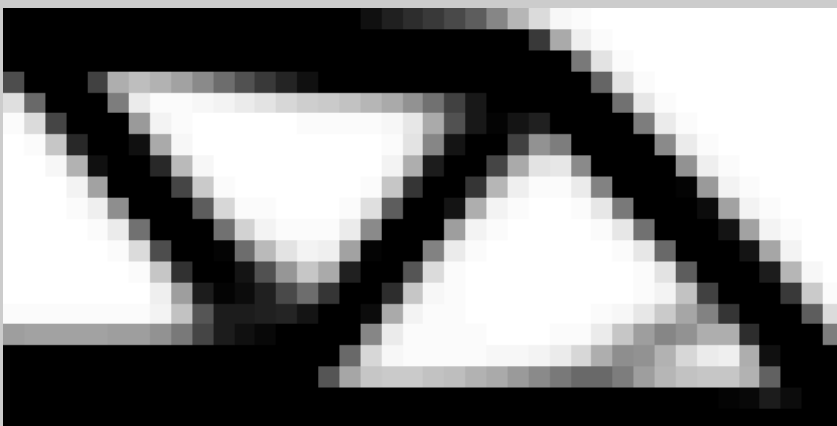
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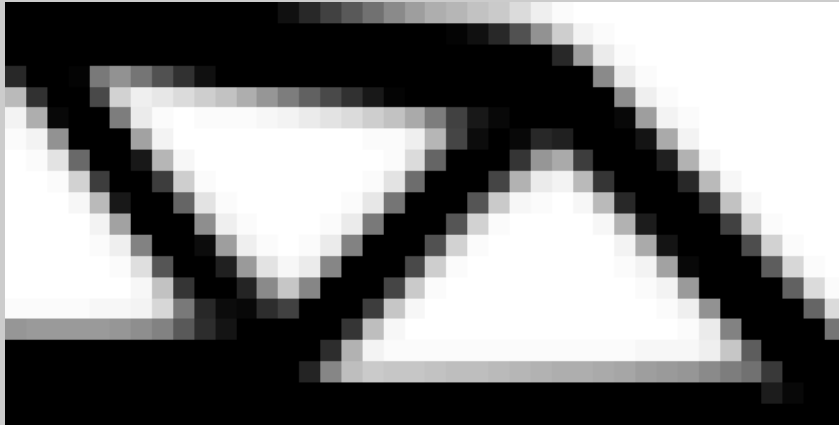
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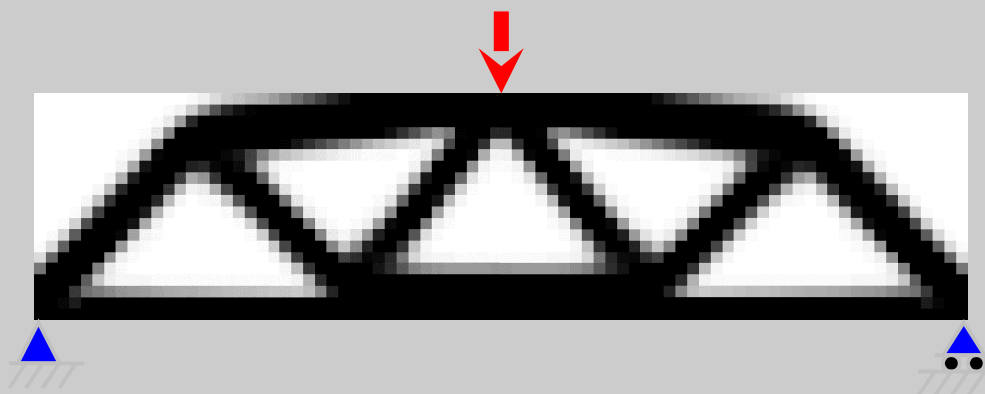
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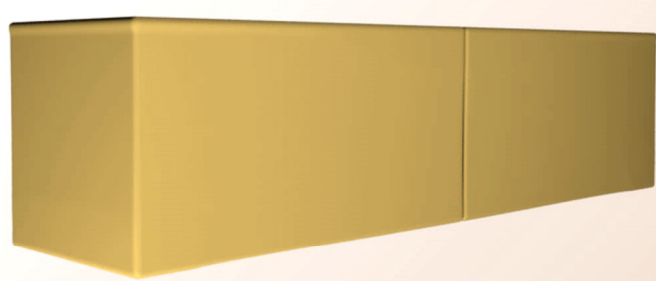
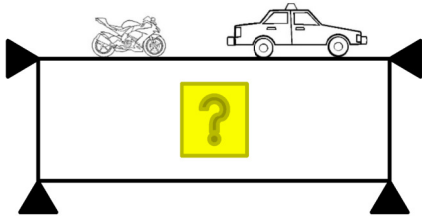
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## XXI century – Optimized projects

### Project of a 3D bridge:



Better performance (stiffness)  
Lower volume (effective use of material)

[m2do.ucsd.edu/](http://m2do.ucsd.edu/)

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## How it all started...

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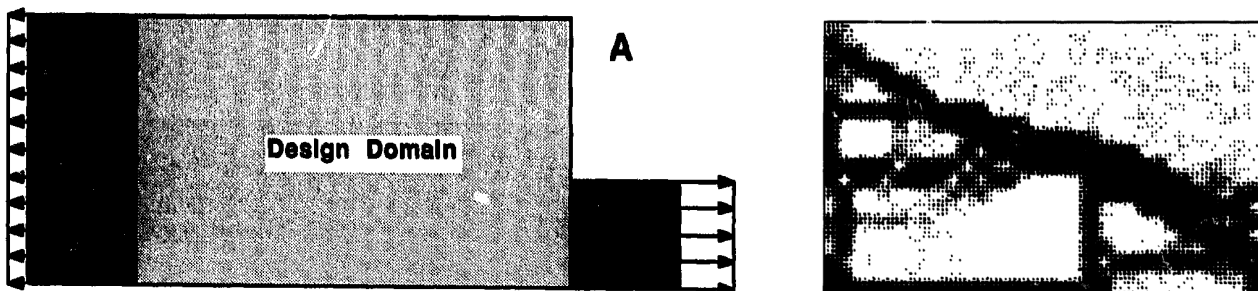
M. P. Bendsøe and N. Kikuchi, “Generating optimal topologies in structural design using a homogenization method”, *Computer Methods in Applied Mechanics and Engineering* 71, pp.197–224, 1988.



M.P. Bendsøe



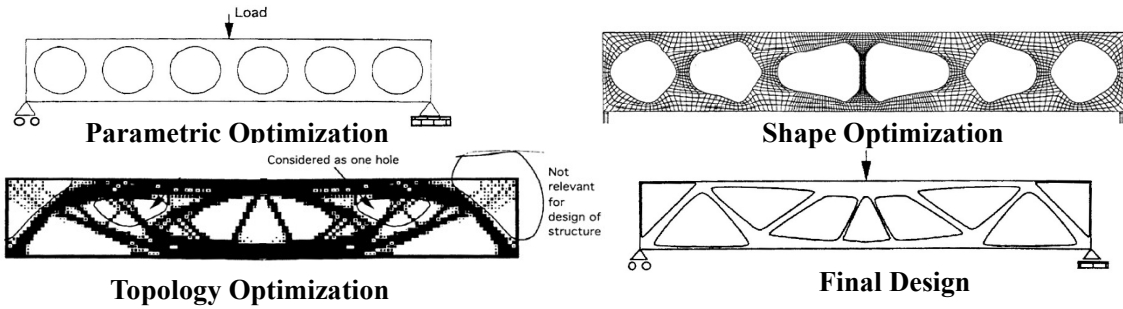
Noboru Kikuchi



In 2018 it was celebrated 30 years!!!

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“MBB Beam - Messerschmitt-Bolkow-Blohm” – Airbus 319 (1989)

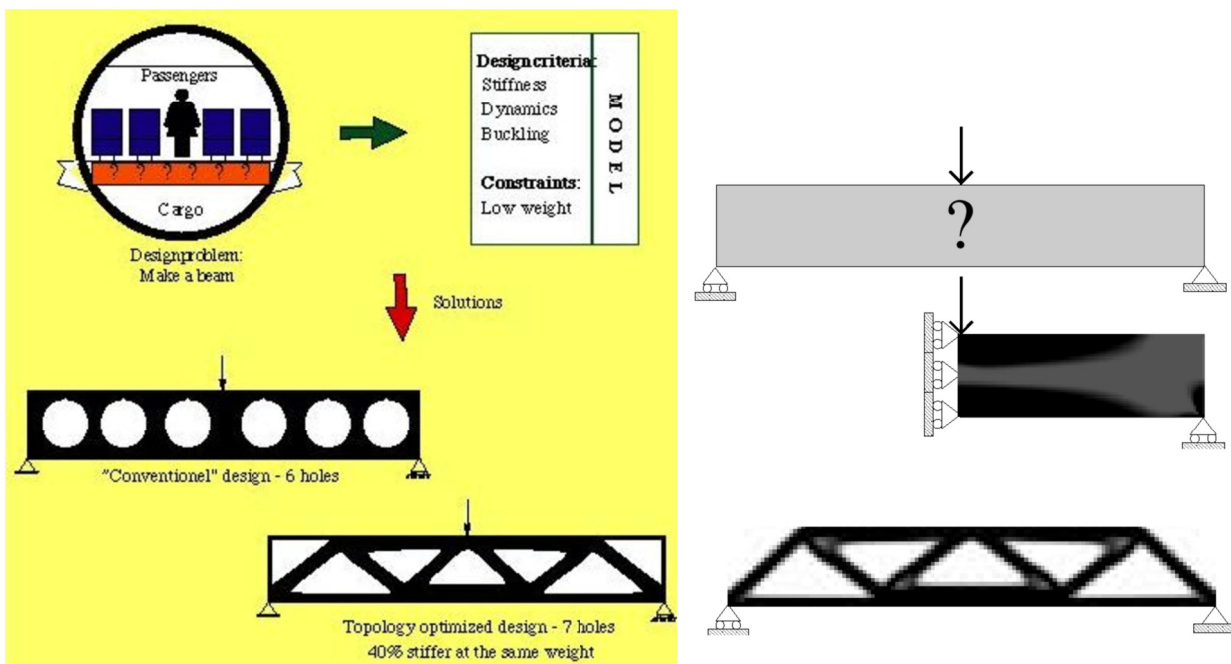


	Volume	Deflection	Max. Stress	Weight Reduction
Initial, unfeasible design	1.07	10.1	292	
Parametric optimization	1.02	10.1	248	4.7%
Shape optimization	0.95	10.1	372	11.2%
Topology optimization	0.65	10.1	227	39.2%
Final design	0.58	10.1	305	45.7%

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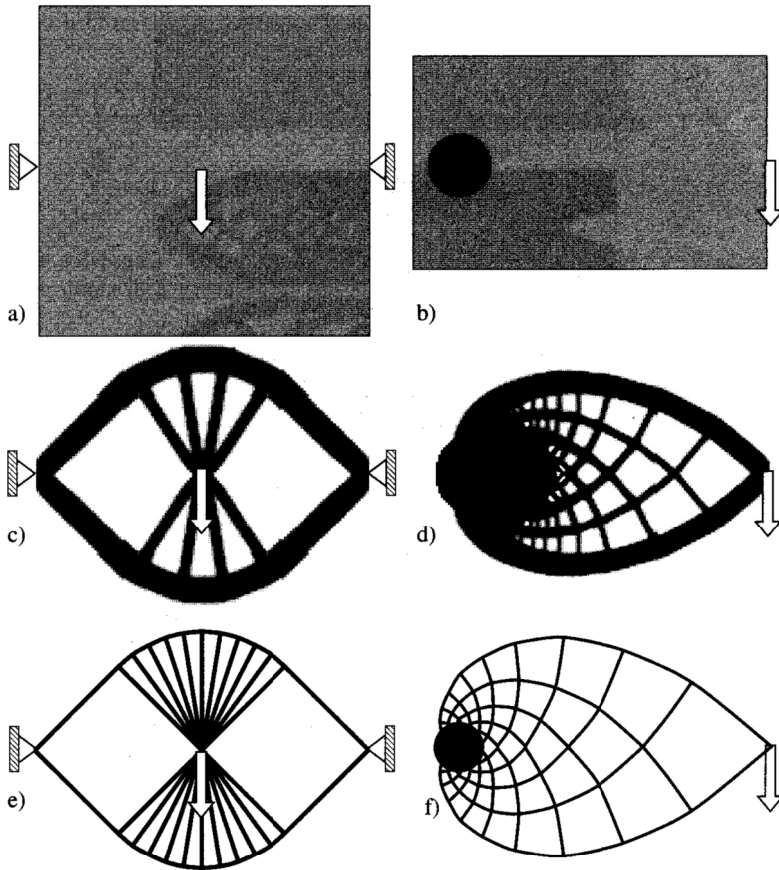
## Examples

“MBB - Messerschmitt-Bolkow-Blohm” beam



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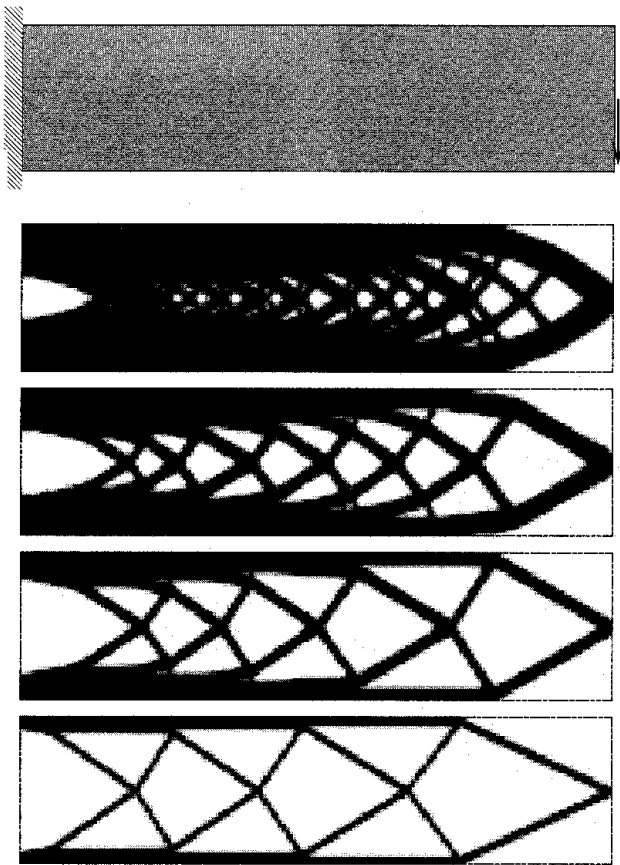


Start domain

Result of Topology Optimization

Analytical result (Michell, 1904)

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Lowering the volume restriction

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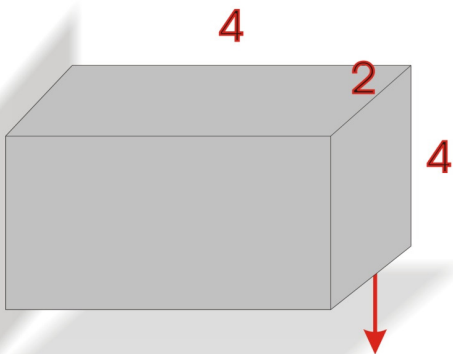
- Structural problems in general:
  - \* maximization of resonance frequency (or buckling load);
  - \* minimization of response frequency;
  - \* maximization of the absorption of impact energy;
  - \* etc...
- Flexible Mechanisms;
- MEMS (“Microelectromechanical Systems”)
- Piezoelectric transducers (actuators, engines);
- Electromagnetic devices:
  - \* maximization of the ratio torque/volume in electrical engines;
  - \* maximization of the receptivity and emissivity in antennae;
- Projects of composite materials with desirable properties;

Check out: <http://www.topopt.dtu.dk>.

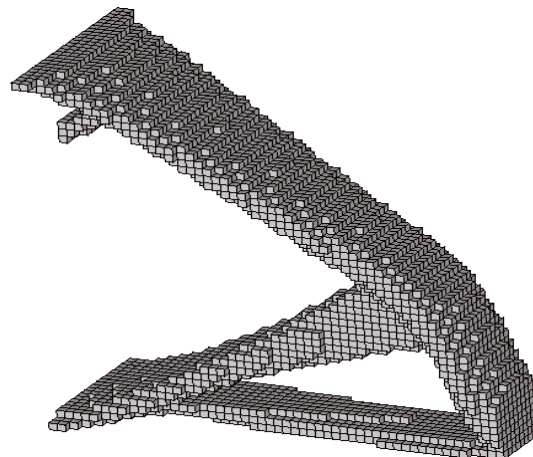
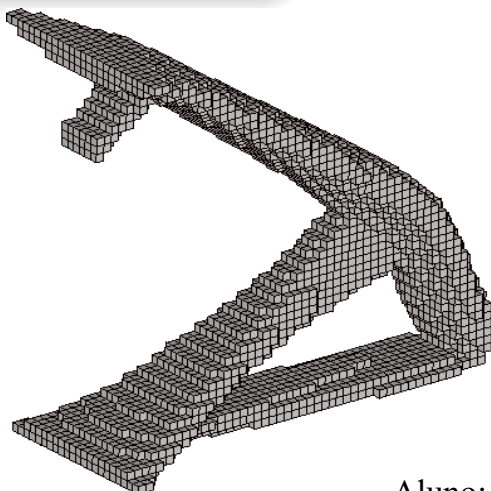
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## Example

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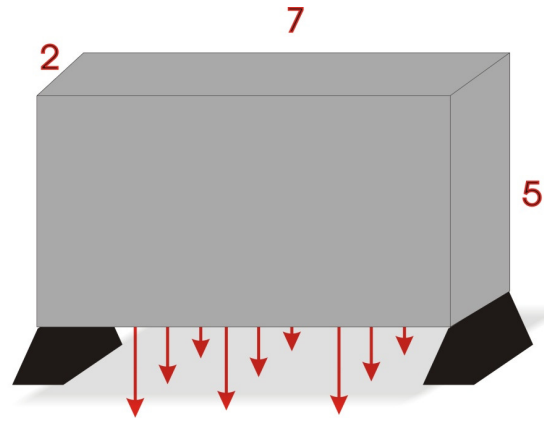


Start domain: 65.000 finite elements of the “brick” type  
CPU time: 37 hours  
Volume restriction: 35%

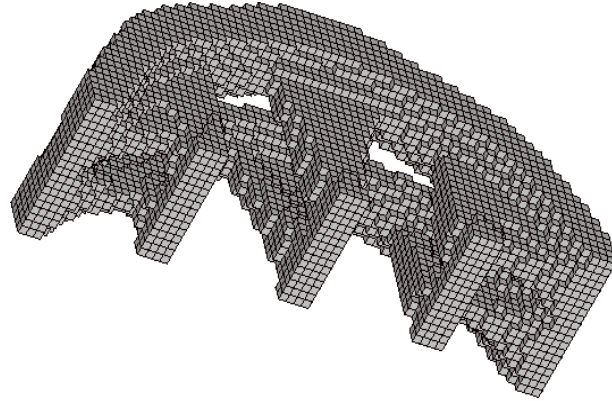
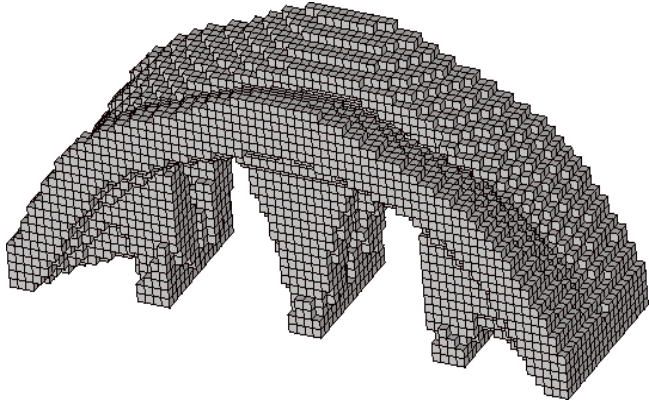


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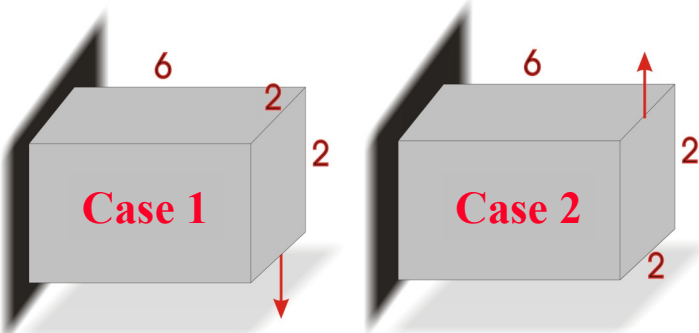


Start domain: 70.000 finite elements of “brick” type  
CPU time: 42 hours  
Volume restriction: 35%

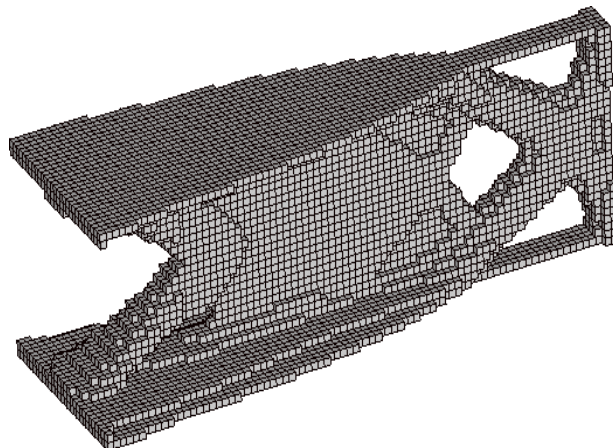
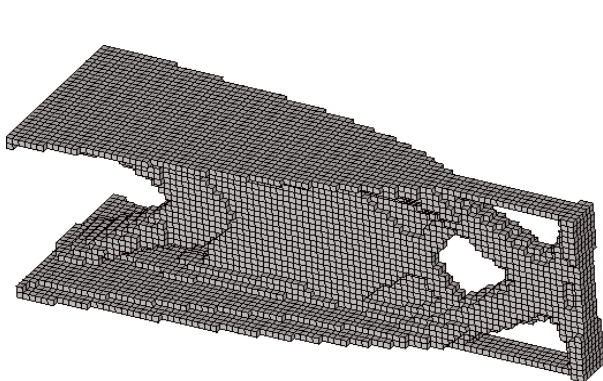


Aluno: James Edward Shooter

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Start domain: 73.000 finite elements of “brick” type  
CPU type: 48 hours  
Volume restriction: 35%

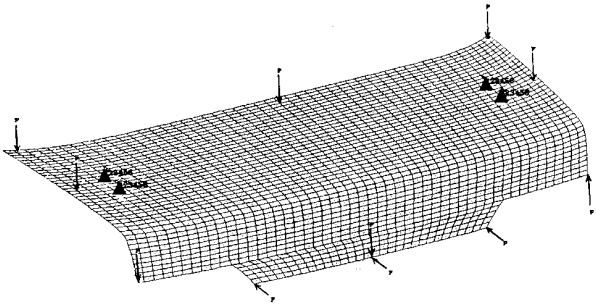


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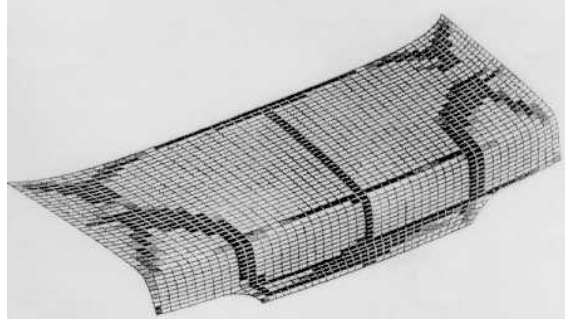
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## Optimal sheet reinforcement with maximal ratio stiffness/volume

Automobile hood



Loads



Optimized reinforcement

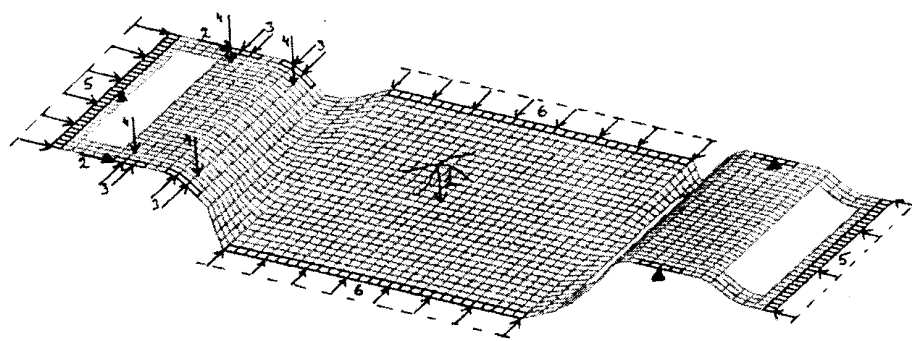
(Bendsoe&Kikuchi 1988)

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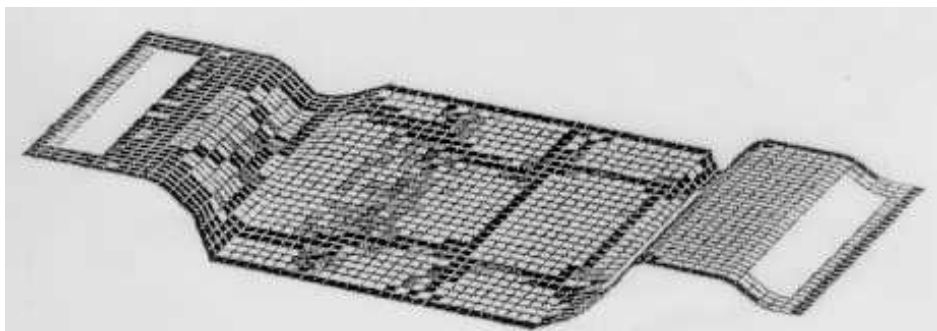
# Example

Automobile chassis

Loads

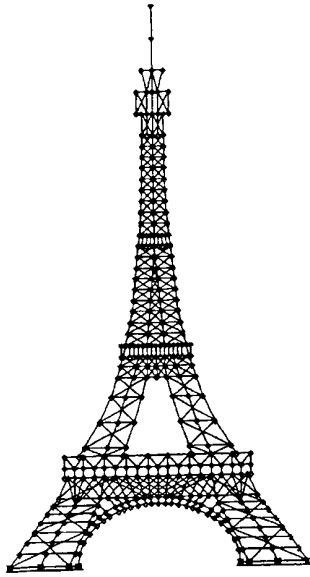


Optimized reinforcement

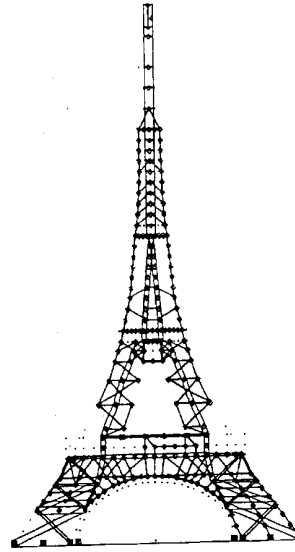


(Bendsoe&Kikuchi 1988)

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Initial project



Optimized project

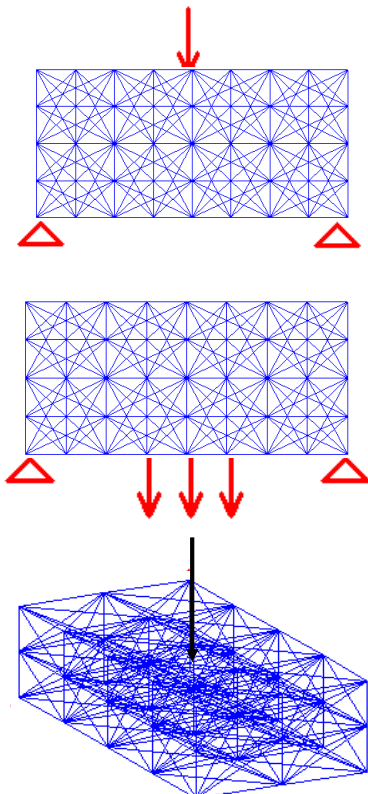
(Bendsoe&Kikuchi 1988)

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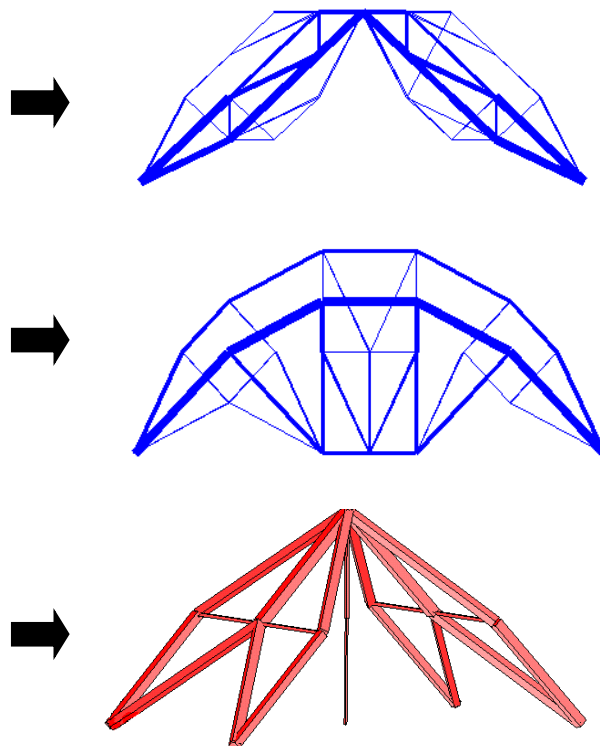
## Example results of OT application to trusses

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Initial project domain

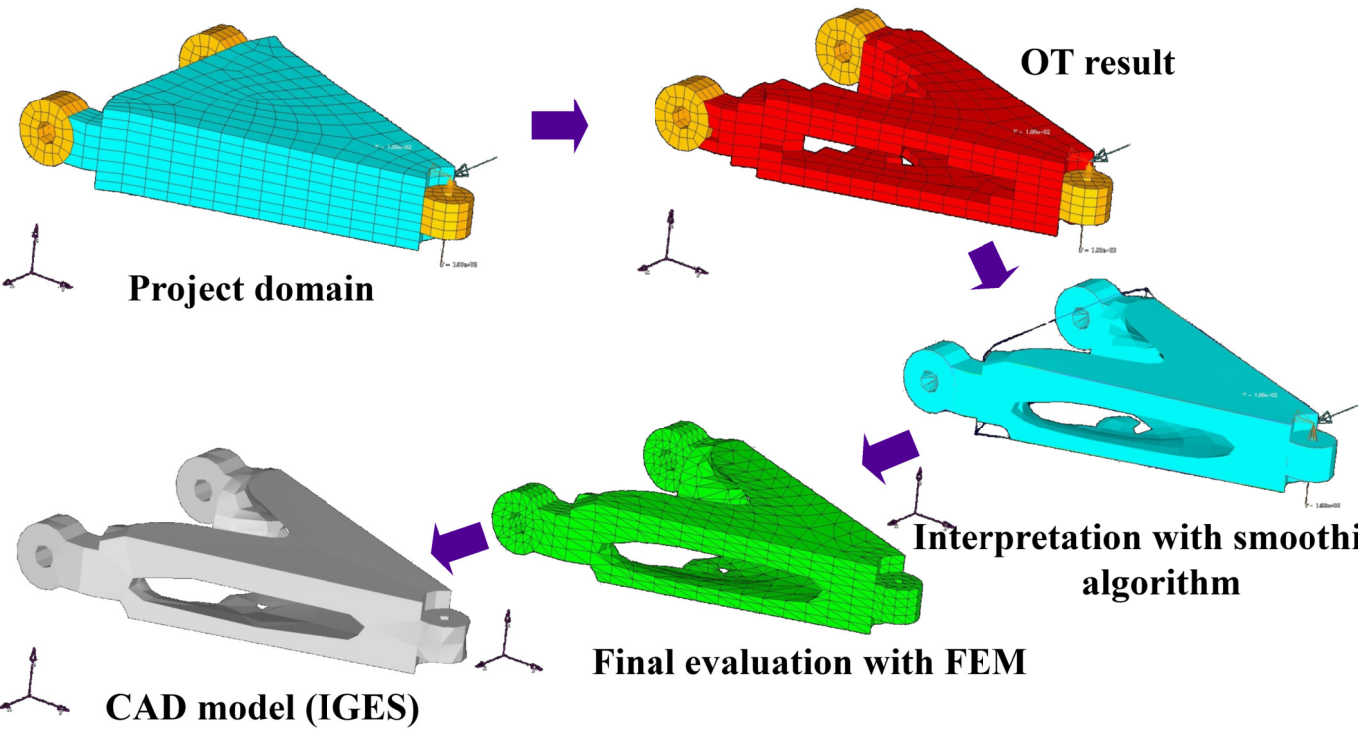


OT result



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## Example 2: Truck front suspension arm.

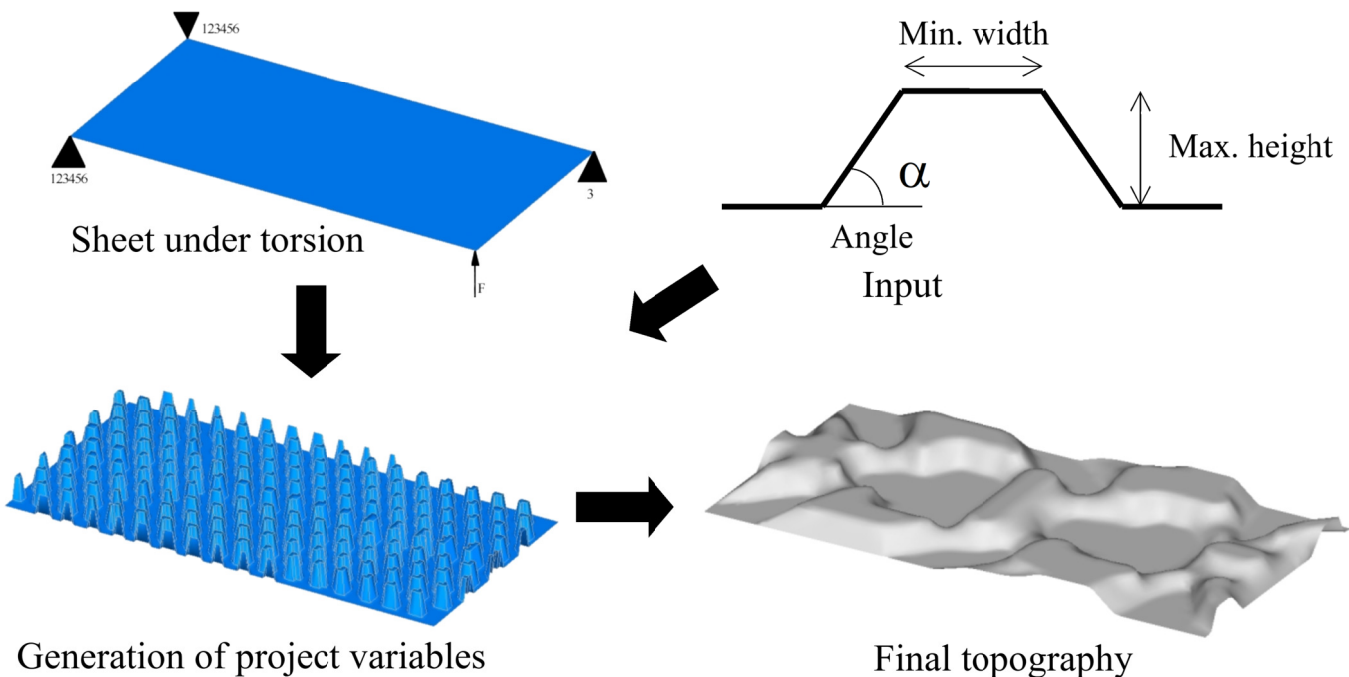


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## Topography Optimization

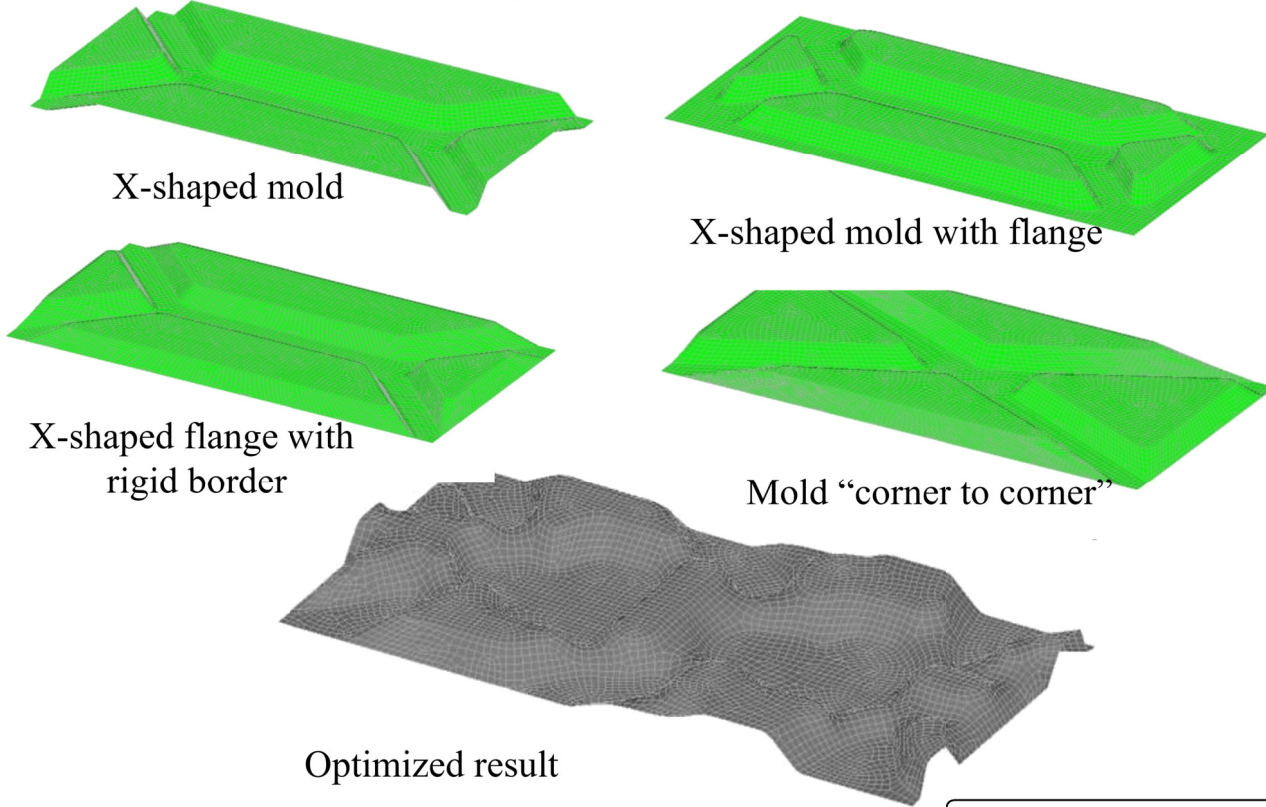
Objective: Find the distribution of reinforcement pattern in sheet and shell structures



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Relevance → Ex.: Sheet subject to torsion



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Comparison of performance amongst the solutions

Solution	Max. Deflection	Max. Stress
Optimized	1.17	196
X-shaped	2.23	267
X-shaped with flange	4.41	644
X-shaped with rigid border	10.57	520
Corner-to-corner	6.47	434

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- The domain occupied by the weld is discretised by finite elements, in the case of continuous welding (ACM), as much as in the case of point by point (MIG)
- This domain is defined as project domain in topology optimization.

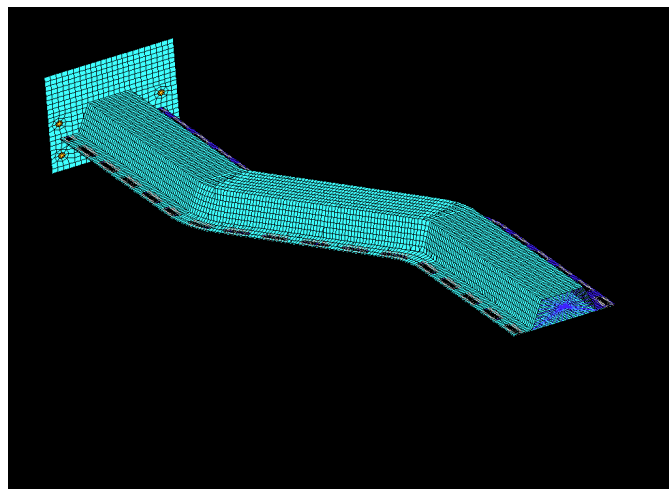
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## Example

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- Model: fixed beam subject to flexion and torsion.
- Initial configuration: 34 MIG welds - weld domain was discretised by 260 solid elements.

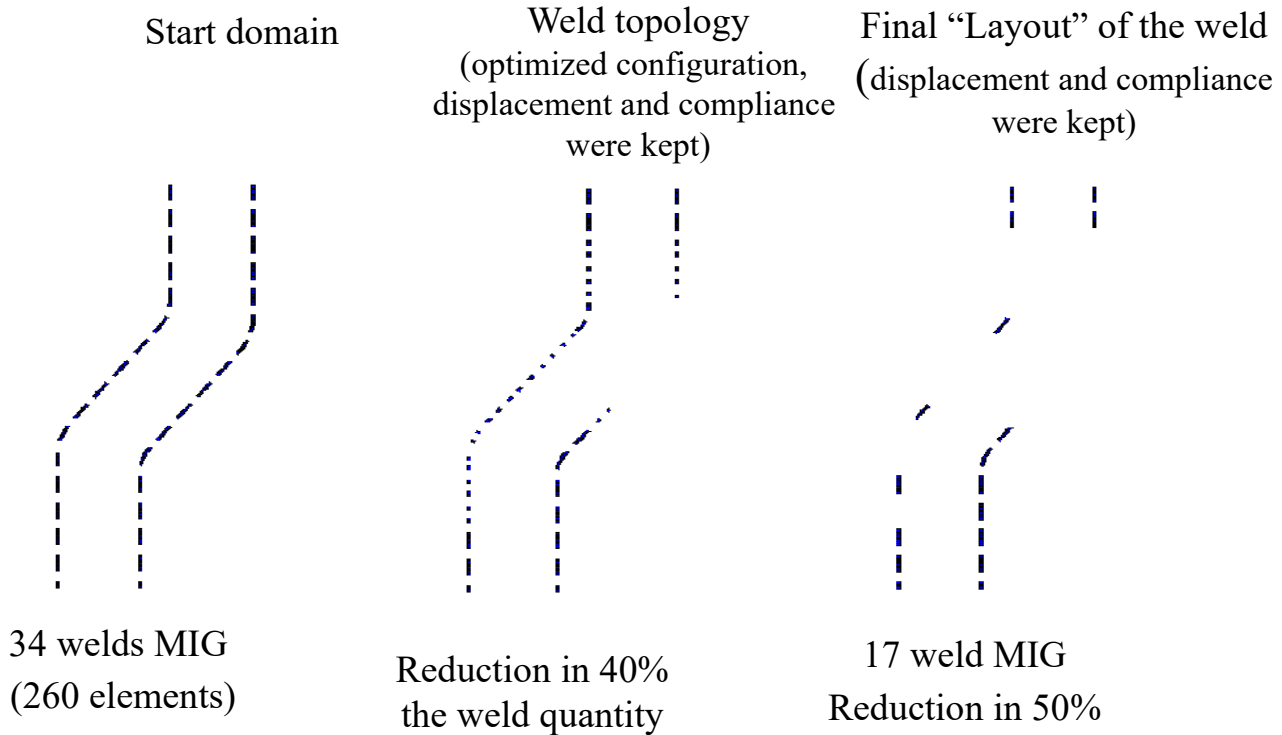
Objective: Keep the same original stiffness, restricting the volume fraction of solid elements that comprise the weld in 30%.



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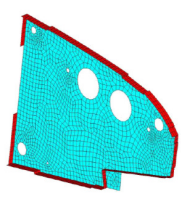
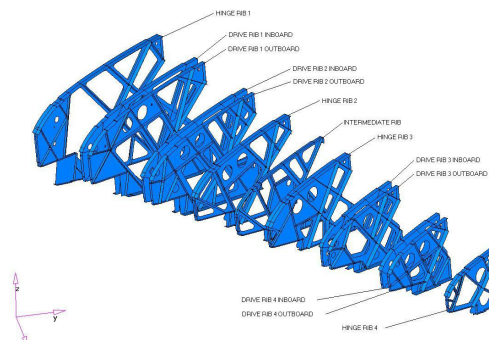


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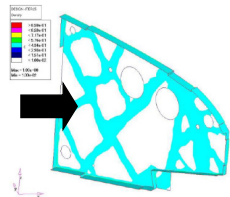
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## Industrial applications

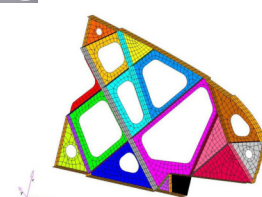
### Application: Airbus A380



Project domain



OT solution

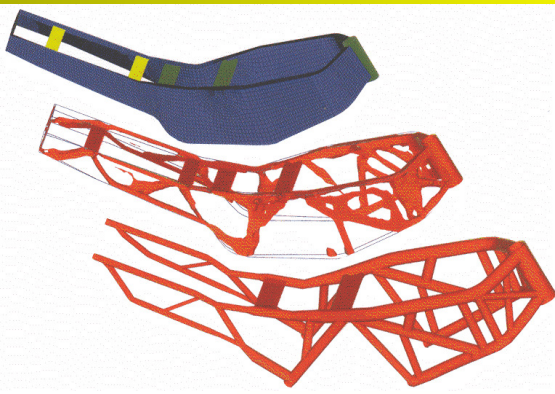


Verification and interpretation



Manufacturing

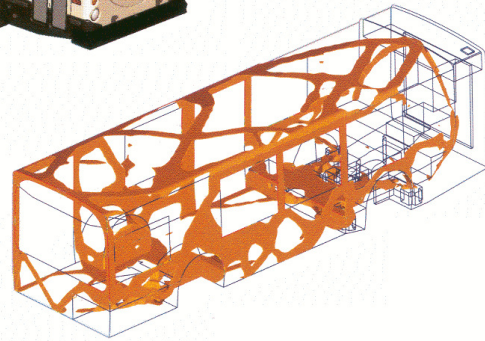
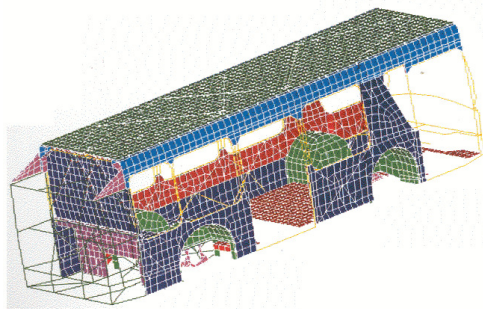
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Motorcycle chassis



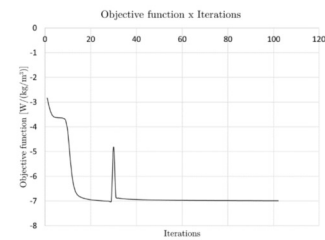
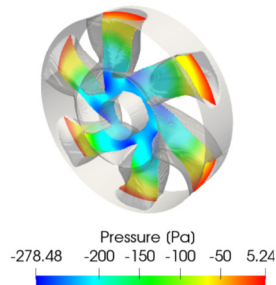
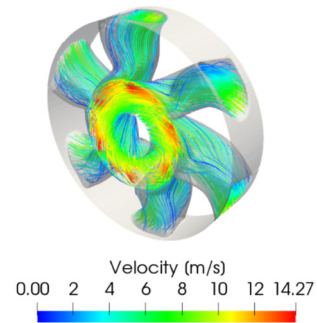
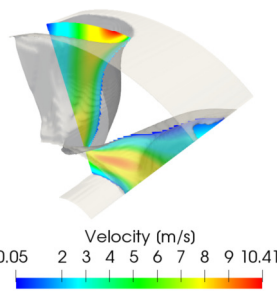
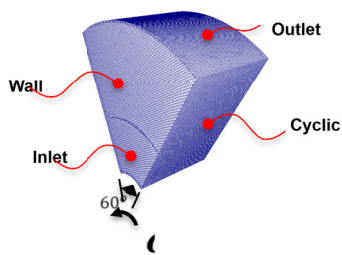
Bus bodywork



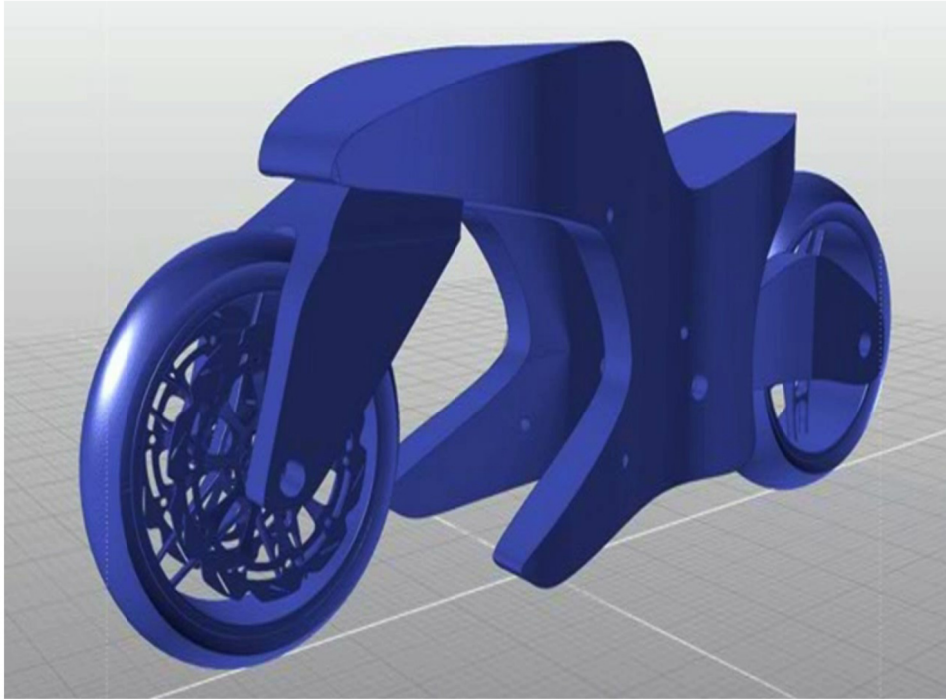
## Topology Optimization of Fluids

### Impeller Optimization

Numerical examples:  
60° sector + 50%  
volume constraint +  
1000 rpm



## Vehicle



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## Large Scale Structures

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### Qatar Convention Center (2008)



Dubai



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# Optimized Project in Engineering

## Soccer stadium



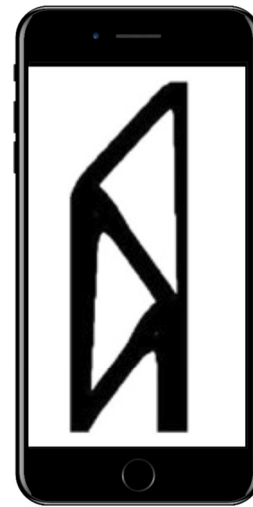
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# Optimized Project in Engineering



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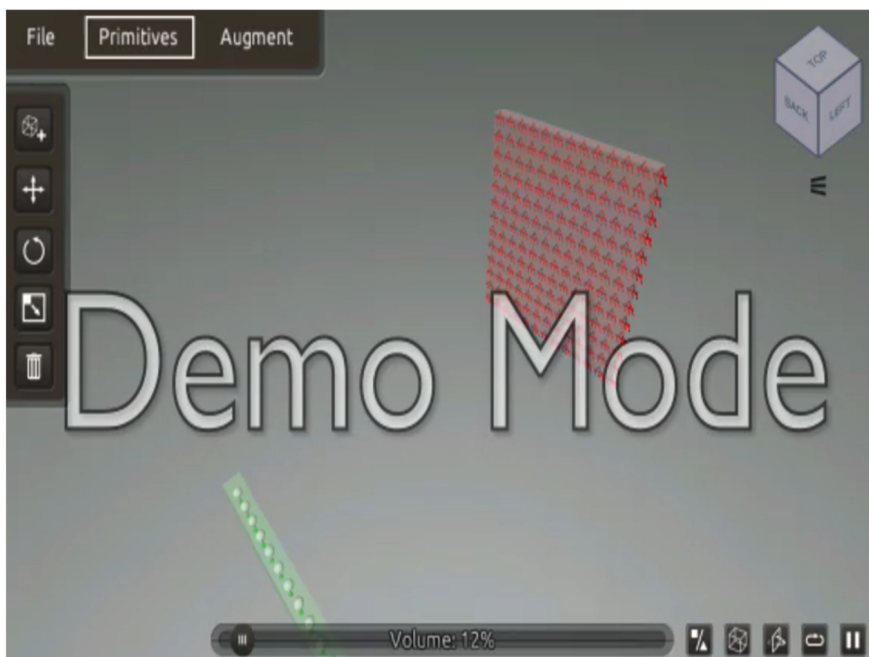
## TopOpt:



<http://www.topopt.mek.dtu.dk> Prof. Dr. Emilio C. Nelli Silva

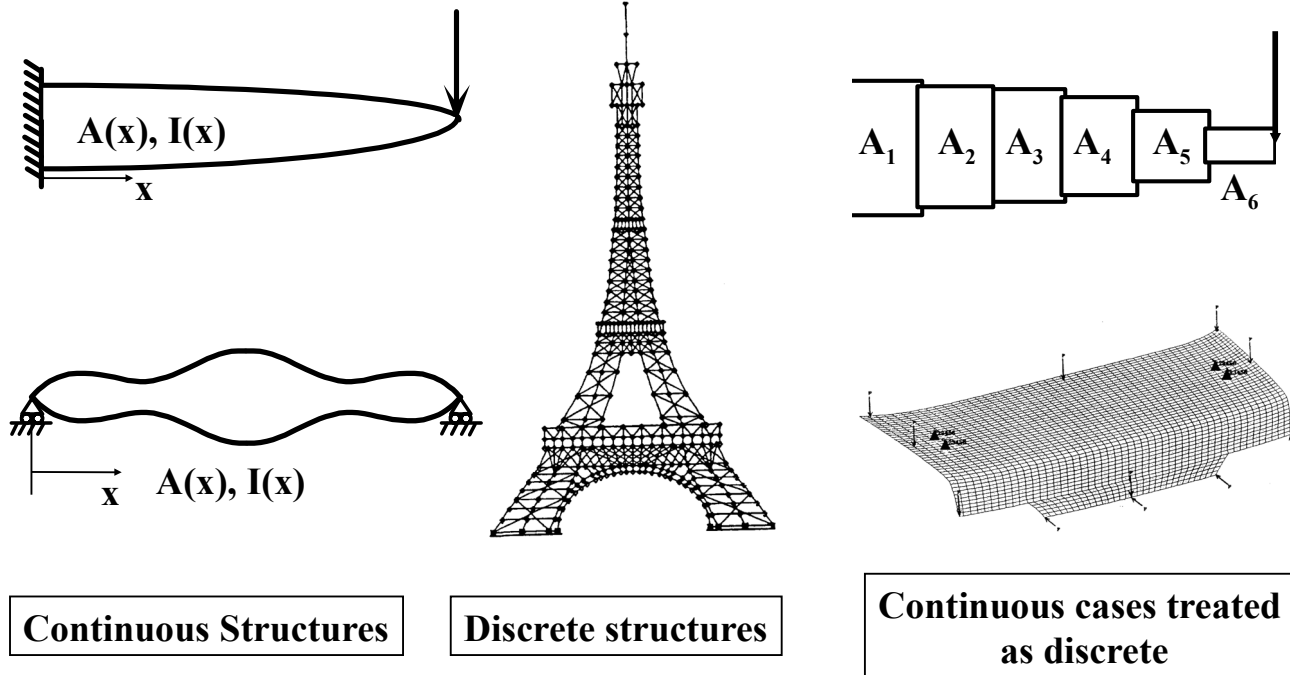
# Optimization using Smartphones

## TopOpt 3D:



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## • Classification of the Types of Structures in Optimization:



Continuous Structures

Discrete structures

Continuous cases treated as discrete

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## • Project variables (“Design Variables”): parameters that might be altered in the optimization procedure.

\* Continuous variables (more usual)

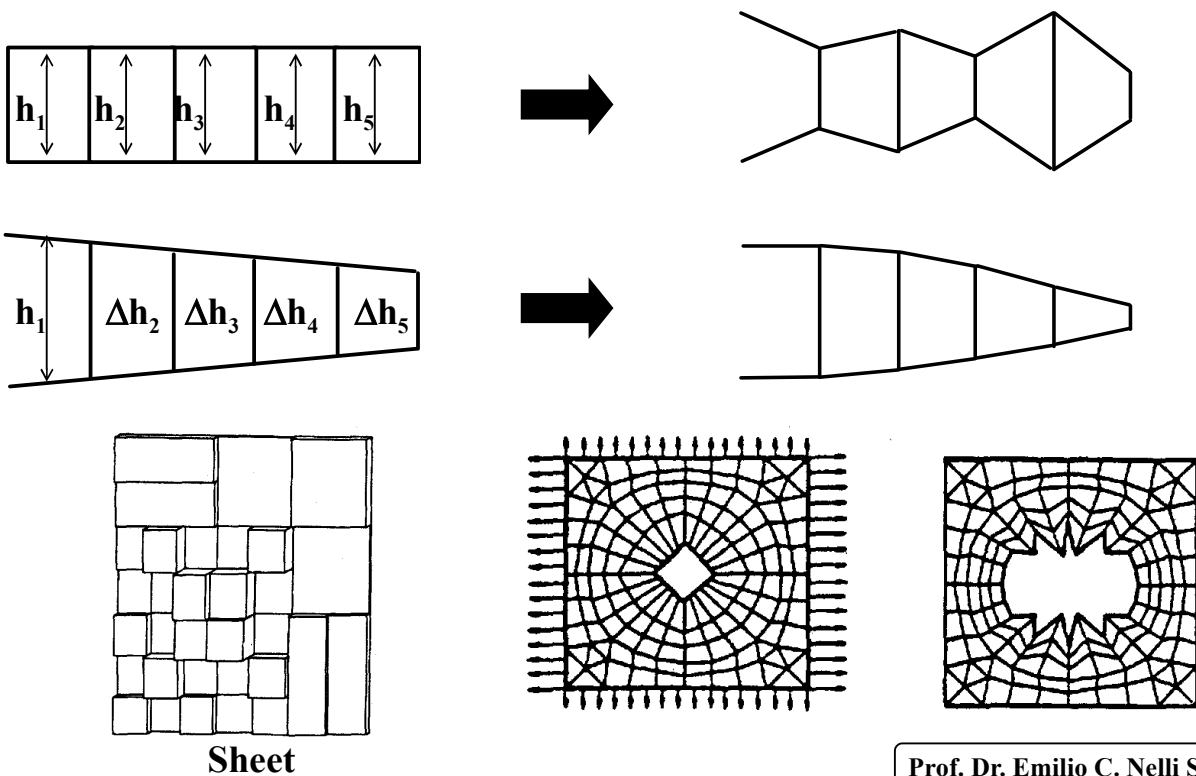
- Distributed parameter:  $A(x) \Rightarrow$  Area function
- Discrete parameter:  $A_1, A_2, \dots, A_n$  where  $A_i$  assume any real value.

\* Discrete variables:  $A_i \in \{1, 1.5, 3, 2, 5, 7\}$  (isolated values)

The solution employs methods based on the theory of Integer Programming  $\Rightarrow$  Complex algorithms and problems of difficult solution.

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The choice of project variables is critical for the success of the optimization.



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- Objective function (OF): specifies what is desirable to be optimized.

Types of OF:

- \* Simple:  $\max f$
- \* Multi-criteria or Multiobjective:  $\begin{cases} \max w_1 f_1 + w_2 f_2 \\ \max \frac{f_1}{f_2} \end{cases}$

Important:  $\max f \equiv \min -f \equiv \min \frac{1}{f}$   
 $\max |x| \equiv \max x^2$ ;  $\max k \cdot f \equiv \max f$ ;  $\max k + f \equiv \max f$

Formulation of objective function

We must be capable of expression what we want mathematically. Ex.: displacements, stiffness, resonance frequency, mechanical stress, etc...

Beware: How can we express objective functions like drivability and safety of a vehicle??

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- Constraints: limitations of the optimized solution.

Types:

\* Lateral:  $x_{\min_i} \leq x_i \leq x_{\max_i} \quad i = 1..n$

\* Inequality:  $g_j(\mathbf{x}) \geq 0 \quad j = 1..n_g$  where:

\* Equality:  $h_k(\mathbf{x}) = 0 \quad k = 1..n_e \quad \mathbf{x} = \{x_1, x_2, x_3, \dots, x_n\}$

- With respect to  $h_j(\mathbf{x}) = 0$ 
  - \* Complex implementation in some nonlinear optimization algorithms;
  - \* In general, it may be transformed:  
 $h_k(\mathbf{x}) \leq 0$  and  $h_k(\mathbf{x}) \geq 0$

- Constraints must be normalized to avoid numerical issues:

$$g_j(\mathbf{x}) \leq g_{\max_j} \Rightarrow \frac{g_j(\mathbf{x})}{g_{\max_j}} \leq 1 \Rightarrow \bar{g}_j(\mathbf{x}) - 1 \leq 0$$

- One must refrain from using large  $n_g$  and  $n_e$

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# Basic definitions

Constraints classification:

- \* Local: mechanical stress and displacements at a point;
- \* Global: material volume, stiffness and response frequency

State of an inequality constraint:

\* Active:  $g_j(\mathbf{x}) = 0$

\* Inactive:  $g_j(\mathbf{x}) > 0$

At the end of the optimization  $\rightarrow$  active constraints. Inactive constraints may be removed from the initial optimization problem.

Measure of importance of a constraint:

Lagrange Multipliers ( $\lambda_i$ )

If:  $\lambda_i = 0 \Rightarrow$  inactive constraint (unnecessary)  
 $\lambda_i \neq 0 \Rightarrow$  active constraint (necessary) } Single out important constraints

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## Formulation of the Optimization Problem:

$$\begin{array}{ll}
 \text{Minimize} & f(\mathbf{x}) \\
 \text{Subject to} & g_j(\mathbf{x}) \geq 0 \quad j=1..n_g \\
 & h_k(\mathbf{x}) = 0 \quad k=1..n_e
 \end{array}$$

An optimization problem is linear if:

$$f(\mathbf{x}) = c_1x_1 + c_2x_2 + \dots + c_nx_n; \quad g_j(\mathbf{x}) = d_1x_1 + d_2x_2 + \dots + d_nx_n;$$

$$\text{and: } h_k(\mathbf{x}) = b_1x_1 + b_2x_2 + \dots + b_nx_n$$

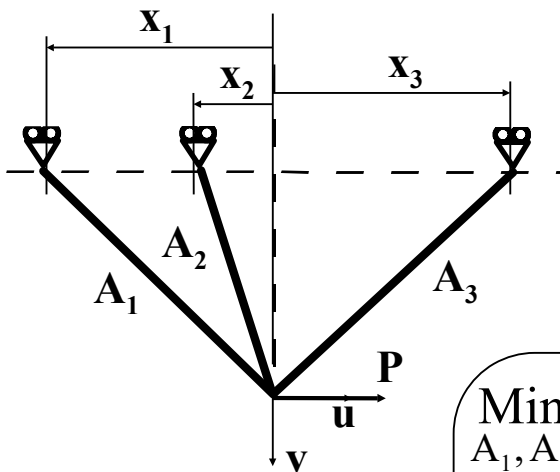
Otherwise it is nonlinear.

If linear, it may be solved with a linear programming method.

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## Example

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$$L_i = \sqrt{x_i^2 + 100}$$

Constraints of balance equations (Castigliano Theorem)

Constraints of mechanical stress

$$\text{Min } m = 0,29(A_1L_1 + A_2L_2 + A_3L_3)$$

$$A_1, A_2, A_3, x_1, x_2, x_3$$

Subject to

$$\begin{cases}
 k_{11}(A_i, x_i, L_i)u + k_{12}(A_i, x_i, L_i)v - 10.000 = 0 \\
 k_{12}(A_i, x_i, L_i)u + k_{22}(A_i, x_i, L_i)v = 0
 \end{cases}$$

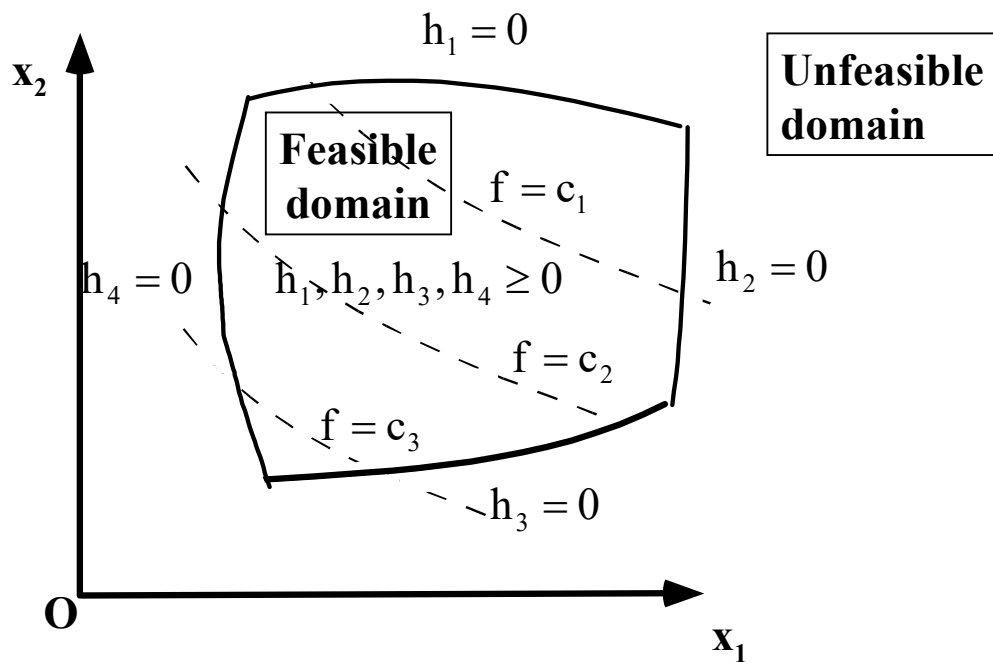
$$\begin{cases}
 30.000 - \sigma_i(u, v, x_i, L_i) \geq 0 \\
 \sigma_i(u, v, x_i, L_i) + 30.000 \geq 0
 \end{cases}$$

$$A_i - 0,1 \geq 0$$

$$i = 1,2,3$$

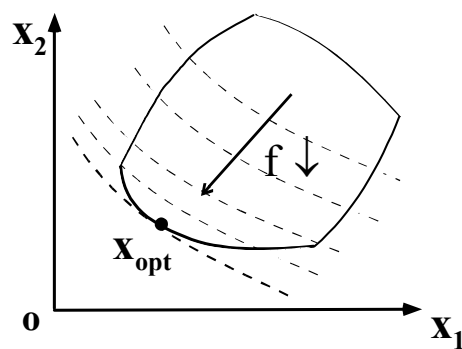
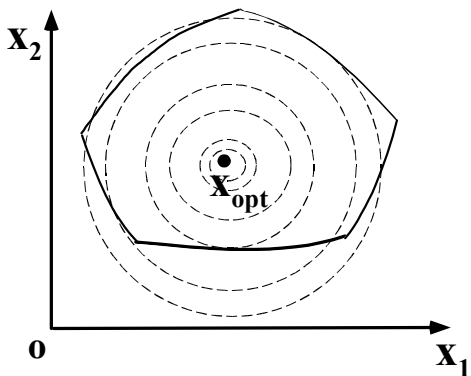
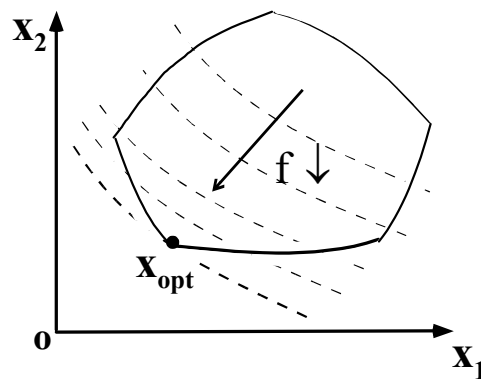
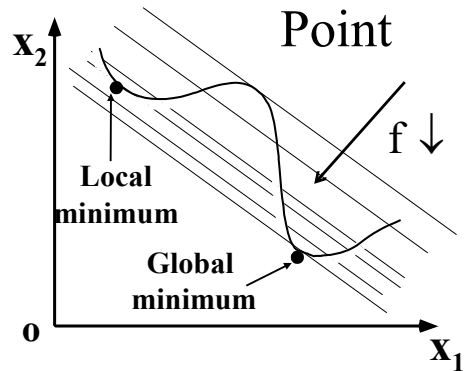
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- Feasible and unfeasible domain:



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- Location of the Optimal Point



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