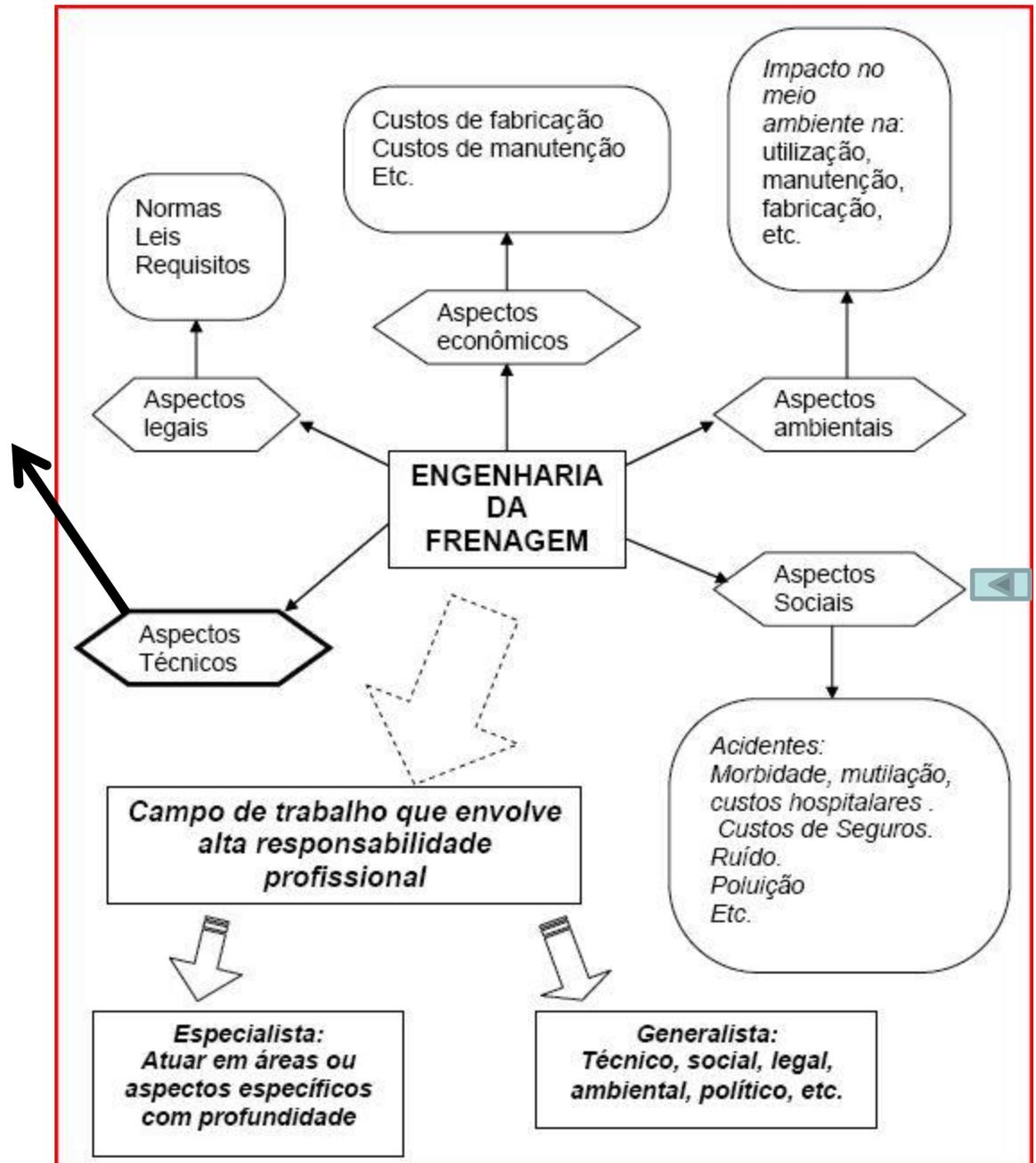


FREIO X
SEGURANÇA

EVITAR O
ESCORREGAMENTO
DAS RODAS

QUASE SEMPRE O
FREIO ESTÁ
ENVOLVIDO NO
ACIDENTE

QUE PARADA
EFICIENTE



ECE REGULATION No.13 & EEC DIRECTIVE 71/320

SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS	CARS & BUSES	VANS & TRUCKS	TRAILERS
NOTATION: Vehicle Category No. of Passenger Seats (excl.driver) FL = Fully laden UL = Unladen Max. Vehicle Weight (Tonnes)	M1 ≤8 ≤5	M2 >8 >5	M3 >8 >5
	N1 ≤3½	N2 3½-12	N3 >12
			O1 O2 O3 O4 NBR 10966 ≤¾ ¾-3½ 3½-10 >10
SERVICE BRAKE (SB) Acting on all wheels, properly distributed and symmetric across vehicle. Braking distribution in accordance with Annex 10 and 75/524, respectively.			
1. TYPE - 0 TEST in Neutral NBR 10967 Cold brakes FL & UL MFDD Pedal Effort	Prescribed Speed km/h Stopping Distance m ≤ MFDD m/sec ² ≥ Pedal Effort daN ≤	80 60 60 80 60 60 0.1V + V ² /150 5.8 50	60 60 60 80 60 60 0.15V+V ² /130 5.0 70
2. TYPE - 0 TEST in Gear. Cold brakes FL & UL		Effectiveness tests from various speeds (30% to 80% of Vmax) - check vehicle behaviour.	
3. TYPE - 0 TEST in Gear. Prescribed Speed = 80% Vmax, but km/h ≤ Cold brakes FL & UL MFDD Pedal Effort		160 100 90 120 100 90 0.1V + V ² /130 5.0 50	100 90 120 100 90 0.15V+V ² /103.5 4.0 70
		Service Brake is Optional 60 Brake Force ≥ 50% of Trailer max. axle wt. (45% for semi-trailers) using ≤ 6.5 bar Overrun Brakes permitted for O1/O2 only	

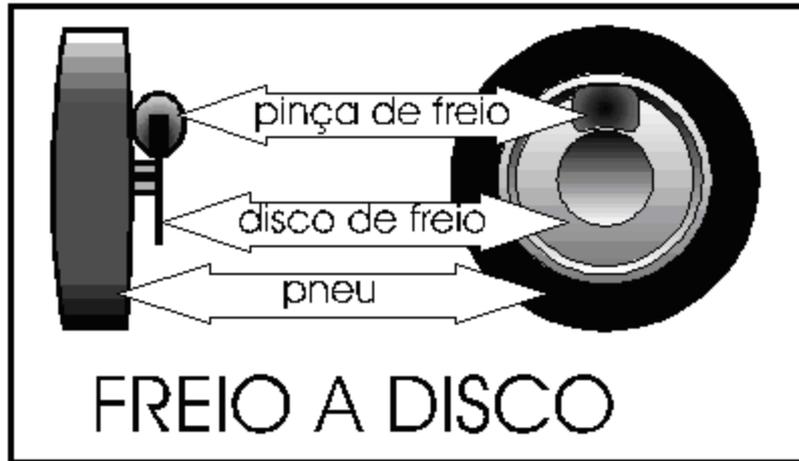
ECE REGULATION No.13 & EEC DIRECTIVE 71/320

SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS	CARS & BUSES	VANS & TRUCKS	TRAILERS	
NOTATION: Vehicle Category No. of Passenger Seats (excl.driver) FL = Fully laden UL = Unladen Max. Vehicle Weight (Tonnes)	M1 ≤8 ≤5	M2 >8 >5	M3 >8 >5 N1 ≤3½ N2 3½-12 N3 >12	O1 ≤¾ O2 ¾-3½ O3 3½-10 O4 >10
SERVICE BRAKE (SB) Acting on all wheels, properly distributed and symmetric across vehicle. Braking distribution in accordance with Annex 10 and 75/524, respectively.				
4. TYPE - I TEST by repeated braking. NBR 10967 V1 = 80% Vmax , but km/h ≤ V2 = 1/2 V1 In gear : Time Interval secs FL No. of Cycles N.B. First snub at 3.0 m/sec ² deceleration; remaining snubs at same Pedal Effort.	120 45 15	100 55 15	60 60 20 60 60 20	
5. TYPE - I TEST by continuous braking. FL				Drag test at 40 km/h for 1.7 km on 7% slope
6. HOT EFFECTIVENESS after Type-I Tests Prescribed Speed and Hot brakes Pedal Effort - FL as Test 1 :	Performance to be ≥ 80% of Test 1 requirement and ≥ 60% of Test 1 achievement.			40 km/h Brake Force ≥ 36% of Trailer max.axle wt. and ≥ 60% of Test 1 achievement

ECE REGULATION No.13 & EEC DIRECTIVE 71/320

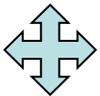
SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS		CARS & BUSES			VANS & TRUCKS			TRAILERS			
NOTATION:	Vehicle Category No. of Passenger Seats (excl.driver) Max. Vehicle Weight (Tonnes)	M1	M2	M3	N1	N2	N3	O1	O2	O3	O4
FL = Fully laden UL = Unladen		≤8	>8	>8	≤3½	3½-12	>12	≤¾	¾-3½	3½-10	>10
SERVICE BRAKE (SB)		Acting on all wheels, properly distributed and symmetric across vehicle. Braking distribution in accordance with Annex 10 and 75/524, respectively.									
7. TYPE - II TEST for long descents NBR 10967											
In gear FL	For M3 N3 O4 vehicles only	Drag test at 30 kp/h for 6 km on 6% slope (or 0.5 m/sec ² by engine alone)									
7A. TYPE - IIA (BIS) TEST for 'Retarders'											
In gear FL	For M3 vehicles (instead of Test 7) - except 'urban buses'	Drag test at 30 kp/h for 6 km on 7% slope (or 0.6 m/sec ² by engine alone) - without using Service, Emergency or Parking Brake									
8. HOT EFFECTIVENESS after Type-II Tests											
Hot brakes FL	Prescribed Speed and Pedal Effort - as Test 1 : Stopping Distance m ≤ MFDD m/sec ² ≥	60 km/h 0.15V + 1.33V ² /130 3.75			60 km/h 0.15V + 1.33V ² /115 3.3			40 km/h ≥ 33% of Trailer max. axle wt.			

Freios mais comuns

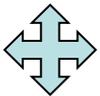
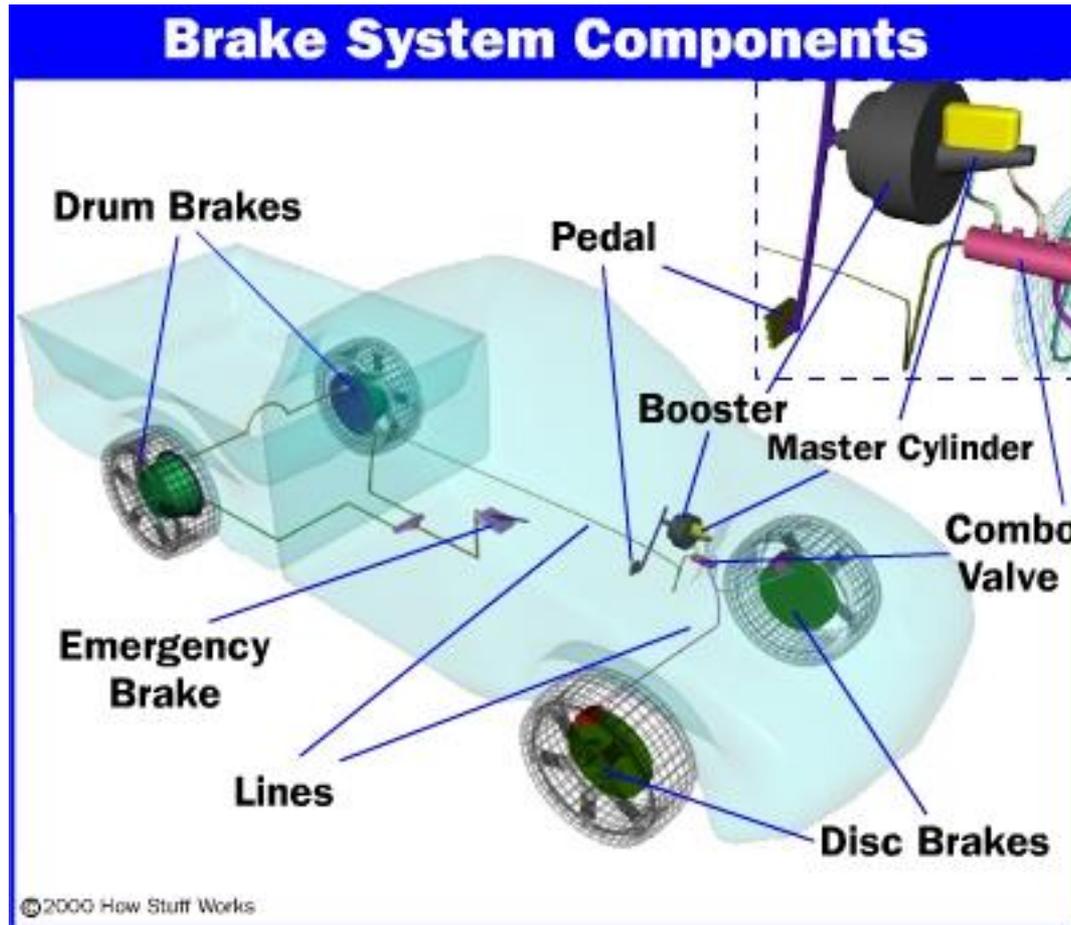


HIDRÁULICOS

PNEUMÁTICOS



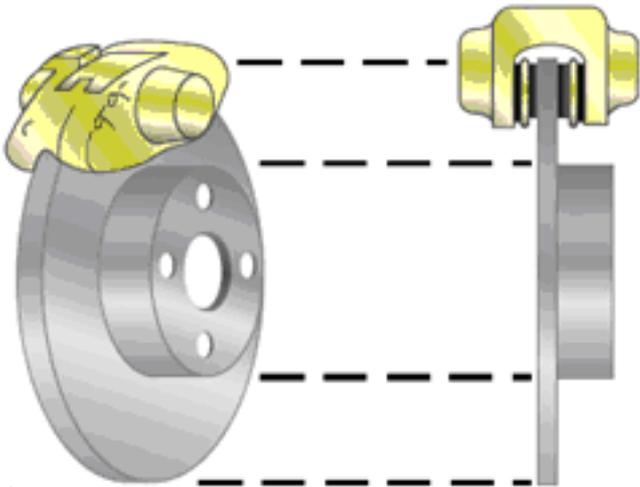
Componentes do sistema de freios



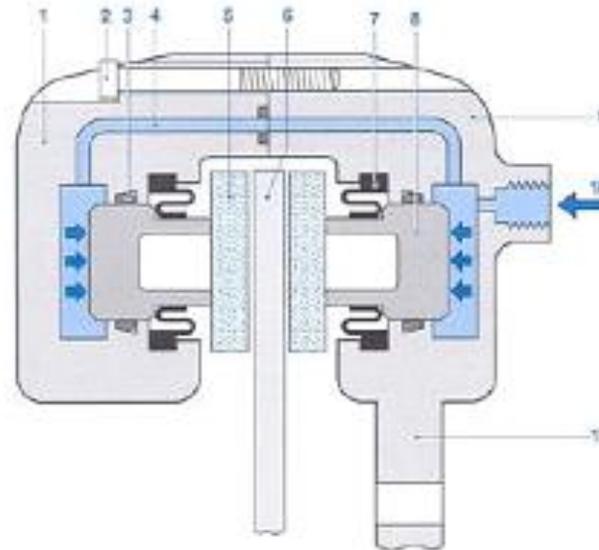
Freio a disco



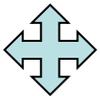
© 2000 How Stuff Works



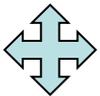
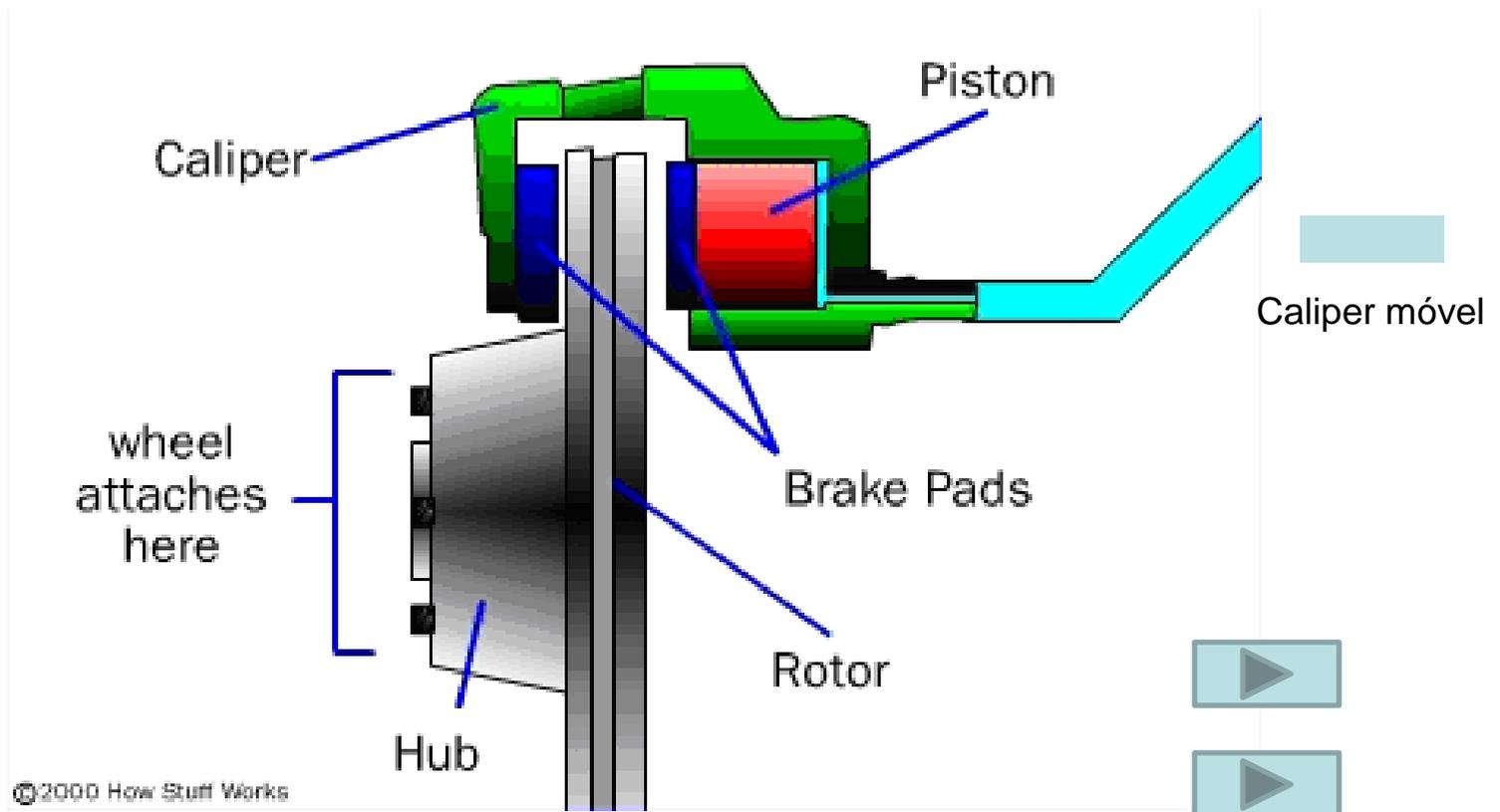
Caliper fixo



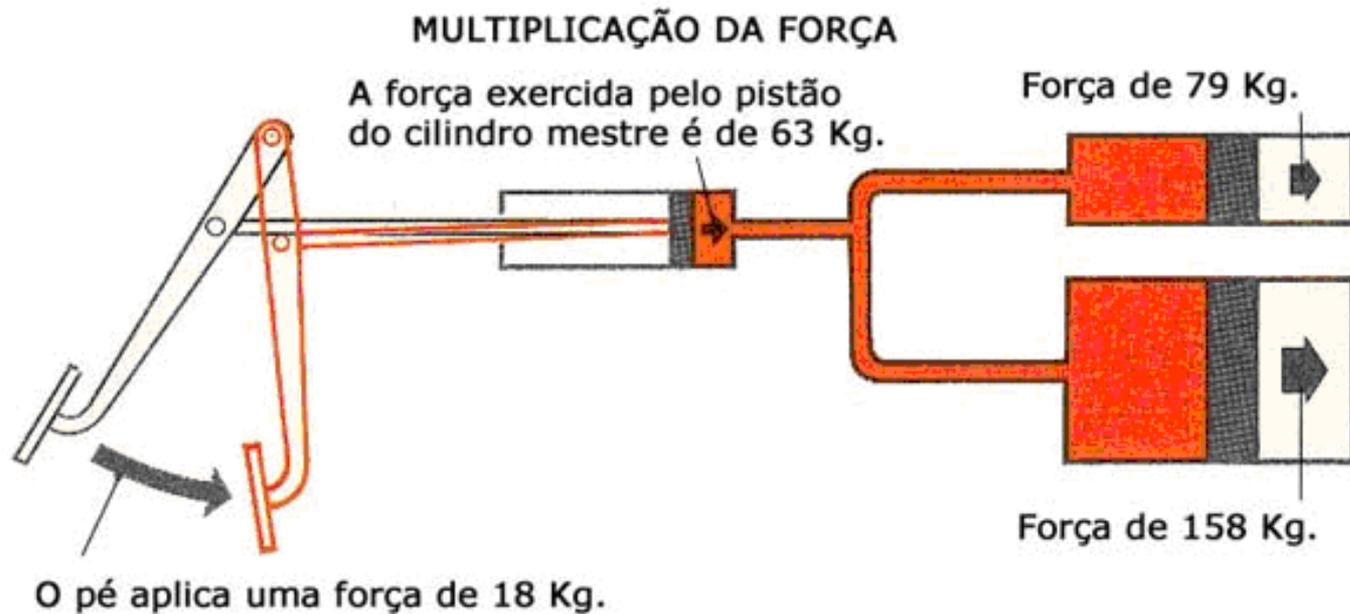
(1) Carcaça externa; (2) parafuso de junção; (3) anel de vedação; (4) canal de fluido; (5) pastilha de freio; (6) disco de freio; (7) guarda pó de borracha; (8) êmbolo; (9) carcaça interna flangelada; (10) furo de alimentação de fluido; (11) flange de montagem.



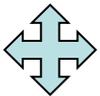
Caliper móvel



Ganho de força

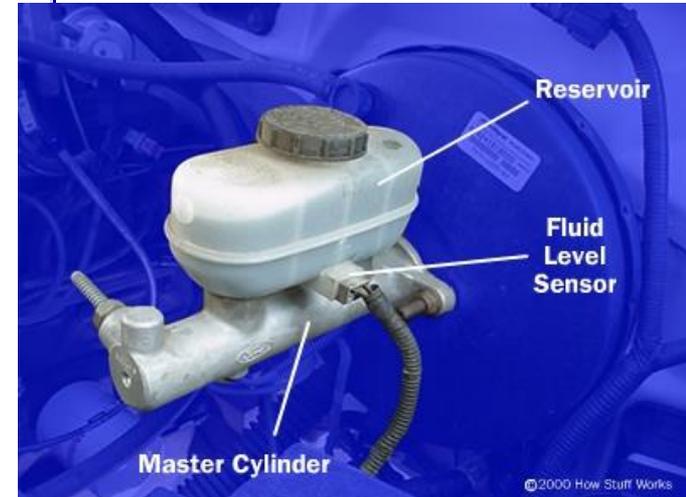
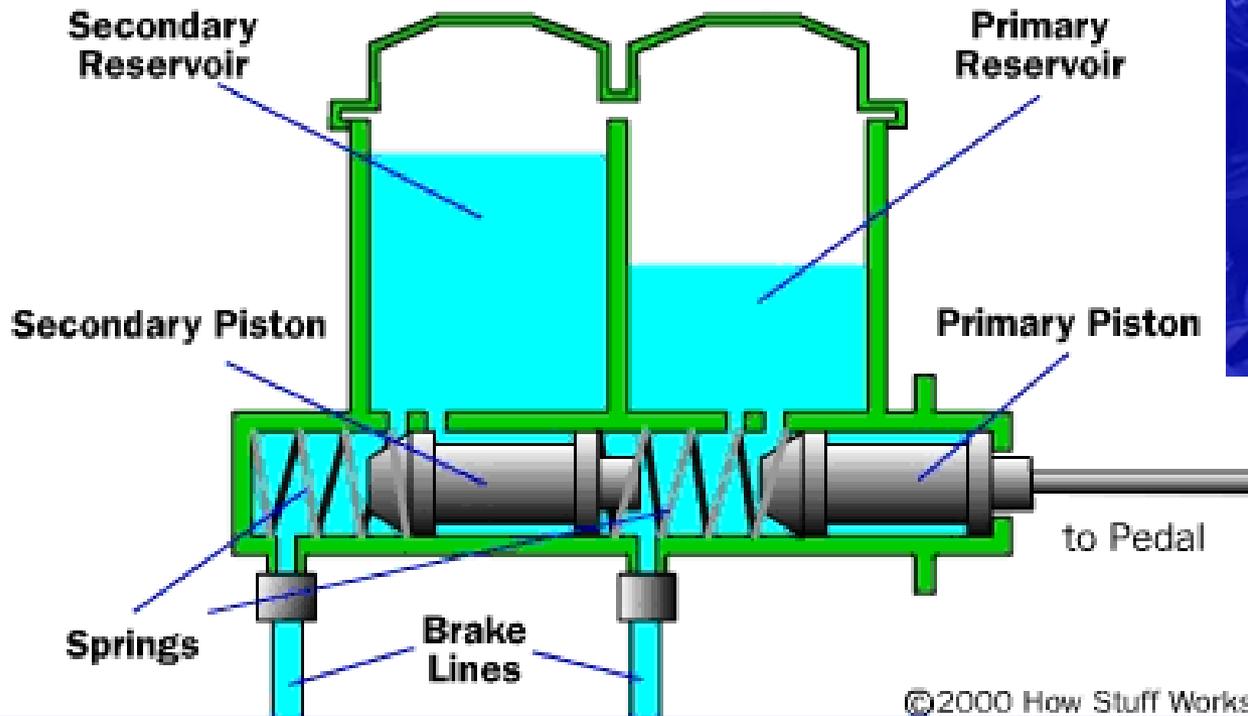


Blaise Pascal
1623 - 1662



Cilindro mestre – duplo circuito

Inside the Master Cylinder



Funcionamento normal

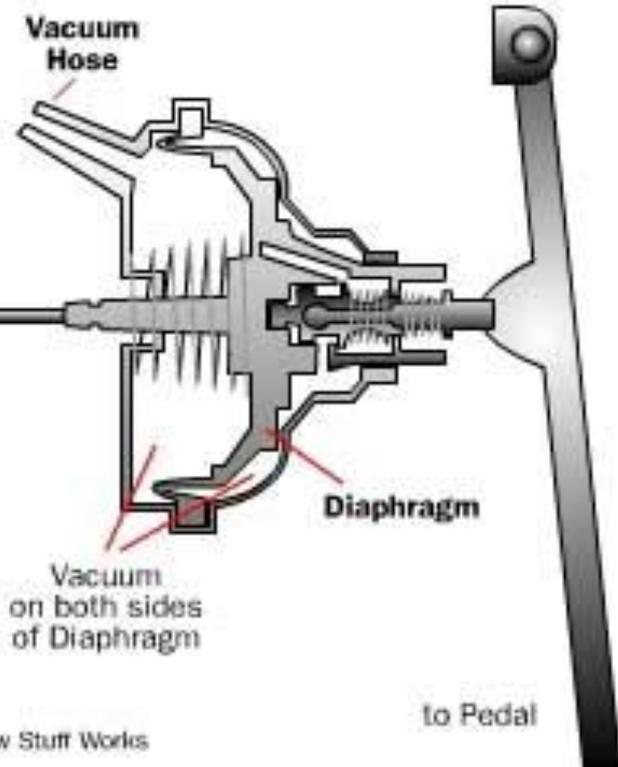
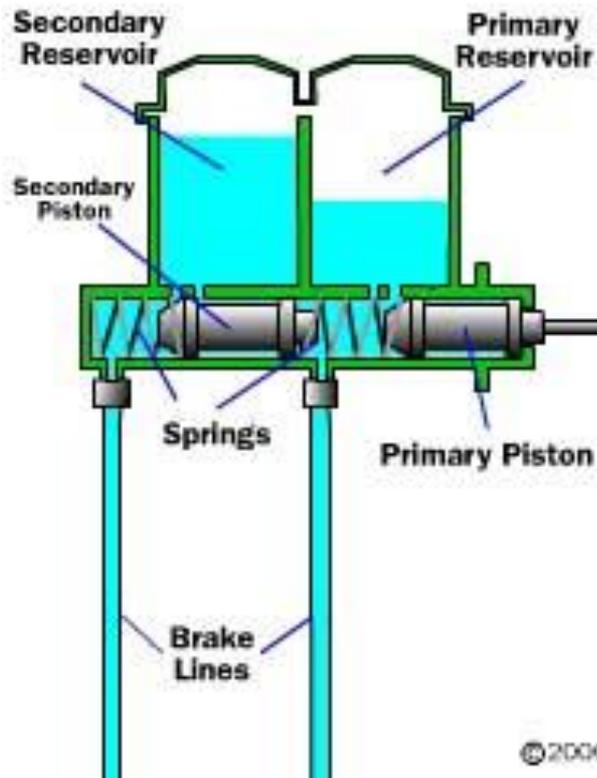
Falha no funcionamento



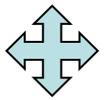
TANDEM MASTER CYLINDER

Master Cylinder

Vacuum Booster

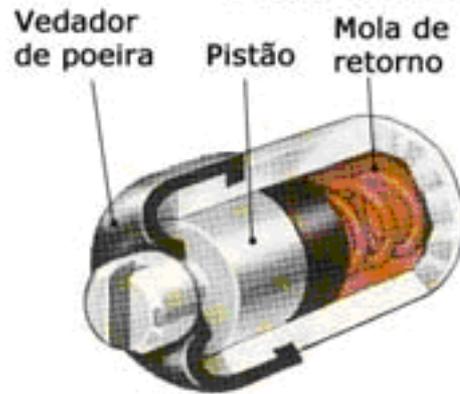


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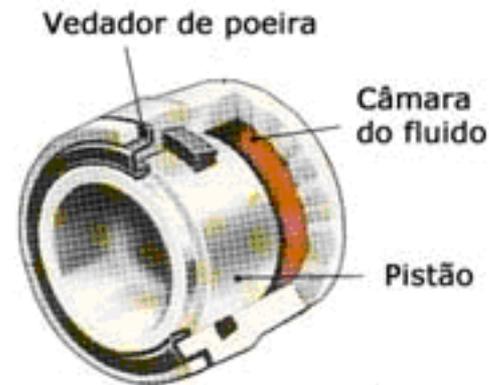


Modelos de cilindros de freio

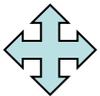
OUTROS MODELOS DE CILINDROS DE FREIO



CILINDRO DE EFEITO SIMPLES
Este modelo possui apenas um pistão, pelo que o cilindro também se move em substituição do segundo pistão.



CILINDRO DO FREIO À DISCO
Dois pistões como na figura, acionam pressão as pastilhas de fricção às duas faces do disco.



Freio a tambor - hidráulico



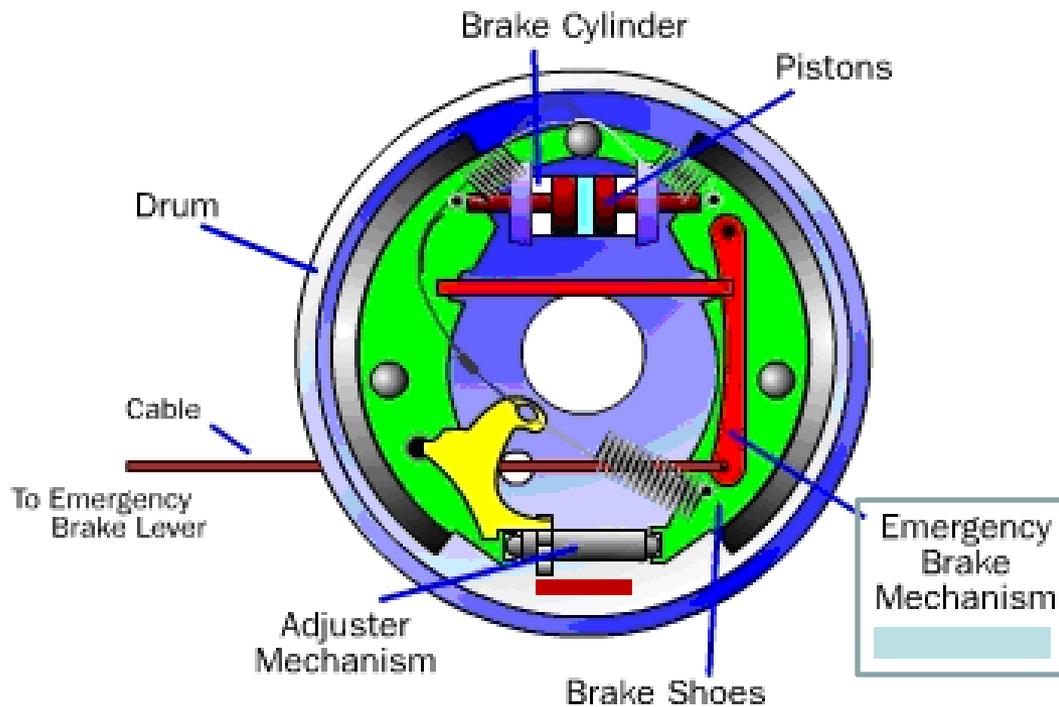
Legenda:

- 1-) Tambor Está preso ao cubo da roda
- 2-) Cilindro É ele quem recebe a pressão exercida pelo motorista sobre o pedal do freio, e se expande, forçando as sapatas contra o tambor.
- 3-) Sapata Peça resistente que é forçada contra o tambor, gerando atrito e ocasionando a frenagem.
- 4-) Lona Reveste externamente a sapata, pois tem maior capacidade de provocar atrito.



Freio a tambor - hidráulico

Drum Brake



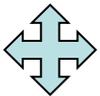
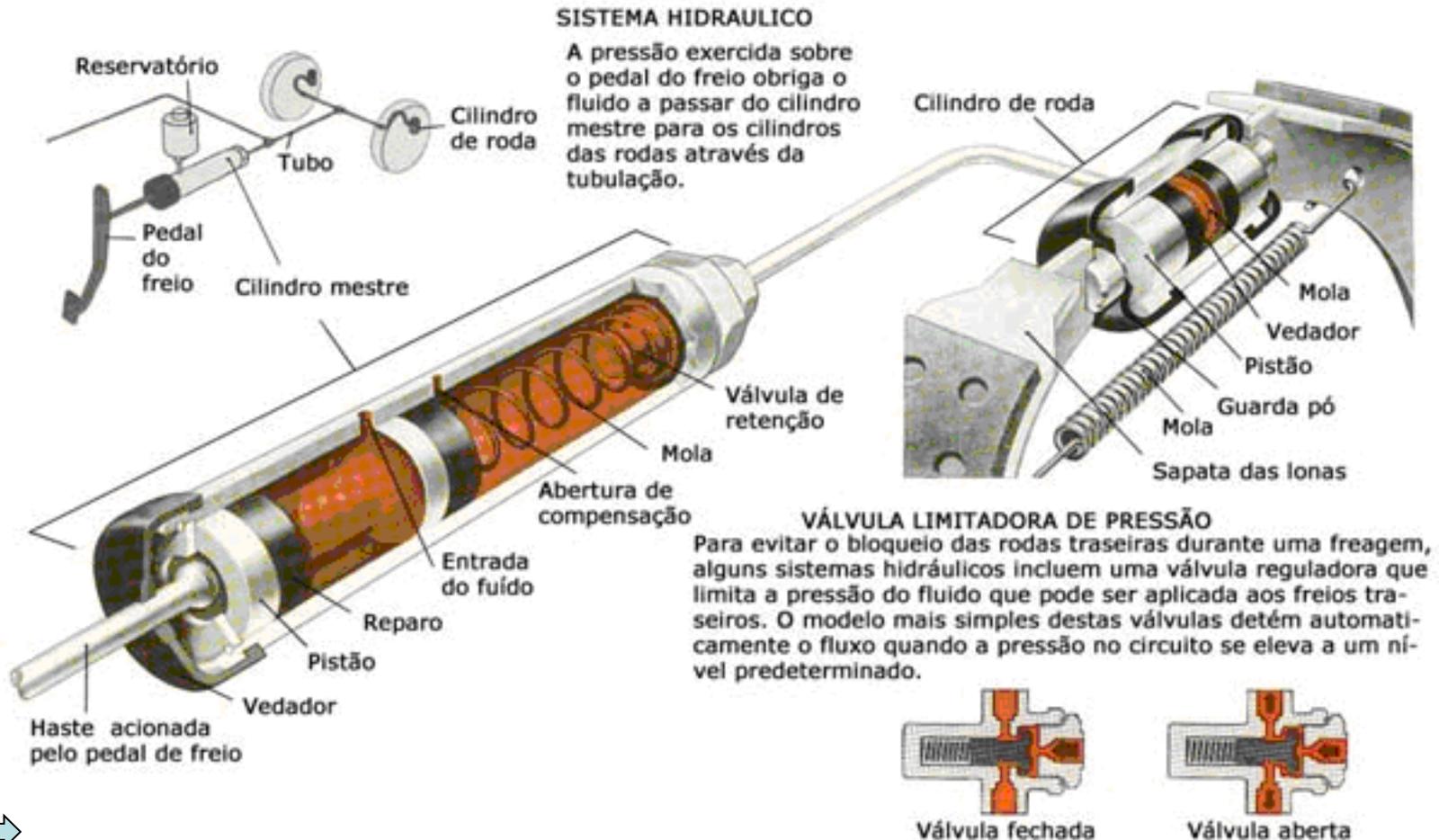
tambor

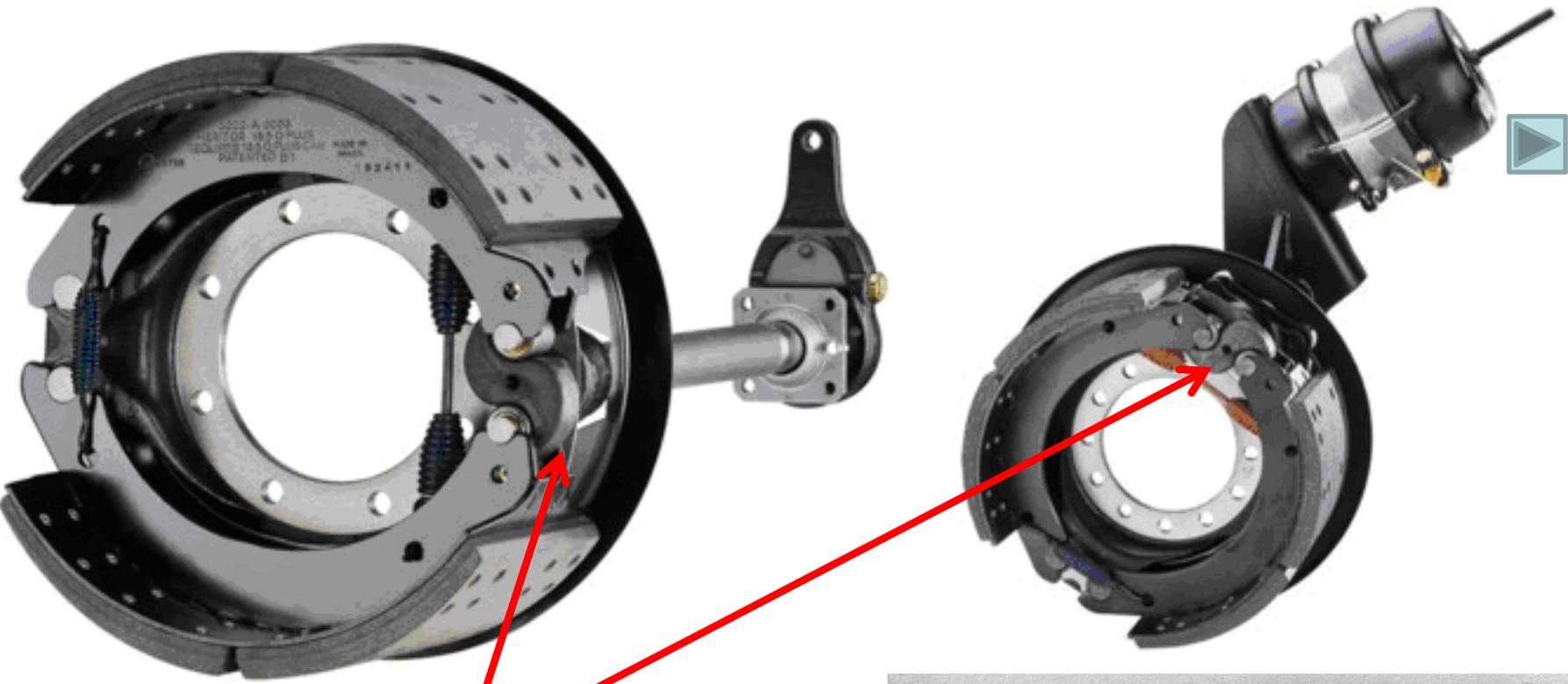


patim



Cilindro mestre e cilindro de roda freio a tambor



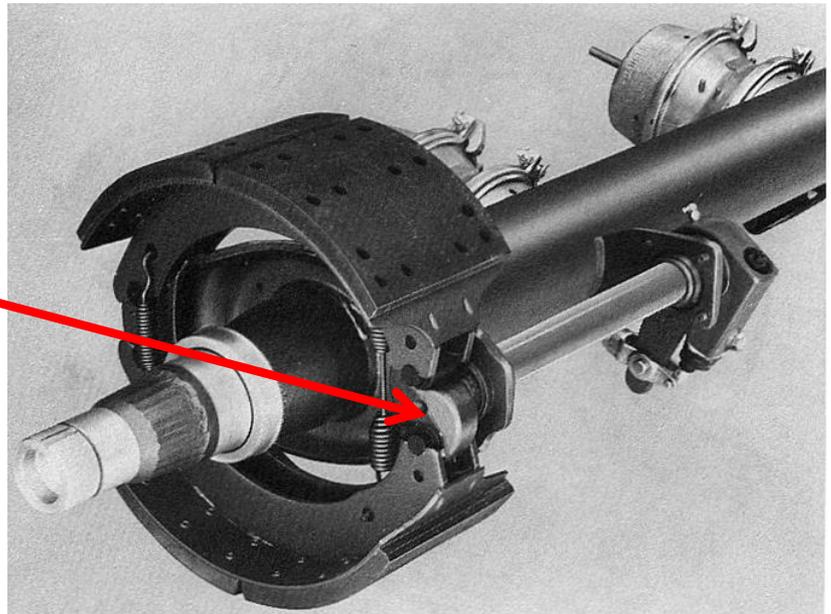


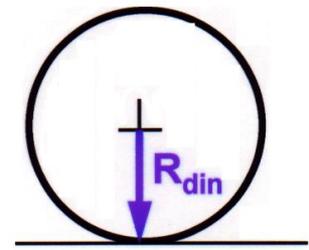
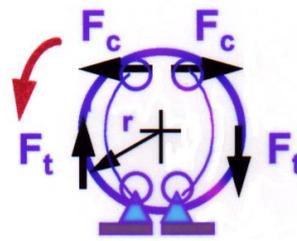
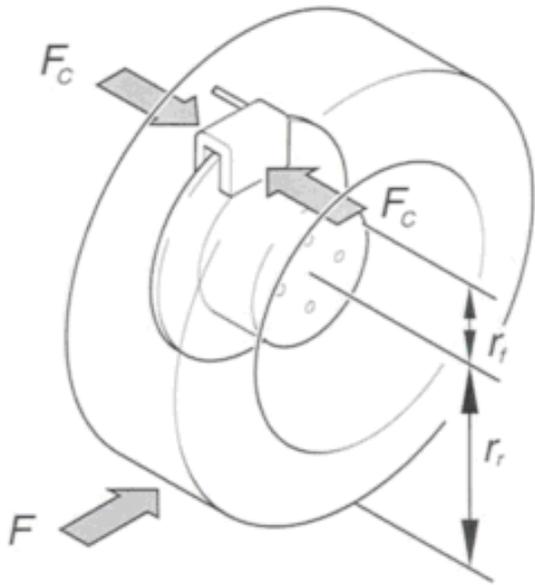
Freio S came

Raio efetivo S came



Frenagem 27





Forças reais:

ex: freio a tambor com acionamento hidráulico

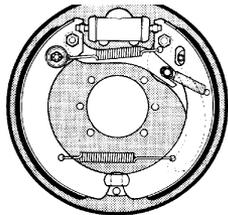
$$F_c = p \cdot A_c$$

fator de freio: $C^* = F_t / F_c$

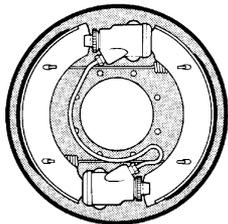
$$F_t = F_c \cdot C^* = p \cdot A_c \cdot C^*$$

torque real: $T_r = 2 \cdot F_t \cdot r = F_r \cdot R_{din}$

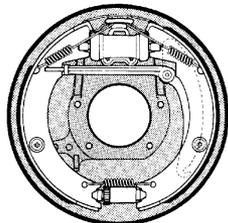
portanto: $F_{ED/ET} = 2 \cdot p \cdot A_c \cdot C^* \cdot r \cdot \eta / R_{din}$ **reta**



Leading-Trailing Shoe Brake



Two-Leading Shoe Brake



Duo-Servo Brake

considerando: $\mu_f = \mu_r = \mu_0$

as forças de freagem serão máximas se:

$$B_f = \mu_f \cdot W b_f = \mu \cdot W b_f = \mu \cdot (W/g) \cdot ((1 - \alpha) + b \cdot \phi) \quad (3)$$

$$B_r = \mu_r \cdot W b_r = \mu \cdot W b_r = \mu \cdot (W/g) \cdot (\alpha - b \cdot \phi) \quad (4)$$

onde: $\phi = H / L$ e $\alpha = L_f / L$

realizando: $\sum F_x : R_i = B_f + B_r \quad (5)$

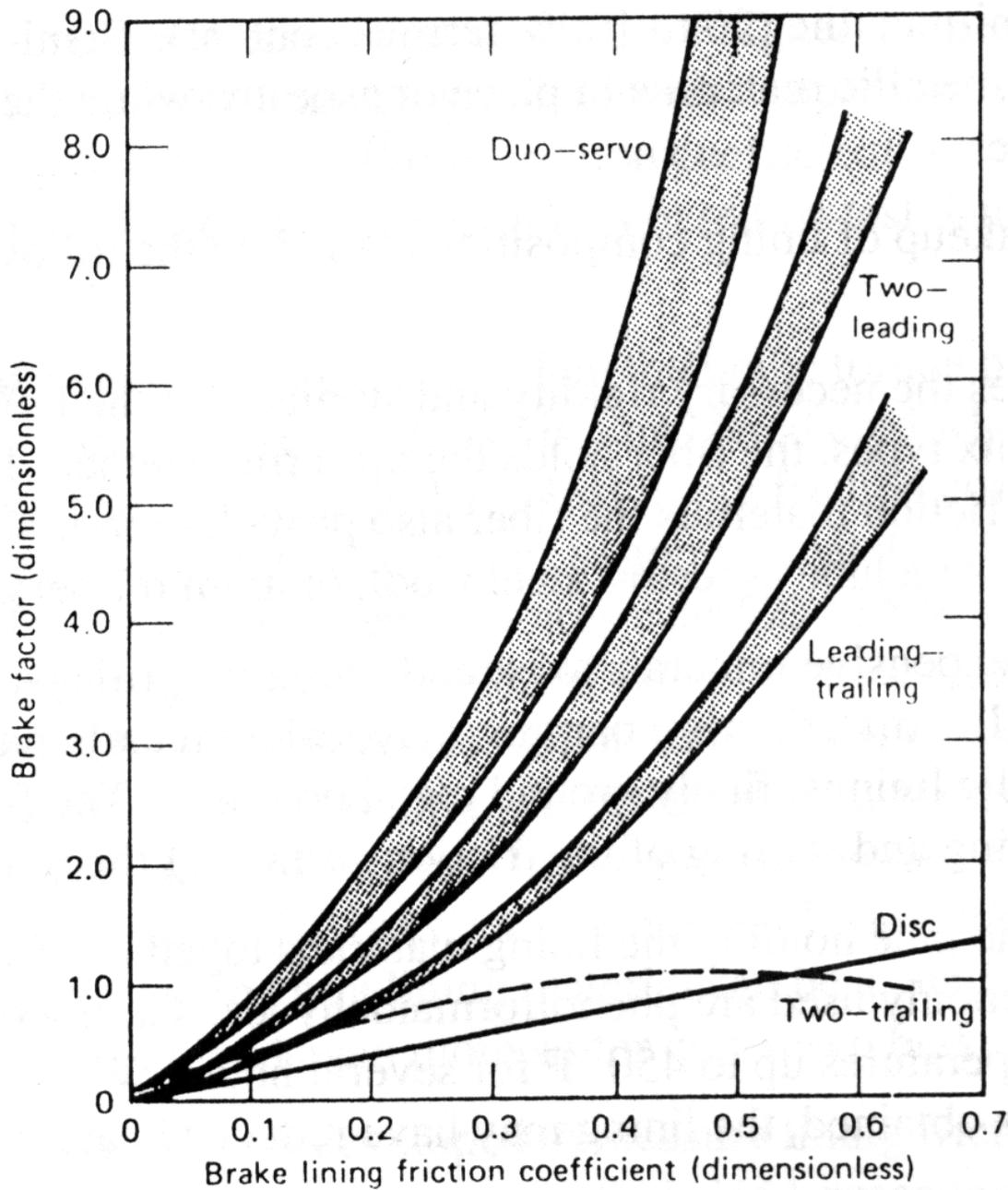
$$W \cdot b/g = \mu \cdot (W b_f + W b_r) = \mu \cdot W$$

tem-se que: $\mu = b/g$

partindo de: $(B_r / W) = (b / g) - (B_f / W) \quad (5):$

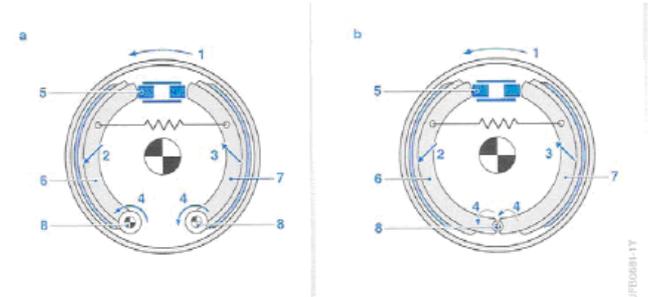
tem-se:

$$B_r / W = (((1 - \alpha) / 2 \cdot \phi)^2 + (B_f / \phi \cdot W))^{1/2} \text{ parábola} - (B_f / W) - ((1 - \alpha) / 2 \cdot \phi)$$



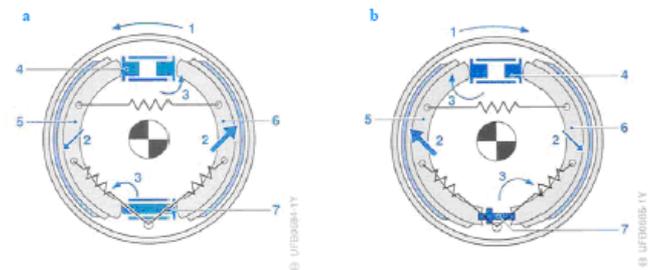
Fator de freio

 Simulador geral



(1) direção de rotação; (2) efeito auto-energizamento; efeito auto-inibimento; (4) torque; (5) cilindro de roda de dupla ação; (6) e (7) sapatas; (8) ponto de ancoragem.

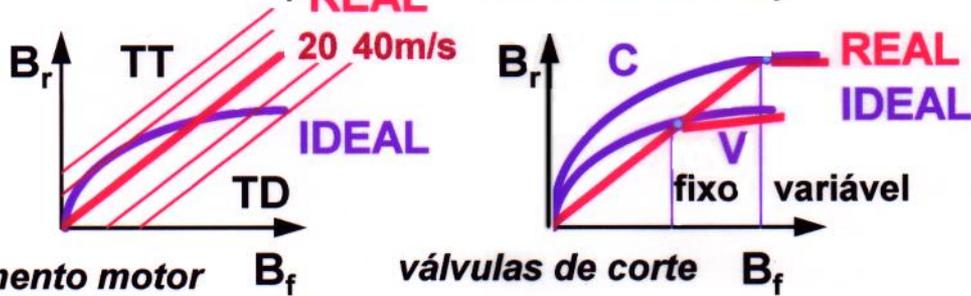
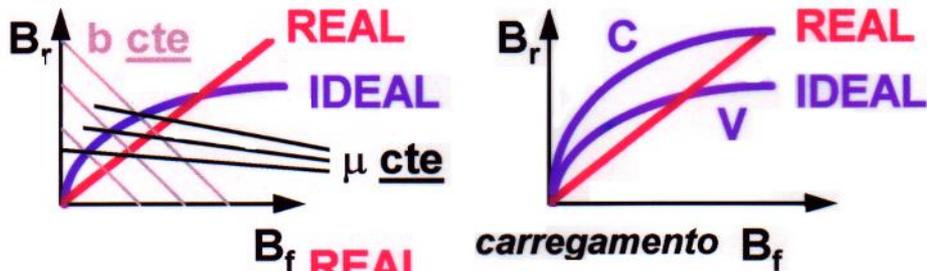
Leadin trailing



(1) direção de rotação; (2) efeito auto-energizamento; (3) torque; (4) cilindro de roda; (5) ponto de apoio; (6) sapatas de freio; (7) pino de pressão.

Duo servo

Distribuição das forças de frenagem

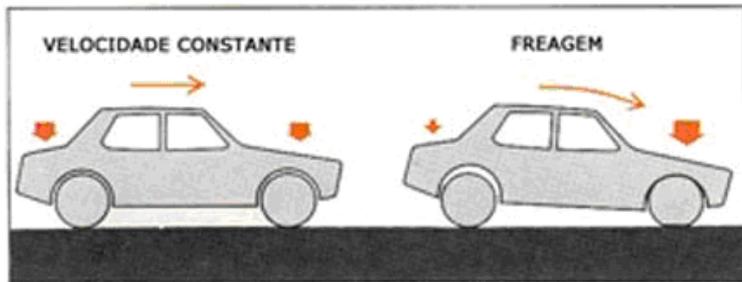


momento motor B_f

válvulas de corte B_f

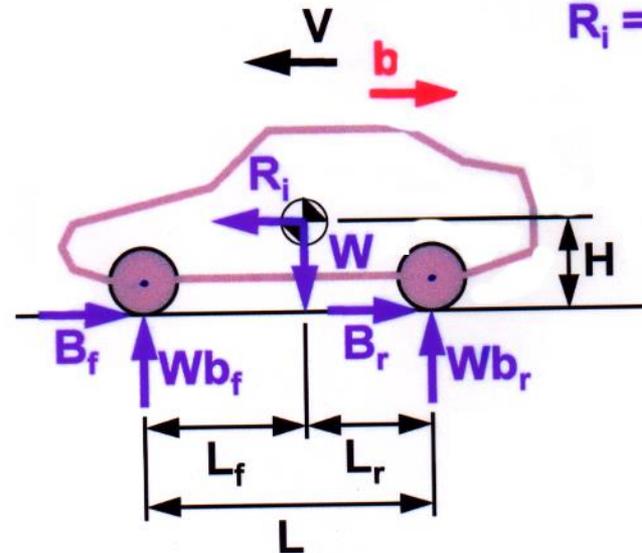
$$R_i = m \cdot b = W \cdot b / g$$

balanceamento ideal x balanceamento real



A velocidade constante, o peso do automóvel distribui-se mais ou menos uniforme.

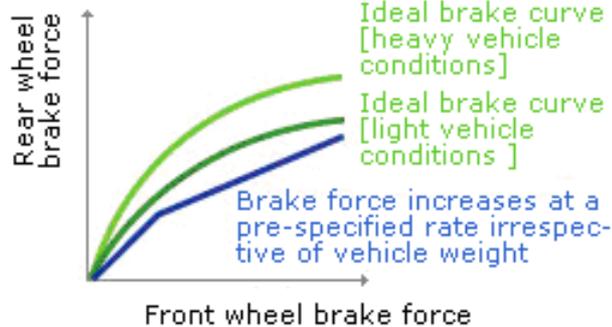
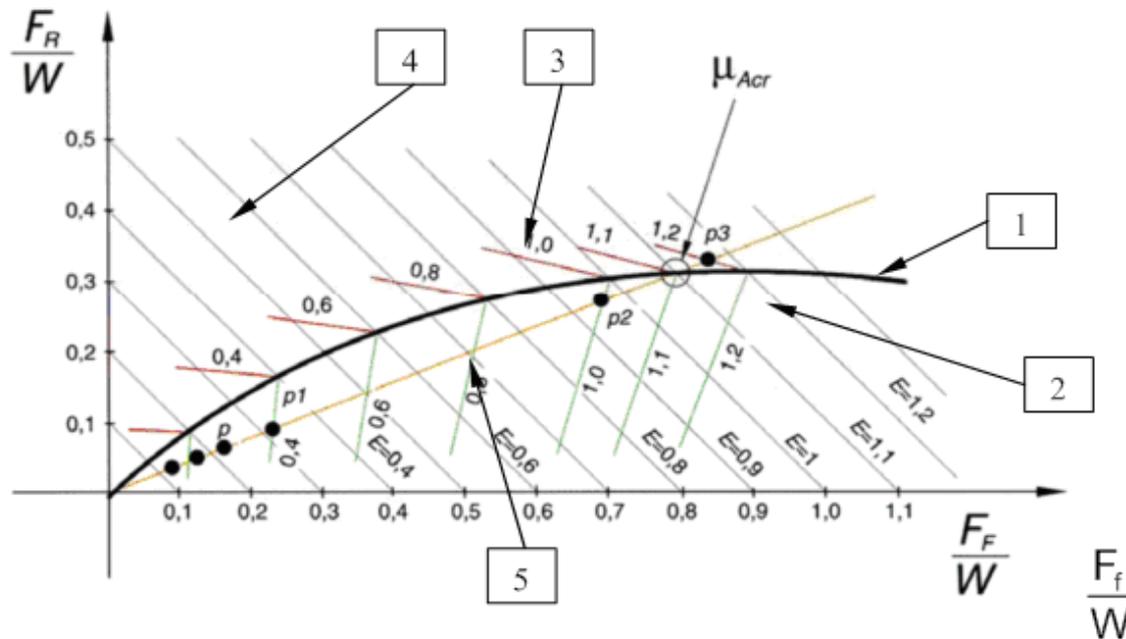
Quando os freios são aplicados, o peso do automóvel é transferido para as rodas dianteiras.



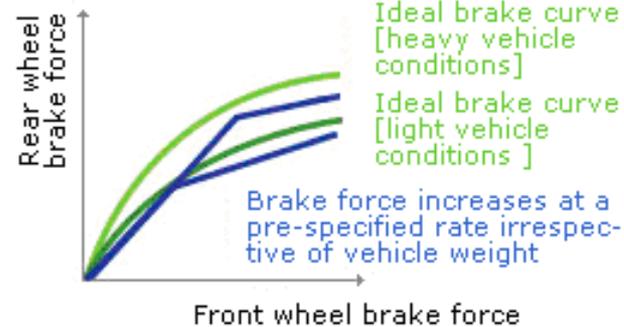
$$\text{condição dinâmica: } Wb_f = (W/g) \cdot (L - L_f) + b \cdot H / L$$

$$Wb_r = (W/g) \cdot (L_f - b \cdot H) / L$$

Distribuição das forças de frenagem



Existing proportioning value applied

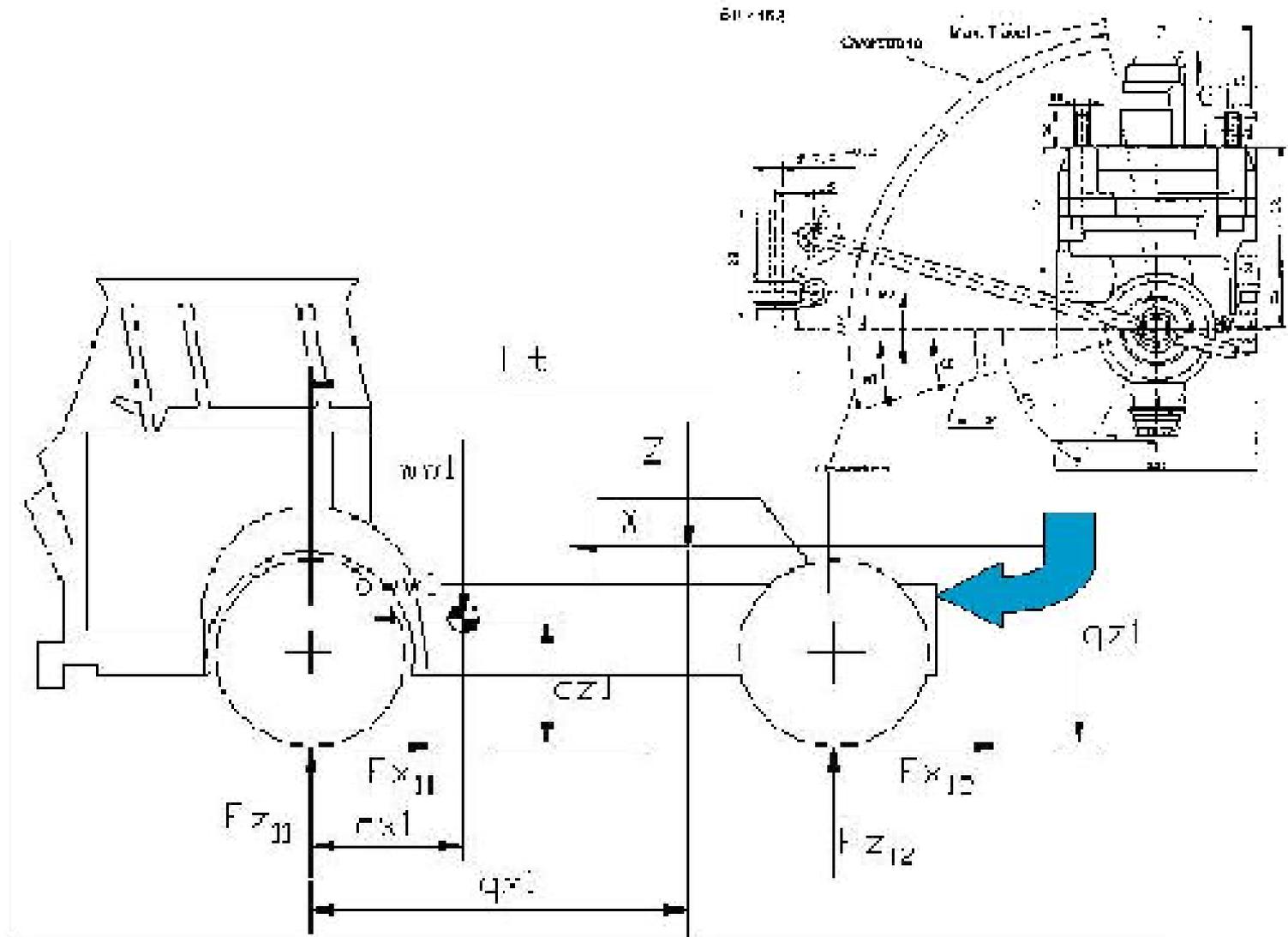


EBD applied

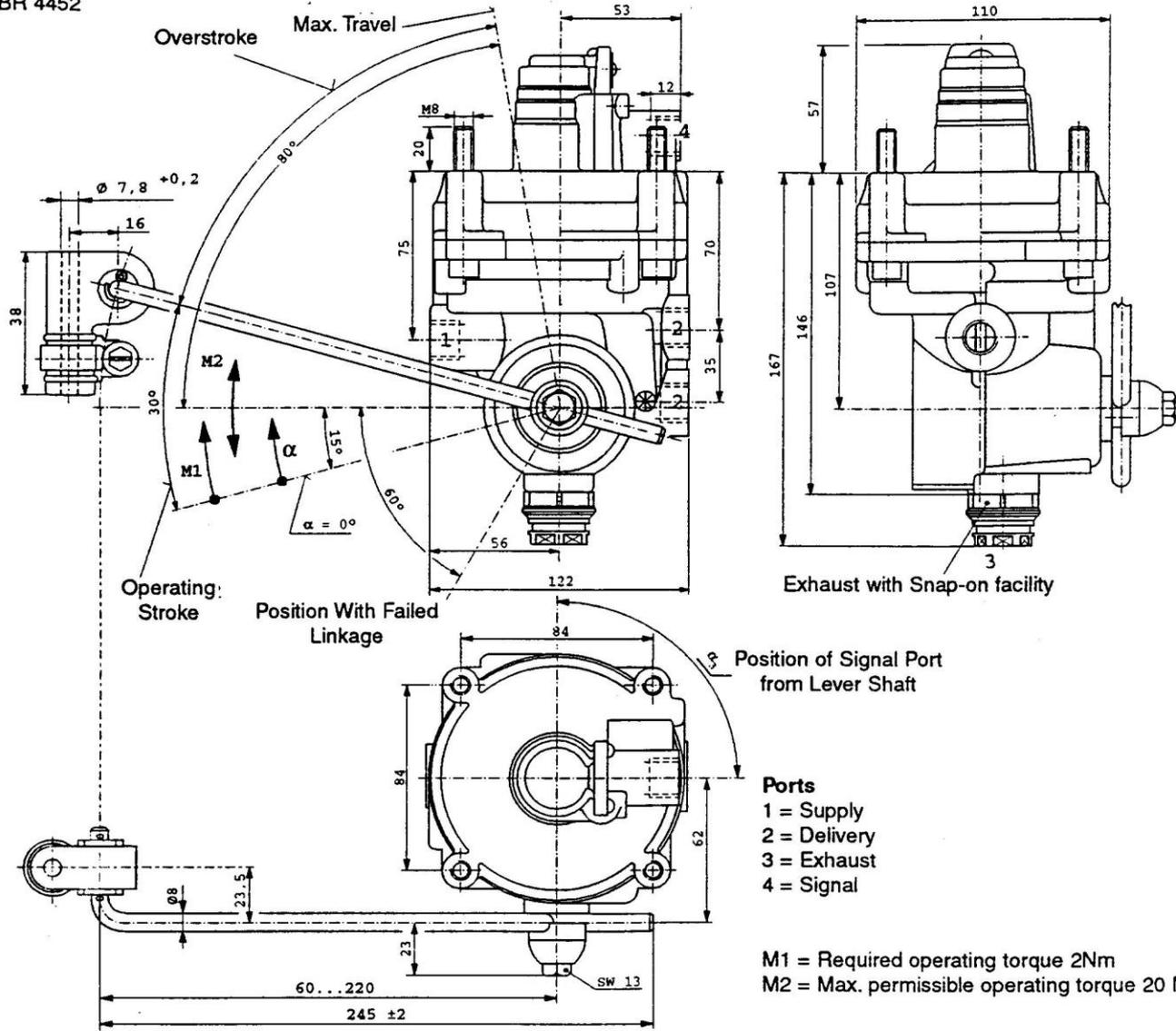
- The closer to the ideal brake curve for a particular vehicle weight, the bigger larger the brake force.



Cavalo mecânico



BR 4452



- Ports**
- 1 = Supply
 - 2 = Delivery
 - 3 = Exhaust
 - 4 = Signal

M1 = Required operating torque 2Nm
M2 = Max. permissible operating torque 20 Nm

Curvas características da válvula ALB

Diagrama A: P_2/P_4 (alfa = 0 e alfa = 30)

Diagrama B: P_2 com alfa ($P_4 = 7,5$ bar)

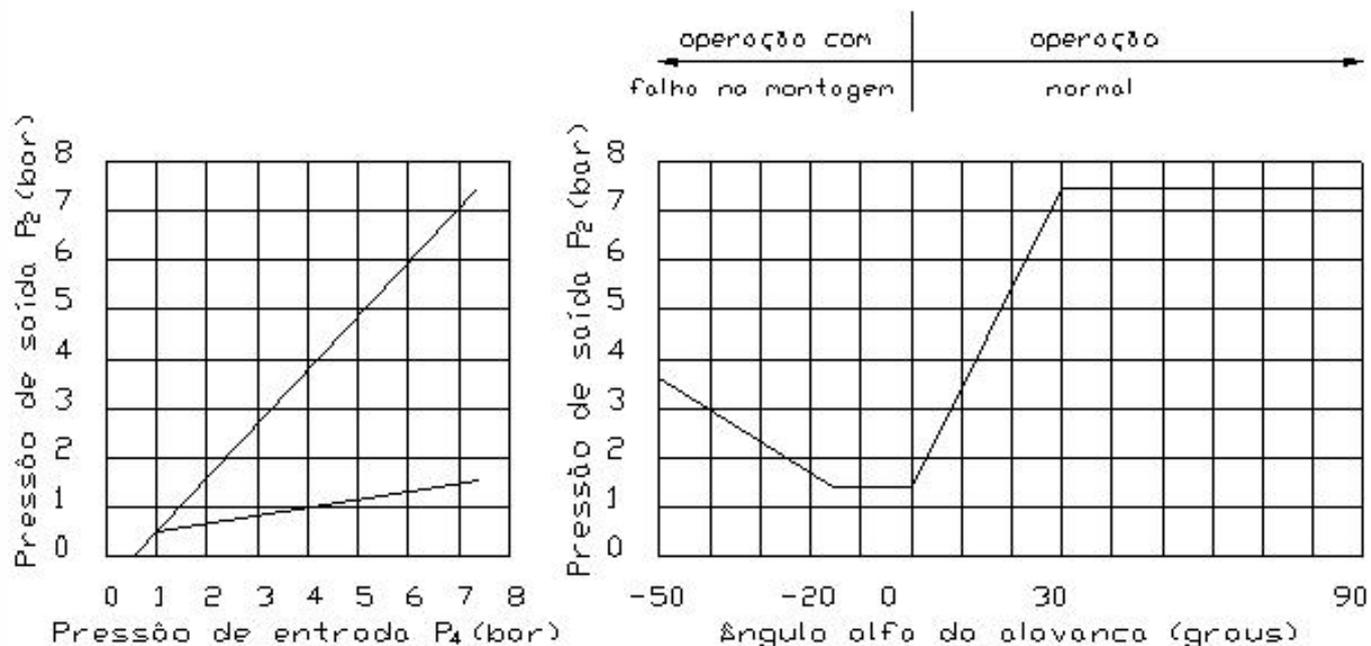
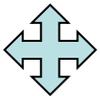
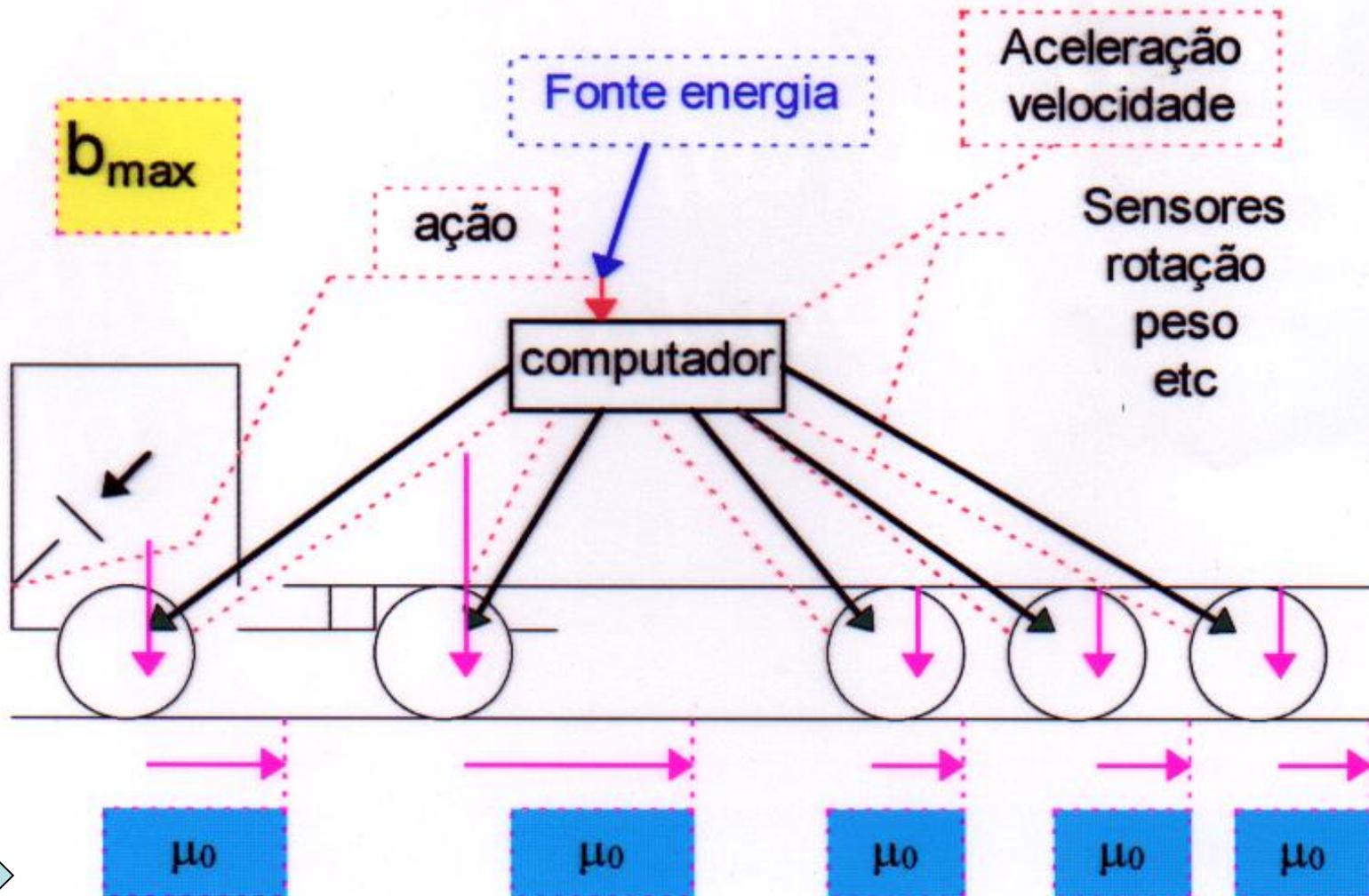


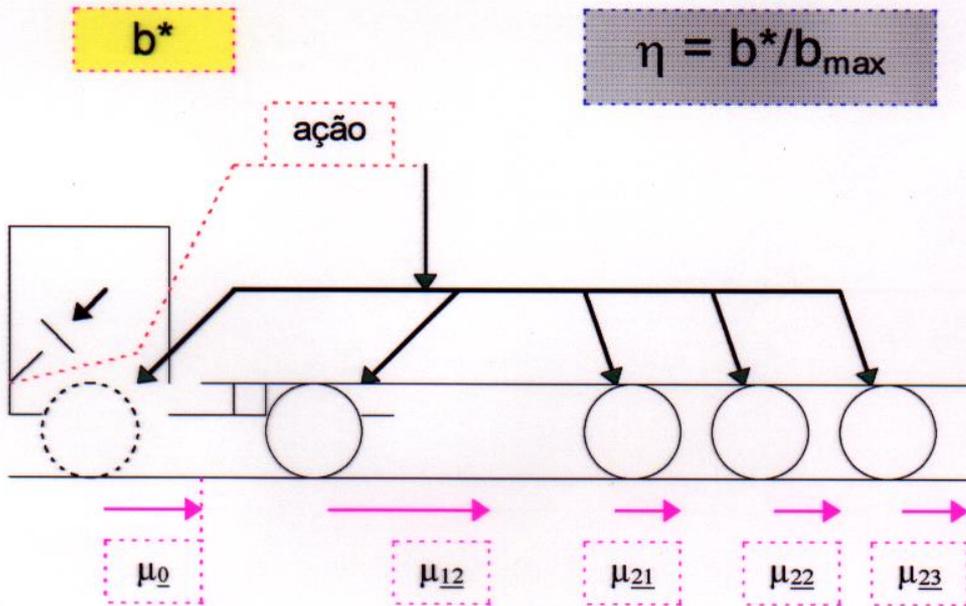
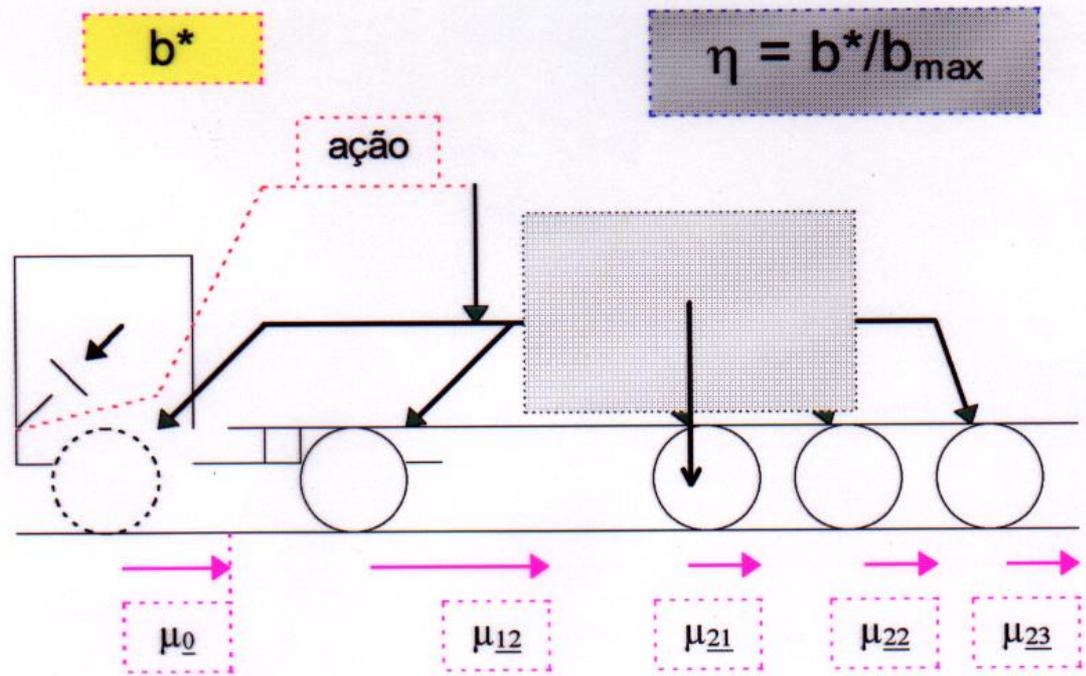
Diagrama A

Diagrama B

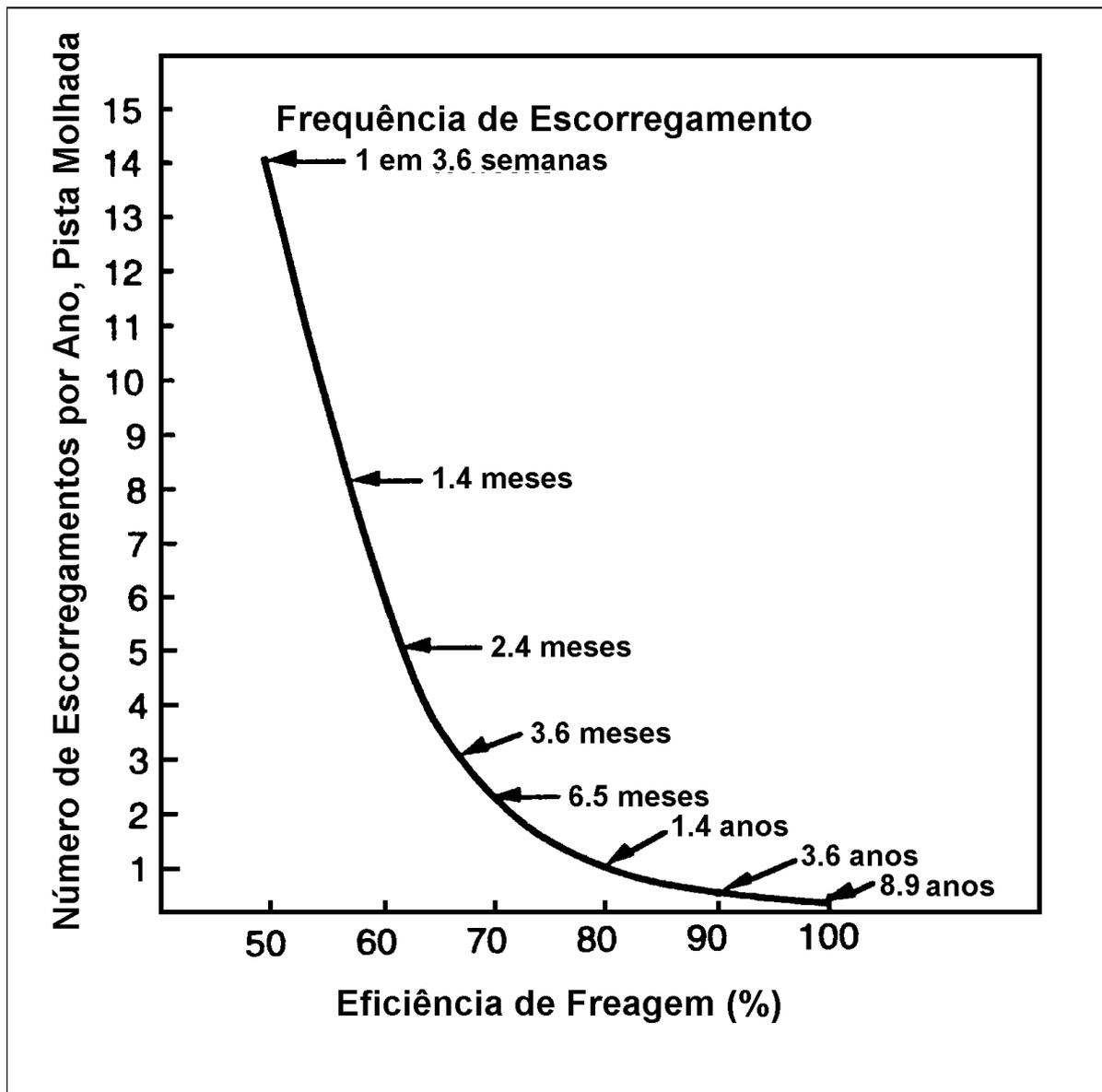
Veículo com freio de alto desempenho



eficiência



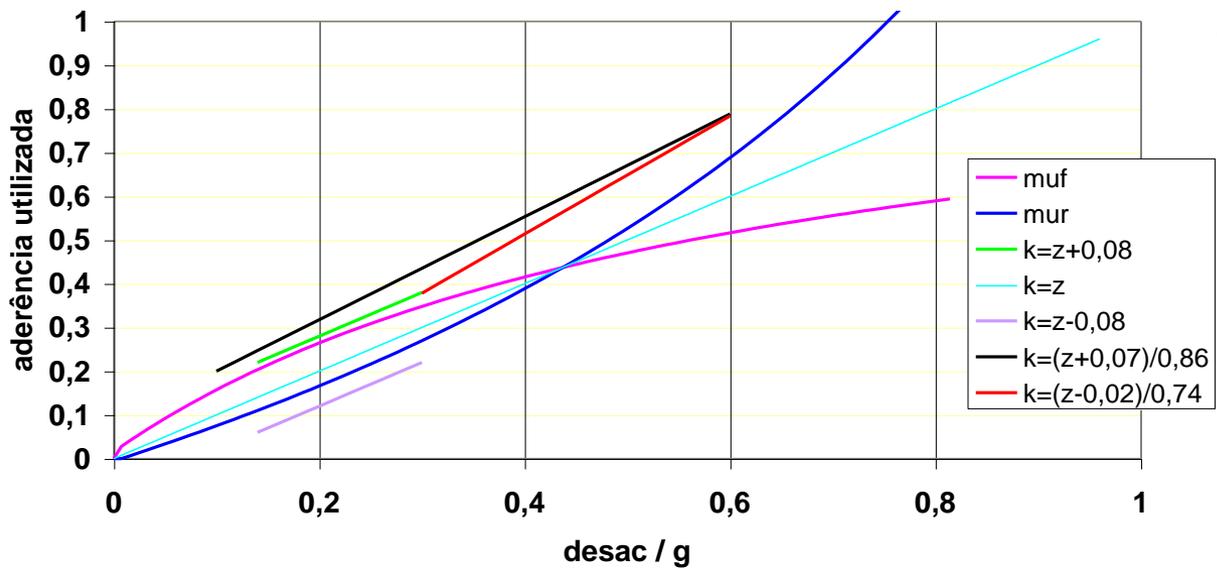
Eficiencia x segurança



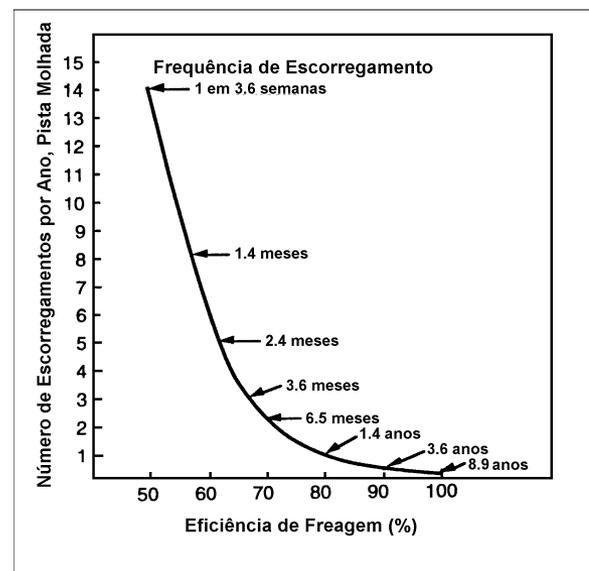
Veículo exemplo carregado - com válvula sensível à carga - Anexo 10
ECE-R13



carregado Vazio c/ Vazio s/

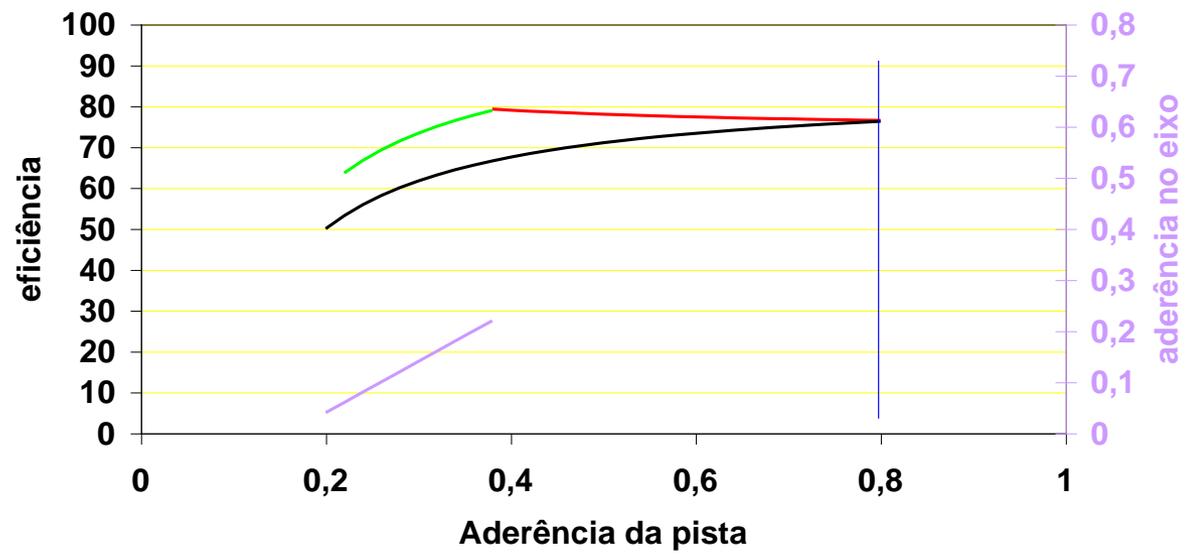


NBR 14354
AGO 1999



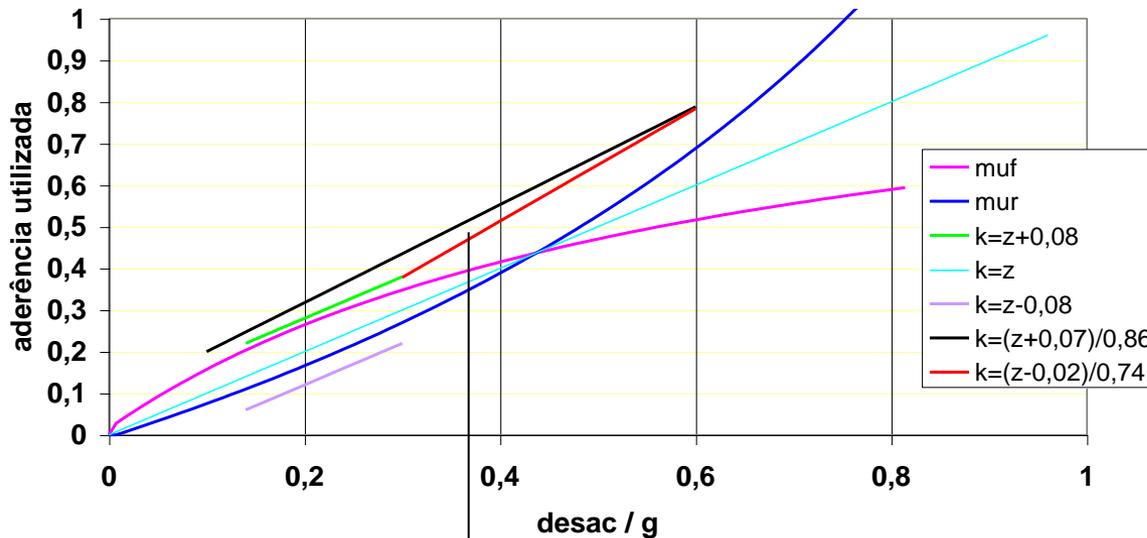
Colóquio freios Caxias

Anexo 10 ECE-R13 - fronteiras de eficiência



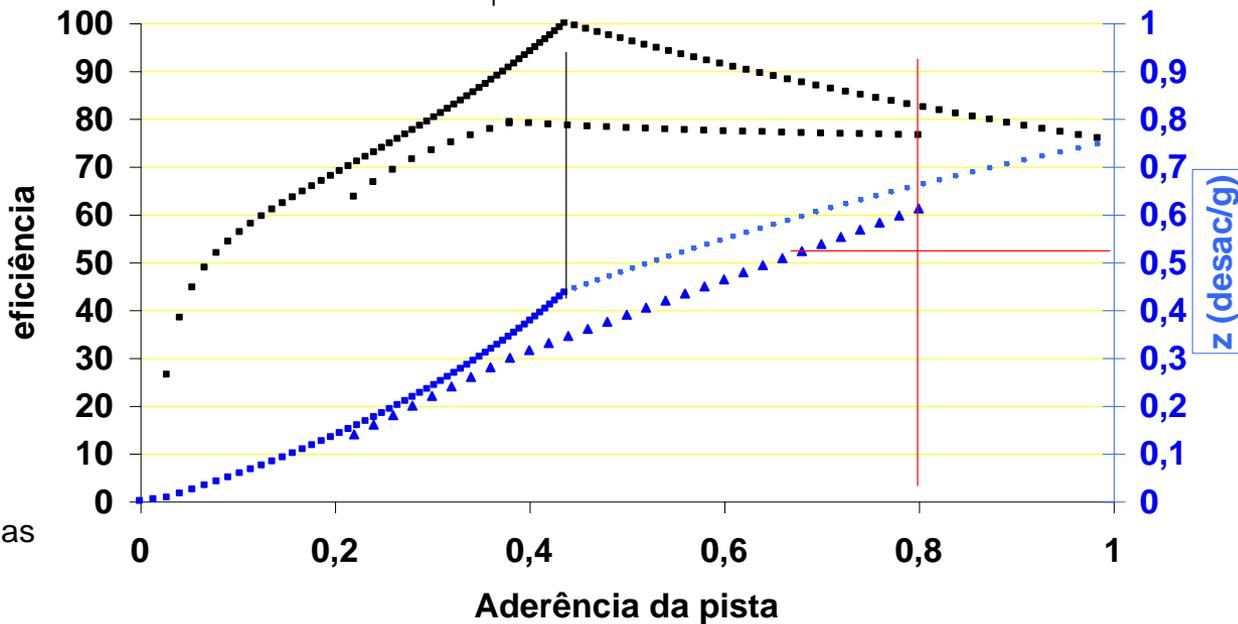
— $k=z+0,08$ — $k=(z-0,02)/0,74$ — $k=(z+0,07)/0,86$ — $k=z-0,08$

Veículo exemplo carregado - com válvula sensível à carga - Anexo 10 ECE-R13



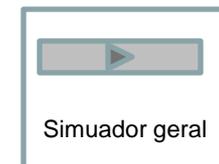
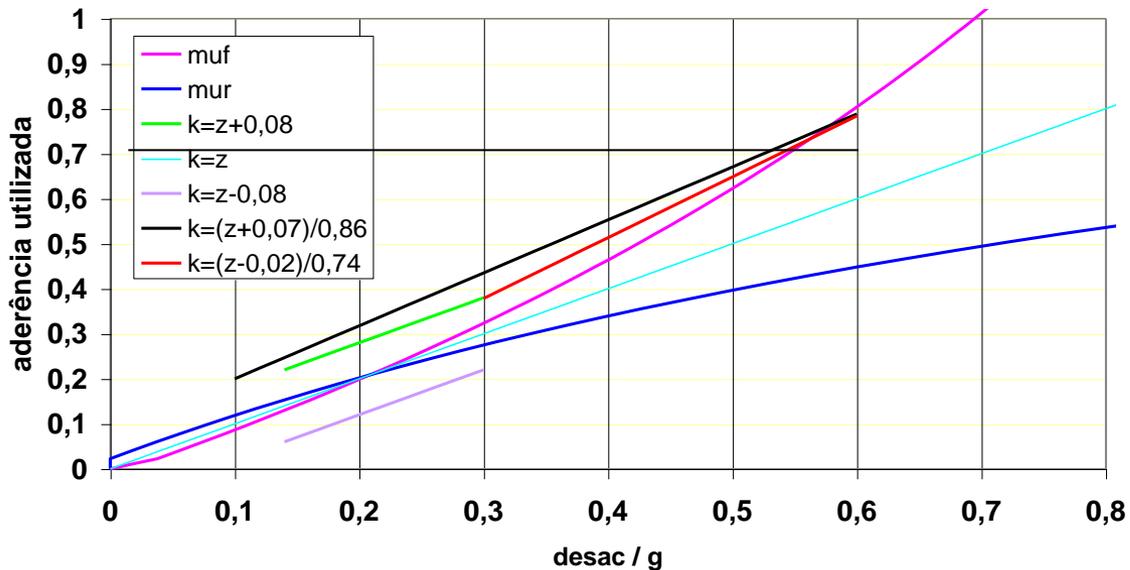
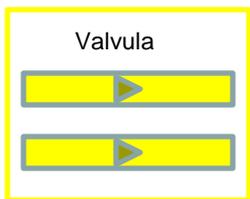
NBR 14354
AGO 1999

Veículo exemplo carregado - c/ válvula LSV - Anexo 10 ECE-R130

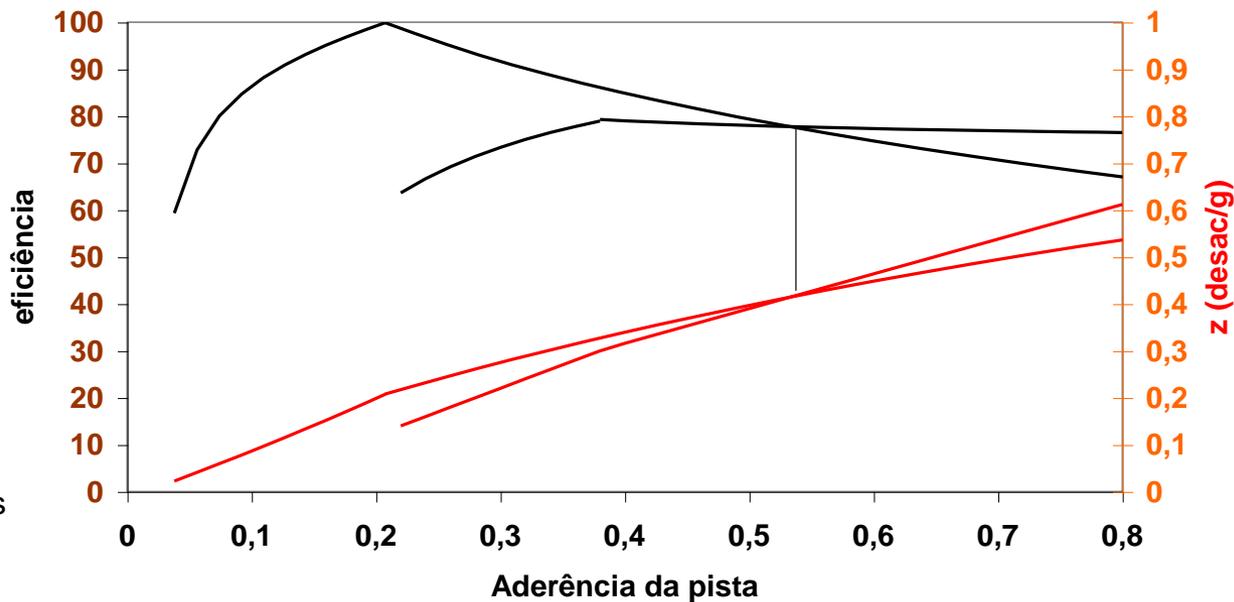


Colóquio freios Caxias

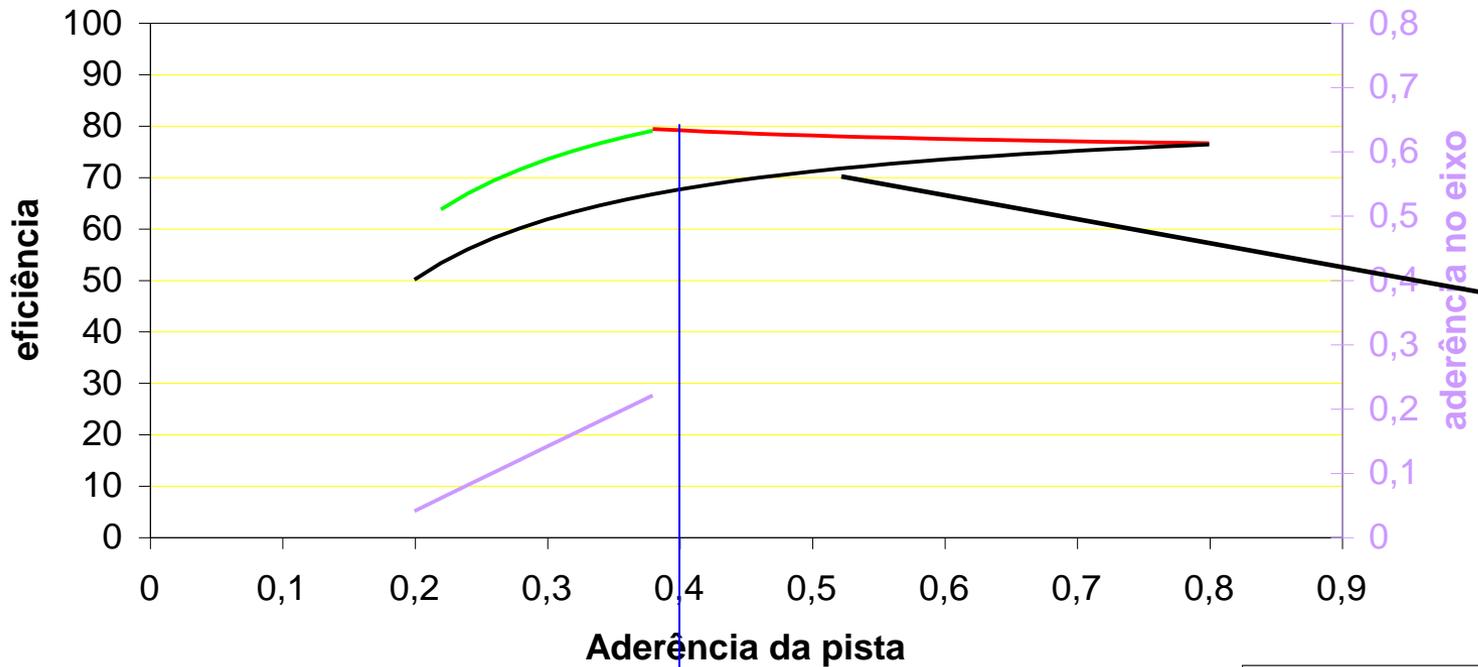
Veículo exemplo vazio - com válvula sensível à carga - Anexo 10 ECE-R13



ECE R13 anexo 10 - veículo vazio - sem válvula sensível à carga



Anexo 10 ECE-R13 - fronteiras de eficiência

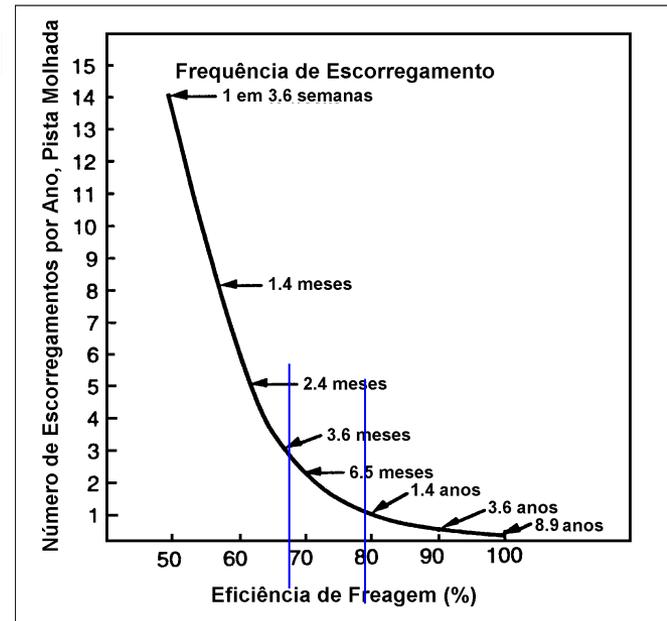


— $k=z+0,08$
 — $k=(z-0,02)/0,74$
 — $k=(z+0,07)/0,86$
 — $k=z-0,08$

Uso do ABS

NBR 14353

AGO 1999



COMPATIBILIDADE CAVALO - SEMIREBOQUE

- Em veículos combinados, deve haver compatibilidade de potência de frenagem entre o cavalo e o semi-reboque, caso contrário, haverá deslizamento das rodas de um deles prematuramente gerando movimentos instáveis com o jackknife e o swing trailer.



ABNT 14354



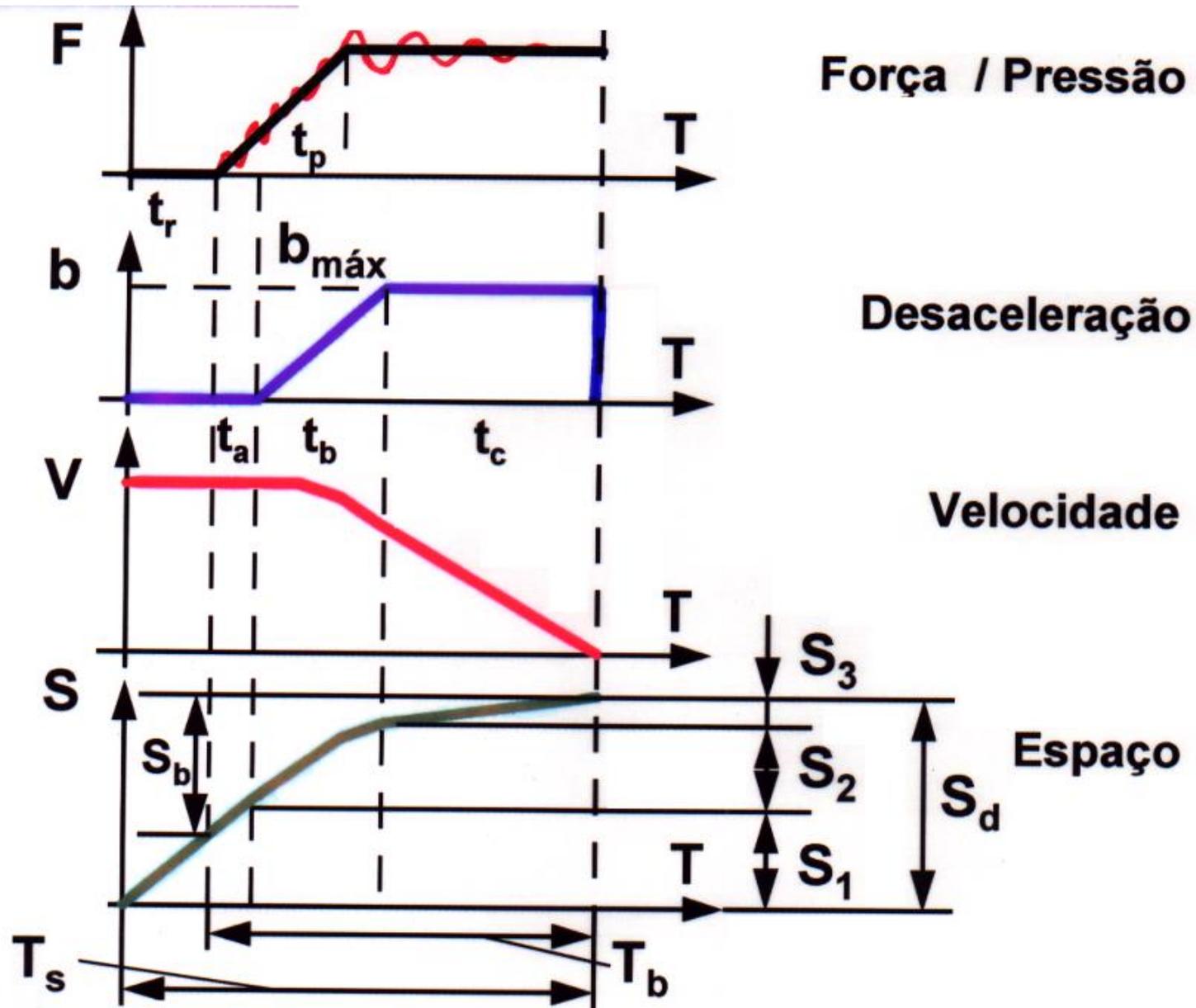
Compatibilidade semi-reboque



Compatibilidade cavalo



Compatibilidade semi-reboque



Espaço e tempo de parada quando a desaceleração é constante – sem efeito da resistência do ar – $t_c = \text{constante}$

EFICIÊNCIA DE 100%

$$S = \frac{\varphi_b}{g \times (\mu + f)} \times ((V_i^2 - V_f^2) \div 2)$$

$$t = \frac{\gamma_b}{g \times (\mu + f)} \times (V_i - V_f)$$

$$b = \frac{g}{\gamma_b} (\mu + f)$$

$$\gamma_b = 1.05$$

$$\mu = 0.8$$

$$V_i = 60 \text{ km / h}$$

$$f = 0.01$$

$$S = 18.35$$

$$t = 2.20$$

$$b = 7.56$$



Espaço e tempo de parada quando a desaceleração é constante – com efeito da resistência do ar – $tc = \text{constante}$

$$\rho = 1,22$$

$$A_f = 6$$

$$\gamma_b = 1.05$$

$$\mu = 0.8$$

$$V_i = 60 \text{ km / h}$$

$$f = 0.01$$

$$S = \frac{m \times \gamma_b}{2 \times C} \log_e \left(1 + \frac{C \times (V_i^2 - V_f^2)}{W \times (\mu + f)} \right)$$

$$C = 1/2 \times \rho \times C_d \times A_f \quad C_d = 1$$

$$S = 17,29$$

$$t = 2,12$$

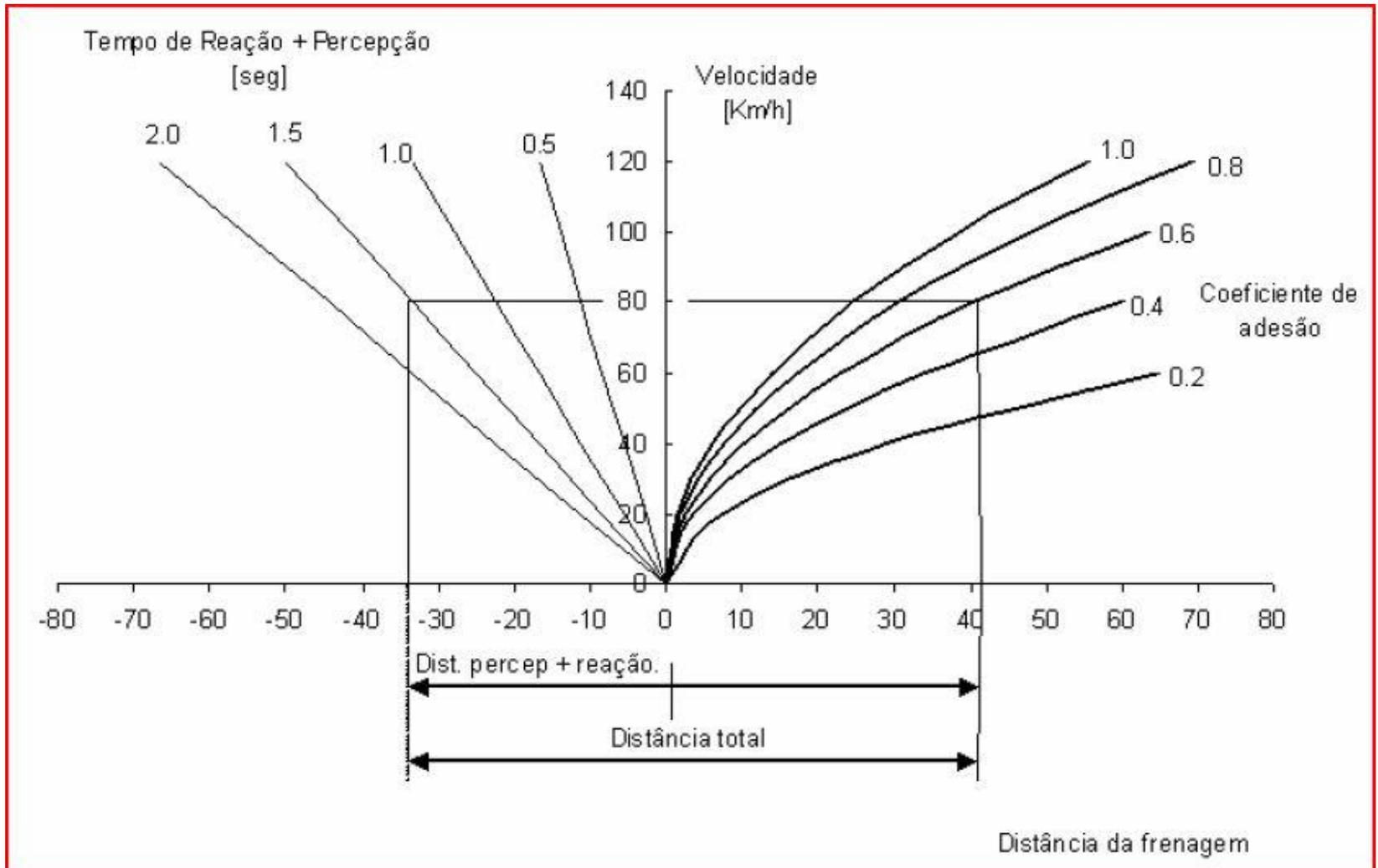
$$b = 8,52$$

$$t = \frac{\gamma_b}{g} \times \sqrt{\frac{W}{C \times (\mu + f)}} \times tg^{-1} \left((V_i - V_f) \times \sqrt{\frac{C}{W \times (\mu + f)}} \right)$$

$$b = \frac{g}{\gamma_b} \times \left(\mu + f + \frac{R_a}{W} \right)$$

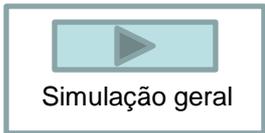
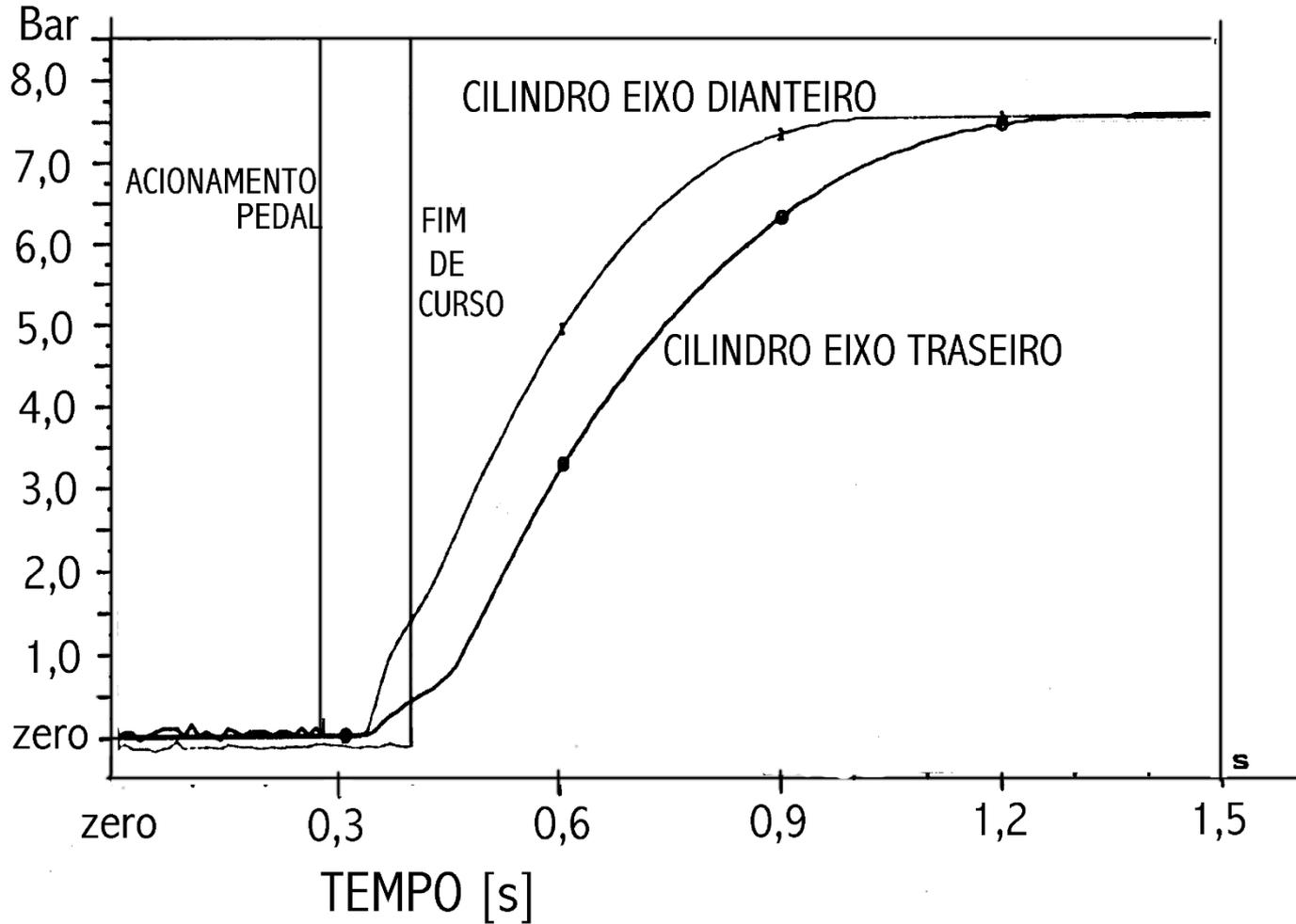


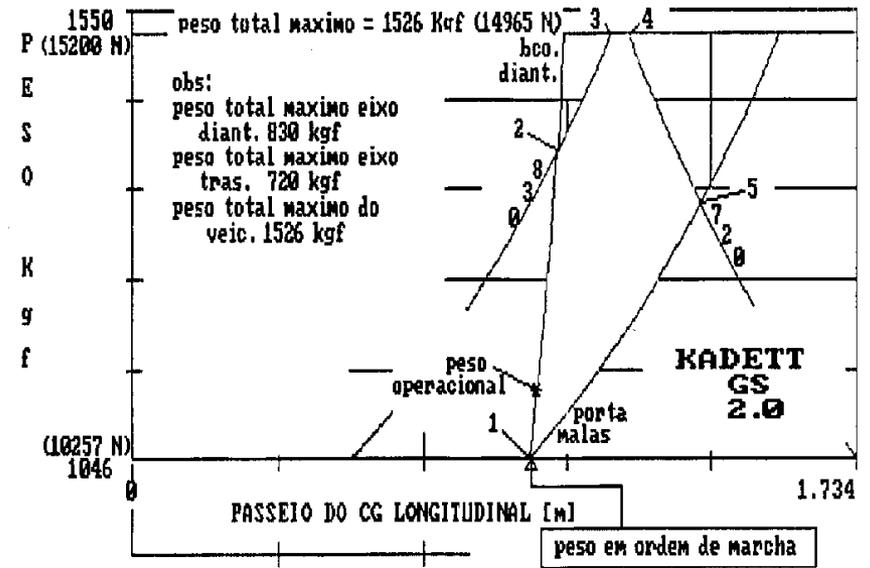
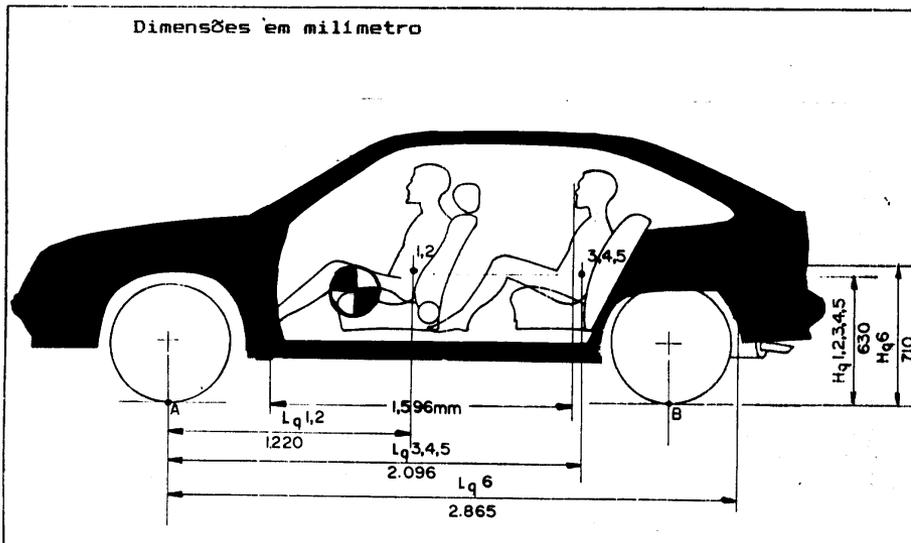
Espaço de parada com a reação do motorista



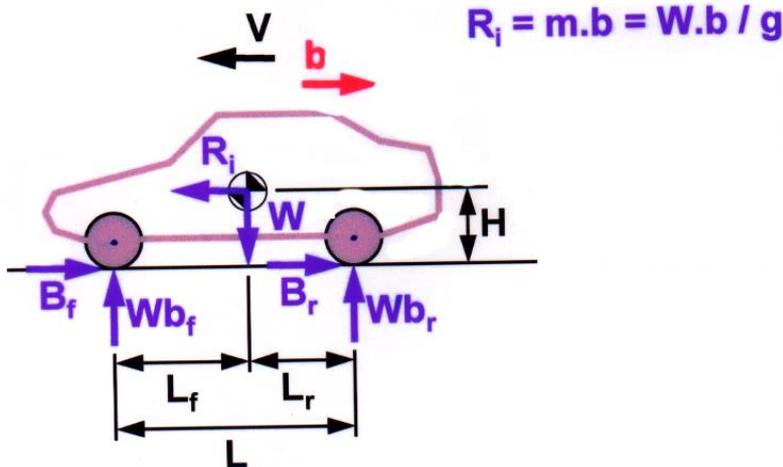
Tempo de resposta do freio

[ÔNIBUS]

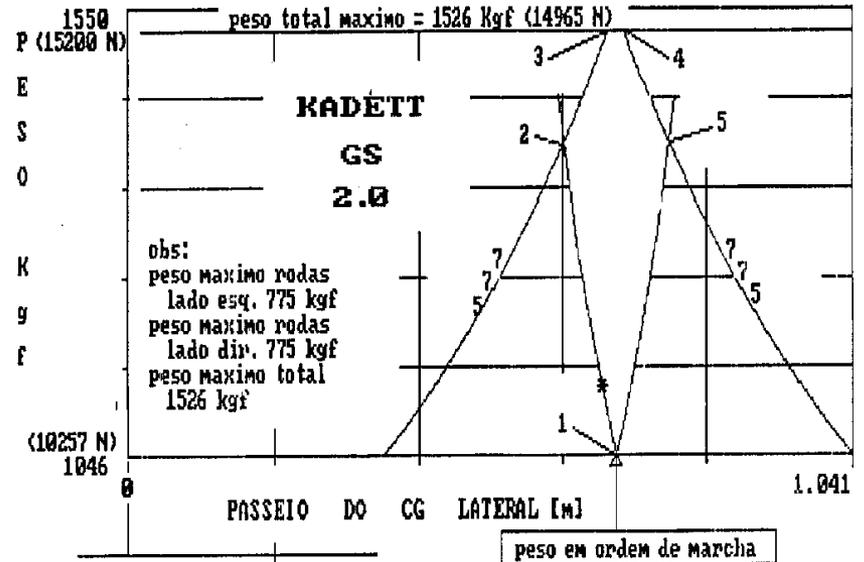




- Passeio do c.g. na longitudinal

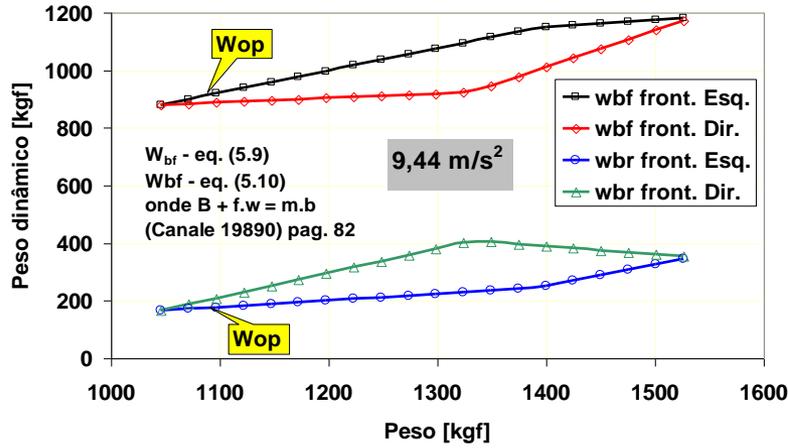


condição dinâmica: $Wb_f = (W/g) \cdot (L - L_f) + b \cdot H / L$
 $Wb_r = (W/g) \cdot (L_f - b \cdot H) / L$

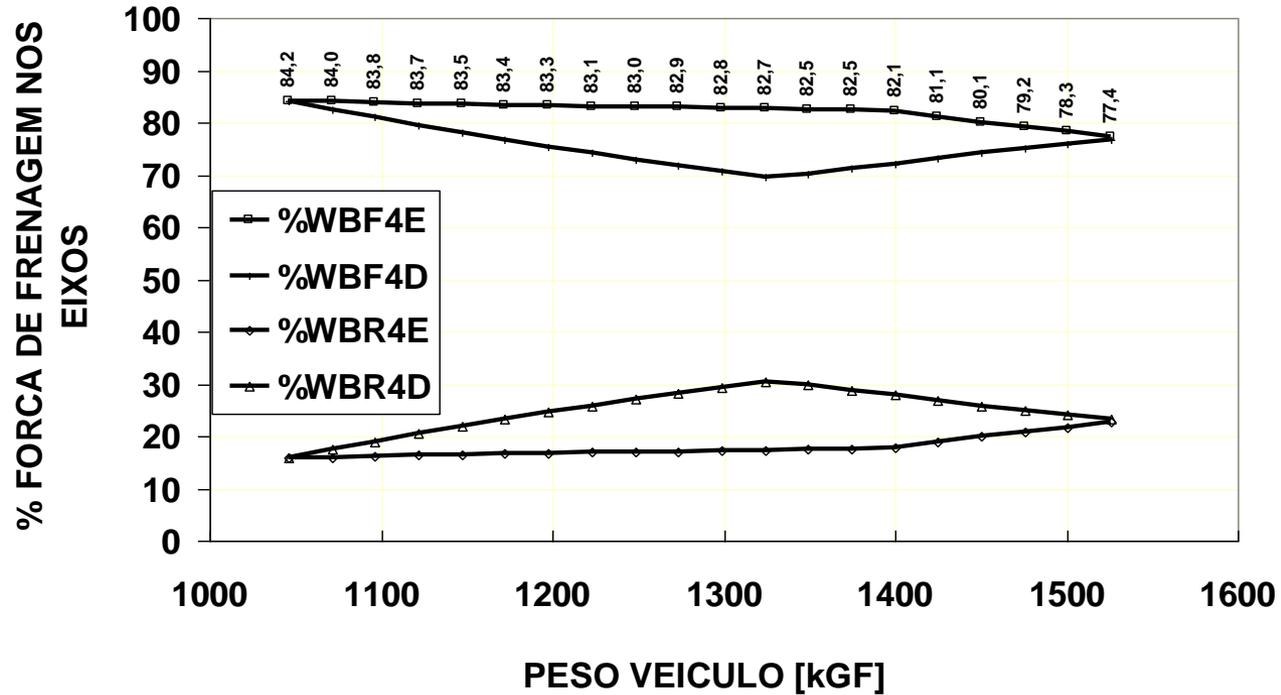


- Passeio do c.g. na transversal (lateral)

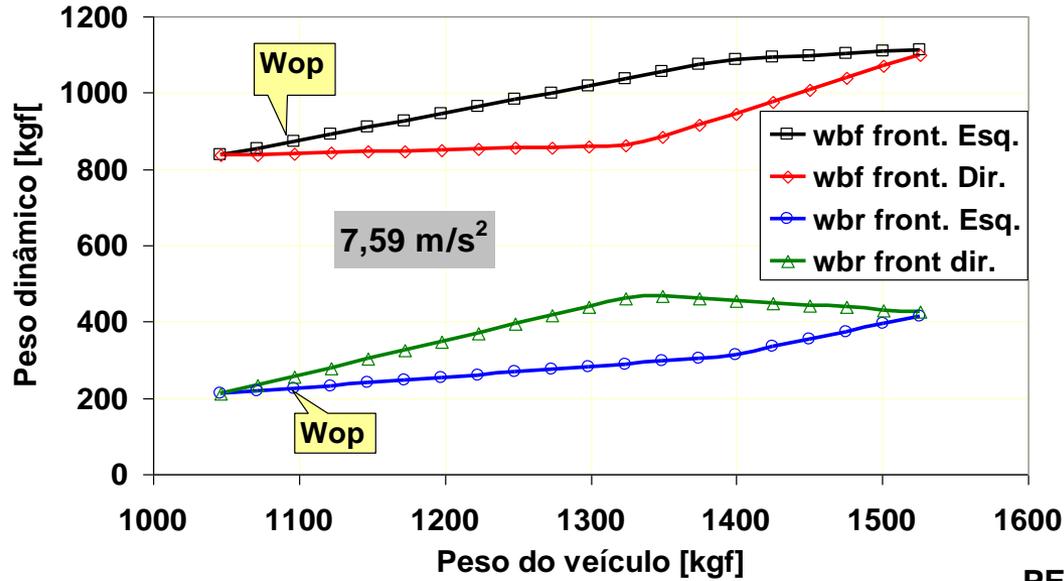
**PESO DINÂMICO x PESO DO VEÍCULO - Pista
asfalto/concreto - ótimas condições [1,0]**



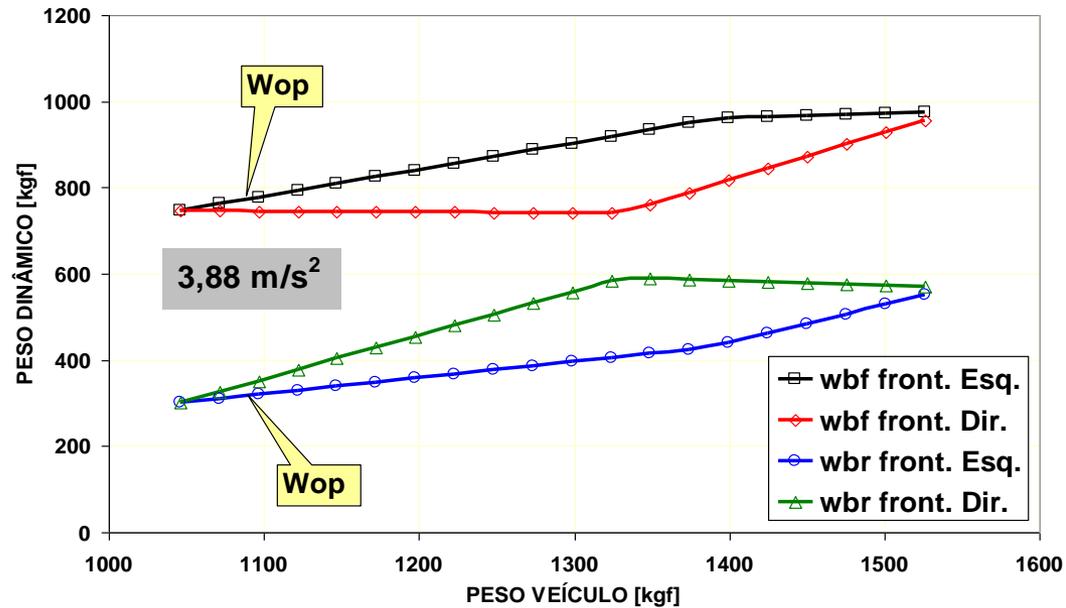
**DISTRIBUIÇÃO ÓTIMA DAS FORÇAS DE FRENAGEM NOS EIXOS DO
VEÍCULO
COEFICIENTE DE ADESÃO =1,0**



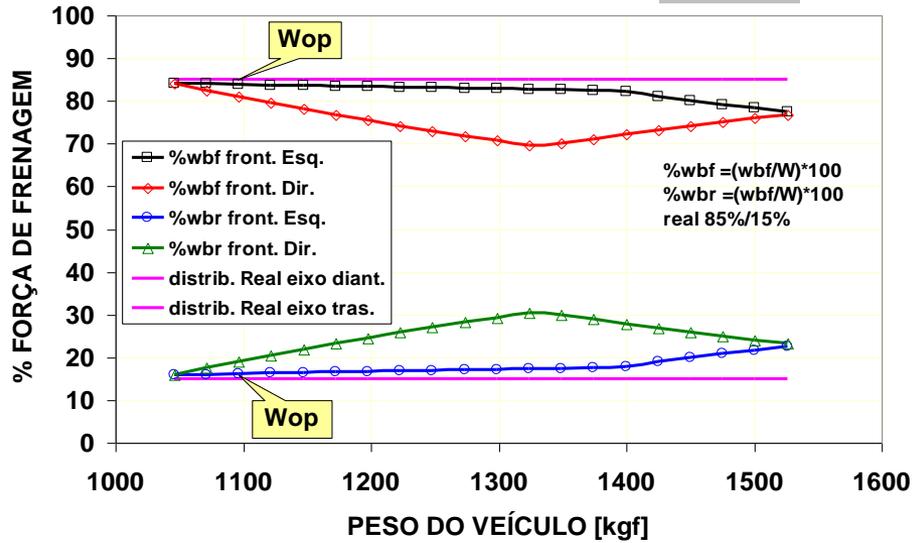
PESO DINÂMICO x PESO DO VEÍCULO - Pista asfalto/concreto - [0,8]



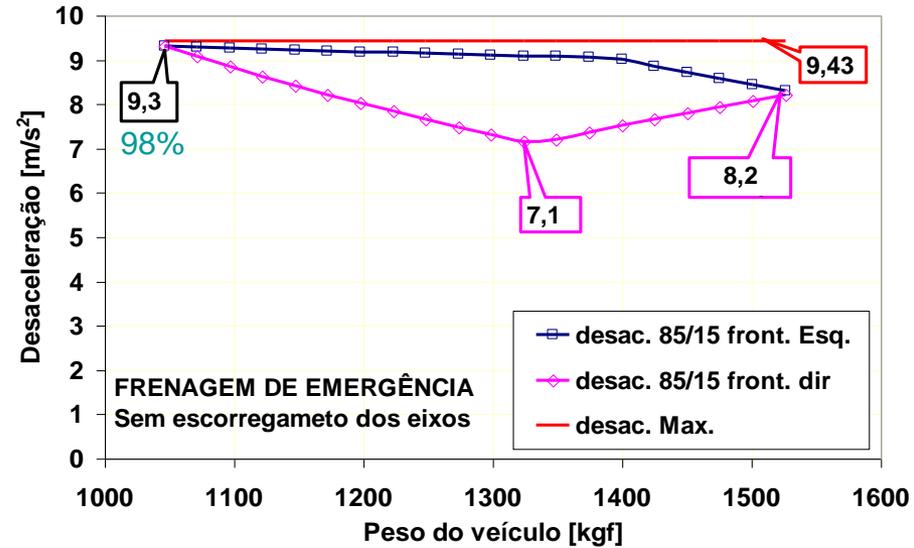
PESO DINÂMICO x PESO DO VEÍCULO [kgf] - pista asfalto/concreto molhada [0,4]



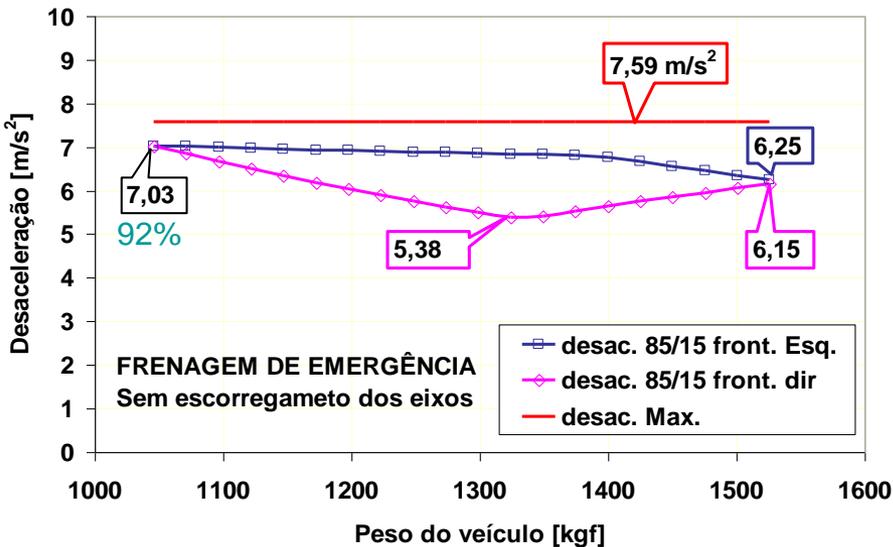
DISTRIBUIÇÃO REAL E IDEAL 9,44 m/s²



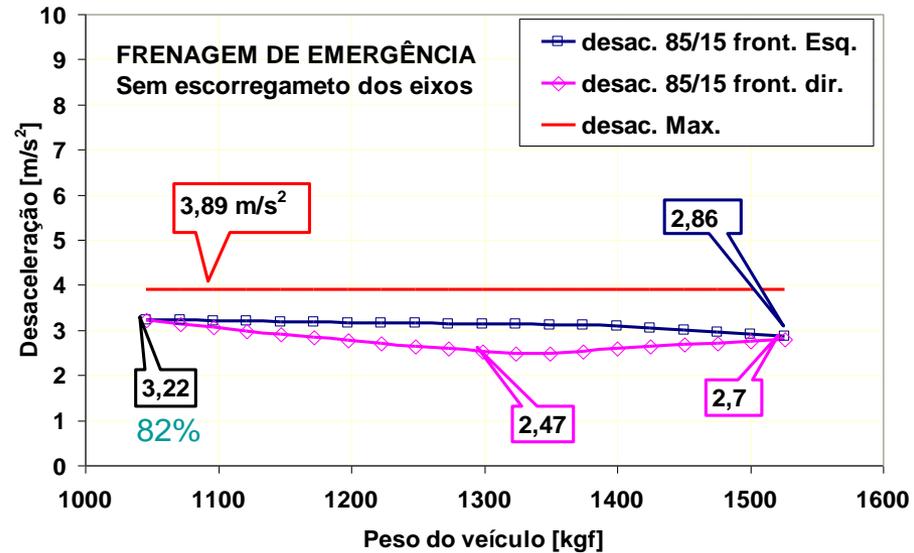
Desaceleração x Peso do veículo [$\pi_0 = 1,0$]



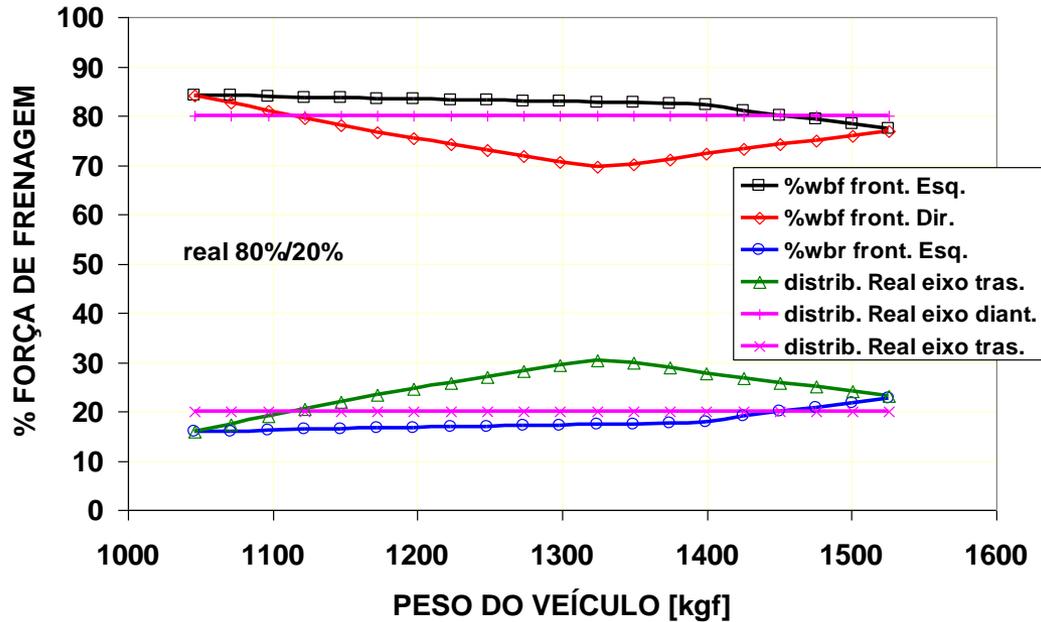
Desaceleração x Peso do veículo [$\pi_0 = 0,8$]



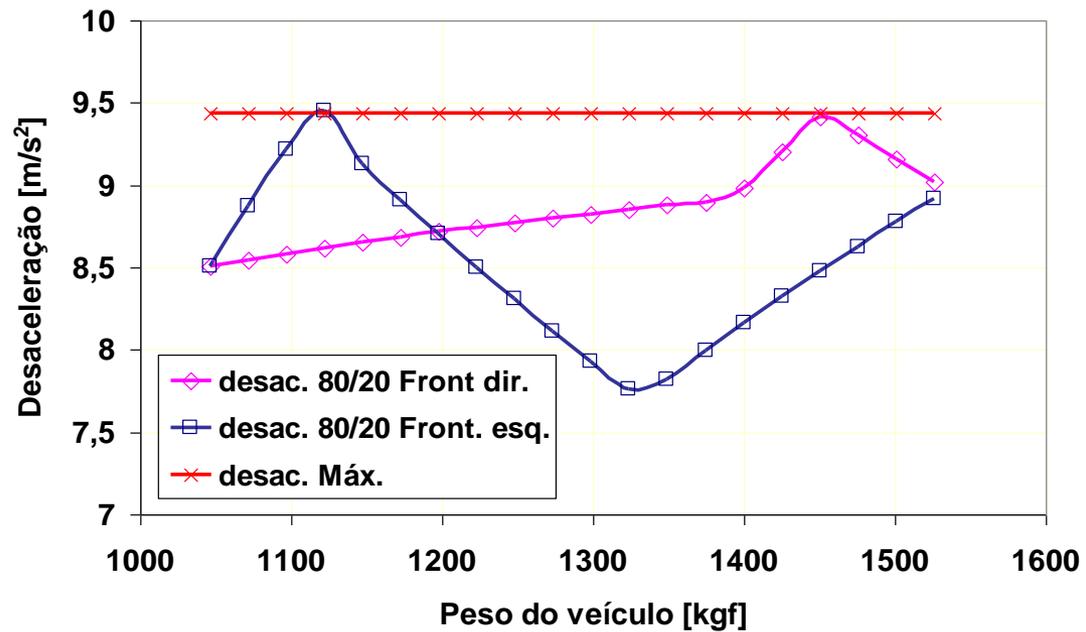
Desaceleração x Peso do veículo [$\pi_0 = 0,4$]



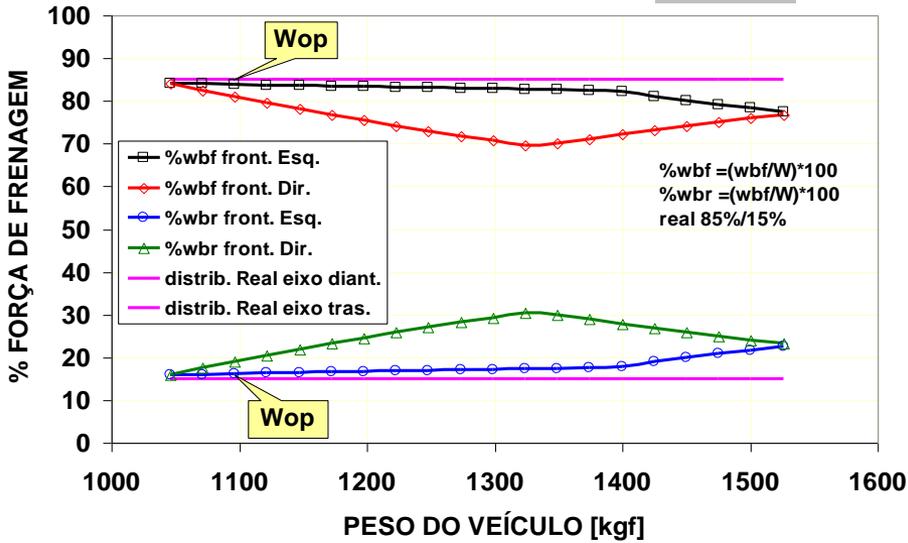
DISTRIBUIÇÃO REAL E IDEAL



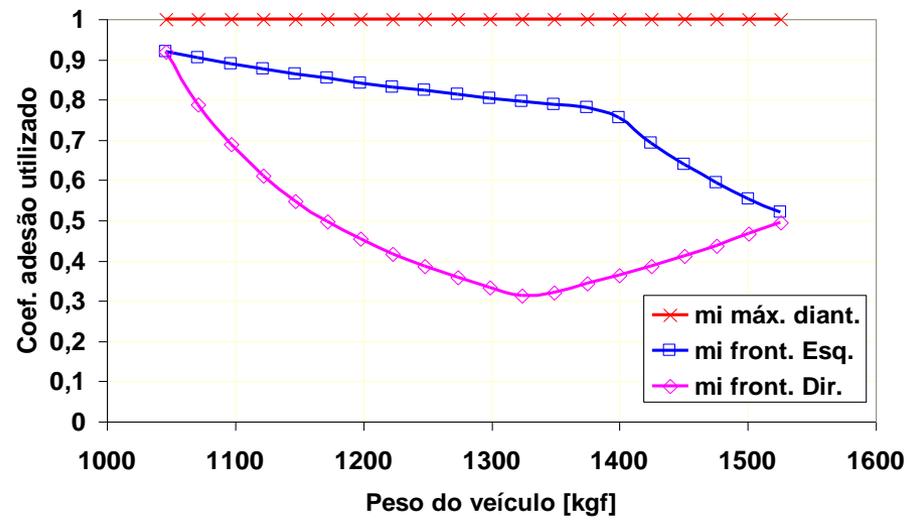
Desaceleração x Peso do veículo [$\pi_0 = 1,0$]



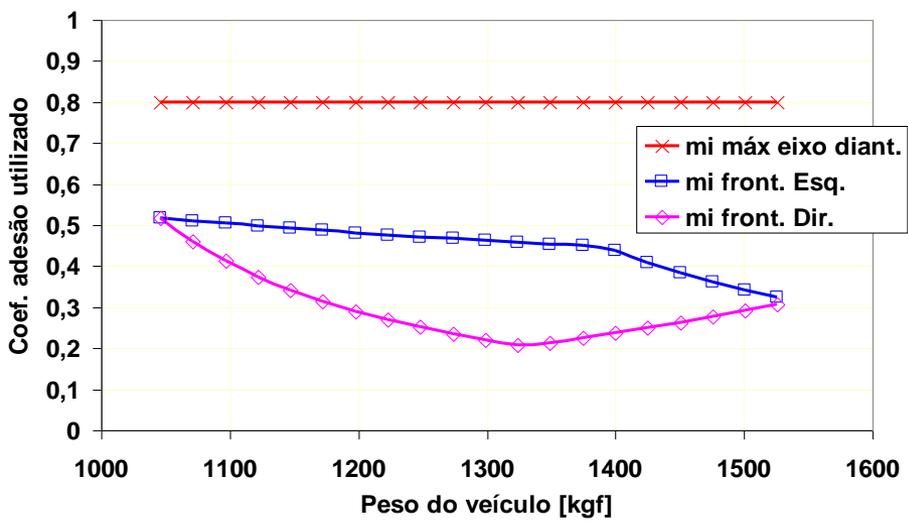
DISTRIBUIÇÃO REAL E IDEAL 9,44 m/s²



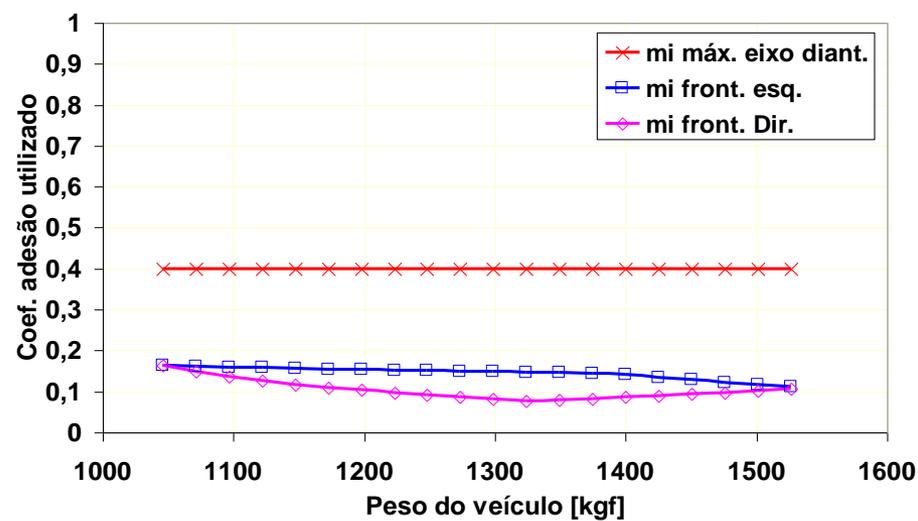
COEFICIENTE DE ADESÃO USADO x PESO DO VEÍCULO [$\pi_0 = 1,0$] - 85%/15%



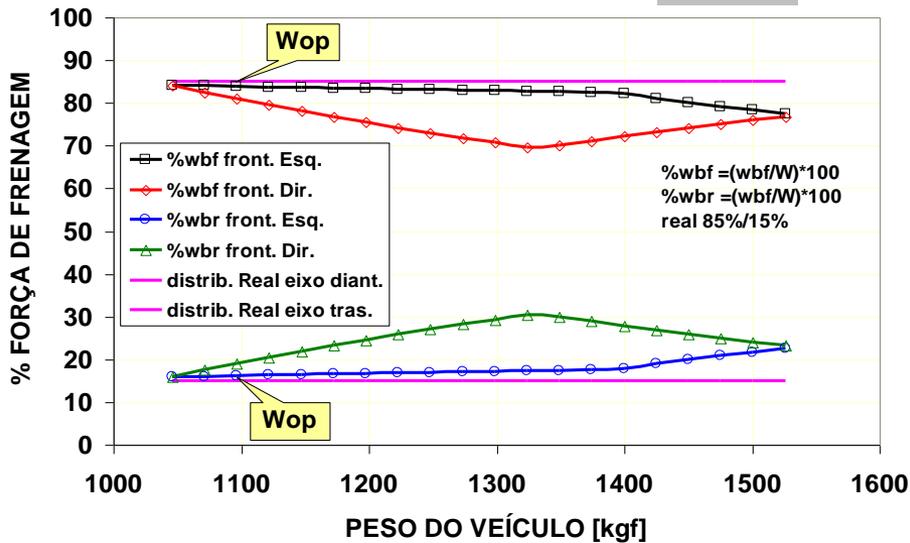
COEFICIENTE DE ADESÃO USADO x PESO DO VEÍCULO [$\pi_0 = 0,8$] - 85%/15%



COEFICIENTE DE ADESÃO USADO x PESO DO VEÍCULO [$\pi_0 = 0,4$] - 85%/15%

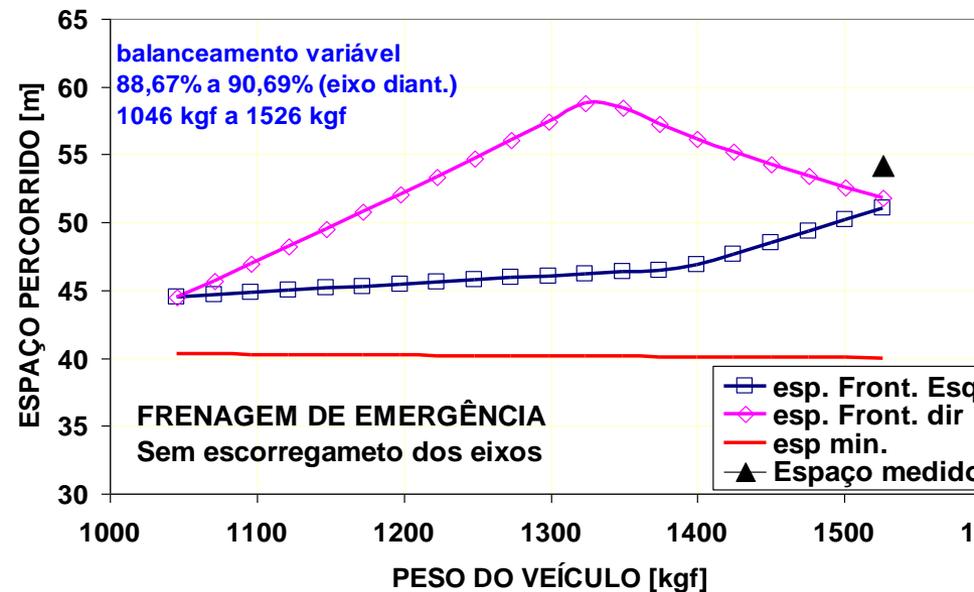


DISTRIBUIÇÃO REAL E IDEAL 9,44 m/s²



ESPAÇO PERCORRIDO x PESO DO VEÍCULO [$\pi_0 = 0,9$]

Veloc. inicial 96 km/h



ESPAÇO PERCORRIDO x PESO DO VEÍCULO [$\pi_0 = 0,8$]

Veloc. inicial 129 km/h

