

# Exercício 4.121

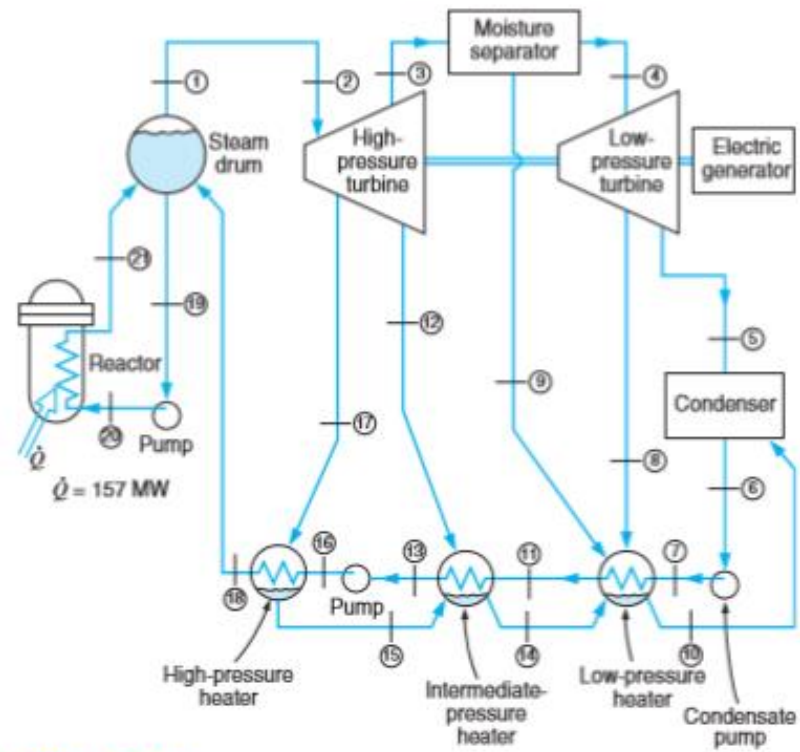
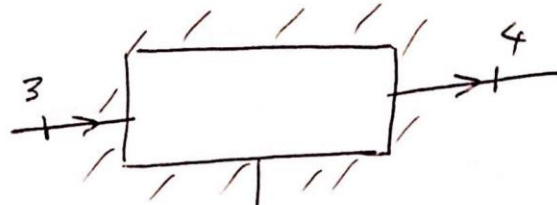


FIGURE P4.121

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a) SEPARADOR



$$\dot{Q} = 0$$

$$\dot{W} = 0$$

MASSA

$$\dot{m}_3 = \dot{m}_4 + \dot{m}_9 \Rightarrow \dot{m}_4 = 58,212 \text{ kg/s}$$

ENERGIA

$$\dot{m}_3 h_3 = \dot{m}_4 h_4 + \dot{m}_9 h_9$$

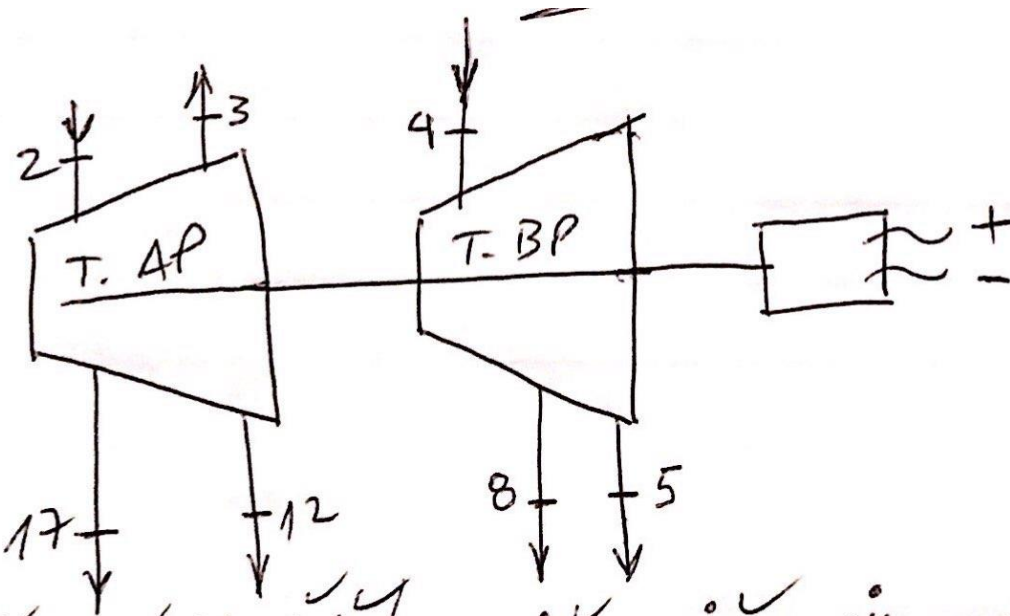
$$h_4 = 2673,9 = h_{f,4} + x_4 h_{g,4} \quad (P=310 \text{ kPa})$$

$\downarrow$   
56418

$\downarrow$   
2160,6

$$x_4 = 0,9755$$

5)



$$\dot{W}_{v.c.} = \dot{m}_2 h_2 - \dot{m}_{17} h_{17} - \dot{m}_{12} h_{12} - \dot{m}_3 h_3$$

$$\dot{W}_{v.c.} = 18394 \text{ kW} //$$

$$\dot{m}_4 = \dot{m}_8 + \dot{m}_5 \Rightarrow \dot{m}_5 = 55,44 \text{ kg/s}$$

$$\dot{W}_{v.c.} = \dot{m}_4 h_4 - \dot{m}_5 h_5 - \dot{m}_8 h_8$$

$$\dot{W}_{v.c.} = 22489 \text{ kW}$$

# Exercício 4.125

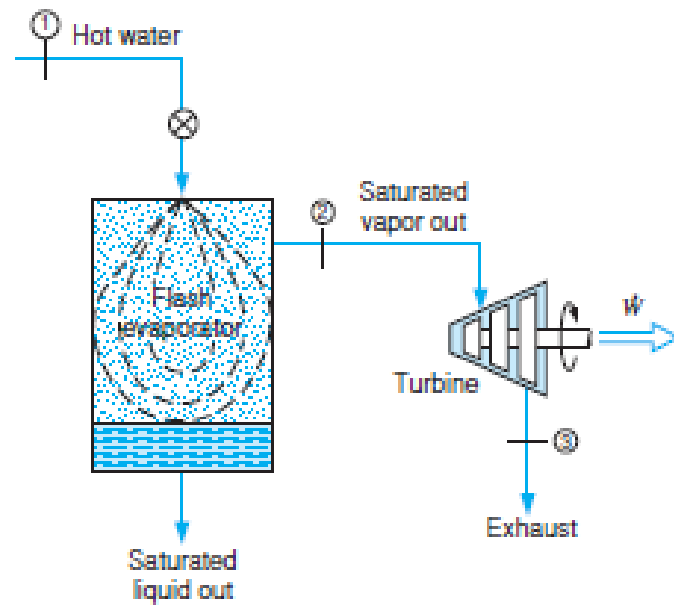
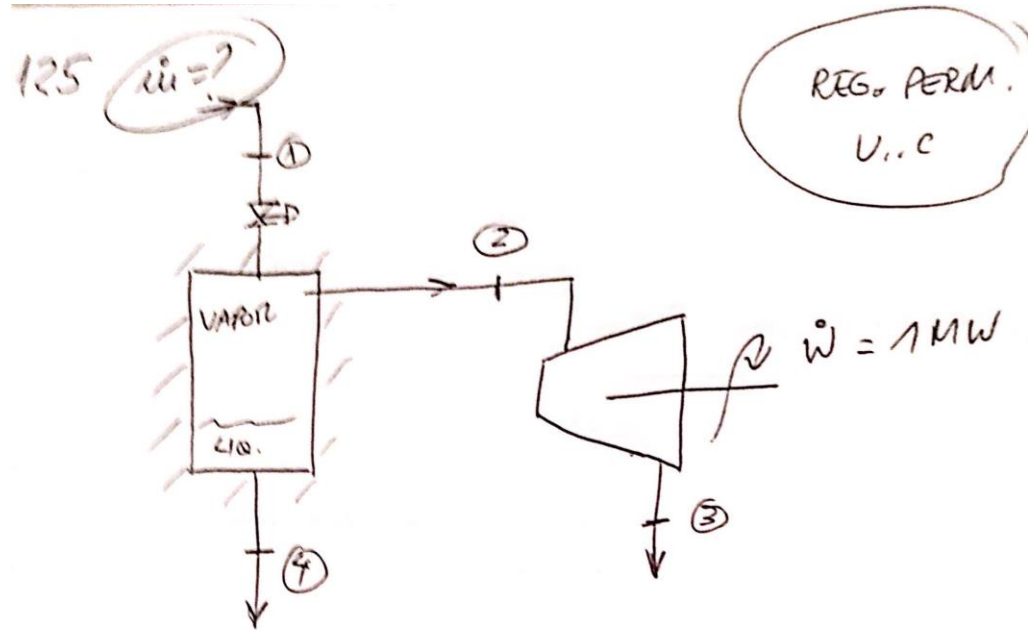


FIGURE P4.125



① :  $P = 1,5 \text{ MPa}$   
 $T = 180^\circ\text{C}$   
 $h_1 = 763,5 \text{ kJ/kg}$

② :  $P = 400 \text{ kPa}$   
 $x = 1,0$   
 $h_2 = 2738,6 \text{ kJ/kg}$

④ :  $P = 400 \text{ kPa}$   
 $x = 0,0$   
 $h_4 = 604,74 \text{ kJ/kg}$

$$\textcircled{1} : P = 1,5 \text{ MPa}$$

$$T = 180^\circ\text{C}$$

$$h_1 = 763,5 \text{ kJ/kg}$$

$$\textcircled{3} \quad h_3 = 191,83 + 0,9 \times 2392,8$$
$$P = 10 \text{ kPa}, \quad x = 0,9 \quad = 2345,4$$

$$\textcircled{2} : P = 400 \text{ kPa}$$

$$x = 1,0$$

$$h_g = 2738,6 \text{ kJ/kg}$$

$$\textcircled{4} : P = 400 \text{ kPa}$$

$$x = 0,0$$

$$h_f = 604,74 \text{ kJ/kg}$$

$$\text{Na turbina : } \dot{w} = \dot{m}_2 (h_2 - h_3)$$

$$\dot{m}_2 = \frac{1000 \text{ kW}}{(2738,6 - 2345,4)} = \underline{\underline{2543 \text{ kg/s}}}$$

No evaporador:

$$\dot{m}_1 h_1 = \dot{m}_2 h_2 + \dot{m}_4 h_4$$

$$h_1 = \frac{\dot{m}_2}{\dot{m}_1} h_2 + \frac{\dot{m}_4}{\dot{m}_1} h_4 \Rightarrow x = \frac{(h_1 - h_4)}{(h_2 - h_4)} = 0,074$$

CS Digitalizada com CamScanner

$$\Rightarrow \dot{m}_1 = \frac{2543}{0,074} = \underline{\underline{34.364 \text{ kg/s}}}$$



# Exercício 4.153

- 4.153 An insulated spring-loaded piston/cylinder device, shown in Fig. P4.153, is connected to an air line flowing air at 600 kPa and 700 K by a valve. Initially, the cylinder is empty and the spring force is zero. The valve is then opened until the cylinder pressure reaches 300 kPa. Noting that  $u_2 = u_{line} + C_v(T_2 - T_{line})$  and  $h_{line} - u_{line} = RT_{line}$ , find an expression for  $T_2$  as a function of  $P_2$ ,  $P_0$ , and  $T_{line}$ . With  $P_0 = 100$  kPa, find  $T_2$ .

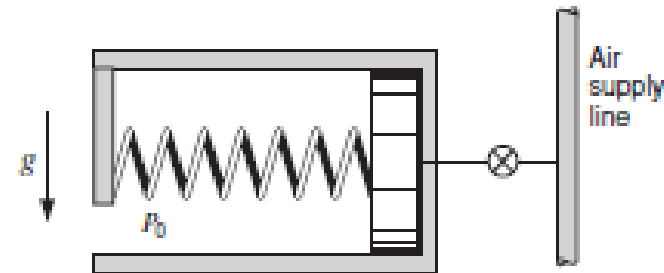
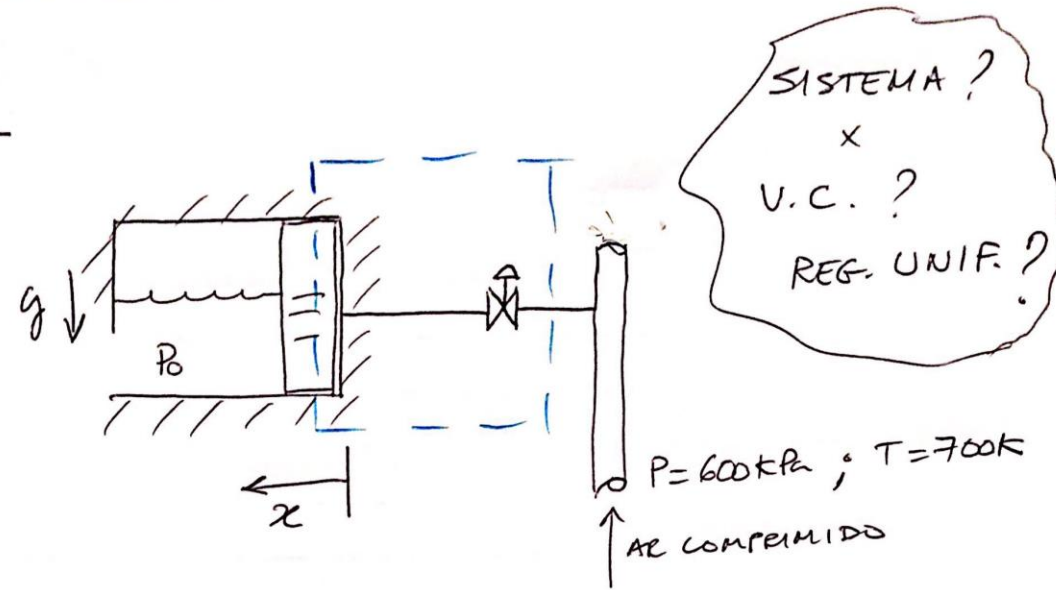


FIGURE P4.153



.C. → MASSA DE AR ENTRE O PISTÃO E A ENTRADA DA VÁLVULA

→ REGIÃO UNIFORME? HIPÓTESE FUNDAM.??

OK

MASSA :  $m_2 - m_1 = m_e$  (1)

ENERGIA :  $m_2 h_2 - m_1 h_1 = m_e h_e - \int P dV$  (2)

→ REGIME UNIFORME? HIPÓTESE FUNDAM.??

MASSA :  $m_2 - m_1^{\overset{0}{\cancel{}}} = m_e$  (1) OK

ENERGIA :  $m_2 u_2 - m_1^{\overset{0}{\cancel{}}} u_1 = m_e h_e - \int P dV$  (2)

$m_2 = m_e$  (1')

$m_2 u_2 = m_2 h_e - \int P dV$  (2')

mas  $\int_1^2 P dV = \frac{1}{2} (P_1 + P_2) (V_2 - V_1)$

Pressão na fronteira

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Sabemos que  $P_1 = P_0$ ,  $V_1 = 0$ ,

então 
$$\int_1^2 P dV = \frac{1}{2} (P_2 + P_0) V_2$$

em (2') :

$$u_2 u_2 = u_2 h_e - \frac{1}{2} (P_2 + P_0) V_2$$

$$\mu_2 \mu_2 = \mu_2 h_e - \frac{1}{2} (P_2 + P_0) V_2 \quad (2'')$$

HIPOTESE: An comprimido  $\rightarrow$  GAS PERFETTO

$$pV = RT \quad (4a) \quad e \quad dh = c_p dT \quad (4b)$$

$$du = c_v dT \quad (4c)$$

em (2''):

$$\mu_2 (\mu_2 - h_e) = -\frac{1}{2} (P_2 + P_0) V_2$$

$$\mu_2 (\mu_2 - (\mu_e + \underbrace{P_e v_e}_{RT_e})) = -\frac{1}{2} (P_2 + P_0) V_2$$

$$\mu_2 (\underbrace{\mu_2 - \mu_e}_{c_v (T_2 - T_e)} - RT_e) = -\frac{1}{2} (P_2 + P_0) V_2$$

$$(c_v (T_2 - T_e) - RT_e) = -\frac{1}{2} (P_2 + P_0) \left( \frac{V_2}{\mu_2} \right)$$

$$c_v (T_2 - T_e) - RT_e = -\frac{1}{2} (P_2 + P_0) \frac{RT_2}{P_2}$$

$$0,7165 (T_2 - 700) - 0,2870 \times 700 = -\frac{1}{2} (300 + 100) \times \frac{0,287 \times T_2}{300}$$

$$T_2 = 773,7 \text{ K}$$