

Proper Orthogonal Decomposition -POD

PEF5737 - Non-linear dynamics and stability

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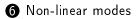
Objetivo

2 Introdução

3 Definitions

4 Geometric interpretation

Applications to linear systems
 Free vibrations
 Forced vibrations







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- 6 Non-linear modes





To present an introduction to the Proper Orthogonal Decomposition (POD) method. Some examples are also presented. This class is based on the paper [1].





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- POD is an empirical method for dynamic analysis and allows conclusions *a posteriori* about the investigated system;
- Besides its use in dynamics of structures, POD is also used in flow analyses (turbulence), image processing among other applications. This class focuses on its use for vibration analyses.





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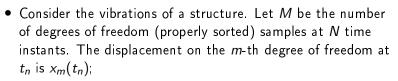
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 Consider the vibrations of a structure. Let M be the number of degrees of freedom (properly sorted) samples at N time instants. The displacement on the m-th degree of freedom at t_n is x_m(t_n);





• Let **X** the response matrix given by:

$$\mathbf{X} = \begin{pmatrix} x_1(t_0) & x_2(t_0) & x_3(t_0) & \dots & x_M(t_0) \\ x_1(t_1) & x_2(t_1) & x_3(t_1) & \dots & x_M(t_1) \\ x_1(t_2) & x_2(t_2) & x_3(t_2) & \dots & x_M(t_2) \\ \vdots & \vdots & \vdots & \vdots & \vdots \\ x_1(t_N) & x_2(t_N) & x_3(t_N) & \dots & x_M(t_N) \end{pmatrix}$$



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• The correlation matrix $\mathbf{R} = \frac{1}{N} \mathbf{X}^{\mathsf{T}} \mathbf{X}$.





- The correlation matrix $\mathbf{R} = \frac{1}{N} \mathbf{X}^{\mathsf{T}} \mathbf{X}$.
- As X is symmetric and real, it can be made a diagonal matrix. The eigenvalues of R are the proper orthogonal values (POVs) and the corresponding eigenvectors are the proper orthogonal modes (POMs).





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Let \mathbf{v} be a normalized POM. Consequently, $\mathbf{R}\mathbf{v} = \lambda\mathbf{v}$. Como $\mathbf{R} = \frac{1}{N}\mathbf{X}^{\mathsf{T}}\mathbf{X}$, we obtain $(\mathbf{X}\mathbf{v})^{\mathsf{T}}(\mathbf{X}\mathbf{v}) = \lambda N \leftrightarrow \frac{1}{N}(\mathbf{X}\mathbf{v})^{\mathsf{T}}(\mathbf{X}\mathbf{v}) = \lambda$





Each row of X can be interpreted as an snapshot (a "photograph"). Defining \mathbf{p}_j as the snapshot at a particular instant t_j , $\mathbf{X}\mathbf{v}$ is given by:





 $\mathbf{X}\mathbf{v}$ can be interpreter as the projection of the experimental data onto \mathbf{v} . Consequently, λ plays the role of a mean squared distance from the origin. In mechanical systems, such as distance is associated with the energy.





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Undamped system under free vibrations $M\ddot{x} + Kx = 0$. Its solution can be written as functions of the natural modes v_i by means of:

$$\mathbf{x}(t) = \sum_{i=1}^{M} A_i \sin(\omega_i t - \phi_i) \mathbf{v_i}$$





• As showed, an eigenvector of **R** (POM) converges to a modal vector. This is valid for low-damped systems.





- As showed, an eigenvector of **R** (POM) converges to a modal vector. This is valid for low-damped systems.
- POD can be used as an empiric scheme for determining the modal shape (a possible alternative do other methods such as, for example, the Circle Adjust Method)





For a harmonically forced system, the POM do not tend to the modal vectors. However, close to the resonance, in which one mode dominates the response and the corresponding POV is much large than the others, the associated POM is a good approximation for the excited mode.

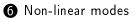




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• A non-linear mode can be interpreted as an invariant manifold in the state-space;





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- Fenny & Kappagantu (1998) deal with synchronous non-linear modes.



The POD use for non-linear systems



• POMs are an optimal linear representation for the non-linear normal modes;





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- The POD technique can be used obtaining reduced-order models for non-linear systems





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- Even for forced vibrations, the modal shapes can be obtained from POD;





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- For synchronous non-linear normal modes, POMs consist of the best linear fit (based on energy criterion);
- Feeny & Kappagantu suggest, as further works, investigations on the POD use for non-synchronous non-linear modes.







B. F. Feeny and R. Kappagantu.

On the physical interpretation of proper orthogonal modes in vibrations.

Journal of sound and vibration, 211:607–616, 1998.

