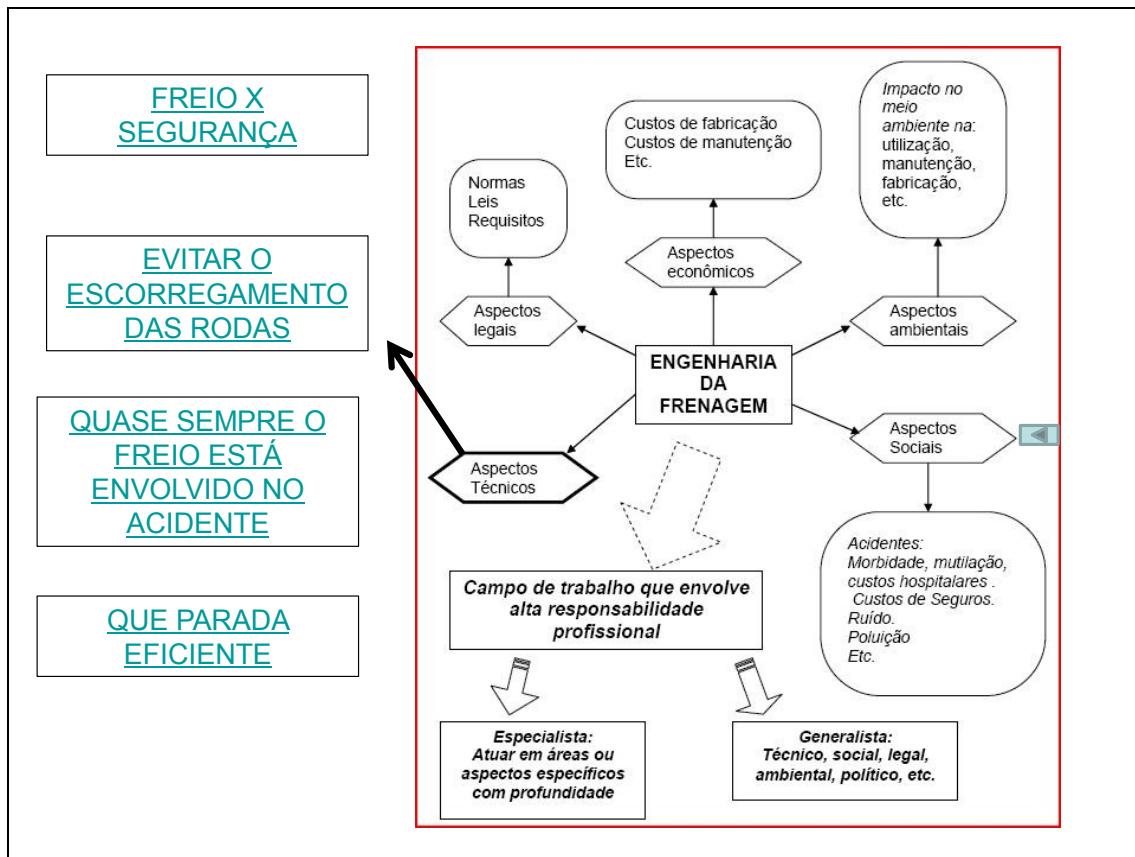




ECE REGULATION No.13 & EEC DIRECTIVE 71/320													
SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS				CARS & BUSES			VANS & TRUCKS			TRAILERS			
NOTATION:				M1	M2	M3	N1	N2	N3	O1	O2	O3	O4
Vehicle Category				≤8	>8	>8				NBR 10966			
No. of Passenger Seats (excl.driver)													
FL = Fully laden				≤5	>5		≤3½	3½-12	>12	≤¾	¾-3½	3½-10	>10
UL = Unladen													
Max. Vehicle Weight (Tonnes)													
SERVICE BRAKE (SB) Acting on all wheels, properly distributed and symmetric across vehicle. Braking distribution in accordance with Annex 10 and 75/524, respectively.													
1. TYPE - 0 TEST in Neutral													
NBR 10967 Prescribed Speed km/h				80	60	60	80	60	60	Service Brake is Optional		60	
Cold brakes Stopping Distance m ≤				0.1V + V ² /150		0.15V+V ² /130						Brake Force ≥ 50% of Trailer max. axle wt. (45% for semi-trailers) using ≤ 6.5 bar	
FL & UL MFDD m/sec ² ≥				5.8		5.0							
Pedal Effort daN ≤				50		70						Overrun Brakes permitted for 01/02 only	
2. TYPE - 0 TEST in Gear.				Effectiveness tests from various speeds (30% to 80% of Vmax) - check vehicle behaviour.									
Cold brakes FL & UL													
3. TYPE - 0 TEST in Gear.													
Prescribed Speed = 80% Vmax, but km/h ≤				160	100	90	120	100	90				
Cold brakes Stopping Distance m ≤				0.1V + V ² /130		0.15V+V ² /103.5							
FL & UL MFDD m/sec ² ≥				5.0		4.0							
Pedal Effort daN ≤				50		70							

ECE REGULATION No.13 & EEC DIRECTIVE 71/320												
SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS		CARS & BUSES			VANS & TRUCKS			TRAILERS				
NOTATION:		M1	M2	M3	N1	N2	N3	O1	O2	O3	O4	
Vehicle Category												
No. of Passenger Seats (excl.driver)		≤8	>8	>8								
FL = Fully laden												
UL = Unladen												
Max. Vehicle Weight (Tonnes)		≤5	>5		≤3½	3½-12	>12	≤3¼	3¼-3½	3½-10	>10	
SERVICE BRAKE (SB)		Acting on all wheels, properly distributed and symmetric across vehicle Braking distribution in accordance with Annex 10 and 75/524, respectively.										
4. TYPE - I TEST by repeated braking.												
NBR 10967 V1 = 80% Vmax , but km/h ≤		120	100	60	120	60	60					
V2 = 1/2 V1												
In gear : Time Interval secs		45	55	60	55	60	60					
FL No. of Cycles		15	15	20	15	20	20					
N.B. First snub at 3.0 m/sec ² deceleration; remaining snubs at same Pedal Effort.												
5. TYPE - I TEST by continuous braking.												
FL								Drag test at 40 km/h for 1.7 km on 7% slope				
6. HOT EFFECTIVENESS after Type-I Tests												
Prescribed Speed and Pedal Effort -												
Hot brakes												
FL as Test 1 :								40 km/h Brake Force ≥ 36% of Trailer max.axle wt and ≥ 60% of Test 1 achievement				

ECE REGULATION No.13 & EEC DIRECTIVE 71/320											
SUMMARY OF BRAKE PERFORMANCE TESTS & REQUIREMENTS		CARS & BUSES			VANS & TRUCKS			TRAILERS			
NOTATION:	Vehicle Category	M1	M2	M3	N1	N2	N3	O1	O2	O3	O4
	No. of Passenger Seats (excl.driver)	≤8	>8	>8							
	FL = Fully laden UL = Unladen	Max. Vehicle Weight (Tonnes)		≤5	>5	≤3½	3½-12	>12	≤¾	¾-3½	3½-10
SERVICE BRAKE (SB)		Acting on all wheels, properly distributed and symmetric across vehicle. Braking distribution in accordance with Annex 10 and 75/524, respectively.									
7. TYPE - II TEST for long descents NBR 10967											
In gear FL	For M3 N3 O4 vehicles only	Drag test at 30 kp/h for 6 km on 6% slope (or 0.5 m/sec ² by engine alone)									
7A. TYPE - IIA (BIS) TEST for 'Retarders'											
In gear FL	For M3 vehicles (instead of Test 7) - except 'urban buses'	Drag test at 30 kp/h for 6 km on 7% slope (or 0.6 m/sec ² by engine alone) - without using Service, Emergency or Parking Brake									
8. HOT EFFECTIVENESS after Type-II Tests											
Hot brakes FL	Prescribed Speed and Pedal Effort - as Test 1 : Stopping Distance m ≤ MFDD m/sec ² ≥	60 km/h 0.15V + 1.33V ² /130 3.75		60 km/h 0.15V + 1.33V ² /115 3.3		60 km/h 0.15V + 1.33V ² /115 3.3		40 km/h ≥ 33% of Trailer max. axle wt.			

Freios mais comuns



Diagrama de um freio a disco. O sistema é composto por uma pinça de freio que se move para comprimir um disco de freio entre duas pastilhas. O conjunto está montado no eixo da roda, que também possui um pneu.

pinça de freio

disco de freio

pneu

FREIO A DISCO



Diagrama de um freio a tambor. O sistema utiliza um regulador que atua sobre duas sapatas de freio, pressionando-as contra as superfícies internas de um tambor. O tambor está montado no eixo da roda, que também possui um pneu.

regulador

sapata de freio

cilindro de roda

pneu

FREIO A TAMBOR

HIDRÁULICOS

PNEUMÁTICOS





HIDRÁULICOS

PNEUMÁTICOS

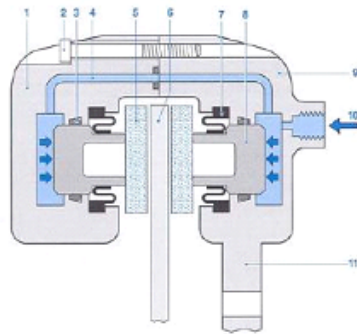


Brake System Components

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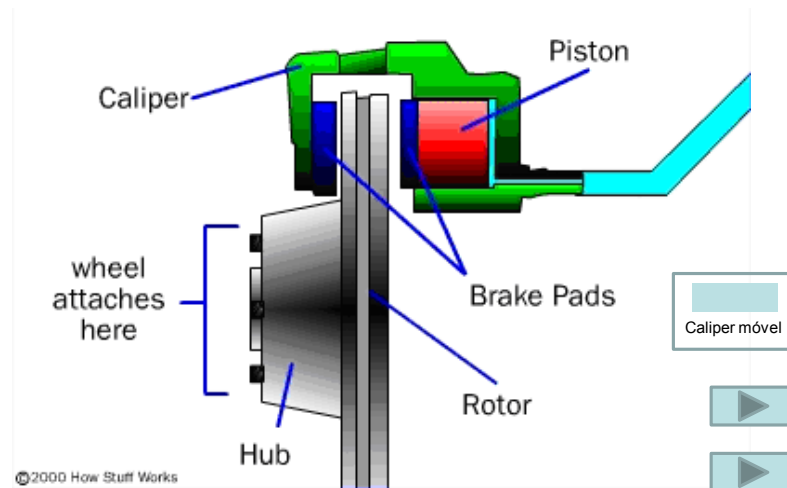
Caliper fixo



(1) Carcaça externa; (2) parafuso de junção; (3) anel de vedação; (4) canal de fluido; (5) pastilha de freio; (6) disco de freio; (7) guarda pó de borracha; (8) êmbolo; (9) carcaça interna flangelada; (10) furo de alimentação de fluido; (11) flange de montagem.



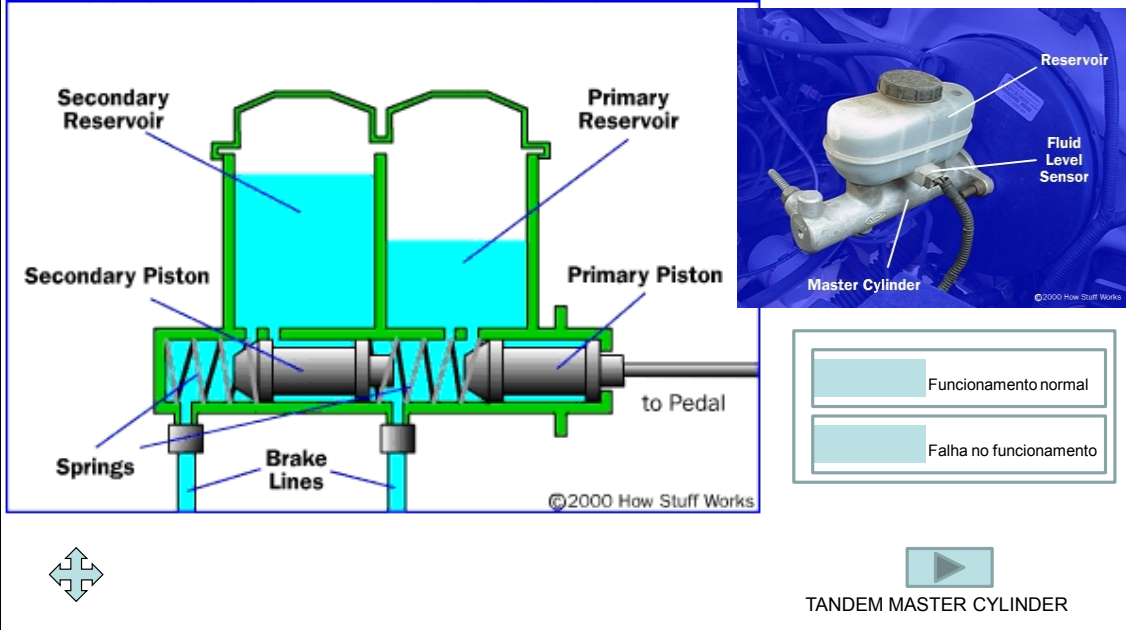
Caliper móvel





Cilindro mestre – duplo circuito

Inside the Master Cylinder





OUTROS MODELOS DE CILINDROS DE FREIO



Freio a tambor - hidráulico

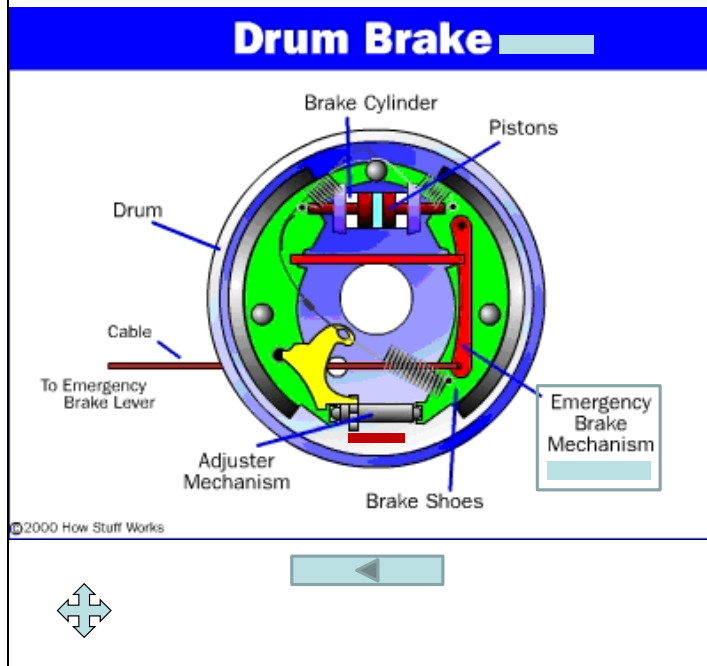


Legenda:

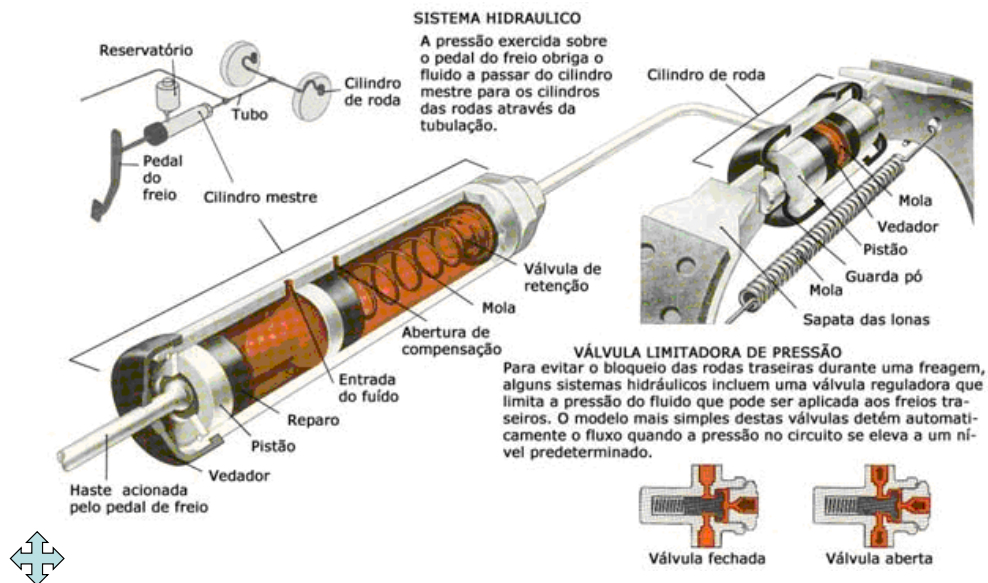
- 1-) Tambor Está preso ao cubo da roda
- 2-) Cilindro É ele quem recebe a pressão exercida pelo motorista sobre o pedal do freio, e se expande, forçando as sapatas contra o tambor.
- 3-) Sapata Peça resistente que é forçada contra o tambor, gerando atrito e ocasionando a frenagem.
- 4-) Lona Reveste externamente a sapata, pois tem maior capacidade de provocar atrito.



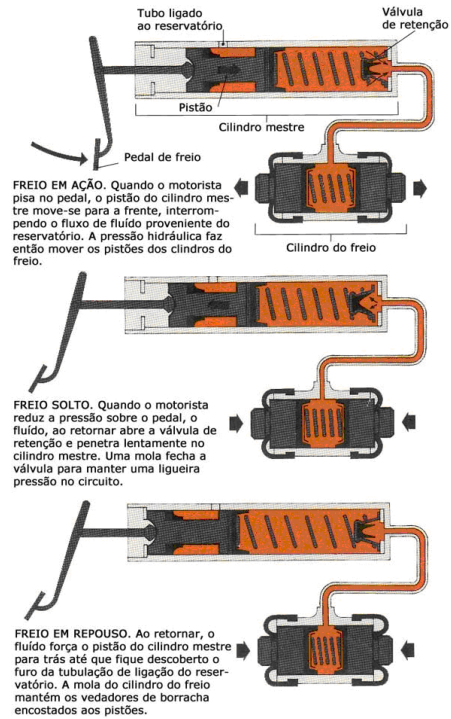
Freio a tambor - hidráulico



Cilindro mestre e cilindro de roda freio a tambor

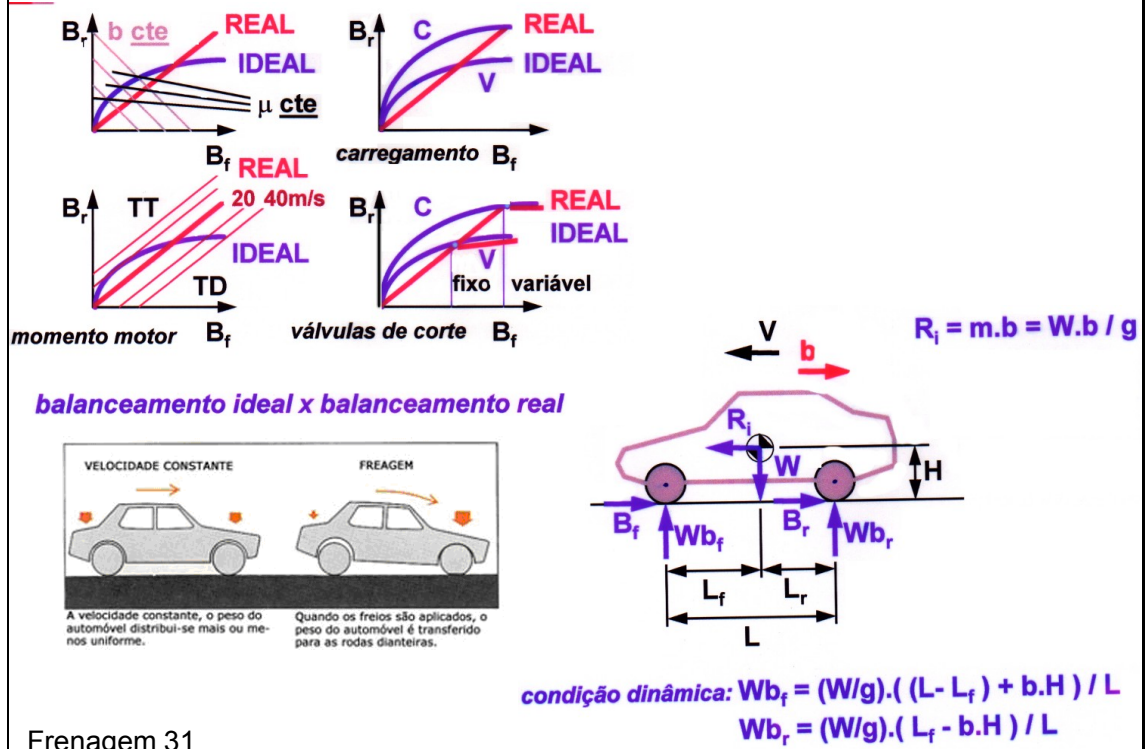


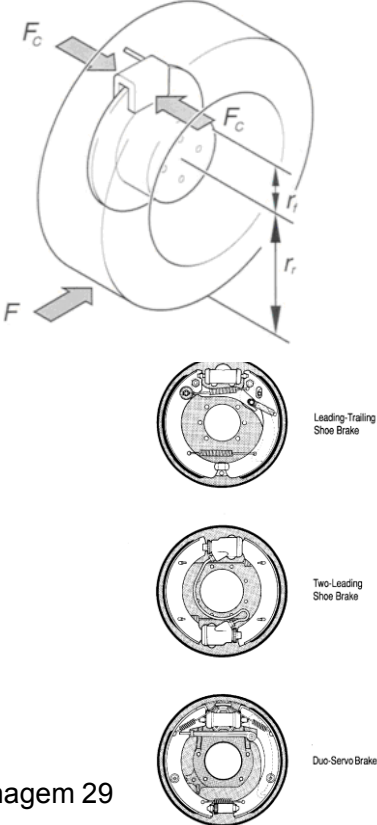
Cilindros – freio a tambor





Distribuição das forças de frenagem

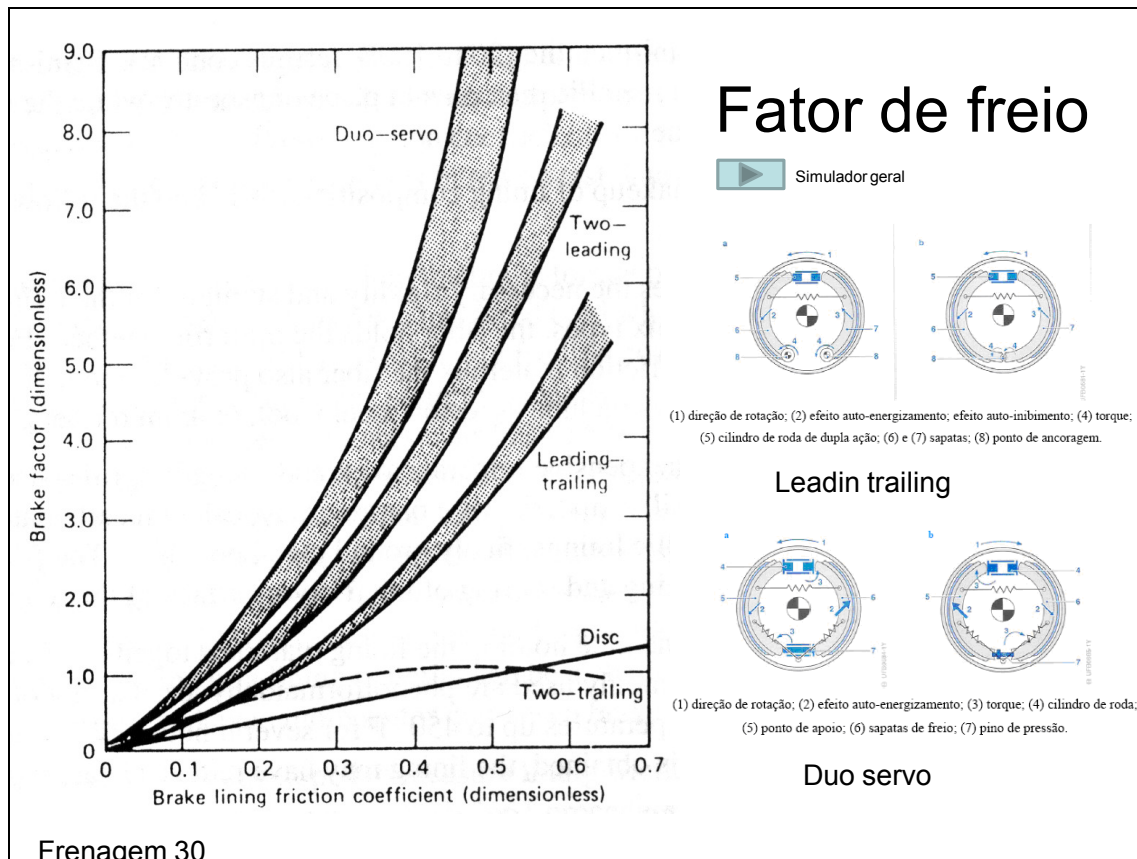




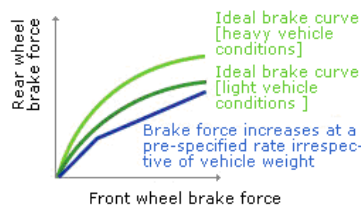
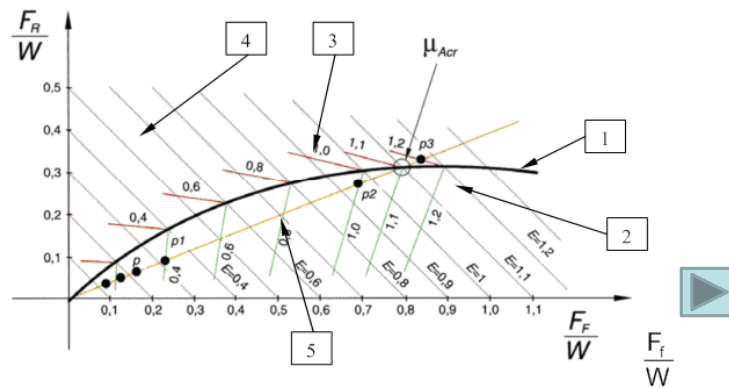
Forças reais:
 ex: freio a tambor com acionamento hidráulico
 $F_c = p \cdot A_c$
 fator de freio: $C' = F_t / F_c$
 $F_t = F_c \cdot C' = p \cdot A_c \cdot C'$
 torque real: $T_r = 2 \cdot F_t \cdot r = F_r \cdot R_{din}$
 portanto: $F_{ED/ET} = 2 \cdot p \cdot A_c \cdot C' \cdot r \cdot \eta / R_{din}$ **reta**

considerando: $\mu_f = \mu_r = \mu_o$
as forças de freagem serão máximas se:
 $B_f = \mu_f \cdot Wb_f = \mu \cdot Wb_f = \mu \cdot (W/g) \cdot ((1 - \alpha) + b \cdot \phi)$ (3)
 $B_r = \mu_r \cdot Wb_r = \mu \cdot Wb_r = \mu \cdot (W/g) \cdot (\alpha - b \cdot \phi)$ (4)
onde: $\phi = H / L$ e $\alpha = L_f / L$
realizando: $\sum F_x : R_i = B_f + B_r$ (5)
 $W \cdot b / g = \mu \cdot (Wb_f + Wb_r) = \mu \cdot W$
tem-se que: $\mu = b / g$
partindo de: $(B_r / W) = (b / g) - (B_f / W)$ (5):
tem-se:
 $B_r / W = (((1 - \alpha) / 2 \cdot \phi)^2 + (B_f / \phi \cdot W))^{1/2}$ **parábola**
 $- (B_f / W) - ((1 - \alpha) / 2 \cdot \phi)$

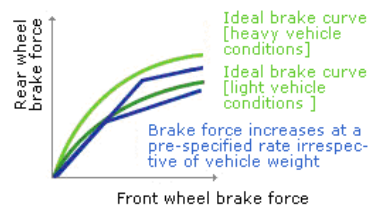
Frenagem 29



Distribuição das forças de frenagem



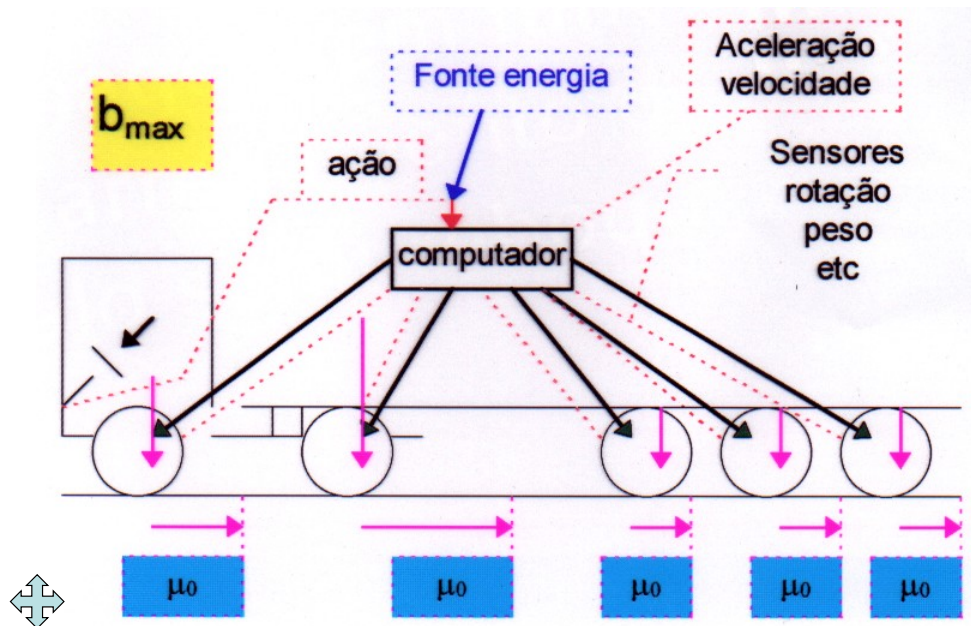
Existing propositioning value applied



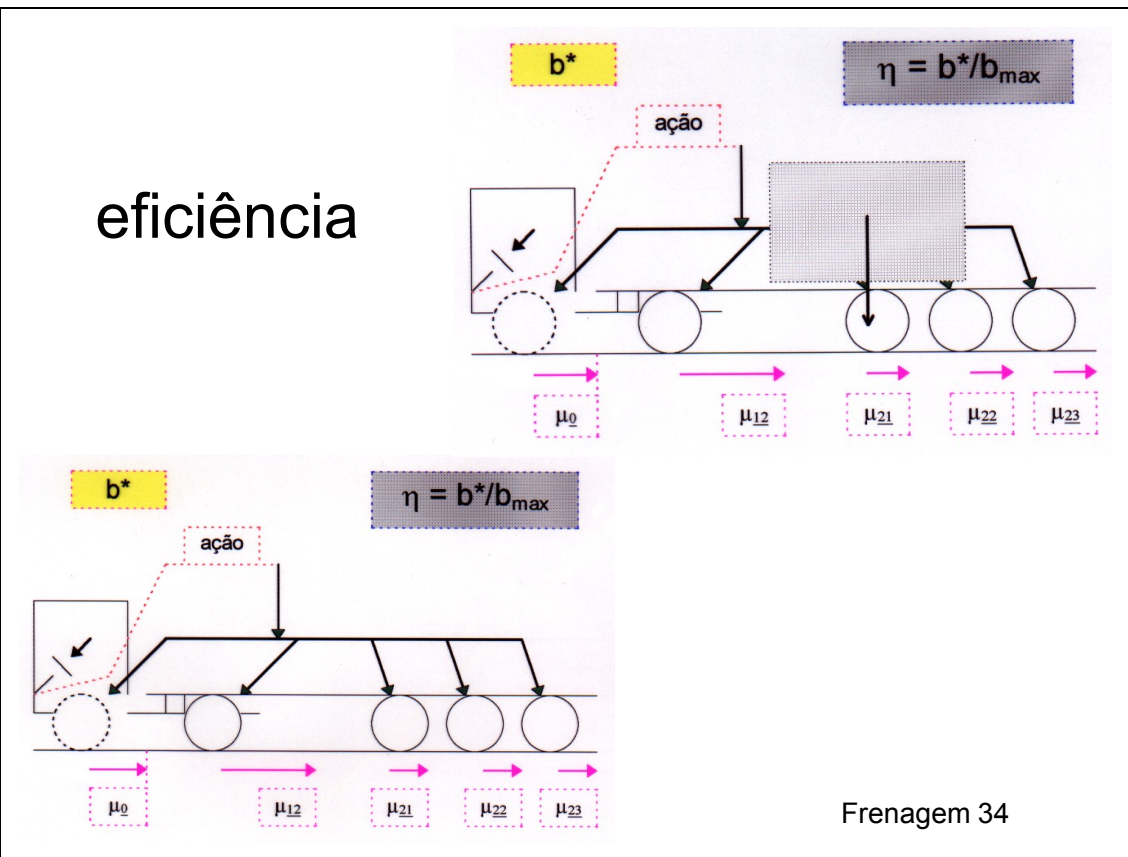
EBD applied

- The closer to the ideal brake curve for a particular vehicle weight, the bigger larger the brake force.

Veículo com freio de alto desempenho

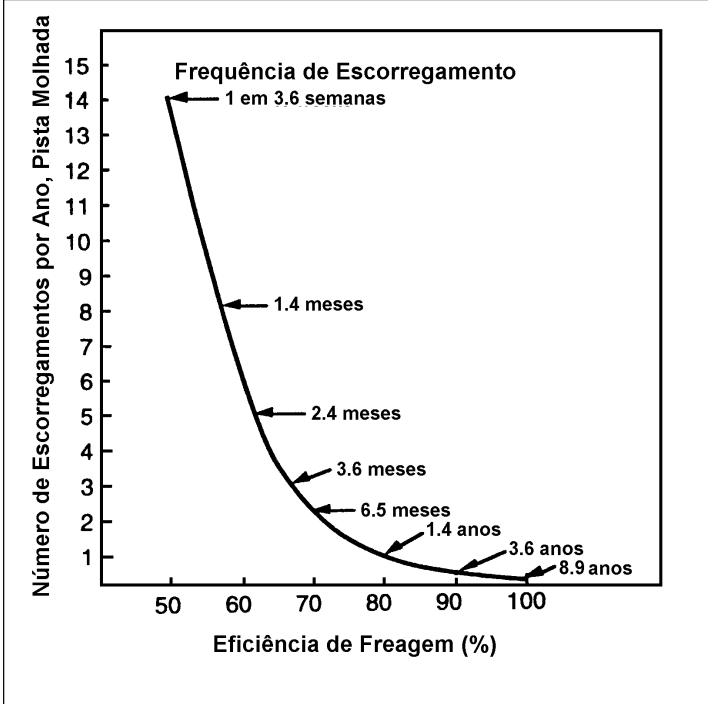


eficiência

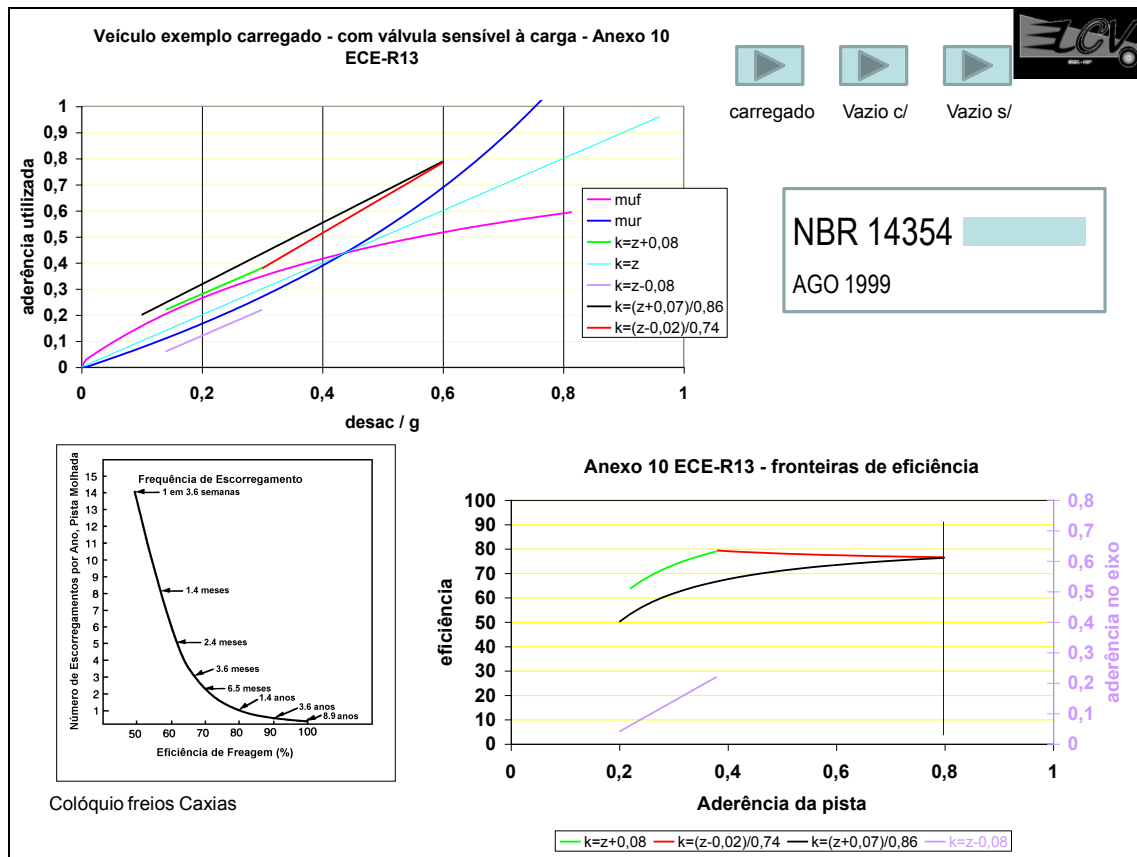


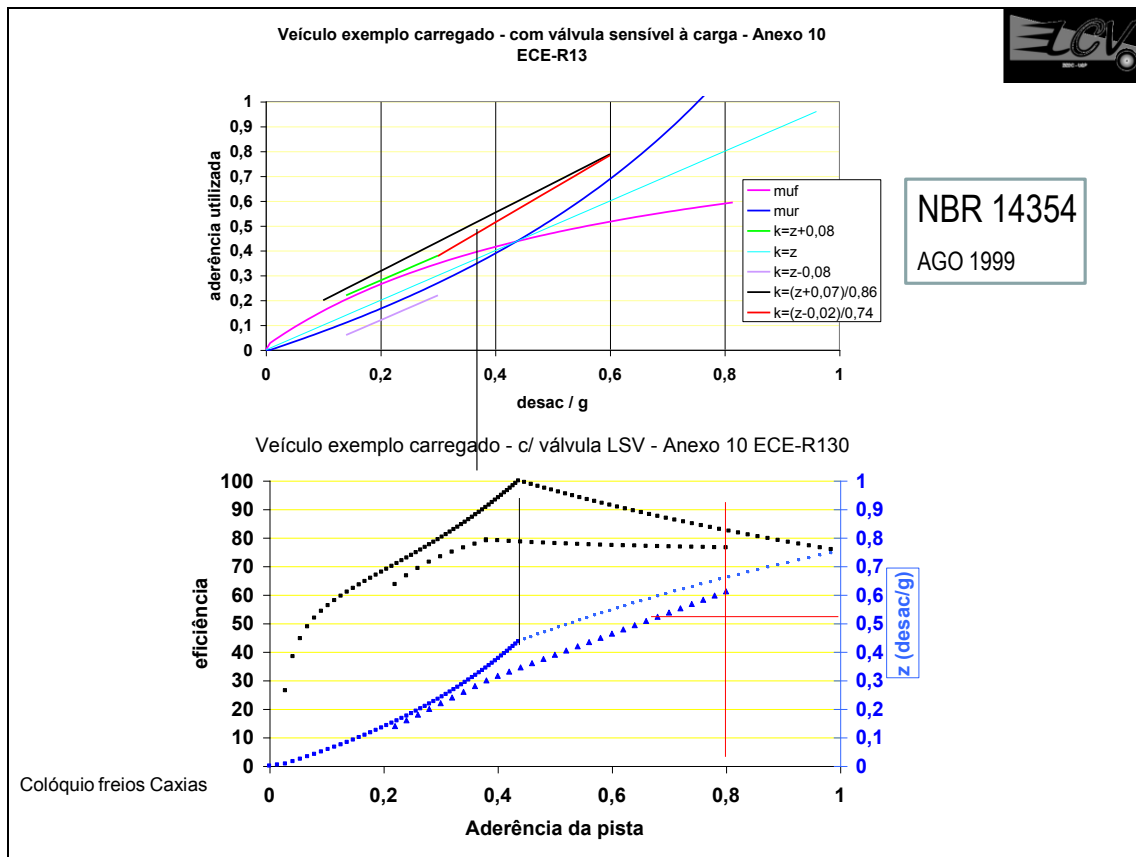
Frenagem 34

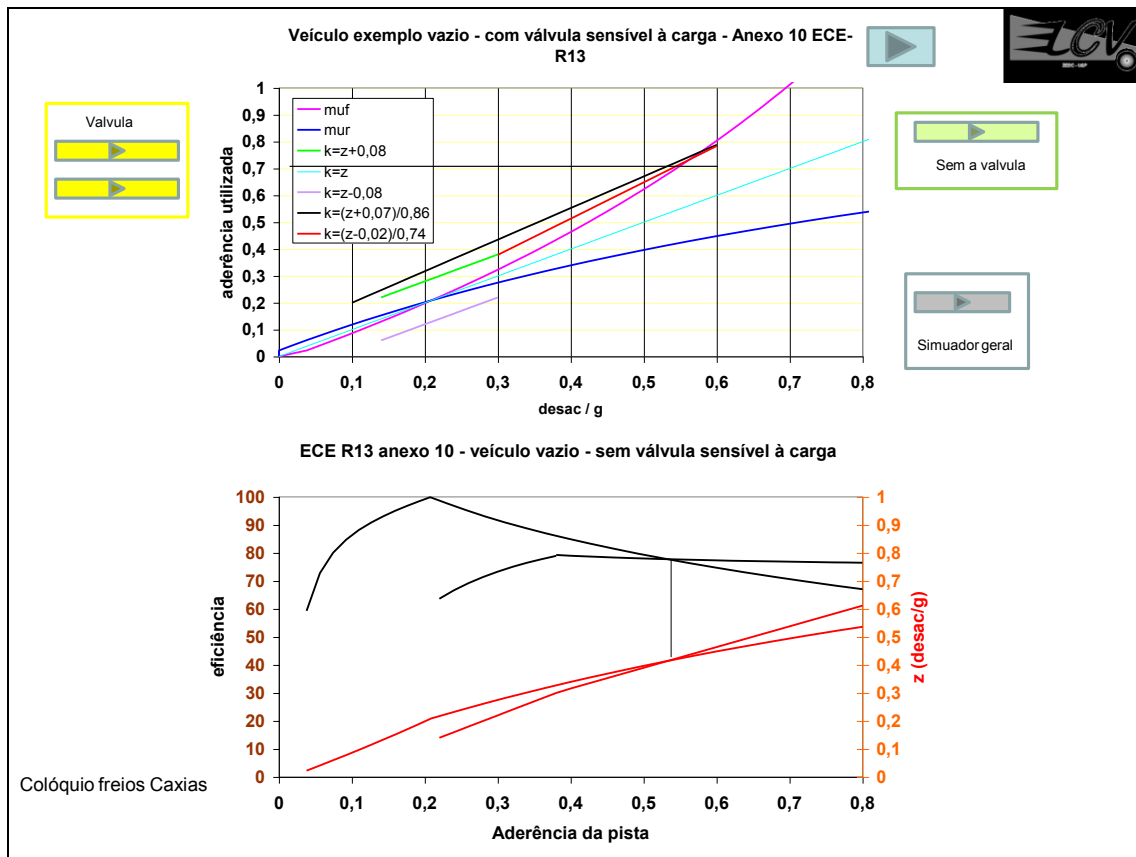
Eficiencia x segurança

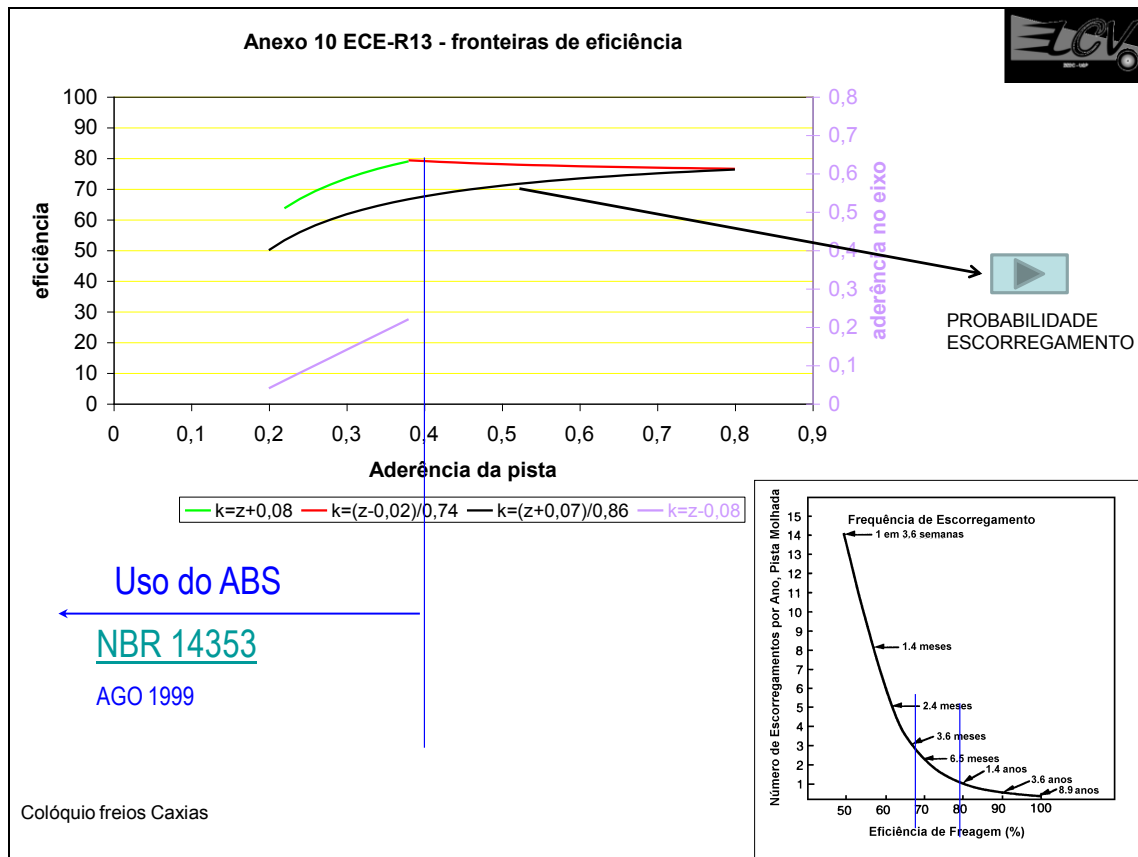


frenagem 35









COMPATIBILIDADE CAVALO - SEMIREBOQUE

- Em veículos combinados, deve haver compatibilidade de potência de frenagem entre o cavalo e o semi-reboque, caso contrário, haverá deslizamento das rodas de um deles prematuramente gerando movimentos instáveis com o jackknife e o swing trailer.



ABNT 14354



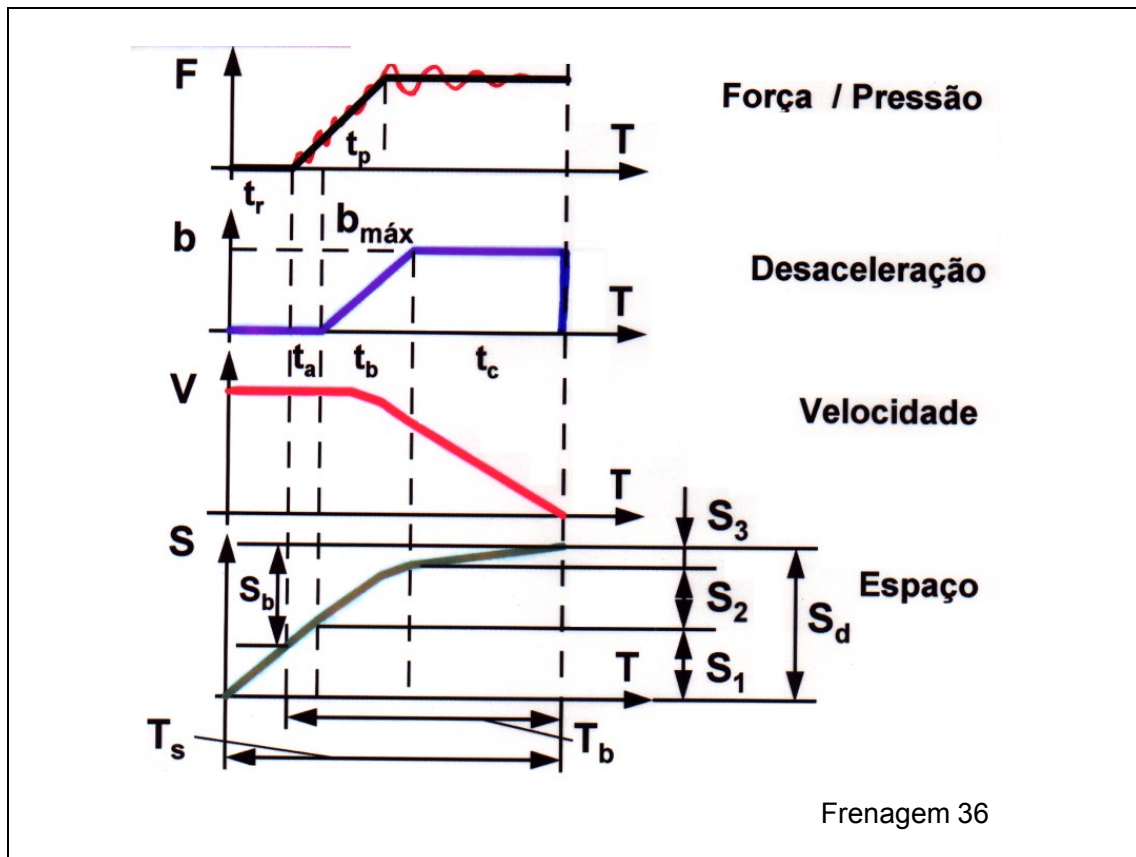
Compatibilidade semi-reboque



Compatibilidade cavalo



Compatibilidade semi-reboque



$$S = \frac{\varphi_b}{g \times (\mu + f)} \times ((V_i^2 - V_f^2) \div 2)$$

$$t = \frac{\gamma_b}{g \times (\mu + f)} \times (V_i - V_f)$$

$$b = \frac{g}{\gamma_b} (\mu + f)$$
$$\begin{aligned} \gamma b &= 1.05 \\ \mu &= 0.8 \\ V_i &= 60 \text{ km} / \text{h} \\ f &= 0.01 \end{aligned}$$

$$\begin{aligned} S &= 18.35 \\ t &= 2.20 \\ b &= 7.56 \end{aligned}$$



$$\begin{aligned} \rho &= 1,22 \\ A_f &= 6 \\ \gamma_b &= 1.05 \\ \mu &= 0.8 \\ v_i &= 60 \text{ km / h} \\ f &= 0.01 \\ S &= 17,29 \\ t &= 2,12 \\ b &= 8,52 \end{aligned}$$

$$S = \frac{m \times \gamma_b}{2 \times C} \log_e \left(1 + \frac{C \times (V_i^2 - V_f^2)}{W \times (\mu + f)} \right)$$

$$C = 1/2 \times \rho \times C_d \times A_f \quad C_d = 1$$

$$t = \frac{\gamma_b}{g} \times \sqrt{\frac{W}{C \times (\mu + f)}} \times tg^{-1} \left((V_i - V_f) \times \sqrt{\frac{C}{W \times (\mu + f)}} \right)$$

$$b = \frac{g}{\gamma_b} \times \left(\mu + f + \frac{R_a}{W} \right)$$



Espaço de parada com a reação do motorista

